



Article A Novel Coupled Meshless Model for Simulation of Acoustic Wave Propagation in Infinite Domain Containing Multiple Heterogeneous Media

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Abstract: This study presents a novel coupled meshless model for simulating acoustic wave propagation in heterogeneous media, based on the singular boundary method (SBM) and Kansa's method (KS). In the proposed approach, the SBM was used to model the homogeneous part of the propagation domain, while KS was employed to model a heterogeneity. The interface compatibility conditions associated with velocities and pressures were imposed to couple the two methods. The proposed SBM–KS coupled approach combines the respective advantages of the SBM and KS. The SBM is especially suitable for solving external sound field problems, while KS is attractive for nonlinear problems in bounded nonhomogeneous media. Moreover, the new methodology completely avoids grid generation and numerical integration compared with the finite element method and boundary element method. Numerical experiments verified the accuracy and effectiveness of the proposed scheme.

Keywords: singular boundary method; Kansa's method; heterogeneous media; acoustic wave; meshless method

MSC: 35J05; 65N35; 65D12

1. Introduction

The propagation of sound waves in fluids and solids is an important issue in science and engineering. In the past few decades, the boundary element method (BEM) has become established as an effective tool for sound propagation analysis, especially for the infinite and semi-infinite domains [1–4], due to the used fundamental solution automatically satisfying the far-field radiation condition. Compared with other well-established meshbased methods, such as the finite element method (FEM) [5–8] and the finite difference method (FDM) [9,10], the BEM can solve acoustic problems merely through boundary discretization. However, it involves a sophisticated mathematical formulation and a tedious estimation of singular and hyper-singular integrals [11,12]. Furthermore, these methods require the use of domain truncation techniques for infinite domain problems for the numerical solution of problems on unbounded domains.

In recent years, various meshless/meshfree methods [13–19] have been proposed to reduce or even eliminate the tasks of mesh generation and singular integration. Among these approaches, the singular boundary method (SBM) [20–23] is a boundary-only discretization meshless technique, which does not require mesh generation and numerical integration. This method is very simple and accurate for the analysis of sound fields in unbounded domains, since it also employs the fundamental solution satisfying the governing equation and the far-field radiation condition [24,25]. Another common meshless scheme is Kansa's method (KS) [26–29], which is based on the radial basis function (RBF). This method does



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Copyright: © 2023 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). not require the fundamental solution, and is suitable for solving arbitrary partial differential equations in bounded domains [30–33].

As can be inferred from above, the SBM and KS have their respective advantages in addressing unbounded homogeneous media and bounded non-homogeneous media. In order to avoid complex computational processes such as mesh generation and singular integral computation using traditional methods such as the FEM and the BEM, this research made a first attempt to couple these two methods (named SBM–KS) for simulating acoustic wave propagation in heterogeneous media. The SBM is adopted to model the homogeneous part of the propagation domain, while KS is employed to model a heterogeneity. A direct coupling strategy between the SBM and KS is presented based on the continuity conditions of velocities and pressures on the interface. The coupling method shows unique advantages in solving such problems compared to existing methods, such as simplicity, accuracy, and being free of mesh and integration.

The organization of this manuscript is as follows: Section 2 briefly describes acoustic wave propagation problems of heterogeneous media. Section 3 introduces the SBM for an unbounded acoustic medium, KS for a heterogeneous acoustic medium, and the coupling strategy of these two methods. In Section 4, two classical numerical examples are provided to verify the accuracy and effectiveness of the proposed methodology. Finally, Section 5 provides some conclusions and remarks.

2. Problem Statement

Consider an unbounded homogeneous medium Ω_1 , containing a subdomain Ω_2 in which the sound velocity is variable (see Figure 1a), and the sound field is excited by a harmonic pressure source at position $s = (x_0, y_0)$. In this regard, the sound waves travel at a constant speed $v_1(x) = v_1$ in Ω_1 and a variable speed $v_1(x)$ in Ω_2 at x = (x, y). Then, the acoustic pressure fields $p_1(x)$ and $p_2(x)$ within homogeneous and heterogeneous media can be described by the following Helmholtz equations:

$$\nabla^2 p_1(\mathbf{x}) + \left[\frac{\omega}{v_1}\right]^2 p_1(\mathbf{x}) = 0, \ \mathbf{x} \in \Omega_1, \tag{1}$$

$$\nabla^2 p_2(\mathbf{x}) + \left[\frac{\omega}{v_2(\mathbf{x})}\right]^2 p_2(\mathbf{x}) = 0, \ \mathbf{x} \in \Omega_2,$$
(2)

where ∇^2 is the Laplace operator, and ω is the angular frequency.



