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An ant colony algorithm applied to lay-up optimization of laminated composite plates

Abstract

Ant colony optimization (ACO) is a class of heuristic algorithms proposed to solve optimization problems. The idea was inspired by the behavior of real ants, related to their ability to find the shortest path between the nest and the food source. ACO has been applied successfully to different kinds of problems. So, this manuscript describes the development and application of an ACO algorithm to find the optimal stacking sequence of laminated composite plates. The developed ACO algorithm was evaluated on four examples for symmetric and balanced lay-up. The results of the first three cases, in which the classical lamination theory was used to obtain the structural response of rectangular plates, were compared to those obtained from the literature using genetic algorithms (GA) and other ACO algorithm. The fourth example investigates the maximization of fundamental frequencies of rectangular plates with central holes, where the structural response was obtained by finite element analysis, showing that this optimization technique may be successfully applied to a broad class of stacking sequence problems for laminated composites.

Keywords

laminated composite material; stacking sequence; ant colony optimization.

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1 INTRODUCTION

Laminated composite materials consist of stacks of layers, each layer usually composed by a matrix of polymeric material and fibers oriented in a specific direction. Figure 1 shows a scheme of a laminate composite material. Laminated composites give the designer the ability to tailor the material according to the application and the structures formed by these materials present high stiffness/mass and strength/mass ratios.

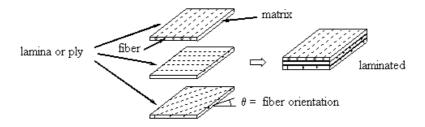


Figure 1 Laminated composite materials

The mechanical properties of composite laminates depend on the material of each layer, the number of layers, the thickness of each layer and the fiber orientations in each layer. The ply thicknesses are often predetermined and the ply orientations are usually restricted to a small set of angles due to manufacturing constraints. This leads to problems of discrete or stacking-sequence optimization. As this optimization problem is very hard to solve, several techniques have been developed. Genetic algorithms (GA) have been examined to solve these kinds of problems by different researchers in many studies [6, 9-13, 15]. Also, other techniques such as fractal branch and bound method were used by Terada et al. [14] and simulated annealing by Akbulut and Sonmez [2]. Recently, a new class of algorithms, the ant colony optimization (ACO), was developed to solve combinatorial optimization problems. ACO was inspired by the observation of the behavior of real ants (Dorigo and Socha [4]). Regarding the ant colony optimization applied to laminated composites, Aymerich and Serra [3] presented the maximization of buckling load using ACO; in Abachizadeh and Tahani [1] a multi-objective optimal design for maximization of fundamental frequency and minimization of cost is studied and Wang et al. [16, 17] proposed a modified ant colony algorithm for stacking sequence optimization of rectangular laminate and composite stiffened panels for improving buckling strength.

This paper addresses the development of an ACO algorithm applied to the lay-up design of composites plates. Four examples are presented to show the performance of the developed algorithm. In the first one the fiber orientation (predefined ply angles) and the material (glass-epoxy or carbon-epoxy layer) are considered as design variables and the material cost is minimized under the constraints of buckling and a maximum weight. In the second example a critical load factor is maximized under a constraint of a maximum of four contiguous plies with the same orientation. Only one material is used (graphite-epoxy layer), being the plies orientations the design variables. This contiguity constrain is to prevent matrix cracking [9, 10, 15]. In these two examples, rectangular plates are analyzed and the Classical Lamination Theory (see Jones [7]) is used to obtain the structural response. The results of the optimized structures are compared to those obtained from other authors, using GA and ACO. In the third and fourth examples, the ply angles of squares plates with a central hole are optimized, aiming maximizing the fundamental frequency. Plates with different diameters of the central hole are analyzed and the structural response is obtained through a commercial finite element code using Mindlin's plate theory.

The main contribution of this paper is to present an ACO algorithm that has a good performance in different optimization problems of composites lay-up design.

2 ANT COLONY OPTIMIZATION (ACO)

Ant colony optimization algorithm is a metaheuristic in which a colony of artificial ants cooperates in finding good solutions. This technique is applied to difficult discrete optimization problems (Dorigo and Stülzle [5]). Based on the communication of real ants, called stigmergy, the simulation of artificial ants in ACO was developed. When the ants walk from-and-to a food source, they deposit in the ground a chemical substance called pheromone. The quantity of pheromone on the grounds forms a pheromone trails. Artificial ants may simulate pheromone laying by modifying appropriate pheromone variables associated with problem states they visit while building solutions to the optimization problem (Dorigo and Socha [4]).

