

# Exergy and Energy Analysis of a Tubular Solar Still with and without Fins: Comparative Theoretical and Experimental Approach

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## Research Article

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1 **Exergy and energy analysis of a tubular solar still with and without fins: Comparative**  
2 **theoretical and experimental approach**

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21 **Abstract**

22 Today, availability of clean water is hard as the residents are expanding and moving  
23 fast to achieve rapid urbanization as a result need for clean water has been raised. Solar stills  
24 are the solution to desalinate to obtain pure water. This paper represents the theoretical and  
25 experimental study of tubular solar still with and without fins. The reading was recorded from  
26 8:00 AM to 6:00 PM. Efficiency of TSS with fins and without fins are 23.39% and 13.76%  
27 respectively. The rate of irreversibility from the basin of TSS with flat is higher than TSS with  
28 finned absorber. Similarly, the rate of irreversibility from water is significantly reduced using  
29 finned absorber. Also, the exergy efficiency of TSS with finned absorber is higher compared  
30 to TSS with flat absorber.

31 **Keywords:** Tubular solar still; fins; desalination

32 **1. Introduction**

33 Nearly about 780 Million of people lack access to water around the world. It is predicted  
34 that 50% of the world population will suffer from water scarcity in 2050. To fulfill the necessity

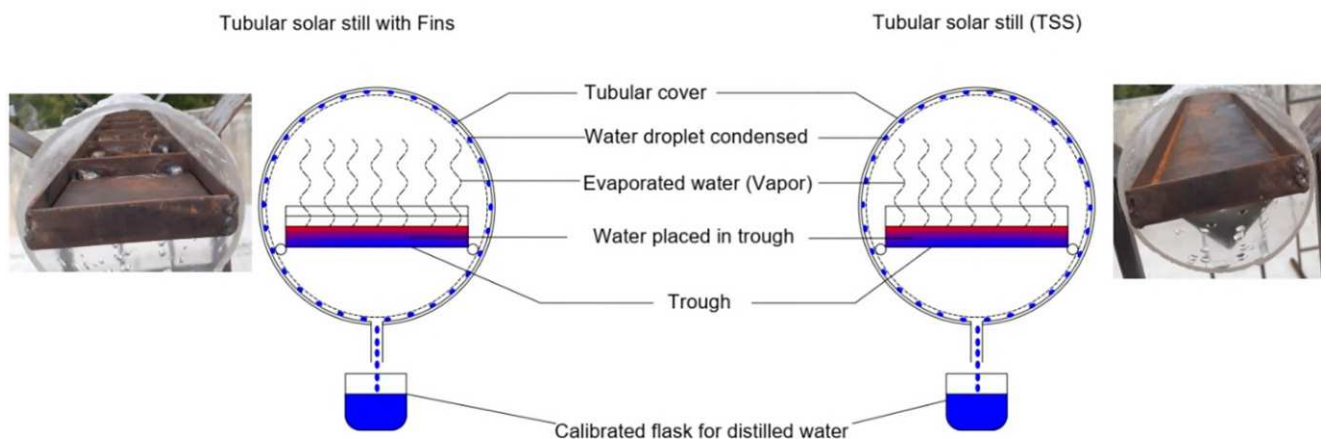
35 of desalination of water can be done .Solar stills can be used to remove salt from the sea water,  
36 solar still desalination is free from pollution and provides high quality pure form of water. The  
37 setup of this solar still desalination can be used in residential areas to overcome water scarcity  
38 in the urban areas. The drawback of solar stills is the low productivity rate and require constant  
39 sunlight for the process to be done effectively (Abdelgaied et al. (2020); Muthu Manokar et al,  
40 (2020); Balachandran et al. (2020); Attia et al. (2020, 2021); Muthu Manokar et al. (2020);  
41 Kumar et al. (2020); Sharshir et al. (2020); Essa et al. (2020); Fath et al. (2003)). For a single  
42 slope and pyramid shaped Fath et al. (2003) carried out a thermo economic analysis. Their  
43 study revealed that, the performance of traditional solar still was significantly higher than the  
44 solar still in the shape of a pyramid. There is an increase of about 30% in the fresh water  
45 produced using conventional solar still than pyramid shaped solar still. Abu-Arabi and Zurigat  
46 (2005) performed a simulation on double glass cover cooling with regenerative effect and  
47 conventional solar still. Their results revealed that the regenerative effect from double glass  
48 cover cooling improved the cumulative yield by 70% than a single slope conventional solar  
49 still. Arunkumar et al. (2013) used a concentric tubular solar still and in addition to that a  
50 parabolic concentrator is attached to focus the incoming solar radiation. Additionally, air- and  
51 water-cooling methods were employed to reduce the cover temperature. Through improved  
52 cooling of air and water flow the productivity of water produced was increased from 2050  
53 ml/day to 3050 ml/day. With continuous cooling water flow in the concentric tube, the fresh  
54 water produced was furthermore increased to 5000 ml/day. Arunkumar et al (2016) enhanced  
55 the productivity of compound parabolic concentrator tubular solar stills. A saline water trough  
56 of rectangular shape was designed and fabricated and this trough is attached along with the  
57 pyramid type and single slope solar still. The integrated solar still has produced an accumulated  
58 yield of 7770 ml/day whereas, the single slope solar still produced a maximum cumulative  
59 yield of 6460 ml/day. Kabeel et al (2019) improved the performance of the tubular solar still  
60 by controlling the cover cooling and water depth. It is found that lowering the water depth  
61 increases the performance, by this the productivity of fresh water rate reached a maximum  
62 value of 5.85 L/m<sup>2</sup>. Elashmawy (2019) describes the performance of the high temperature  
63 stand-alone tubular solar still by changing the thickness and surface cooling. By reducing the  
64 thickness by 40% the productivity and efficiency had been enhanced by 21% and 13.35%  
65 respectively. Elashmawy (2017) conducted three experiments using tubular solar still namely  
66 rectangular trough with a black cloth, half cylindrical trough without cloth and parabolic  
67 concentrator-solar tracking system integrated half cylindrical trough without cloth. The daily  
68 yield is about 4.71,3.6 and 3.53L/m<sup>2</sup> day. Panchal (2015) have proved that the combined

69 application of both black granite gravel and vacuum tubes increased the DBSS fresh water  
70 productivity to 65% and with the application of vacuum tubes alone in DBSS enhanced the  
71 fresh water productivity by 56%. Panchal and Thakkar (2016) had validated the thermal and  
72 experimental analysis carried out on solar still directly coupled with evacuated tubes during  
73 summer and winter climatic conditions. They concluded that the introduction of evacuated  
74 tubes and Polyurethane Foam type insulation material to the experimental model enhanced the  
75 distillate output and also helps in reducing heat loss. Rahbar et al. (2015) proposed new  
76 correlations to predict the fresh water produced and heat transfer coefficient of an TSS using  
77 CFD and theoretical approach. From the characteristic curve of their study, it can be concluded  
78 that on lower cover temperature and higher water temperature, the yield from TSS was higher.  
79 Sarhaddi et al. (2017) carried out experiments on a weir cascade solar still by incorporating  
80 PCM energy storage to estimate the energy and exergy under clear sky condition and semi-  
81 cloudy condition. From the results of exergy and energy analysis, it has been summarized that  
82 the still with PCM is preferred for semi-cloudy days and still without PCM is suitable for sunny  
83 days. Experiments conducted on a typical sunny day with clear sky revealed that the exergy  
84 efficiency of solar still without PCM was slightly lower than semi-cloudy days, whereas, the  
85 energy efficiency reduced using PCM during semi-cloudy condition as it affects the melting  
86 process of PCM beneath the basin. Shanmugam et al. (2018) conducted experiments to study  
87 the yield enhancement of solar still by incorporating nanoparticles and PCM in the basin of the  
88 still model. The distillate yield of SB-SS with wick material by nanoparticles as FWCW and  
89 PCM is 4.120 and 7.460 kg/m<sup>2</sup> day. Sharshir et al. (2016) studied the performance of  
90 continuous solar desalination model comprised of HDH unit and SS with an evacuated solar  
91 water heater unit. The experimental study shows that the distillate productivity of the SS with  
92 exit warm water from HDH is 242% higher than the CSS system and there is about 39% rise  
93 in the gain output ratio. The effect on forced convection on cover cooling of pyramid solar still  
94 was experimentally carried out by Taamneh & Taamneh (2012). A small DC powered fan was  
95 used to cool the entire cover surface. Experimental results revealed that an improvement in  
96 daily freshwater yield of about 2.99 litres per day (25%) is achieved using with forced  
97 convection which is higher compared to the free convection still. Bhaskar and Rai (2018)  
98 investigated the productivity and exergy analysis of tubular solar still operated in active and  
99 passive mode individually. This study showed that the daily fresh water yield of the TSS in  
100 active mode is 52% more than the passive mode and also the TSS with fan had exergy  
101 efficiency of about 133% higher than the TSS in passive mode. Xie et al. (2016) have designed  
102 and constructed a novel conceptual design of low temperature- multi effect desalination system

103 that comprises an array of Tubular Solar Still capable of producing freshwater independently  
 104 to investigate the performance affecting parameters- vacuum pressures, heating conditions and  
 105 evaporation temperatures. Panchal and Mohan (2017) presented a cost effective optimized  
 106 solar still model with its different approaches of augmenting the productivity of solar still by  
 107 adding some modification such as fins, increasing the number of effects and adding energy  
 108 absorbing materials inside the basin.