Figure 1. Schematic diagrams of (**a**) acoustic wave propagation in heterogeneous media and (**b**) nodal distribution for the coupled method.

Notice that the governing equations are the PDEs with constant and variable coefficients in domains Ω_1 and Ω_2 , respectively. The conventional boundary-type methods, such as the BEM, the SBM, and the fundamental solution method, cannot be directly applied to solve variable-coefficient PDEs. Meanwhile, domain-type methods, such as the FEM, the

meshless local Petrov–Galerkin (MLPG) method and Kansa's method, require the truncation of boundaries and the division of grids, which is extremely troublesome when dealing with problems in infinite homogeneous media. It should be noticed that the fundamental solution employed in the SBM automatically satisfies the Sommerfeld radiation condition at infinity:

$$\lim_{r \to \infty} r^{\frac{1}{2}(d-1)} \left(\frac{\partial p(x)}{\partial r} - ikp(x) \right) = 0$$
(3)

where *d* is the spatial dimension, $i = \sqrt{-1}$ is the imaginary unit, *k* is the wave number, and *r* is the distance between point *x* and the sound field's center.

In the present study, the above-mentioned problem was solved by coupling the SBM and KS to overcome the limitations posed separately by each method. The SBM was employed to model the unbounded acoustic medium, while KS was used to model the heterogeneous medium. The coupling between the two approaches was accomplished by utilizing continuity conditions of pressures and velocities on the boundary of the heterogeneous medium. Figure 1b illustrates the schematic diagram of the nodal distribution for the coupled meshless model. The two methods used the same nodes on the interface.

3. Methodology

3.1. SBM for Unbounded Acoustic Medium

Assuming the total number of nodes on the interface is *M*, the sound pressure at point $x \in \Omega_1 \cup \Gamma$ can be calculated by the following SBM formula:

$$\hat{p}_1(\boldsymbol{x}) = \sum_{j=1}^M \alpha_j G(\boldsymbol{x}, \boldsymbol{x}_j) + p_{inc}(\boldsymbol{x}_i, \boldsymbol{s}), \ \boldsymbol{x}_j \in \Gamma$$
(4)

where α_j is the unknown coefficient, \mathbf{x}_j is the boundary node shown in Figure 1b, $p_{inc}(\mathbf{x}_i, \mathbf{s}) = H_0^{(2)}(\frac{\omega}{v_1} || \mathbf{x}_i - \mathbf{s} ||_2)$ represents the incident pressure field generated by a harmonic pressure source at position $\mathbf{s} = (x_0, y_0)$ in the domain Ω_1 , and $G(\mathbf{x}, \mathbf{x}_j)$ is the fundamental solution of the Helmholtz equation, which is given by the following:

$$G(\boldsymbol{x}, \boldsymbol{x}_j) = -\frac{\mathrm{i}}{4} H_0^{(2)} \left(\frac{\omega}{v_1} \| \boldsymbol{x} - \boldsymbol{x}_j \|_2 \right)$$
(5)

where $H_0^{(2)}$ is the zeroth-order Hankel function of the second kind.

To solve the unknown coefficients $\{\alpha_j\}_{j=1}^M$, let *x* in Equation (4) be the boundary node x_j ; we have the following equations for the Dirichlet boundary condition:

$$\hat{p}_{1}(\mathbf{x}_{i}) = \sum_{\substack{j=1\\i\neq j}}^{M} \alpha_{j} G(\mathbf{x}_{i}, \mathbf{x}_{j}) + \alpha_{i} p_{ii} + p_{inc}(\mathbf{x}_{i}, \mathbf{s}), \quad i = 1, 2, \dots, M,$$
(6)

and for the Neumann boundary condition, we have the following:

$$\frac{\partial \hat{p}_1(\boldsymbol{x}_i)}{\partial n_{x_i}} = \sum_{\substack{j=1\\i\neq j}}^M \alpha_j \frac{\partial G(\boldsymbol{x}_i, \boldsymbol{x}_j)}{\partial n_{x_i}} + \alpha_i q_{ii} + \frac{\partial p_{inc}(\boldsymbol{x}_i, \boldsymbol{s})}{\partial n_{x_i}}, \quad i = 1, 2, \dots, M,$$
(7)

where x_i and x_j denote the *i*th and *j*th boundary nodes, p_{ii} and q_{ii} are the origin intensity factors when the source point and the field point coincide (i.e., i = j), which can be computed using the following formulas in references [34,35]:

$$u_{ii} = \frac{\mathrm{i}}{4} - \frac{1}{2\pi} \left(\ln\left(\frac{L_i}{2\pi}\right) + \ln\left(\frac{k}{2}\right) + \gamma \right),\tag{8}$$

$$q_{ii} = \frac{1}{L_i} - \sum_{\substack{j=1\\j \neq i}}^{N} \zeta_{ji} \frac{\partial G_0(\boldsymbol{x}_i, \boldsymbol{s}_j)}{\partial \boldsymbol{n}_s}, \qquad (9)$$

where L_i is the influence range of the boundary point x_i (see Figure 2), γ is the Euler constant, $G_0(x_i, x_j)$ the fundamental solution of the Laplace equation, as follows:





For the convenience of coupling calculations, Equations (6) and (7) can be written in the following matrix forms:

$$\hat{\mathbf{p}}_1 = \mathbf{G}\boldsymbol{\alpha} + \mathbf{p}_{\mathrm{in'}} \tag{11}$$

$$\hat{\mathbf{q}}_1 = \mathbf{H}\boldsymbol{\alpha} + \mathbf{q}_{\rm in}.\tag{12}$$

3.2. Kansa's Method for Inhomogeneous Acoustic Medium

For the closed domain Ω_2 with the boundary Γ , we chose a set of *N* collocation points, including *Ni* internal points and *M* boundary points, as shown in Figure 1b. According to the basic idea of KS, the sound pressure in subdomain Ω_2 can be approximated using a linear combination of RBFs, as follows:

$$\hat{p}_2(\mathbf{x}) = \sum_{k=1}^N \beta_k \varphi_k(\mathbf{x}), \tag{13}$$

where β_k is the unknown coefficient to be determined, and $\varphi_k(\mathbf{x})$ is the multiquadric (MQ) RBF function, which is defined as the following:

$$\varphi_k(\mathbf{x}) = \sqrt{r_k^2 + c^2},\tag{14}$$

where $r_k = ||\mathbf{x} - \mathbf{x}_k||_2$ is the distance between nodes \mathbf{x} and \mathbf{x}_k , and c is the shape parameter, which is fixed at 0.5 in this study. The MQ-RBF is a highly sought-after function due to its numerous advantages for various applications. Its main advantages include the following:

(1) Smoothness: it is a smooth function with continuous derivatives of all orders, which is a crucial requirement for many applications. (2) Accuracy: it offers exceptional approximation accuracy for a wide range of functions and can converge faster than other RBFs for some problems, making it an excellent choice for large-scale applications. (3) Scalability: it is computationally efficient, making it a perfect fit for large-scale problems. It has a low computational cost for interpolation and can be easily parallelized. (4) Robustness: it is less sensitive to data outliers than other RBFs, making it a robust choice for applications where data may contain noise or outliers. (5) Universality: it is a universal approximator, meaning it can approximate any continuous function to any desired accuracy, given sufficient data points.

Substituting Equation (13) into Equation (1) for internal nodes, one obtains the following:

$$\nabla^2 \sum_{k=1}^N \beta_k \varphi_k(\mathbf{x}_i) + \left[\frac{\omega}{v_2(\mathbf{x}_i)}\right]^2 \sum_{k=1}^N \beta_k \varphi_k(\mathbf{x}_i) = 0, \ \mathbf{x}_i \in \Omega_2$$
(15)

In addition, the sound pressure and its normal derivative at the boundary nodes satisfy the following equations:

$$\hat{p}_2(\mathbf{x}_i) = \sum_{k=1}^N \beta_k \varphi_k(\mathbf{x}_i), \ \mathbf{x}_i \in \Gamma,$$
(16)

$$\frac{\partial \hat{p}_2(\boldsymbol{x}_i)}{\partial n_{\boldsymbol{x}_i}} = \sum_{k=1}^N \beta_k \frac{\partial \varphi_k(\boldsymbol{x}_i)}{\partial n_{\boldsymbol{x}_i}}, \ \boldsymbol{x}_i \in \Gamma.$$
(17)