For the development of the ACO algorithm, graph G concepts are adopted (see Dorigo and Stülzle [5]). The artificial ants build solutions in stochastic constructive procedures until the connected graph G is complete (C, L), where C are the components and the all connections of components C is in the set L. In a procedure for building a graph, there are two elements associated in this algorithm step. The first one is the pheromone trail τ , associated with the components C and the connections L. It influences in the artificial ants' search process, and in the pheromone update by ants. The second is defined by a heuristic value η or heuristic information and is related to the problem information.

In this work, Ant Colony System (ACS) has been chosen as a tool for stacking sequence optimization. ACS is one of variation of ACO. This metaheuristic has a good performance to solve combinatorial optimization problems and has been successfully applied in many complex discrete problems. ACS algorithm has a framework based on three rules that manage the optimization problem. In this variant, the first procedure is the Tour Construction or pseudorandom proportional rule, defined by Dorigo and Stülzle [5] as

$$j = \begin{cases} \underset{l \in \mathcal{N}_{i}^{k}}{\arg \max} \left\{ \left[\tau_{il} \right] \left[\eta_{il} \right]^{\beta} \right\}, & q \leq q_{0}, \\ J\left(p_{ij}^{k} \right), & q > q_{0}, \end{cases}$$

$$(1)$$

where q is a random variable uniformly distributed in [0, 1], q_0 is a parameter for the best possible move $(0 \le q_0 \le 1)$, k is an ant, β is a parameter which determines the relative influence of the heuristic information, η_{il} is the heuristic information value, i, j are the initial and the next choice or candidate, l is a candidate solution, N_i^k is the feasible neighborhood of ant k, J is a random variable obtained according to the probability distribution p_{ij}^k (see Eq. (2)) in which the ant k chooses the next solution if $q > q_0$. If $q \le q_0$ it means that the ant is exploiting the learned knowledge based on the pheromone trails and the heuristic information. If $q > q_0$ the ant explores other tours or search around the best-so-far solution. The probability distribution is expressed as follows

$$p_{ij}^{k} = \frac{\left[\tau_{ij}\right]^{\alpha} \left[\eta_{ij}\right]^{\beta}}{\sum_{l \in \mathbf{N}_{i}^{k}} \left[\tau_{il}\right]^{\alpha} \left[\eta_{il}\right]^{\beta}} , \tag{2}$$

where α is a parameter which determines the relative influence of the pheromone trail. The second rule is the Global Pheromone Trail Update, which is given by

$$\tau_{ij} \leftarrow (1 - \rho) \tau_{ij} + \rho \Delta \tau_{ij}^{bs} , \quad \forall (i, j) \in T^{bs}$$
(3)

where ρ is the global pheromone evaporation rate $(0 < \rho \le 1)$, $\Delta \tau_{ij}^{bs}$ is the amount of pheromone the ant k deposits on each best-so-far solution and T^{bs} is a set of the best connections. When this rule is applied, both the evaporation and new pheromone deposit are update only to the best-so-far ant.

Local Pheromone Trail Update, the last rule, is applied during the tour construction. The pheromone evaporation and a new pheromone deposit are updated when an ant is exploiting or exploring the connection according to the pseudorandom proportional rule, given by

$$\tau_{ij} \leftarrow (1 - \xi)\tau_{ij} + \xi\tau_0 \quad , \tag{4}$$

where ξ is the local pheromone evaporation rate $(0 < \xi < 1)$, and τ_0 is the initial pheromone trails value.

The effect of the local updating rule, as explained in Dorigo and Stülzle [5], is that each time an ant uses an arc (i,j) or connection, its pheromone trails τ_{ij} is reduced and, therefore, the arc becomes less attractive for the following ants.

3 ANT COLONY SYSTEM (ACS) APPLIED TO LAMINATED COMPOSITES

The ACS algorithm must be set with some initial parameters, followed by the initial pheromone trail. Dorigo and Stülzle [5] suggest some parameter values for different kind of extensions of ACO. In this work the parameters are set following their recommendations. Table 1 shows the values used for each parameter.