109 The effect of humid air present in the tubular enclosure on heat and mass transfer was  
 110 experimentally studied by Ahsan and Fukuhara (2010). Other similar configuration of solar  
 111 still includes improving the exposure area by attaching hollow and solid fins. In addition to  
 112 the fins, additional materials such as wick material, PCM, ethanol, solar pond, etc. can further  
 113 increase the fresh water. The proposed model provided new outputs for the tubular solar still.  
 114 Finally, it is concluded that the daily and hourly production of the Tubular solar still can be  
 115 accurately predicted.

116 **2. Experimental setup and procedure**



117 Fig. 1. Schematic diagram and experimental test rig of TSS with fins (left) and without fin  
 118 (right)  
 119

120 The schematic diagram and experimental test rig photograph of TSS with fins (left) and the  
 121 TSS without fins (right) is depicted in Fig 1. This experimental setup consists of a transparent  
 122 tube made up of glass, a steel rectangular water basin called trough and a calibrated flask to  
 123 collect the freshwater produced. The glass tube allows the penetration of solar irradiance from  
 124 any direction which helps in augmenting the evaporation process in this desalination system.  
 125 The trough containing saline water is placed in the transparent glass tube. The trough is coated  
 126 in black color in order to reduce the reflection of solar irradiance by absorbing all the solar  
 127 irradiance transmitted through the outer transparent glass tube. Solar thermal heat produced by  
 128 the solar radiation are absorbed by the saline water in the trough. As a result of heating, the

129 saline water gets heated and evaporated. The evaporated water vapour get condensed on the  
130 inner surface of the glass tube due to the release of latent heat of evaporation. The condensed  
131 water flows down by the effect of gravity and collected at the bottom of tube as a freshwater.  
132 Two experimental models of TSS in which one of the models having fins attached with trough  
133 and another without fins in trough are used and a comparative experimental study between the  
134 two models is carried out. Fins used in the trough helps in boosting the desalination process  
135 because of the increased surface area of absorber and enhanced greenhouse effect within the  
136 still.

### 137 **3. Results and discussion**

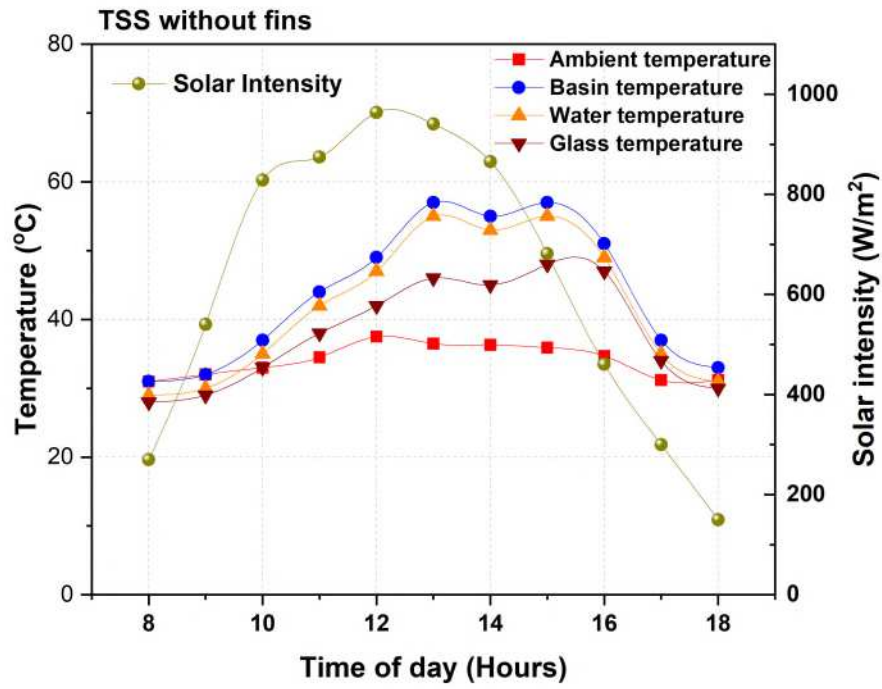
138 The experimental data recorded such as solar radiation, ambient temperature, cover, basin, and  
139 water temperature from the modified tubular solar still using flat and finned absorber is  
140 presented in this section with a detailed discussion. Using the empirical correlations, the  
141 instantaneous thermal and exergy efficiencies are determined. In addition to the predicted yield  
142 is correlated to the experimental results obtained from the study. Furthermore, a comparison of  
143 different solar still using fins, and phase change materials were made to justify the present  
144 experimental investigation.

#### 145 **3.1. Thermal analysis**

146 In this section, a comparative interpretation between the theoretical and experimental  
147 study of TSS with and without fins are carried out. The hourly variation of operating parameters  
148 for TSS with and without fins including solar intensity, glass temperature, basin temperature,  
149 water temperature and ambient temperature are plotted in graph as shown in Fig. 2 and Fig. 3.  
150 Maximum solar intensity of about  $963.7 \text{ W/m}^2$  was attained at midday and its starts decreasing  
151 gradually. While solar intensity starts decreasing during evening, the temperature of TSS's  
152 glass, basin and water starts increasing around evening. The TSS integrated with fins reacts  
153 faster and higher to solar intensity than the TSS without fins. The distillate output rate of any  
154 solar still is determined by the temperature of water inside still and the performance of still is  
155 also depend upon many factors such as air temperature inside still(cavity between the basin  
156 and glass cover area), lower glass cover temperature, absorber plate temperature, surface area  
157 of absorber etc.,

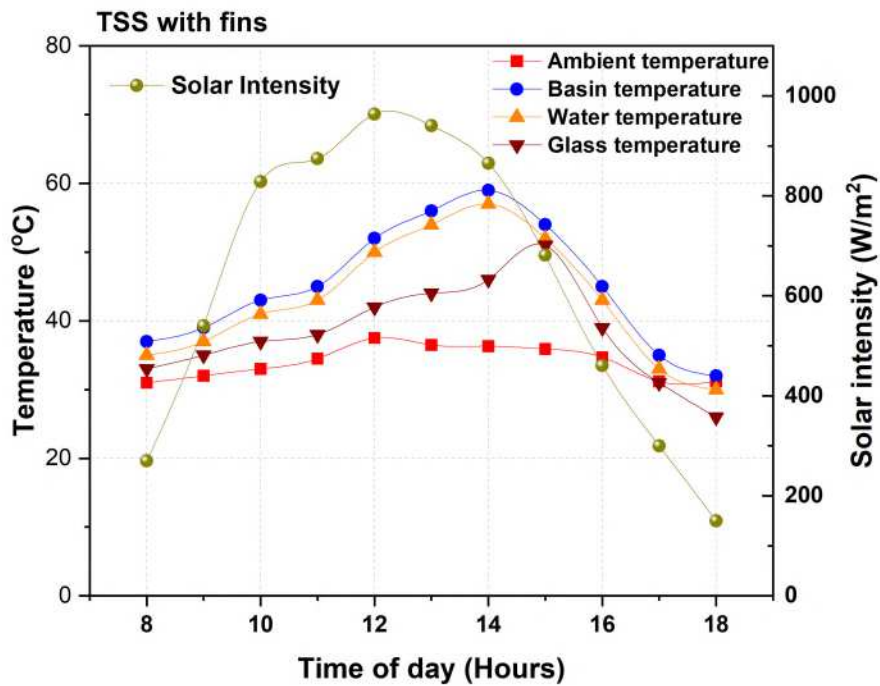
158 Fig. 2 shows the solar intensity of TSS without fins are measured starting from 8:00  
159 AM – 6:00 PM are presented in the above figure. Solar intensity ranges from  $150 \text{ W/m}^2$  -  
160  $1000 \text{ W/m}^2$ . At the beginning of the day, solar intensity of is about  $270 \text{ W/m}^2$  and the  
161 temperature is about  $20 \text{ }^\circ\text{C}$ . The ambient, basin, water, glass temperature are about 31, 31, 29,

162 28 °C and their solar intensity is about 400 W/m<sup>2</sup>-450 W/m<sup>2</sup>.Solar intensity reaches as high as  
 163 963.7W/m<sup>2</sup> at the noon time and the ambient temperature is about 37.5 °C. The ambient, water,  
 164 glass temperature reaches its peak at around 3pm and then gradually decreases, also the solar  
 165 intensity drops to 150 W/m<sup>2</sup> around 6:00 AM.



166  
 167 Fig. 2 Hourly variation in solar intensity, ambient, basin, water, and glass temperature  
 168 recorded from TSS without fins





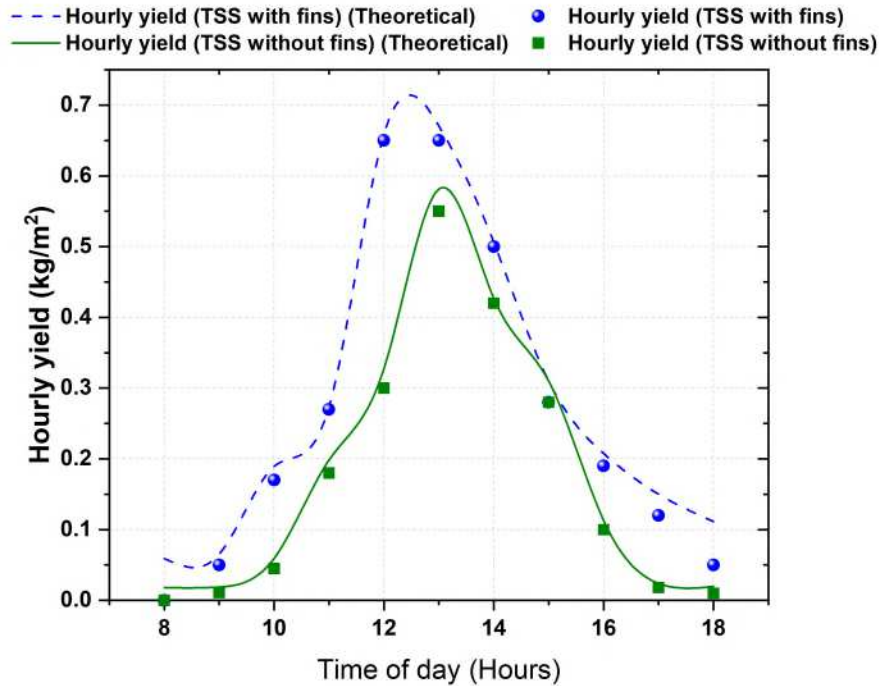
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170 Fig. 3 Hourly variation in solar intensity, ambient, basin, water, and glass temperature

171 recorded from TSS with fins

172 Fig. 3 shows the solar intensity of TSS with fins, the ambient temperature is lower due to lower  
 173 solar intensity in the day time. so that the basin, water and glass temperature remains lower.