Equations (15)–(17) can be rewritten in the following matrix forms:

$$\mathbf{R}\boldsymbol{\beta} = \mathbf{0},\tag{18}$$

$$\hat{\mathbf{p}}_2 = \mathbf{B}\boldsymbol{\beta},\tag{19}$$

$$\hat{\mathbf{q}}_2 = \mathbf{F}\boldsymbol{\beta}.\tag{20}$$

3.3. Coupled Model Dymamic System

This study proposed a direct coupling strategy between the two methods, under the condition that the nodes used in the SBM model matched the boundary nodes used in KS. Note the following:

$$\frac{\partial p(\mathbf{x})}{\partial n_{\mathbf{x}}} = -\mathrm{i}\rho\omega v(\mathbf{x}),\tag{21}$$

where the coupled system can be established by employing the continuity of pressure and velocity on the interface between the two media, which can be expressed as follows:

$$p_1(x) = p_2(x)$$
 or $\mathbf{p}_1 = \mathbf{p}_2$, $x \in \Gamma$, (22)

$$\frac{\partial p_1(\mathbf{x})}{\partial n_x} = -\frac{\partial p_2(\mathbf{x})}{\partial n_x} \quad \text{or} \quad \mathbf{q}_1 = \mathbf{q}_2, \quad \mathbf{x} \in \Gamma.$$
(23)

Considering all the nodes within the domain Ω_2 , and using the above continuity conditions, Equations (11), (12), and (18)–(20) can be combined to form a total linear system, namely, the following:

$$\begin{bmatrix} \mathbf{G} & -\mathbf{B} \\ \mathbf{H} & \mathbf{F} \\ \mathbf{0} & \mathbf{R} \end{bmatrix} \begin{bmatrix} \alpha \\ \beta \end{bmatrix} = \begin{bmatrix} -\mathbf{p}_{\text{in}} \\ -\mathbf{q}_{\text{in}} \\ \mathbf{0} \end{bmatrix}$$
(24)

4. Numerical Results and Discussion

In this section, we examine the proposed SBM–KS by three numerical examples, including single and multiple heterogeneous inclusion materials. To assess the numerical errors, the following absolute error was employed:

Absolute error =
$$|p_{num}(\mathbf{x}) - p_{ana}(\mathbf{x})|$$
, (25)

where $p_{num}(x)$ and $p_{ana}(x)$ represent the numerical and analytical solutions at point x, respectively. Note that p can be a real part or an imaginary part of the sound pressure.

Example 1. We consider an infinite homogenous fluid medium with a circular inclusion of radius 1.0 m [36]. The sound velocities are 1500 m/s and 2500 m/s in the infinite fluid medium and the circular inclusion, respectively. Both media have a same density of 1000 kg/m³. The pressure source is placed at ($x_0 = -5 \text{ m}$, $y_0 = 0 \text{ m}$).

To numerically solve this problem, the SBM–KS chose 100 interface nodes and 688 internal nodes. Figure 3 shows the comparison of the analytical solution [37], and numerical results obtained by the proposed SBM–KS and COMSOL software under a frequency of 1000 Hz. Absolute errors of these two methods are also provided in Figure 4. In the simulation, the finite element method with 7780 elements used the perfectly matched layer. We can see from Figure 3 that the numerical results obtained from the SBM–KS and the COMSOL FEM are in good agreement with the analytical solutions. It can also be clearly observed that the curve of our method completely coincides with the curve of the analytical solution, while the FEM has certain errors. Moreover, it can be noted that the calculation accuracy of the proposed method is at least two orders higher than that of the FEM. In this example, the condition number of the proposed approach is 1.426×10^{11} . The SBM has a small condition number, but KS has a large condition number [38], which leads to a large value of the condition number for the final coefficient matrix. However, the method can still obtain accurate numerical results.



Figure 3. Comparison of the numerical and analytical solutions for hydrodynamic pressures along the common interface under a frequency of 1000 Hz.



Figure 4. Absolute errors of the SBM–KS and the COMSOL Multiphysics finite element methods.