Table 1 ACS parameters (present work)

Parameter	m	α	β	q_0	ξ	ρ	
Value	5	1	2	0.9	0.1	0.1	

The parameter m represents the number of ants in a tour construction. The other parameters from Table 2 are those defined by Eqs. (1), (2) and (3).

As already mentioned, the ACO is based on the construction of a graph G(C, L) that the ants build the solutions following the state rule and the roulette wheel. The initial stacking sequence is chosen randomly and the pseudorandom rule is initialized following the Eq. (1). Aymerich and Serra [3] explain that a candidate solution is then constructed step-by-step by choosing probabilistically the orientations of the stacks of the laminate. This tour construction is repeated applying the two sub-rules from Eq. (1) until the end of constructing the solution. Based on an ACO applied to a travel salesman problem (TSP) benchmarking, Wang et al. [16, 17] presented the ACO modeled as a multi city-layer system. An ant in the city i chooses the next city j probabilistically in according to Eq. (1), building the stacking sequence optimization. The reduction of amount of pheromone minimizes the risk of stagnation in local optima and the new feasible combinatorial lay-up candidates can be explored. The Global Pheromone Trail Update process focuses on the best solution from the beginning of the stacking laminate sequence. For the laminate with best orientations, available by the objective function, the amount of pheromone is deposited for this specific ant trail. Abachizadeh and Tahani [1] related that this rule, which acts as positive feedback, makes the search for the best solution more direct.

4 NUMERICAL EXAMPLES

The developed ACO algorithm is tested on three cases shown below. All the parameters of ACO algorithm for these studies are in according to Table 1 and the laminates are symmetric and balanced.

4.1. Case 1- Hybrid laminated cost minimization with buckling and weight as constraints

The problem addressed here consists in minimizing the cost of a simply support laminated composite rectangular plate, subjected to biaxial compressive loads (see Fig. 2). The laminate is symmetric and balanced and it can be made of two types of materials. The same problem was also solved by Girard [6] using a GA. The possibilities for the orientations of the plies are: 0°, +45°, -45° and 90°. The constraints are the maximum weight of 85 N and a minimum limit for the critical buckling factor (λ_{\min}).

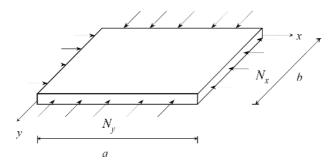


Figure 2 Plate under biaxial compressive loads (case 1)

The materials are carbon/epoxy (CE) with cost of 8 monetary unit per kilogram (U/Kg) and glass/epoxy (GE) with 1 U/Kg. The characteristics and properties of those materials are presented in Table 2.

	E_1 (GPa)	E_2 (GPa)	G_{12} (GPa)	n_{12}	Ply thickness e (mm)	Mass density ρ (Kg/m ³)
Carbon-Epoxy (CE)	138.0	9.0	7.1	0.3	0.127	1605
Glass-Epoxy (GE)	43.4	8.9	4.55	0.27	0.127	1993

Table 2 Lamina's material properties (case 1)

This problem was solved for laminates with 48, 52 and 60 plies. The plate has length a =0.9144 m and width b = 0.762 m. The ply thickness e is the same for all laminae. Gravity acceleration is considered as 9.9 m/s² as adopted by Girard [6]. The buckling load factor represents the critical buckling load divided by the applied load, and is given by (see Jones [7])

$$\lambda_{cb} = \min_{p,q} \left(\frac{\pi^2 \left[D_{11} \left(\frac{p}{a} \right)^4 + 2 \left(D_{12} + 2 D_{66} \right) \left(\frac{p}{a} \right)^2 \left(\frac{q}{b} \right)^2 + D_{22} \left(\frac{q}{a} \right)^4 \right]}{\left(\frac{p}{a} \right)^2 N_x + \left(\frac{q}{b} \right)^2 N_y} \right)$$
 (5)

where D_{ij} are the coefficients of the laminate bending stiffness matrix based on the classical lamination theory (CLT), p and q determine the amount of half waves in the x and y-direction, respectively.

The values for minimum critical buckling factor (λ_{\min}) and the biaxial compressive load applied to the laminates are shown in Table 3.