174 The ambient temperature increases from 9:00 AM to 12:00 PM, while the ambient temperature  
 175 reaches its maximum value of 37.5 around 12:00 PM. The solar intensity increases and reaches  
 176 a maximum value of 963.7 W/m<sup>2</sup> during the noon. The basin and water temperature attains their  
 177 maximum temperature around 59, and 57 °C at 2:00 PM. The glass temperature reaches a  
 178 maximum value of 51 °C at 3:00 PM. In noon time the ambient temperature decreases to 31.2  
 179 °C so that the solar intensity also decreases and reaches a lower value of 150 W/m<sup>2</sup>. Thus, the  
 180 basin, water and glass temperature also decrease.



181

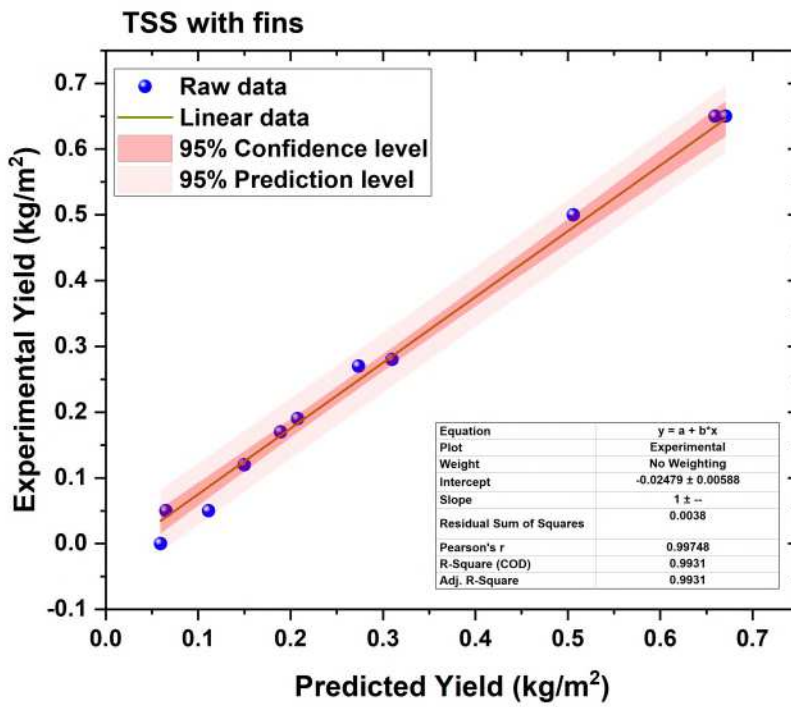
182 Fig. 4 Hourly variation of theoretical and experimental distillate yield for TSS with and  
 183 without fins.

184 The experimental and theoretical results on hourly fresh water production from TSS  
 185 using flat and finned absorber is plotted in Fig. 4. It is seen from Fig. 4 that using a flat absorber,  
 186 the hourly fresh water produced is lower as compared to finned absorber. There is a gradual  
 187 increase in the yield from the sun rise and reaches the maximum during the peak solar intensity.  
 188 It is also seen that the experimental and theoretical yield are in agreement in both the cases.  
 189 From the Fig. 4, it is clear that the theoretical distillate yield is always greater than the  
 190 experimental distillate yield from the TSS. The maximum theoretical freshwater yield value  
 191 achieved by the TSS with fins is  $0.67 \text{ kg/m}^2$  and by the TSS without fins is  $0.58 \text{ kg/m}^2$ . The  
 192 maximum experimental freshwater yield value achieved by the TSS with fins is  $0.65 \text{ kg/m}^2$  and  
 193 by the TSS without fins is  $0.55 \text{ kg/m}^2$  which shows that the fins present in the TSS will naturally  
 194 augment the freshwater yield due to the enhanced surface area of absorber. These fins increased  
 195 the rate of absorption of heat in the basin due to the increased surface area in the basin by the  
 196 water. The presence of fins in the basin furthermore distributes the heat throughout the water  
 197 for augmenting the rate of evaporation from the surface of water. With simultaneous increase  
 198 in the rate of evaporation inside the enclosure, the amount of water produced from the solar  
 199 still is increased.

200

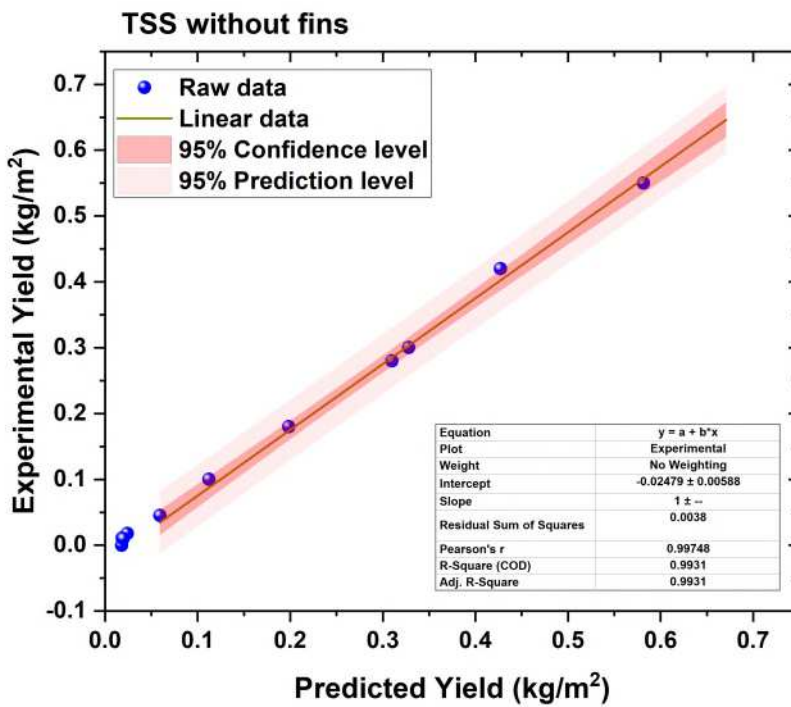
201 Fig. 5 and 6 shows the results of predicted and measured yield of TSS without and with  
 fins on the absorber respectively. The predicted yield from solar still is measured using

202 Equation (1) to (3). It can be seen that the experimental yield produced from the TSS in well  
 203 agreement with the predicted yield with a confidence level of 95%.



204  
 205

Fig. 5 Predicted and measured yield of TSS with fins



206  
 207  
 208

Fig. 6 Predicted and measured yield of TSS without fins

209 The hourly yield from the tubular solar still under both the cases can be mathematically  
210 expressed as [43],

$$211 \quad m_e = \frac{h_e \times A_w \times (T_w - T_g)}{h_{fg}} \quad (1)$$

212 The influential parameter for determining the yield of fresh water from solar still were partial  
213 difference in pressure, evaporative heat transfer coefficient, temperature difference and  
214 convective heat transfer coefficient as mathematically expressed in Equation (2) and (3).

215 Mathematically, the EHTC is estimated as (Shukla and Sorayan (2005)),

$$216 \quad h_e = \frac{16.273 \times 10^{-3} \times h_c \times (P_w - P_g)}{(T_w - T_g)} \quad (2)$$

217 In the similar way, the CHTC is mathematically expressed as (Shukla and Sorayan (2005)),

$$218 \quad h_c = 0.884 \left\{ (T_w - T_g) + \frac{(P_w - P_g)(T_w + 273.15)}{(268.9 \times 10^{-3} - P_w)} \right\}^{1/3} \quad (3)$$

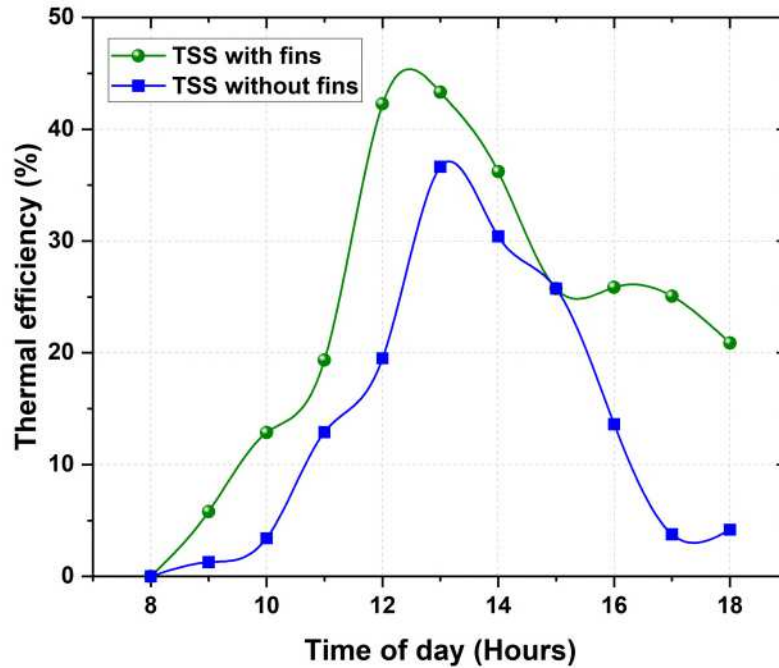
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220

