

Identification of Thermal Parameters by Treating the Inverse Problem

Abdelaaziz Ghafiri
Laboratory of Electronics,
Signal Processing and
Modelling Physics
Department of Physics
Ibn Zohr University
Agadir ,Morocco

Jamal Chaoufi
Laboratory of Electronics,
Signal Processing and
Modelling Physics
Department of Physics
Ibn Zohr University
Agadir ,Morocco

Claude Vallee
Institut P'
University of Poitiers
Poitiers, France

El hanafi Arjdal
Laboratory of Electronics,
Signal Processing and
Modelling Physics
Department of Physics
Ibn Zohr University
Agadir, Morocco

Jean Christophe Dupre
Institut P'
University of Poitiers
Poitiers, France

Arnaud Germaneau
Institut P'
University of Poitiers
Poitiers, France

Kossi atchonouglo
Department of Physics
University of Lomé
Lome ,Togo

Hassan Fatmaoui
Laboratory of Electronics,
Signal Processing and
Modelling Physics
Department of Physics
Ibn Zohr University
Agadir ,Morocco

ABSTRACT

The aim of this work is to identify the tensor of thermal conductivity and volumetric heat of an anisotropic material with conductivity tensor diagonal, the heat conduction follows the linear Fourier law. The temperature field in the plate is obtained by solving the analytical heat equation. The solution of the direct problem is simulated by applying the Finite Element Method 2D. The inverse problem is solved by returning the intermediate step of the MEF. An optimization method based on conjugate gradient algorithm has enabled us to develop an identification of thermophysical parameters procedure.

Keywords

Identification; Thermophysical parameters; Inverse problem; Finite Element Method; Infrared Camera; Projected Conjugate Gradient Method

1. INTRODUCTION

Improving the characterization of anisotropic materials is currently a major challenge for designing new structures or infrastructures. The thermophysical parameters identification allows to introduce materials whose characteristics meet specific needs or optimize existing systems.

Since the 1980s, several studies have given considerable impetus to inverse methods in thermal: Hensel applies reverse engineering theories [1] Kurpisz determines the heat flux [2], Ballis measuring the thermal diffusivity of cell carbon foam [3]. Some authors have studied the sensitivity of the parameters λ and ρc [3] [4] [5]. In the literature, few studies intended to. the simultaneous identification of the specific

heat and conductivity Atchongolo presented a method for identifying thermal parameters for isotropic materials [6]

The approach of this study is based on the exploitation of temperature fields obtained by the analytical resolution of the heat equation. The Inverse Problem is solved by crossing back the equations obtained by the Finite Element Method for solving the Direct P identify the thermal parameters of an material with thermal conductivity tensor is

$$\text{diagonal } \lambda = \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix}.$$

To validate our calculation approach we compared the temperature fields determined, analytically and numerically, with the experimental fields temperatures measured by an infrared camera on a plate of polymethylmethacrylate (PMMA), these measures were carried out within the institute P' of the University of Poitiers.

2. DIRECT PROBLEM

2.1. Position of the problem

In order to identify the thermal characteristics of a material, we performed a procedure both experimental and numerical. We consider a rectangular thin solid plate of length L , width l and thickness e . The plate is homogeneous, its thickness is small and the length is very close to the width. We will assume that the temperature distribution is two-dimensional.

Let us consider that the plate occupies the interval $[0, L]$ from the Ox axis, and $[0, l]$, by the Oy axis. The temperature distribution within the plate is a function $T(x, y, t)$ of

coordinates (x, y) and time t . At time $t=0$, the temperature distribution is known, we will denote it $T_0(x, y)$. A constants heat flux q_1 is imposed on the side bounded by $x=L$ (denoted Γ_3) and q_2 is imposed on the side bounded by $y=l$ (denoted Γ_4). The other sides (denoted Γ_1 and Γ_2) are well protected against any convective, radiative or conductive currents. Figure 1 illustrates the domain $\Omega=[0, L] \times [0, l]$ occupied by the plate and the boundary conditions.