Numbers of plies	Threshold buckling factor	Load	
N_L	λ_{\min}	$N_x({ m N/m})$	$N_y \left({ m N/m} \right)$
48	150	175	175
52	250	175	175
60	375	175	175

Table 3 Minimum values for critical buckling factor and compressive load

The optimization problem can be formulated as

Find:
$$\left\{\theta^{k}, mat^{k}\right\}$$
, $\theta^{k} \in \left\{0_{2}, \pm 45, 90_{2}\right\}$, $mat^{k} = \left\{\text{CE, GE}\right\}$, $k = 1$ to n

Minimize: material cost

Subject to: $\lambda_{cb} \geq \lambda_{\min}$
 $W \leq W_{\max} = 85N$

Symmetric and balanced laminated

(6)

where θ^k is the orientation and mat^k is the material of each stack of the laminate, and n the total number of stacks. Each stack is composed of two layers with the same orientation but opposite signs (for instance, $\pm 45^{\circ}$) to guarantee balance. Also, considering the symmetry of the laminate, n corresponds to the total number of layers divided by four $(n = N_L/4)$. λ_{cb} represents the critical buckling factor and W is the weight of the laminate in Newtons (N). To respect the symmetry condition, it is only operated with the half of the plies of the laminate, the other half being with the same angles. For the stacking sequence balance, the laminate is made considering always two contiguous laminae, in which the angles are the same but with opposite signs (for example, $+45^{\circ}$ and -45° , assigned for ± 45). It notices that for the angles of 0° and 90° , its respective negative pairs are equals, so its representation is 0_2 and 90_2 , respectively.

The constraints on λ_{cb} and W are taken into account by penalization. The penalizations and the evaluation function had been adapted from Girard [6] and are given by

$$F(\mathbf{x}) = \begin{cases} f(\mathbf{x}) - \Delta b \, g_{sum} & \text{if} \quad g_{\min} = 0 \\ f(\mathbf{x}) + \left| \Delta p \, g_{\min} \right| & \text{if} \quad g_{\min} \neq 0 \end{cases}$$

$$g_{\min} = \min\left(0; \, g_1(\mathbf{x}); \, g_2(\mathbf{x})\right)$$

$$g_{sum} = g_1(\mathbf{x}) + g_2(\mathbf{x})$$

$$g_1(\mathbf{x}) = \frac{\lambda_{cb}}{\lambda_{\min}} - 1 + \Delta g$$

$$g_2(\mathbf{x}) = 1 - \frac{W(\mathbf{x})}{W_{\max}} + \Delta g$$

$$\Delta b = 0.000001$$

$$\Delta p = 1$$

$$\Delta g = 0.01$$
(7)

where ${\bf x}$ represents the vector with the design variables, $f({\bf x})$ is the function to minimize, in this case the cost, g_1 is the constraint equation for λ_{cb} , g_2 is the constraint equation for W, Δb is a bonus factor, Δp is a penalization factor, Δg is a relaxation factor on constraints and $F({\bf x})$ the penalized function. It is worthwhile to notice that the constraint violation, already considering the relaxation, occurs when $g_1 < 0$ and/or $g_2 < 0$. The condition $g_{\min} = 0$ corresponds that both the constraints are satisfied, thus a bonus is applied to the function. Otherwise it is penalized.

Table 4 shows the stacking sequence of the laminated plate obtained by Girard [6] using a GA and the comparative results obtained by the ACO of the present work. In the stacking sequences, the figures that are underlined correspond to the layers of glass-epoxy (GE), and the remaining figures to carbon-epoxy (CE). \bar{n}_f represents the average number of objective function evaluations needed to find the best solution and SD the correspondent standard deviation over 100 independent runs.

N_L	λ_{\min}	Method	Stacking sequence	Material (CE/GE)	Cost (U)	Weight (N)	λ_{cb}	$\overline{n}_f(\mathrm{SD})^*$
48	150	GA	$[\pm 45_3 \ / \ \pm 45_9]_s$	12/36	19.99	79.73	165.56	23945
52	250	GA	$[\pm 45_6 \ / \ \pm 45_7]_s$	24/28	32.21	82.64	259.47	27345
60	375	GA	$[\pm 45_{15}]_{\rm s}$	60/0	68.19	84.39	442.79	24984
48	150	ACO	$[\pm 45_3 \ / \ 90_4 \ /$	12/36	19.98	79.73	190.23	23744
			$\pm 45_7]_{ m s}$					(1080)
52	250	ACO	$[90_2 \ / \ \pm 45_5 \ / \ 90_2$	24/28	32.21	82.63	283.09	25581
			$/$ $\pm 45_5$ $/$ $0_2]_{ m s}$					(1225)
60	375	ACO	$[\pm 45_{15}]_{\rm s}$	60/0	68.16	84.36	443.02	25039
								(935)