Table. 1 Daily yield of different research work done in TSS by various researchers

S. No	Literature	Study	Fresh water produced	Location
1	Fath et al (2003)	Pyramid and single slope solar still	2.6 l/m <sup>2</sup> d	Aswan, Egypt
2	Abu-Arabi and Zurigat (2005)	Regenerative solar still	4.15kg/m <sup>2</sup>	Marmul, Oman
3	Arunkumar et al (2013)	Cover cooling of tubular solar still with water and air medium	5000 ml/day	Coimbatore, India
4	Arunkumar et al (2016)	Parabolic concentrators on tubular solar still	7770ml/day	Coimbatore, India
5	Kabeel et al (2019)	Tubular solar still with cover cooling – effect on water depth	5.85 l/m <sup>2</sup>	Tanta, Egypt
6	Elashmawy (2019)	Tubular solar still with cover cooling technique	2.4 l/m <sup>2</sup>	Hail, Saudi Arabia
7	Elashmawy (2017)	Tubular solar still with parabolic concentrator	4.21 l/m <sup>2</sup>	Hail, Saudi Arabia
8	Panchal (2015)	ETC integrated double slope solar still	-	Patan, India
9	Panchal & Thakkar (2016)	ETC integrated solar still	0.81kg	Patan, India
10	Rahbar et al. (2015)	Computational analysis on tubular solar still – CFD approach	0.99kg/m <sup>2</sup> h	-
13	Sarhaddi et al. (2017)	Weir cascaded solar still	1.08kg/m <sup>2</sup> .h	Zahedan,Iran
14	Shanmugan et al. (2018)	Nano coated absorber plate and PCM	7.46 kg/m <sup>2</sup> (summer) 4.12 kg/m <sup>2</sup> (Winter)	Chennai, India
15	Sharshir et al. (2016)	Continuous desalination using wick and shallow reservoir solar still	37 l/day	Kafrelshiekh, Egypt
16	Taamneh & Taamneh (2012)	Pyramid type solar still	2.99 l/per	Mashad,
17	Bhaskar & Rai (2018)	Tubular solar still	0.168 L	Allahabad, India
18	Xie et al (2016)	Multi stage tubular solar still	0.40 kg/hr	Chengdu, China
19	Panchal & Mohan (2017)	Methods adopted in finned solar still	1.05 kg/m <sup>2</sup> h	-
20	Ahsan & Fukuhara (2010)	Tubular solar still	NA	Fukui, Japan
21	Rabhi et al (2017)	Pin fins with external condenser	3.49 kg/m <sup>2</sup>	Gafsa-Tunisia
22	El-Sebaili & El-Naggar (2017)	Finned single slope solar still	5.4 kg/m <sup>2</sup>	Tanta, Egypt
23	El-Sebaili et al (2015)	Fin configuration on solar still	5.37 kg/m <sup>2</sup>	Tanta, Egypt
24	Velmurugan et al (2008)	Single basin solar still with fin for enhancing productivity.	2.81 kg/m <sup>2</sup>	Madurai, India
25	Velmurugan et al (2008a)	Industrial effluent desalination using fins in solar still	2.77 kg/m <sup>2</sup>	Madurai, India
26	Rajaseenivasan & Srithar (2016)	CO <sub>2</sub> mitigation on solar still using square and circular fins	4.55 kg/m <sup>2</sup>	Madurai, India
27	Alaian et al. (2016)	Pin fins and wick inside single slope solar still	4820 ml/m <sup>2</sup>	Mansoura, Egypt
28	Manokar et al. (2017)	Acrylic solar still with pin fins	2.64 kg/m <sup>2</sup>	Chennai, India

29	Muthu Manokar & Prince Winston (2017)	Comparative analysis on galvanized iron and Acrylic solar still with pin fins	2.34 kg/m <sup>2</sup>	Chennai, India
30	Panomwan Na Ayuthaya, R.t al (2013)	The thermal performance of an ethanol solar still with fin plate to increase productivity.	3.5 kg/m <sup>2</sup>	Thailand
31	Jani & Modi (2018)	Circular and square hollow fins in single slope solar still	1.49 kg/m <sup>2</sup> (circular fin) 0.94 kg/m <sup>2</sup> (square fin)	Valsad, India
32	Srivastava & Agrawal (2013)	Single slope solar still with extended porous fins	7 kg/m <sup>2</sup>	Rewa, India
33	Yousef et al (2019)	Pin fin heat sink PCM based energy storage in solar still	3.9 kg/m <sup>2</sup>	Alexandria, Egypt
34	Appadurai & Velmurugan (2015)	Solar pond integrated solar still	3	NA
35	Omara et al. (2011)	Corrugated absorber single slope solar still	3.5 kg/m <sup>2</sup>	Kafrelshiekh, Egypt



224

225 Fig. 7 Instantaneous variations in thermal efficiency of TSS with and without fins.

226 The instantaneous hourly changes in thermal efficiency of the TSS using flat absorber  
 227 and finned absorber is plotted in Fig. 7. The instantaneous thermal efficiency of the solar still  
 228 is calculated using Equation (4) and as follows,

$$229 \quad \text{Instantaneous Thermal efficiency, } \eta_{thermal} = \frac{m_e \times h_{fg}}{I(t) \times A_w \times 3600} \times 100 \quad (4)$$

230 From the graph, it can be noted that the efficiency of the solar still without fins reaches  
 231 the peak value during mid-day and gradually falls around evening. The thermal efficiency of  
 232 the TSS having fins also reaches the maximum value during the mid-day same as the solar still  
 233 without fans but the TSS with fins maintains the thermal efficiency for a significant time period  
 234 in evening of that experiment day. The peak value of thermal efficiency for solar still without  
 235 fins reached approximately 36.65 %, whereas for the solar still with fins it attains a peak value  
 236 of about 43.13 %. Hence, the usage of fins in the TSS has a remarkable effect on the freshwater  
 237 production and also helps in boosting the vapor to entrap inside the tubular enclosure.

### 238 3.3. Rate of irreversibility from water, glass and basin

239 The rate of irreversibility of water, glass and basin using finned absorber and flat absorber is  
 240 mathematically expressed in Equation (5-7). The total rate of irreversibility is the summation  
 241 of destruction of exergy and loss of exergy.

242 The rate of irreversibility from glass is mathematically given as (Sarhaddi et al. (2017)),

243 
$$I_{r,g} = \alpha_g E_{sun} + U_b \times (T_b - T_w) \left(1 - \frac{T_a}{T_b}\right) \quad (5)$$

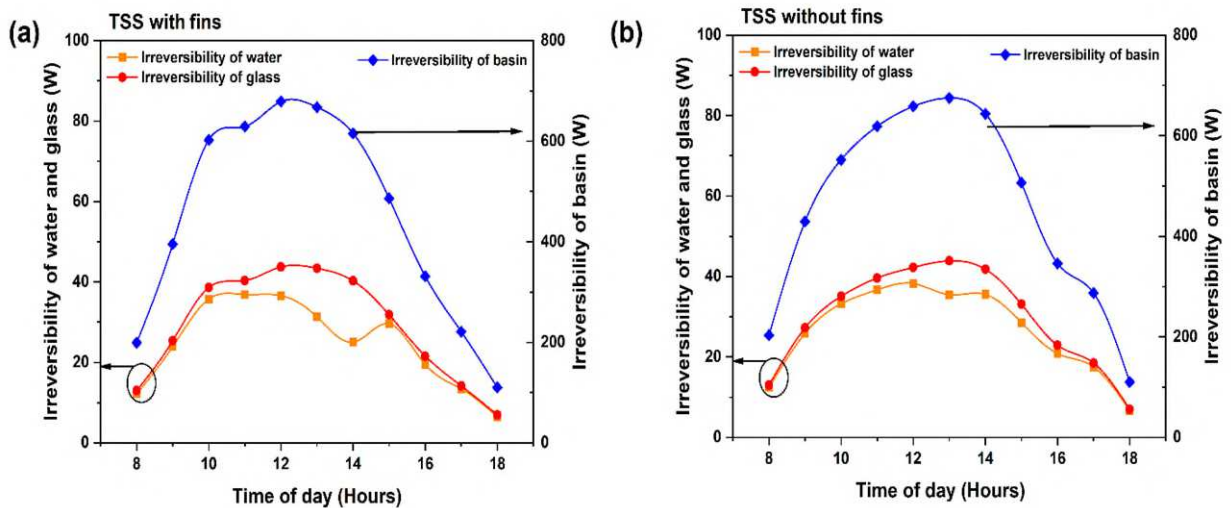
244 The rate of irreversibility from water is mathematically given as (Sarhaddi et al. (2017)),

245 
$$I_{r,g} = \tau_g \alpha_w E_{sun} + U_b \times (T_b - T_w) \left(1 - \frac{T_a}{T_b}\right) - E_{evap} \quad (6)$$

246 The rate of irreversibility from basin is mathematically given as (Sarhaddi et al. (2017)),

247 
$$I_{r,g} = \tau_g \tau_w \alpha_b E_{sun} + U_b \times (T_b - T_w) \left(1 - \frac{T_a}{T_b}\right) \quad (7)$$

248 Fig. 8 (a, b) shows the variations of irreversibility of water, glass, and basin of water, glass and  
 249 basin of TSS using flat and finned absorber. It is clear that the irreversibility of basin is higher  
 250 in both the case and the lower irreversibility occurs on water and glass. Also, on increased solar  
 251 intensity falling on the solar still increased the irreversibility of each component. The average  
 252 irreversibility rate of water, glass and basin using flat absorber is found as 26.45, 29.45 and  
 253 457.2 W respectively, whereas, for a finned absorber it is found as 24.6, 29.02 and 448.8 W  
 254 respectively. It is observed that the irreversibility rate of finned absorber is reduced as  
 255 compared to that of solar still using flat absorber. Also, from Fig. 8 (a) and (b) it is depicted  
 256 that the irreversibility rate of water and glass were closer till reaching the peak solar intensity.  
 257 From the previous literatures (Sarhaddi et al. (2017)) it is found that the irreversibility of solar  
 258 still can be reduced by modifying the design of the absorber plate.