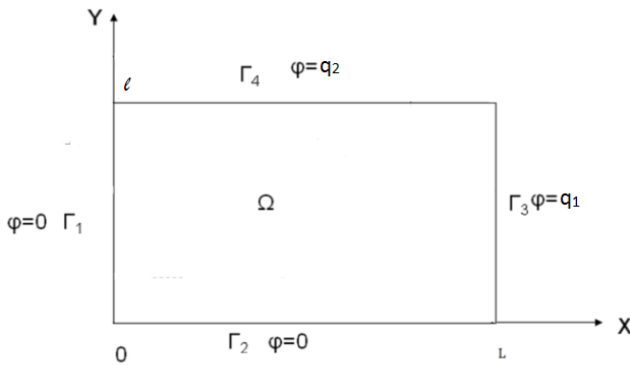


Fig 1: Rectangular plate and boundary conditions

Let ρc be the heat capacity per unit volume (ρ being the specific mass and c the heat capacity per unit mass). Let $\lambda = \begin{bmatrix} \lambda_{11} & 0 \\ 0 & \lambda_{22} \end{bmatrix}$ be the thermal conductivity tensor. According to Fourier's prescriptions, this symmetrical tensor is assumed to be positive definite. The temperature distribution $T(x, y, t)$ in the plate is governed by the diffusion equation [7]:

$$\rho c \frac{\partial T}{\partial t} = \lambda_1 \frac{\partial^2 T}{\partial x^2} + \lambda_2 \frac{\partial^2 T}{\partial y^2} \quad \text{in } \Omega = [0, L] \times [0, l]$$

The above boundary conditions read:

$$\lambda_1 \frac{\partial T}{\partial x}(0, y, t) = 0 \quad \lambda_1 \frac{\partial T}{\partial x}(L, y, t) = q_1$$

$$\lambda_2 \frac{\partial T}{\partial y}(x, 0, t) = 0 \quad \lambda_2 \frac{\partial T}{\partial y}(x, l, t) = q_2$$

The initial condition reads:

$$T(x, y, 0) = T_0(x, y) = 0.$$

2.2. Analytical resolution of the direct problem

A particular solution of the heat equation and the boundary conditions is

$$\theta(x, y, t) = \frac{1}{\rho c} \left(\frac{q_1}{L} + \frac{q_2}{l} \right) t + \frac{q_1}{2\lambda_1 L} \left(x^2 - \frac{L^2}{3} \right) + \frac{q_2}{2\lambda_2 l} \left(y^2 - \frac{l^2}{3} \right)$$

but this solution does not satisfy the initial condition.

we will use these functions $\frac{q_1}{2\lambda_1 L} \left(x^2 - \frac{L^2}{3} \right)$ and $\frac{q_2}{2\lambda_2 l} \left(y^2 - \frac{l^2}{3} \right)$

instead functions $\frac{q_1}{2\lambda_1 L} x^2$ and $\frac{q_2}{2\lambda_2 l} y^2$ because they are null mean (in x , respectively in y).

the function $v(x, y, t) = T(x, y, t) - \theta(x, y, t)$ is the solution of the heat equation with all the homogeneous boundary conditions and the initial condition

$$v(x, y, 0) = -\frac{q_1}{2\lambda_1 L} \left(x^2 - \frac{L^2}{3} \right) - \frac{q_2}{2\lambda_2 l} \left(y^2 - \frac{l^2}{3} \right)$$

The method of separation of variables leads to search $v(x, y, t)$ in the form of a double Fourier series

$$v(x, y, t) = \sum_{m=0}^{\infty} \sum_{n=0}^{\infty} A_{mn} \exp\left[-\frac{\pi^2 t}{\rho c} \left(\frac{\lambda_1 m^2}{L^2} + \frac{\lambda_2 n^2}{l^2} \right)\right] \cos \frac{m\pi x}{L} \cos \frac{n\pi y}{l}$$

At time 0, the exponential is worth 1, the coefficients A_{mn} are determined by the initial condition.

$$\sum_{m=0}^{\infty} \sum_{n=0}^{\infty} A_{mn} \cos \frac{m\pi x}{L} \cos \frac{n\pi y}{l} = -\frac{q_1}{2\lambda_1 L} \left(x^2 - \frac{L^2}{3} \right) - \frac{q_2}{2\lambda_2 l} \left(y^2 - \frac{l^2}{3} \right)$$