Table 4 Comparative ACO (present work) versus GA (Girard [6])

Analyzing the stacking sequences obtained by ACO and GA, both algorithms found the best configurations with carbon/epoxy plies on the external surfaces. These configurations satisfy the constraints of buckling, due to a higher bending stiffness and lower weight. Although the results for angle orientations present small difference between the two methods, the stacking sequence of materials and respective obtained costs reached similar results. For laminated plate with 60 layers, the stacking sequence results found by ACO and by GA are identical. Considering laminates with 48 and 52 plies, the stacking sequence obtained by ACO did not find all orientations angles in ±45° and the critical buckling factor were also slightly higher than GA. The numbers of evaluations of the objective function were also similar for either methods, being slightly lower for the ant colony optimization in two situations ($N_L = 48$ and $N_L = 52$).

4.2. Case 2 - Critical load factor maximization

This problem was also investigated by Aymerich and Serra [3]. They applied an ant colony optimization metaheuristic to find the lay-up design of laminated plates aiming the maximization of the critical load factor λ_c . The optimization problem can be defined as

Find:
$$\theta^k , \quad \theta^k \in \left\{0_2, \pm 45, 90_2\right\}, \ k{=}1 \text{ to } n$$

$$Maximize: \quad \lambda_c = \min(\lambda_{cb}, \lambda_{cf})$$

$$- Max. \ of \ 4 \ contiguous \ plies \ with \ the \ same \ orientation$$

$$- Symmetric \ and \ balanced \ laminated \equiv{(8)}$$

^{*} Girard [6] did not present the standard deviation of \overline{n}_f for the GA algorithm.

where λ_{cb} is the critical buckling load factor (see Eq. (5)), λ_{cf} is the critical failure factor based on allowable strains (defined further), θ^k is the orientation of each stack and n is the total number of stacks. As presented in Eq. (8), the critical failure factor λ_c is the smallest between λ_{cb} and λ_{cf} .

The critical failure load factor λ_{cf} is defined as

$$\lambda_{cf} = \min_{k} \left[\min \left(\frac{\varepsilon_1^u}{S_f | \varepsilon_1^k|}, \frac{\varepsilon_2^u}{S_f | \varepsilon_2^k|}, \frac{\gamma_{12}^u}{S_f | \gamma_{12}^k|} \right) \right]$$
(9)

where ε_i^u , i=1,2 and γ_{12}^u , are the ultimate strains; ε_i^k , i=1,2 and γ_{12}^k are the strains in the material principal directions of the kth lamina and S_f is the safety factor, set as 1.5. The material properties of the layers are presented in Table 5.

Table 5 Graphite-epoxy lamina's properties (case 2)

Elastic material properties				Ultimat	te strains	
E_1 (GPa)	E_2 (GPa)	G_{12} (GPa)	v_{12}	$\mathbf{\epsilon}_1^u$	$oldsymbol{arepsilon}_2^u$	${\color{gray}\gamma_{12}^u}$
127.59	13.03	6.41	0.3	0.008	0.029	0.015

The characteristics of the plate analyzed in this case, which are composed only by graphite/epoxy material are exposed in Table 6. The N_u loading corresponds to 25% of N_x .

Table 7 shows the results obtained for this case, compared to the ones obtained by Aymerich and Serra [3]. Despite the fact that optimal stacking sequences are different from those obtained by Aymerich and Serra [3], the correspondent load factors λ_{cb} and λ_{cf} are quite close. The differences are less than 2.5%.