259  
 260 Fig. 8 Irreversibility of water, glass, and basin of TSS using (a) flat absorber and (b) finned  
 261 absorber

### 262 3.4.Exergy efficiency

263 The exergy efficiency of the solar still is mathematically expressed as follows (Petla (2003);  
 264 Hepbasli (2008)),

265 
$$\eta_{exergy\ efficiency} = \frac{E_{out}}{E_{in}} \times 100 \quad (8)$$



266 The exergy output is mathematically expressed as (Petla (2003); Hepbasli (2008)),

267 
$$E_{out} = E_{evaporation} = \frac{m_e}{3600} \times A_w \times h_{fg} \times \left(1 - \frac{T_a}{T_w}\right) \quad (9)$$

268 Where,

269  $h_{fg}$ - Latent heat of vaporization (kJ/kg)

270  $T_w$ - Water temperature (K)

271 The exergy input is mathematically expressed as (Petla (2003); Hepbasli (2008)),

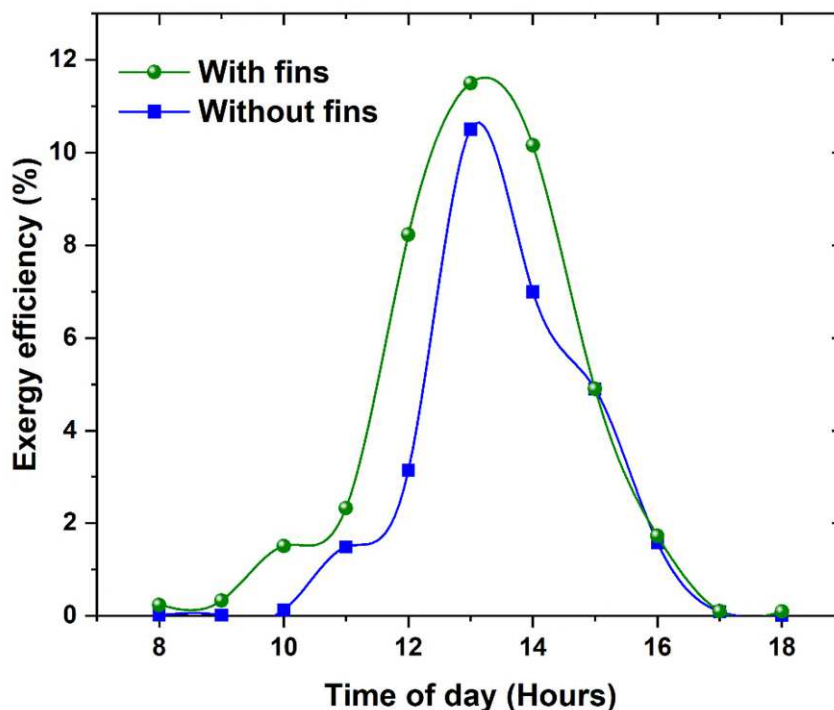
272 
$$E_{in} = E_{sun} = A_w \times I(t) \times \left[1 - \frac{4}{3} \left(\frac{T_a}{T_{sun}}\right) + \frac{1}{3} \left(\frac{T_a}{T_{sun}}\right)^4\right] \quad (10)$$

273 Where,

274  $T_{sun}$ - Temperature of sun ( $T_{sun}=6000$  K)

275  $T_a$ - ambient temperature (K)

276 The exergy efficiency of the solar still increases with respect to time and the amount of  
277 solar radiation falling on the system. It is seen that the exergy efficiency of the both the solar  
278 still increases as the solar radiation increased and reaching the maximum of 11.8 and 10.6 %  
279 for finned and flat absorber respectively. During the start of experiment till reaching the  
280 maximum solar intensity the exergy efficiency of finned absorber TSS produced exergy  
281 efficiency.



282

283 Fig. 9 Instantaneous variations on exergy efficiency from TSS using flat and finned absorber

284 **4. Conclusions**

285 The performance analysis of TSS with and without fins are investigated in this study.  
286 The findings of experimental and theoretical study show that the fins integrated with the basin  
287 of TSS augmented the performance and thermal efficiency higher than the TSS without fins.  
288 The distillate yield of TSS with fins is experimentally and theoretically higher than the TSS  
289 without fins. A cumulative distillate gain of 53.08 % implying an hourly thermal efficiency  
290 gain of 69.9% are recorded for the TSS with fins compared with the TSS without fins. The  
291 maximum daily distillate production has been found to be 2.93 l per day for TSS with fins. The  
292 use of fins in the basin of TSS enhanced the amount of heat absorbed by the absorber due to  
293 increase in the surface area of absorber plate which in turn results in the higher freshwater  
294 production compared to the TSS without fins. The rate of irreversibility is slightly reduced  
295 from the TSS using finned absorber while compared to the flat absorber. Similarly, by attaching  
296 fins in the absorber plate, the exergy efficiency is improved from the solar still while compared  
297 to solar still with flat absorber.

298 **Ethical Approval**

299 Not Applicable

300 **Consent to Participate**

301 Not Applicable

302 **Consent to Publish**

303 Not Applicable

304 **Authors Contributions**

305 Conceptualization, Methodology, Resources, Formal analysis, Writing - original draft  
306 preparation, review and editing, Supervision and investigation were carried out by Ravishankar  
307 Sathyamurthy, Abd Elnaby Kabeel, Ali Chamkha

308 Writing - original draft preparation, review and editing were carried out by Hemanth Arun  
309 Kumar, Hariprasath Venkateswaran, Athikesavan Muthu Manokar, Ramani Bharathwaaj,  
310 Sathiyaseelan Vasanthaseelan

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313 **Competing Interests**

314 The authors declare that there is no competing interest

315 **Availability of data and materials**

316 Not Applicable

317

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- 430

# Figures

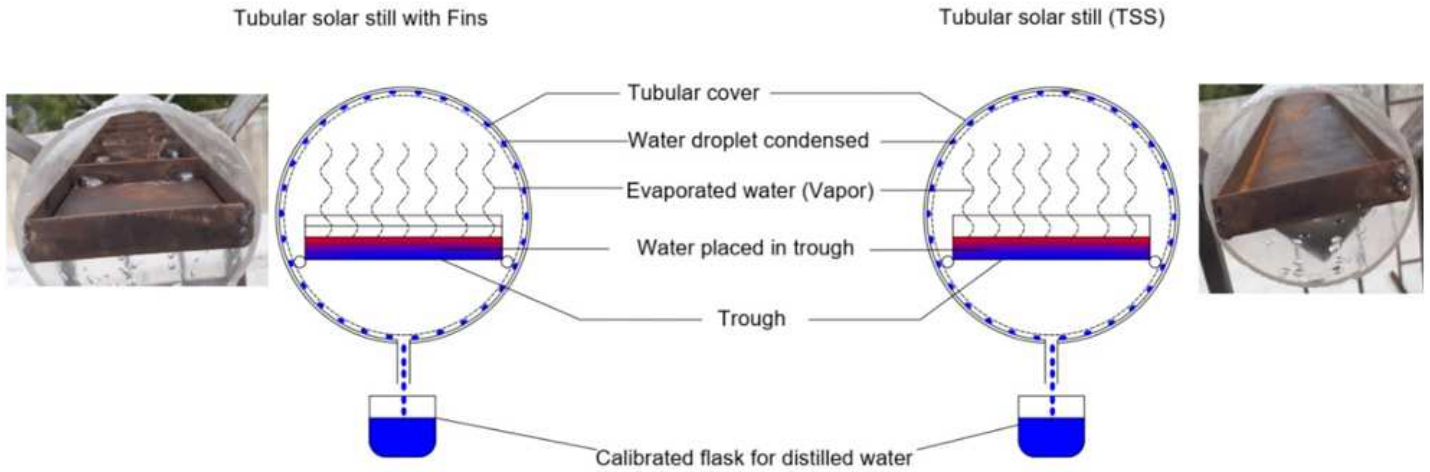


Figure 1

Schematic diagram and experimental test rig of TSS with fins (left) and without fin (right)