Example 2. This example considers a non-homogeneous circular region of radius 1 m centered at point (0, 0), which is embedded in an unbounded fluid medium. The pressure source with a frequency of 1000 Hz is placed at $(x_0 = -2.5 \text{ m}, y_0 = 0 \text{ m})$. Both media have a same density of 1000 kg/m³. The outer fluid medium allows sound waves to travel at 1500 m/s, while the non-homogeneous medium allows sound waves to travel at the following:

$$v_2(x,y) = 1500 + 150 \left[1 + \sin\left(\pi\sqrt{x^2 + y^2} + \frac{\pi}{2}\right) \right],$$
 (26)

which is illustrated in Figure 5.



Figure 5. Velocity distribution in a heterogeneous domain for Example 2.

In practical problems, the node distribution may be scattered and uneven. As a meshless technique, the proposed SBM–KS can address the non-uniform node distribution in a leisurely manner. In order to test the effect of node distribution on the calculation accuracy, distributions of regular and irregular nodes were investigated in the calculation, as shown in Figure 6. It includes 100 interface nodes and 688 internal nodes.



Figure 6. Nodal distributions: (a) regular nodes and (b) irregular nodes.

Figure 7 displays the profiles of the analytical response in the domain $[-2, 2] \times [-2, 2]$, and Figures 8 and 9 depict the absolute errors of the proposed method under regular and irregular nodes, respectively. It was noted that the numerical solutions are in good agreement with the analytical one for the regular and irregular distributions of nodes, and the numerical errors are small. In this example, the condition number of the proposed approach is 4.665×10^{11} .



Figure 7. Analytical responses in the domain $[-2, 2] \times [-2, 2]$: (a) real part, (b) imaginary part.



Figure 8. Distributions of absolute error in the domain $[-2, 2] \times [-2, 2]$ with regular nodes: (a) the real part with (b) the imaginary part.



Figure 9. Distributions of absolute error in the domain $[-2, 2] \times [-2, 2]$ with irregular nodes: (a) the real part with (b) the imaginary part.

Example 3. The last example considers a complex sound propagation problem in a heterogeneous medium, shown in Figure 10. The domain Ω_1 is an infinite domain, in which the speed of sound is $v_1 = 1500 \text{ m/s}$ and the density is $\rho_1 = 1000 \text{ kg/m}^3$. The pressure source is placed at $(x_0 = -5 \text{ m}, y_0 = 0 \text{ m})$. The bounded domains Ω_2 , Ω_3 , and Ω_4 are all heterogeneous media, and their boundaries can be expressed as the following parameter forms:

$$\Gamma_2 = \left\{ \left(x = r_2(\theta) \cos \theta, y = 2 + r_2(\theta) \sin \theta \right) \middle| r_2(\theta) = \sqrt[3]{\cos(3\theta)} + \sqrt{2 - \sin^2(3\theta)}, \ 0 \le \theta \le 2\pi \right\},\tag{27}$$

$$\Gamma_3 = \left\{ \left(x = r_3(\theta) \cos \theta, y = -2 + r_3(\theta) \sin \theta \right) \middle| r_3(\theta) = e^{\sin \theta} \sin^2 \theta + e^{\cos \theta} \cos^2 \theta, \ 0 \le \theta \le 2\pi \right\},\tag{28}$$

$$\Gamma_3 = \{ (x = 4 + r_4(\theta) \cos \theta, y = r_4(\theta) \sin \theta) | r_4(\theta) = 1, \ 0 \le \theta \le 2\pi \}.$$
(29)



Figure 10. Geometry of the heterogeneous medium in Example 3.

In the present study, we assumed that the density of the three heterogeneous media is the same as that of the infinite medium, while these three media allow sound waves to travel at the following velocities:

$$v_2(x,y) = 1500 + 200 \left[1 + \sin\left(\pi\sqrt{x^2 + (y-2)^2}/r_2(\overline{\theta}) + \frac{\pi}{2}\right) \right], \ (x,y) \in \Omega_2,$$
(30)

$$v_3(x,y) = 1500 + 100 \left[1 + \sin\left(\pi\sqrt{x^2 + (y+2)^2}/r_3(\overline{\theta}) + \frac{\pi}{2}\right) \right], \quad (x,y) \in \Omega_3,$$
(31)

$$v_4(x,y) = 1500 + 150 \left[1 + \sin\left(\pi\sqrt{(x-4)^2 + y^2}/r_4(\overline{\theta}) + \frac{\pi}{2}\right) \right], \ (x,y) \in \Omega_4,$$
(32)

where $\overline{\theta}$ denotes the azimuth angle of the point (*x*, *y*), functions r_2 , r_3 , and r_4 have are given in Equations (27)–(29). The velocity variations in the domain $[-8, 8] \times [-6, 6]$ are shown in Figure 11.