With our choice of functions with zero mean in x and in y , the coefficient A_{00} is null and only the coefficients A_{0m} and A_{n0} ($m \geq 1, n \geq 1$) are not null.

$$v(x, y, t) = \sum_{m=1}^{\infty} A_{m0} \exp\left(-t \frac{\lambda_1 \pi^2 m^2}{\rho c L^2}\right) \cos \frac{m\pi x}{L} + \sum_{n=1}^{\infty} A_{0n} \exp\left(-t \frac{\lambda_2 \pi^2 n^2}{\rho c l^2}\right) \cos \frac{n\pi y}{l}$$

We integrate twice by parts

$$\frac{m\pi}{L} \int_0^L x^2 \cos \frac{m\pi x}{L} dx = 2 \frac{L^2}{m\pi} (-1)^m$$

Such as $\int_0^L \cos^2 \left(\frac{m\pi x}{L} \right) dx = \frac{L}{2}$

thus $\frac{L}{2} A_{m0} = -\frac{q_1}{\lambda_1} \frac{L^2}{m^2 \pi^2} (-1)^m$ and $\frac{l}{2} A_{0n} = -\frac{q_2}{\lambda_2} \frac{l^2}{n^2 \pi^2} (-1)^n$

Finally the temperature in the plate is

$$T(x, y, t) = \frac{1}{\rho c} \left(\frac{q_1}{L} + \frac{q_2}{l} \right) t + \frac{q_1 L}{2\lambda_1} \left(x^2 - \frac{L^2}{3} \right) + \frac{q_2 l}{2\lambda_2} \left(y^2 - \frac{l^2}{3} \right) - \frac{2q_1 L}{\lambda_1} \sum_{m=1}^{\infty} \frac{(-1)^m}{m^2 \pi^2} \exp\left(-t \frac{\lambda_1 \pi^2 m^2}{\rho c L^2}\right) \cos \frac{m\pi x}{L} - \frac{2q_2 l}{\lambda_2} \sum_{n=1}^{\infty} \frac{(-1)^n}{n^2 \pi^2} \exp\left(-t \frac{\lambda_2 \pi^2 n^2}{\rho c l^2}\right) \cos \frac{n\pi y}{l}$$

In the next part we will give an alternative method for solving the heat equation using the finite element method (FEM).

2.3. Resolution of the Direct problem by Finite Element Method(FEM)

The above Partial Differential Equation (1) admits a variational formulation. The latter consists in multiplying the heat conduction equation by a regular test function T^* , and to integrate it over the whole domain Ω . This weak formulation leads to solve the equivalent problem [5]: Among

all temperature fields satisfying the initial condition $T(x, y, 0) = T_0(x, y)$, find the one for which

$$\int_{\Omega} \rho c T^* \frac{\partial T}{\partial t} dx dy + \int_{\Gamma_1} T^* \varphi_s dy + \int_{\Omega} (\text{grad } T^*) \cdot (\lambda \text{ grad } T) dx dy = 0$$

and this for any test function T^* .

An approximate solution of the diffusion problem presented in paragraph 2.3 can be obtained by choosing a finite number of appropriate test functions. We present next some details about this approach for obtaining the temperature field in the framework of Finite Element Method.

The Finite Element Method [8] is applied for the space discretization of the partial differential equations. We proceed to the discretization of the domain Ω into N_t three nodes triangular elements Ω^e . The total number of nodes is denoted N [9]. Let us denote $[A^e]$ the $n^e \times n_t$ positioning matrix defined by the following rule: a component $[A^e]_{ij}$ is equal to 1 if the node numbered globally j on the structure coincides with the node numbered locally i on the element e , it is equal to 0 otherwise. The matrix $[B^e]$ is the gradient matrix of element (e).

Therefore, this boundary value problem reduces to the ordinary differential equation (ODE).

$$[C] \left[\frac{dT}{dt} \right] + \sum_e [B^e A^e] \lambda [B^e A^e]^t [T] = [F]$$

We can summarize this ODE issued from a weak formulation associated to a spatial discretization as [6]:

$$[C] \left\{ \frac{dT}{dt} \right\} + [K] \{T\} = \{F\}$$

where

- $\{T\}$ is the nodal temperature vector,
- $[C]$ is the overall capacity matrix,
- $[K]$ is the overall conductivity matrix,
- $[F]$ is the overall nodal flux vector.