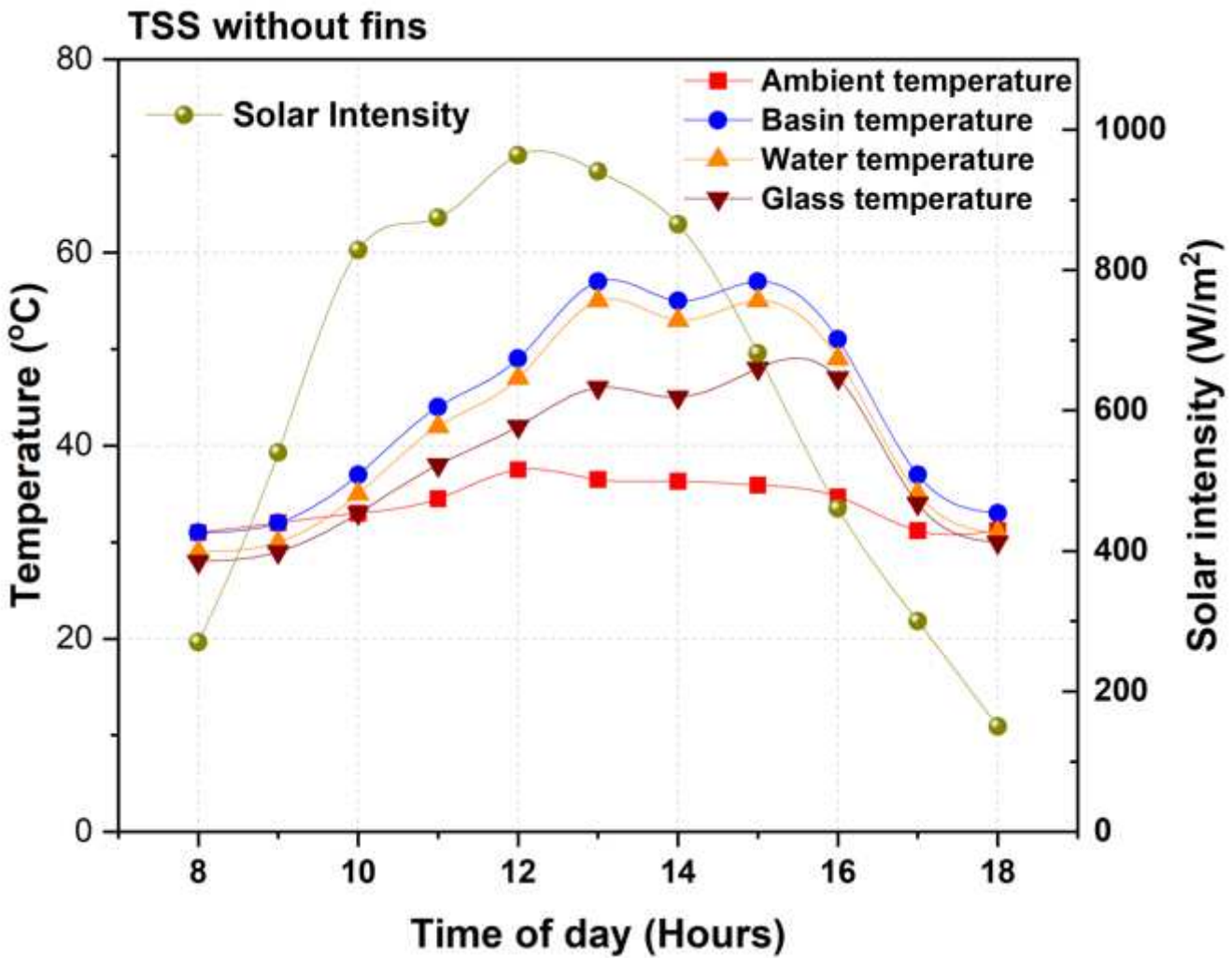


Figure 2

Hourly variation in solar intensity, ambient, basin, water, and glass temperature recorded from TSS without fins

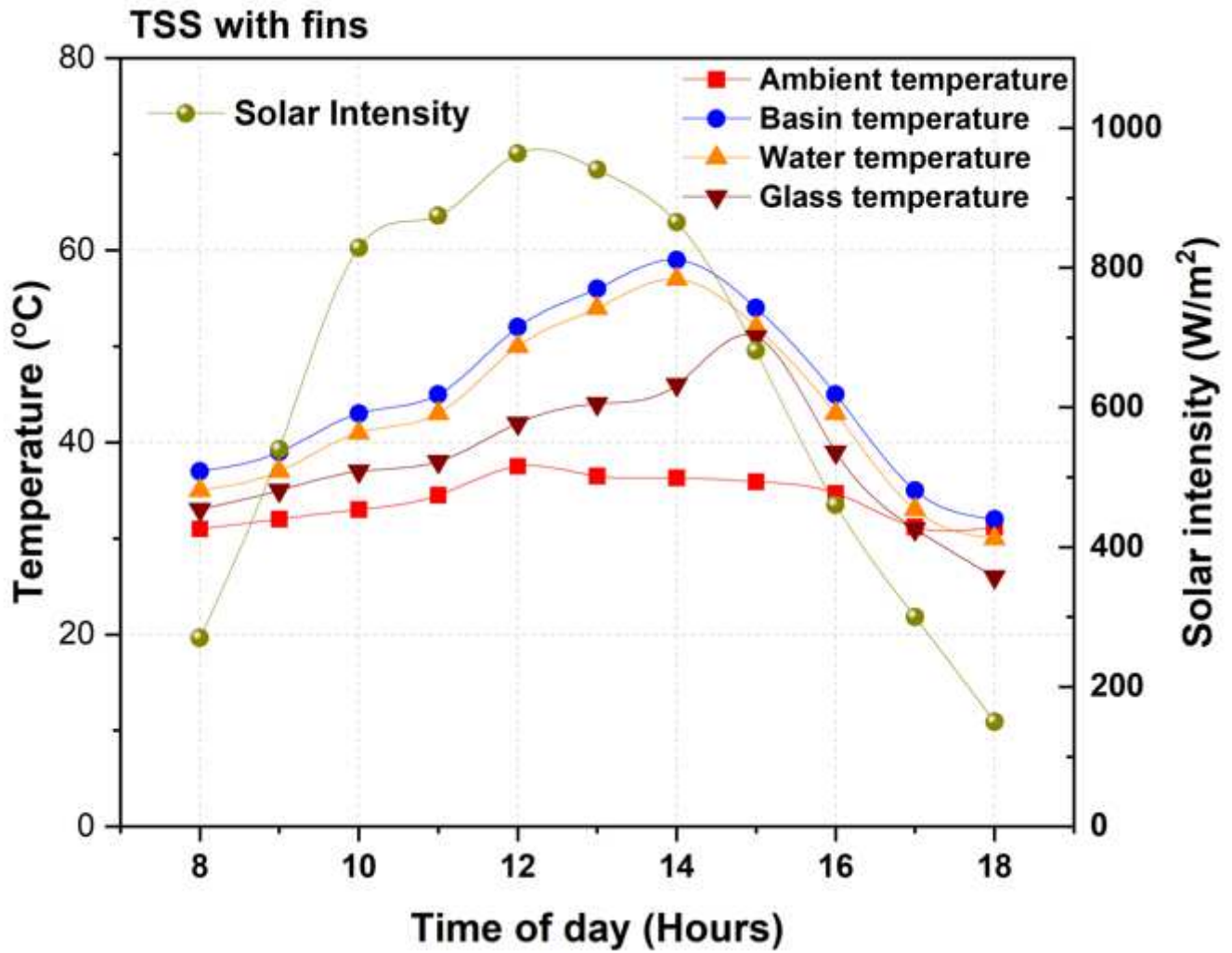


Figure 3

Hourly variation in solar intensity, ambient, basin, water, and glass temperature recorded from TSS with fins



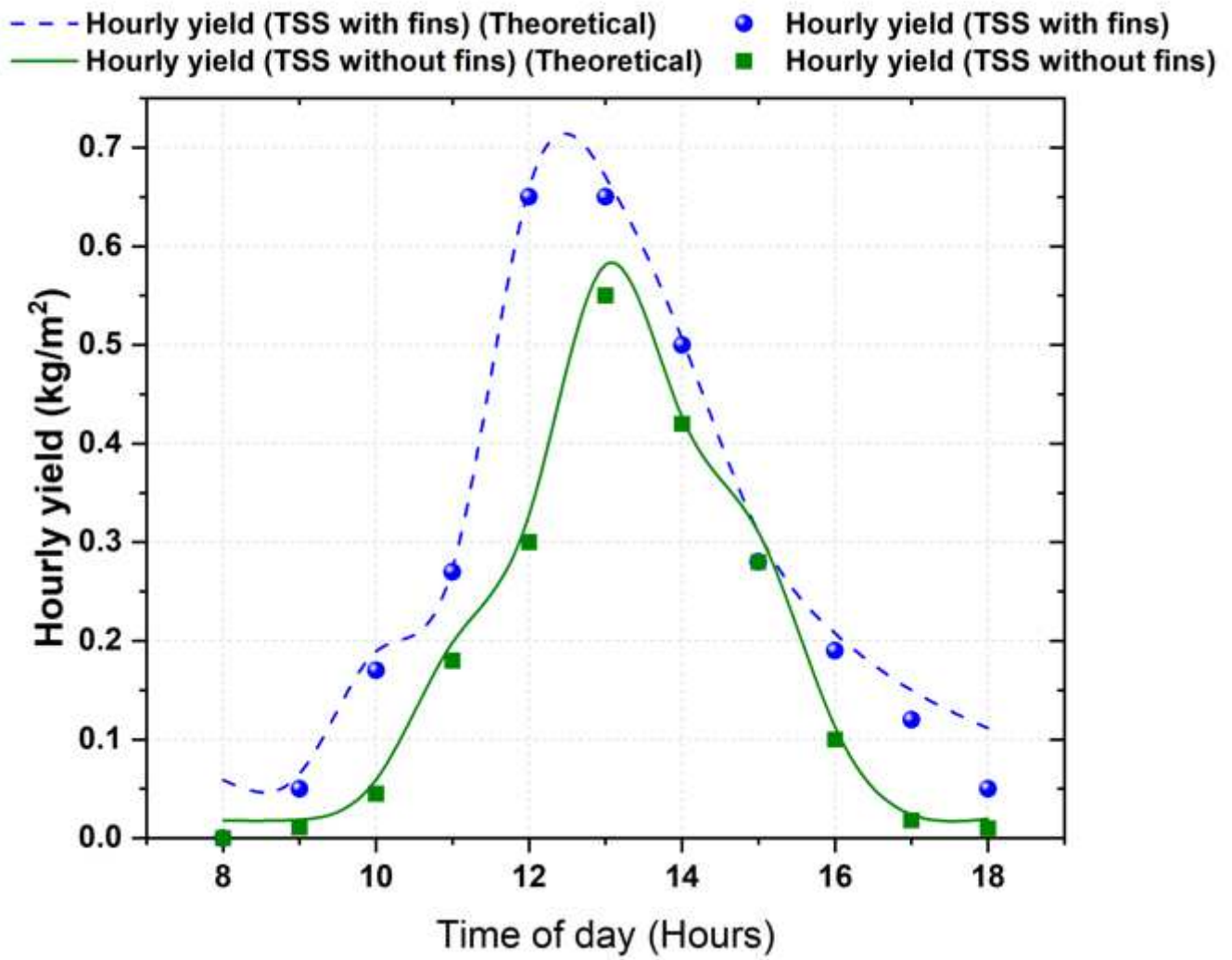


Figure 4

Hourly variation of theoretical and experimental distillate yield for TSS with and without fins.