Table 6 Geometric characteristics and loading of the laminated plate (case 2)

Geometry				Loading	
Number of plies	Thickness	Length	Width	$N_x \left({ m N/m} \right)$	$N_{y}\left(\mathrm{N/m}\right)$
N_L	t (mm)	$a~(\mathrm{mm})$	b (mm)	1.1 (1./ 111)	1.9 (1.711)
48	0.127	508	127	175	43.75

Stacking Sequence	Reference	Load Factor	
		Buckling (λ_{cb})	Failure ($\lambda_{\!c\!f}$)
$[\pm 45_2/90_2/\pm 45_3/0_2/\pm 45/0_4/\pm 45/0_2]_s$	ACO (reference [3])	12743.45	12678.78
$[\pm 45/90_2/\pm 45_4/(0_2/\pm 45/0_2)_2]_{\mathrm{s}}$	ACO (reference [3])	12725.26	12678.78
$[90_2/\pm 45_5/(0_2/\pm 45/0_2)_2]_{ m s}$	ACO (reference [3])	12674.85	12678.78
$[\pm 45_3/90_4/\pm 45_2/0_2/\pm 45/0_4]_s$	ACO (present work)	12459.75	12690.69
$[90_2/\pm 45_4/(0_2/\pm 45)_3/0_2]_{ m s}$	ACO (present work)	12418.12	12690.69
	ACO (present work)	12634.43	12690.69

Table 7 Comparison ACO (present work) versus ACO (Aymerich and Serra [3])*

4.3 Case 3 - Maximization of the Fundamental Frequency

The purpose of the problems presented in this section is to find the optimal stacking sequence to maximize the fundamental frequency of composite laminated plates made of graphite-epoxy. The optimization problem is formulated as

Find:
$$\theta^k$$
, $\theta^k \in \{0_2, \pm 15, \pm 30, \pm 45, \pm 60, \pm 75, 90_2\}$, $k=1$ to n

Maximize: ω (fundamental frequency)

Subject to: Symmetric and balanced laminate (10)

Two kinds of structures are analyzed. First (case 3.1, subsection 4.3.1), rectangular plates with different ratios length/width are optimized. Second (case 3.2, subsection 4.3.2), square plates with a central hole of diameter D are analyzed, where different values for D are considered.

In case 3.1, during the optimization process, the structural response of the plates is obtained using the Classical Lamination Theory (CLT). Next, the optimal lay-up design found is checked using a finite element method (FEM) code, aiming to validate the FEM results and the analytical responses. In case 3.2, the structural responses are computed using the FEM code.

4.3.1 Case 3.1 - Rectangular Plates

For rectangular plates, using the Classical Lamination Theory, the fundamental frequency is calculated by (Jones [7])

$$\omega_{mn}^2 = \frac{\pi^4}{\rho h} \left[D_{11} \left(\frac{m}{a} \right)^4 + 2 \left(D_{12} + 2D_{66} \right) \left(\frac{m}{a} \right)^2 \left(\frac{n}{b} \right)^2 + D_{22} \left(\frac{n}{b} \right)^4 \right]$$
(11)

where ω_{mn} is the natural frequency of the vibration mode (m, n), h is the total thickness of the laminate and ρ is the mass density average over the thickness direction, which is given by

^{*} problem based on Kogiso et al. [8] using GA search procedures.

$$\rho = \frac{1}{h} \int_{-h/2}^{h/2} \rho^{(k)} \, \mathrm{d}z = \frac{1}{NL} \sum_{k=1}^{NL} \rho^{(k)} \,, \tag{11}$$

where $\rho^{(k)}$ is the mass density of the material in the k-th layer and N_L is the total numbers of plies of the laminate.

The geometric characteristics of the laminates are presented in Table 8. The ratio a/b varies from 0.2 to 2 with increment of 0.2. The material properties of graphite-epoxy layers (T300/5208) are reported in Table 9.

Table 8 Laminated plate characteristics

Number of plies	Ply thickness	Length/Width	Width
N_L	e (m)	a/b	<i>b</i> (m)
8	2.5×10^{-4}	0.2 to 2.0	0.25

Table 9 Graphite-epoxy layer's properties (cases 3.1 and 3.2)

Elastic mat	Mass density			
E_1 (GPa)	E_2 (GPa)	G_{12} (GPa)	u_{12}	$r \ ({ m Kg/m}^3)$
181	10.3	7.17	0.28	1600

The results obtained with the ACO algorithm developed in this work are presented in Table 10. This table also shows the results provided by Abachizadeh and Tahani [1] for the same case, also using ACO. The last two columns of the Table 10 show the value of the fundamental frequencies obtained by the finite element code for the optimal design and the percentage difference between the numerical and analytical results.