Solving our problem of transient conduction by FEM leads into solving the first order ODE for which the initial condition is: $[T_0]^t = [T_1(0) \ T_2(0) \ \dots \ T_N(0)]$. The temperature field in the material will be approximated at the nodes.

2.4. Results of the direct problem

In this part, and to validate our approach to calculating we compared the analytical and numerical results with experimental temperatures for PMMA plate with the following characteristics [10]

- Dimensions of the plate:
 $L = 0,148m$ and $l = 0.041$
- Discretization step: $\Delta x = L / 148$ and $\Delta y = l / 41$
- Thermal conductivity tensor:
 $\lambda = \begin{bmatrix} 0.17 & 0 \\ 0 & 0.17 \end{bmatrix} (\text{W/m}^\circ\text{C})$

- Specific heat $\rho c = 1,666.10^6 \text{ J / m}^3 / ^\circ\text{C}$
- Flows $q_1 = 256 \text{ W / m}$ and $q_2 = 0$
- Time step: $\Delta t = 1 \text{ s}$

Figure 2 below illustrates the evolution of experimental and simulated temperatures of the plate, at a node with respect to time.

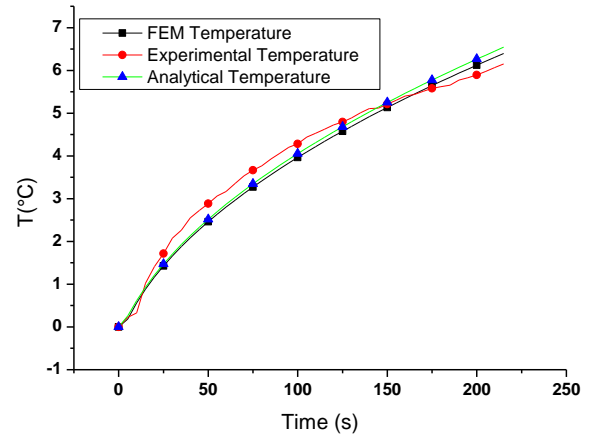


Fig 2: Evolution of temperature at point of coordinates (0.147, 0.02)

Figure 3 below illustrates the evolution of experimental and simulated temperatures of the plate, in a horizontal line at time $t=100 \text{ s}$ and $y=0.02 \text{ m}$.

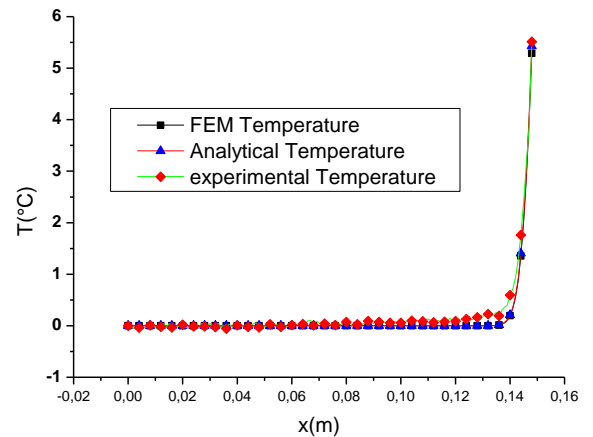


Fig 1: Changes of the temperature in function of the abscissa x (t=100s, y=0.02m)

Note that the simulated analytical temperature evolution curves fit the experimental ones, This comparison validates the developed direct approach.

Next, in the inverse problem and to put in the experimental conditions, a noise with standard deviation $\sigma = 0.02^\circ\text{C}$ is imposed on the temperature according to the precision current infrared cameras

3. INVERSE PROBLEM

3.1. Position of the problem

In the inverse problem we enjoy the results obtained from the analytical solution of the direct problem to estimate the

thermo physical parameters of a given material. Once the temperature function $T(x, y, t)$ is recorded in a series of points on the surface of the sheet, repeatedly, the least squares method is applied to estimate thermodynamic parameters. The method of least squares leads to constrained optimization process: Reduce the cost function

$$J(\rho c, \lambda) = \frac{1}{2} \int_0^{t'} \left\| [C] \left\{ \frac{dT}{dt} \right\} + \sum_e [B^e A^e] \lambda [B^e A^e]^t \{T\} - \{F\} \right\|^2 dt$$

Under the positive constraint for ρc , and the diagonal constraint for λ .