### TSS with fins

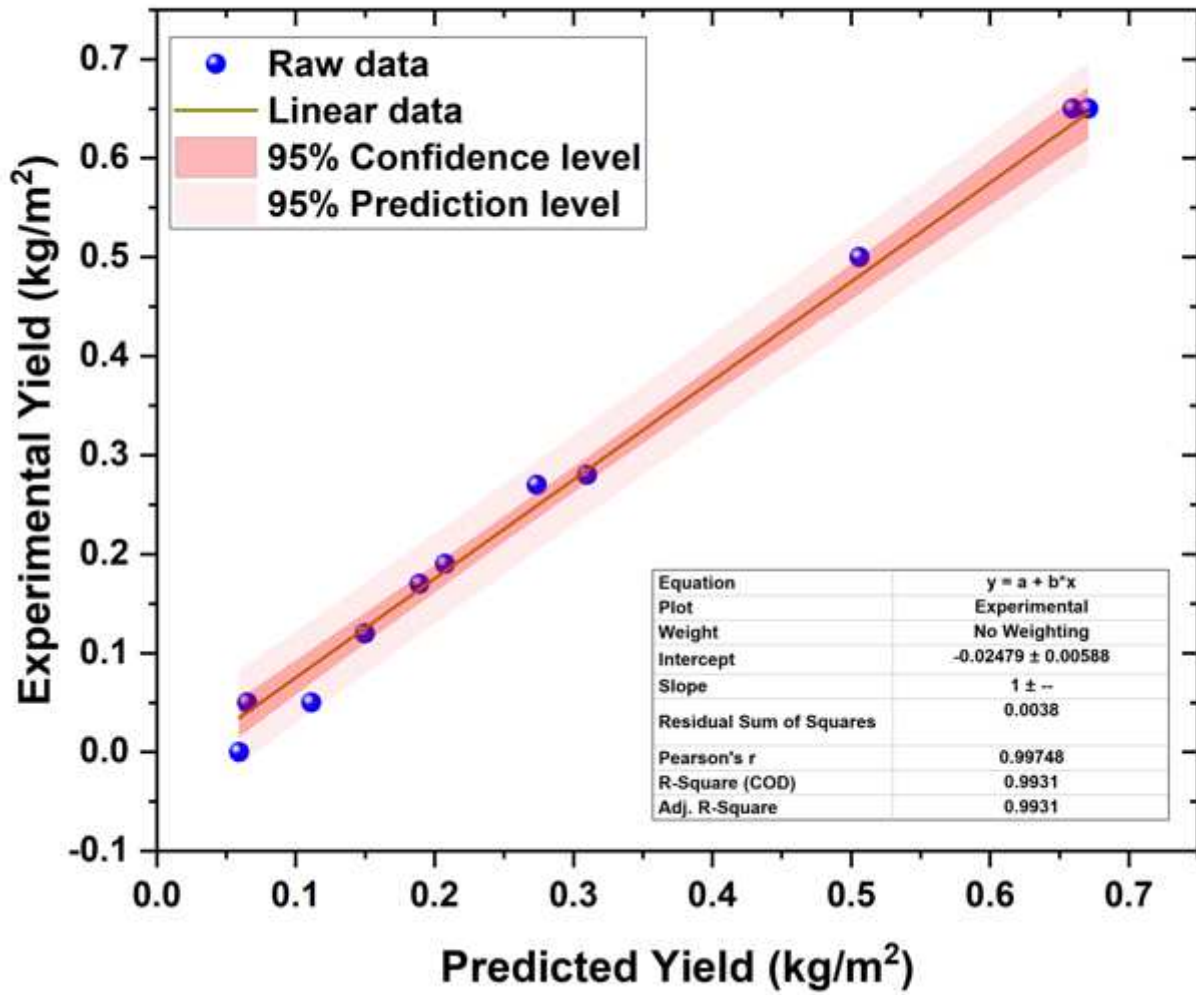


Figure 5

Predicted and measured yield of TSS with fins

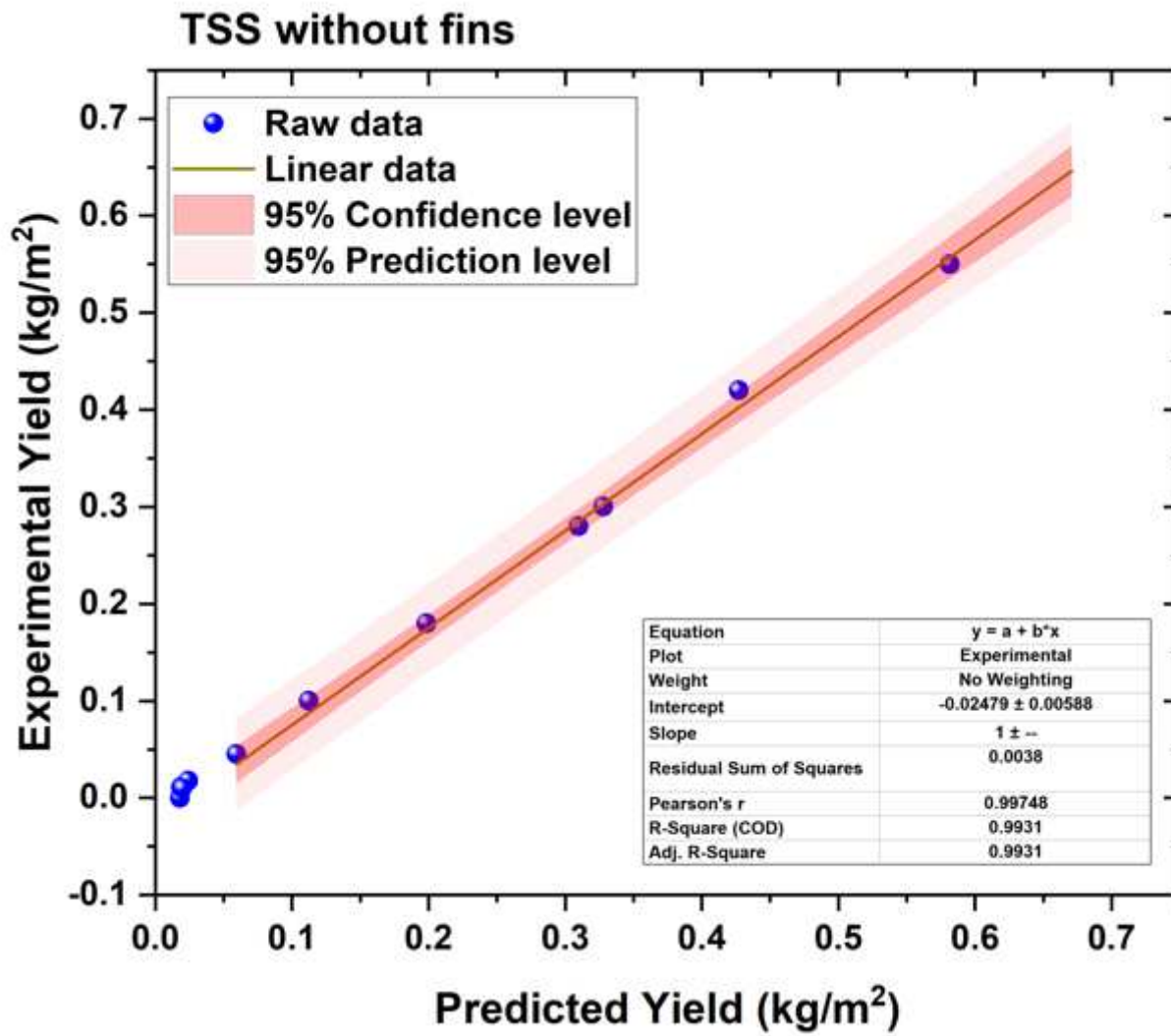


Figure 6

Predicted and measured yield of TSS without fins

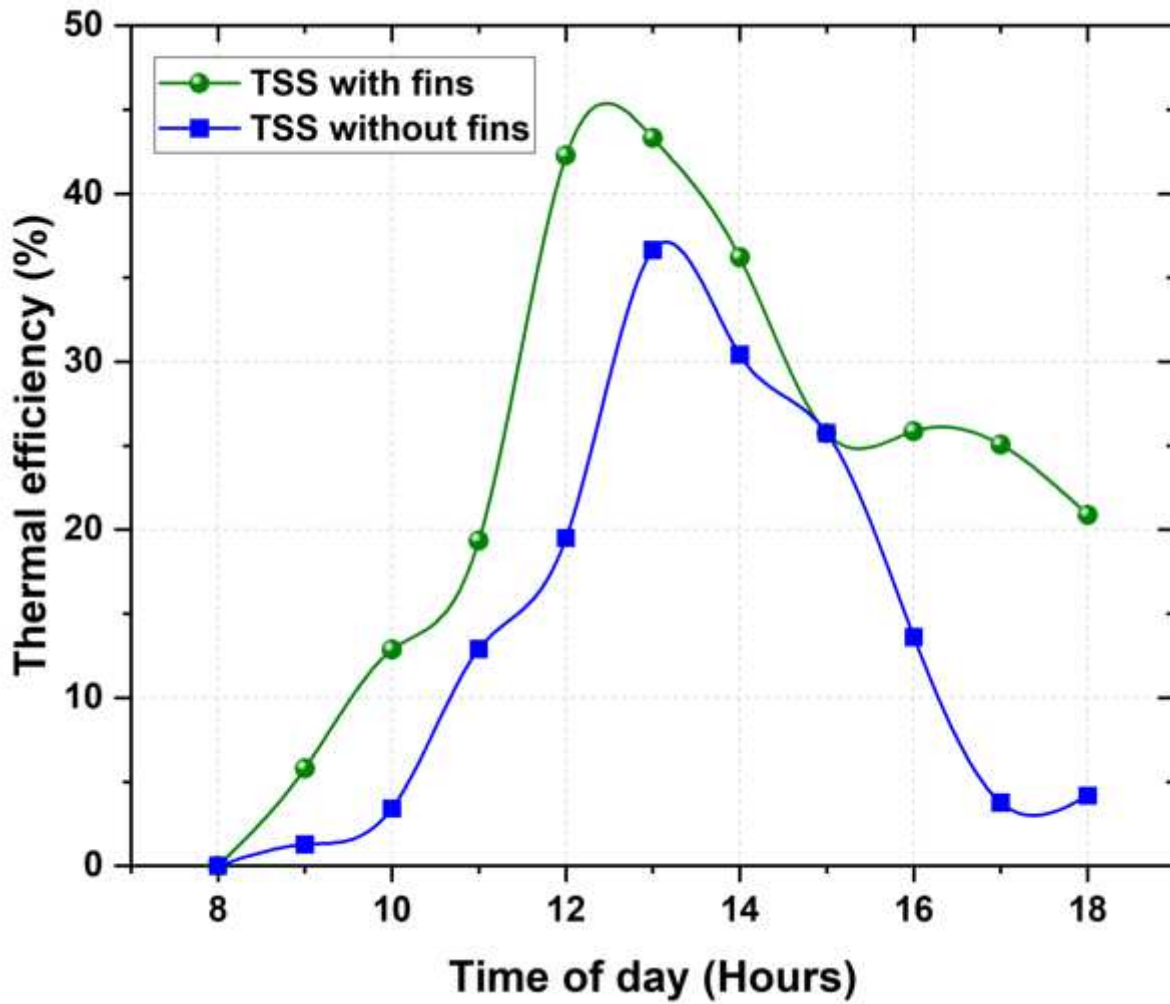


Figure 7

Instantaneous variations in thermal efficiency of TSS with and without fins.