Table 10 Optimum stacking sequence for maximum fundamental frequency (case 3.1)

	ACO - Abachizadeh and Tahani [1]		ACO – Pre	sent Work	Finite elements	D:ff
a/b	Stacking sequence	ω (rad/s)	Stacking	ω	ω (rad/s)	Difference %
		(====/=)	sequence	(rad/s)	(====/=)	, ,
0.2	$[0]_{4s}$	24390	$[0]_{4s}$	24389.88	24150.68	0.98
0.4	$[0]_{4s}$	6170	$[0]_{4s}$	6170.01	6138.48	0.51
0.6	$[\pm 15]_{2s}$	2801	$[\pm 15]_{2s}$	2801.00	2772.14	1.03
0.8	$[\pm 30]_{2s}$	1797	$[\pm 30]_{2s}$	1797.21	1745.66	2.87
1.0	$[\pm 45]_{2s}$	1413	$[\pm 45]_{2s}$	1413.00	1362.76	3.56
1.2	$[\pm 45]_{2s}$	1189	$[\pm 45]_{2s}$	1189.11	1148.69	3.4
1.4	$[\pm 60]_{2s}$	1078	$[\pm 60]_{2s}$	1077.86	1051.49	2.45
1.6	$[\pm 75]_{2s}$	1016	$[\pm 75]_{2s}$	1016.42	1007.89	0.84
1.8	$[90]_{4s}$	1003	$[90]_{4s}$	1002.46	1001.29	0.12
2.0	$[90]_{4s}$	996	$[90]_{4s}$	996.32	995.19	0.11

4.3.2 Case 3.2 - Square Plates with Central Hole

Consider a composite square plate with side dimensions of 0.25 m and a central hole of diameter D, as shown in Fig. 3. The diameter varies between 0.0 to 0.16 m by an increment of 0.04 m. The parameters are identical to those of previous simulations. Table 11 shows the results for this problem. Optimal solution matched \pm 45° for all layers as expected due to symmetry of the problem and geometry, therefore the results are consistent. $\overline{n}_{\!\scriptscriptstyle f}$ represents the average number of objective function evaluations that the algorithm needed to find the best solution and SD the correspondent standard deviation. Figure 4 presents the first vibration mode for a plate with diameter D = 0.04.

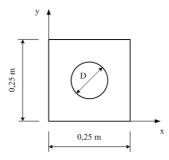


Figure 3 Square plate with a central hole

Table 11 Optimal results obtained for case 3.2

$D\left(\mathbf{m}\right)$	Stacking sequence	w (rad/s)	$\overline{n}_f \left(\mathrm{SD} \right)$
0.00	$[\pm 45]_{2s}$	1362.76	13.3 (1.0)
0.04	$[\pm 45]_{2s}$	1350.26	12.3(1.0)
0.08	$[\pm 45]_{2s}$	1409.70	9.0 (0.8)
0.12	$[\pm 45]_{2s}$	1632.75	10.5~(0.6)
0.16	$[\pm 45]_{2s}$	2138.23	14.8(0.5)

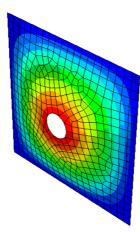


Figure 4 The first vibration mode of a plate with a central hole of diameter $D=0.04\mathrm{m}$

5 CONCLUSIONS

In this paper, the Ant Colony Optimization (ACO) was applied to the optimization of laminated composite plates. The formal representation that the artificial ants use in a problem formulation for ACO was applied to the analysis of laminated composite plates. Many numerical simulations were carried out and three cases are selected to evaluate the performance of metaheuristic procedures. The tests were divided in two parts. First, the developed ACO algorithm was compared with other optimization techniques. Finally, free vibration analyses were carried out using ACO and a FEM code to solve laminated composites with complex geometry. Based on this formulation, the ACO was implemented to minimize the cost or maximize the buckling load factor, considering the constraints. ACO was also applied to rectangular and squared composite plates to obtain their highest fundamental frequencies, which were used as reference data. The stacking sequence was obtained from ACO and simulated in a commercial finite element code. Subsequently, the ACO algorithm, developed in Matlab, was coupled with a FEM code. Finally, composite square plates with central holes were simulated, and the results agreed with those expected for this geometry. Based on these numerical cases, the ACO has been successfully applied in the layup composite laminate plate optimization and the proposed connection approach provides an efficient tool to optimize complex laminated composite structures.

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