Figure 11. Velocity distribution in the heterogeneous domain for Example 3.

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In this example, the proposed method was used to solve the problem of sound propagation in an infinite domain with three heterogeneous media. In the calculation, the SBM was employed to simulate the external sound field Ω_1 , while KS was used to approximate the sound field in heterogeneous media Ω_2 , Ω_3 , and Ω_4 . The two methods were coupled by employing the continuity conditions of pressure and velocity on the interfaces Γ_2 , Γ_3 , and Γ_4 .

In order to obtain accurate and reliable numerical results, a total of 6422 nodes were used in the study. On each interface, 400 nodes were evenly arranged according to the angle. In domains Ω_2 , Ω_3 , and Ω_4 , 1360, 2616, and 1246 nodes were configured, respectively. The node distribution is shown in Figure 12. The proposed SBM–KS was used to calculate the sound field with two different frequencies. The FEM results were also obtained using the COMSOL Multiphysics software to compare with our method. In the simulation, the FEM used 242,444 domain elements and 2846 boundary elements to achieve the reliable solutions.



Figure 12. Node distribution of the coupling method.

Figures 13 and 14 give numerical results of sound pressure and sound pressure level at frequencies 1000 Hz and 1500 Hz. From these figures, it can be observed that the sound wave propagates regularly when it does not encounter heterogeneous materials. However, after passing through heterogeneous materials, the waveform changes. The higher the frequency, the more noticeable the impact effect. In addition, it can be observed from Figures 13 and 14 that the results of the two methods basically have the same trend from a global perspective, and can reveal the propagation law of sound waves. In terms of details, the numerical solutions of the two methods differed slightly. It should be clarified that this example did not have an analytical solution to verify the computational accuracy of the two methods, but the previous two examples indicated the reliability of the proposed SBM–KS approach.

In this example, the condition numbers of the proposed approach are 2.177×10^{19} and 4.408×10^{19} for 1000 Hz and 1500 Hz, respectively. Compared with the previous two examples,

the geometry investigated in this example is more complex, including three heterogeneous media with irregular boundaries in an infinite domain. Note that the condition number increased sharply as the frequency increased, and the number of nodes increased.

The proposed method in this study successfully solved the problem of infinite domain acoustic wave propagation involving multiple heterogeneous media. It provides a new and simple mesh-free numerical technique for the efficient and accurate numerical simulation of such problems, and also serves as a reference for validating the numerical effectiveness of other methods.



Figure 13. Profiles of sound pressure and sound pressure level obtained using the SBM–KS approach and COMSOL FEM at a frequency of 1000 Hz.



(a) SBM-KS

(b) FEM

Figure 14. Profiles of sound pressure and sound pressure level obtained using the SBM–KS approach and COMSOL FEM at a frequency of 1500 Hz.

5. Conclusions

In this study, a novel coupled algorithm was presented for the analysis of acoustic wave propagation in heterogeneous media, based on the SBM and KS. The proposed model can accurately solve problems of heterogeneous media containing localized regions with varying medium parameters, for which the application of the SBM is not suitable. The new methodology completely avoids grid generation and numerical integration, and greatly exerts the respective advantages of the two methods.

Numerical examples investigated the sound propagation problems through single and multiple heterogeneous materials. Numerical results demonstrated that the proposed scheme is accurate and reliable for simulated acoustic wave propagation in heterogeneous media. On the one hand, the method eliminates the preprocessing process in the FEM, such as mesh division and perfect matching layer setting. On the other hand, it is superior to the traditional FEM in terms of accuracy and efficiency. In addition, compared with the BEM and MFS coupling methods, the calculation of singular integrals and the selection of fictitious boundaries are avoided completely. Therefore, the proposed methodology can be considered a competitive candidate for solving this type of problem.

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