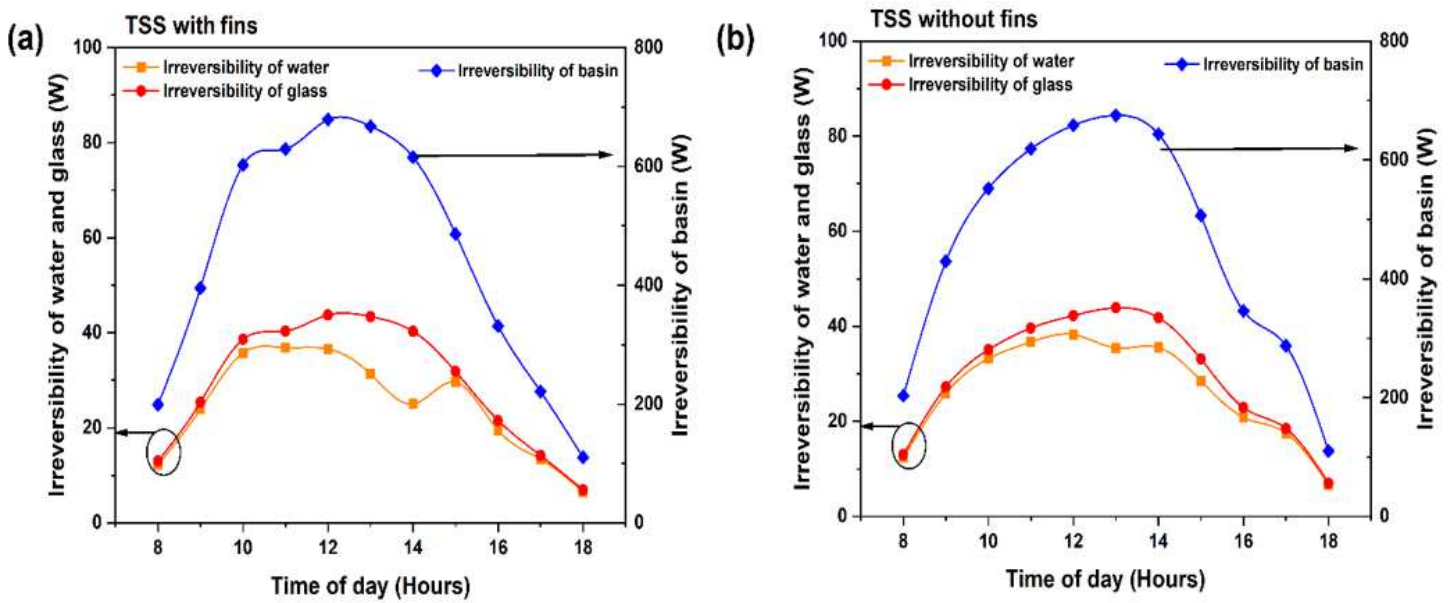


Figure 8

Irreversibility of water, glass, and basin of TSS using (a) flat absorber and (b) finned absorber

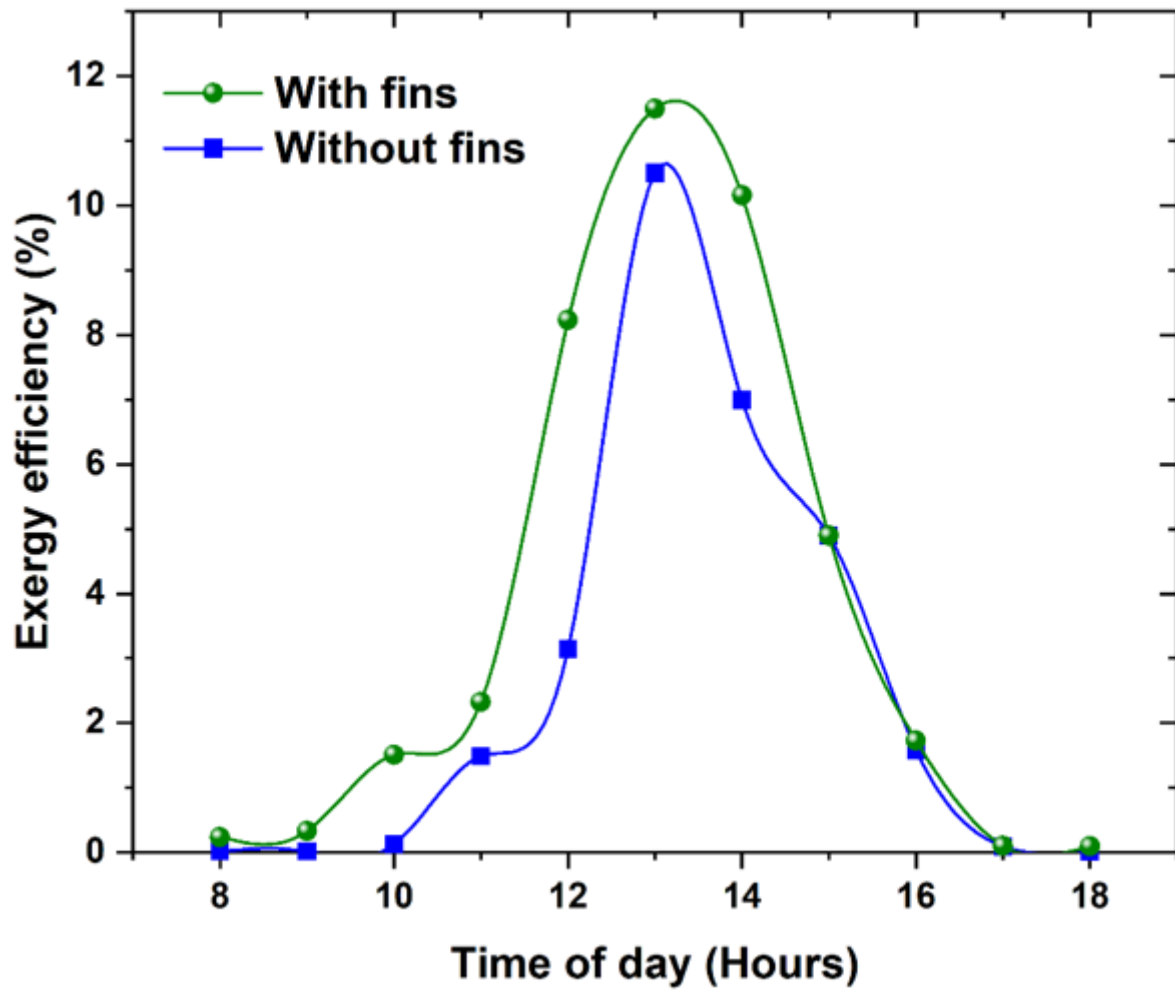


Figure 9

Instantaneous variations on exergy efficiency from TSS using flat and finned absorber