To minimize this function by a steepest descent method, we need to express its gradient with respect to the conductivity tensor λ :

$$R = \frac{1}{2} \int_0^{t'} \sum_e [B^e A^e] [U \{T\}' + \{T\} (U)'] [B^e A^e]^t dt$$

with

$$U = [C] \left\{ \frac{dT}{dt} \right\} + [K] \{T\} - \{F\}$$

We will need also its derivative with respect to the specific

$$\text{heat: } r = \frac{\partial J}{\partial(\rho c)}$$

3.2. Identification algorithm

As steepest descent method to minimize the function $J(\rho c, \lambda)$, we implement the projected conjugate gradient method which consists in constructing iteratively a sequence converging to the minimum.

The algorithm of this method can be summarized as follows

1. Initialize λ by λ_0 and ρc by $(\rho c)_0$,
 Deduce the initial values r_0 and R_0 of r and R ,
 Initialize a sequence of scalars d_i by $d_0 = -r_0$ and a sequence of directions D_i by $D_0 = -R_0$.
2. At iteration i
 calculate μ_i and ν_i which minimize
 $J((\rho c)_i + \mu d_i, \lambda_i + \nu D_i)$ with respect to μ and ν
 $(\rho c)_{i+1} = (\rho c)_i + \mu_i d_i$
 $\lambda_{i+1} = \lambda_i + \nu_i D_i$
3. if $\|r_{i+1}\| < \varepsilon$ and $\|R_{i+1}\| < \varepsilon$ stop, otherwise
 $d_{i+1} = -r_{i+1}$
 $\beta_i = \frac{(R_{i+1} - R_i)' R_i}{R_i' R_i}$
 $D_{i+1} = -R_{i+1} + \beta_i D_i$
 $i = i + 1$ and return to step 2.

In step 3, the test stop is based on a preassigned small scalar ε , the calculus of β_i follows the prescription of Polak and Ribière [11].

3.3. Identification results

The Conjugate Gradient method developed in the last section is applied to the simulated temperature fields obtained by analytical resolution of the direct problem. The material is supposed to be anisotropic with diagonal thermal conductivity tensor. The results from our identification algorithm without noise are shown in the table 1 below.

Table 1. Identified Values from simulated temperature fields without noise

Parameters	Values used in the simulation	identified Values
$\lambda_1(\text{W/m}^\circ\text{C})$	0.45	0.4309
$\lambda_2(\text{W/m}^\circ\text{C})$	0.17	0.1687
$\rho c(\text{J/m}^3/\text{C})$	$1.666 \cdot 10^6$	$1.6759 \cdot 10^6$

The table 2 shows the values identified from the noisy temperatures

Table 2. Identified Values from simulated noisy temperature fields

Parameters	Values used in the simulation	identified Values
$\lambda_1(\text{W/m}^\circ\text{C})$	0.45	0.4309 ± 0.0170
$\lambda_2(\text{W/m}^\circ\text{C})$	0.17	0.1638 ± 0.0140
$\rho c(\text{J/m}^3/\text{C})$	$1.666 \cdot 10^6$	$1.6573 \cdot 10^6 \pm 0.027 \cdot 10^6$

4. CONCLUSION

The finite element method meets the requirements imposed by the sample geometry and the boundary conditions. Its application on a homogeneous anisotropic material enabled us to transform the Fourier's heat conduction equation in a first order ordinary differential equation. Therefore, the resolution of the direct problem needs solely a time integration algorithm. The developed algorithm allows us to simulate the temperature field in the bidimensional case. The accuracy of the simulations ensured the validity of our approach. Moreover, our code proved to be fast handling, as well for varied geometric dimensions, than for varied boundary and initial conditions.

The identification algorithm is based on the conjugate Gradient method. It allows to characterize the thermal conductivity tensor and the specific heat of materials. The identification results revealed to be in good agreement with the values used in the simulation of the direct problem.

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