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ON RADIATED AND SCATTERED WAVES
FROM A SUBMERGED ELLIPTIC CYLINDER
IN A UNIFORM CURRENT

by

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ABSTRACT

The two-dimensional radiation problem and diffraction problem are discussed for submerged elliptic cylinders when a current is present. It is shown that the impact of the current on the wave amplitudes and wave forces are large. The singularity in the problem, corresponding to a wave travelling upstream with a group velocity equal to the speed of the current, is examined. As expected, this singularity influences the motion strongly. We find, however, that the amplitudes and forces remain finite.

dingly, when one harmonic wave is diffracted by a submerged body, normally three new waves are generated when $\tau < 1/4$ and one new wave when $\tau > 1/4$.

The actual wave problem has been discussed in a recent paper by Grue and Palm (1985) (hereafter called I) for a submerged body of form as a circular cylinder. It turns out, however, that the circular cylinder is no typical representative for a submerged body of arbitrary two-dimensional form. For example, in the diffraction problem it is found that a circular cylinder, in contrast to other bodies, generates only one new wave also for $\tau < 1/4$. In this paper we shall consider a submerged elliptical cylinder. The obvious advantage of this contour is that by changing the eccentricity we obtain bodies varying from a circle to a flat plate.

It will be clear that the case $\tau = 1/4$ is of special interest in this problem. Physically it corresponds to that σ is tuned so that the wave travelling upstream has group velocity equal to the speed of the current. It seems to be generally accepted in the literature that this case will lead to infinite wave amplitudes and hence infinite wave forces. We obtain, however, that the wave amplitudes and wave forces are finite for all values of τ . It is shown, however, that the motion has certain peculiarities near $\tau = 1/4$. Thus in the diffraction problem for an incoming k_2 wave (defined as a wave travelling upstream) the transmitted wave tends towards zero as $\tau \rightarrow 1/4$ whereas a reflected k_1 wave (defined in section 2) is generated, travelling downstream with an amplitude equal to the incoming wave. The corresponding result is obtained for an incoming k_1 wave with the k_1 wave and the k_2 wave changing roles. Due to this fact we may close to $\tau = 1/4$ obtain practically speaking, total reflection. We find also that the

where j is the imaginary unit, and Re_j denotes the real part (with respect to j). Both χ and ϕ satisfy the Laplacian equation

$$\nabla^2 \chi = 0 ; \quad \nabla^2 \phi = 0 \quad (2.2)$$

Assume that the cylinder is oscillating with its centre at

$$x = \text{Re}_j \xi_x \exp(j\sigma t), \quad y+d = \text{Re}_j \xi_y \exp(j\sigma t) \quad (2.3)$$

and rotating with an angular displacement $\vec{\Omega}$. The body boundary conditions applied at the mean position of the body surface S is

$$\frac{\partial \chi}{\partial n} = -n_x \quad (2.4)$$

where $\frac{\partial}{\partial n}$ denotes the normal derivative, and

$$\frac{\partial \phi}{\partial n} = \vec{n} \cdot [j\sigma \vec{\alpha} + U(\vec{\alpha} \cdot \nabla) \nabla \chi] + U(\vec{\Omega} \times \vec{n}) \cdot \nabla(\chi + x) \quad (2.5)$$

(see Newman 1978, eq. 3.28). Here

$$\vec{\alpha} = \vec{\xi} + \vec{\Omega} \times \vec{r}, \quad \vec{\xi} = (\xi_x, \xi_y) \quad (2.6)$$

\vec{n} is the normal vector of the body, and \vec{r} is the vector from the centre of the ellipse to a point on the surface (see fig.1).

The linearized boundary condition at $y = 0$ is obtained by combining the dynamic and kinematic boundary conditions which gives

$$U^2 \frac{\partial^2 \chi}{\partial x^2} + g \frac{\partial \chi}{\partial x} = 0 \quad (2.7)$$

$$(j\sigma - U \frac{\partial}{\partial x})^2 \phi + g \frac{\partial \phi}{\partial y} = 0 \quad (2.8)$$

To solve (2.2) with the proper boundary conditions we shall transform the mathematical problem to an integral equation. This may be achieved by expressing ϕ (and χ) as a source distribution over the boundary of the submerged body. For a detailed derivation of the integral equation, we refer to I, and here only give the necessary

and a bar denotes complex conjugate. With

$$v = \frac{\sigma^2}{g} \quad , \quad \tau = \frac{U\sigma}{g} \quad (2.15)$$

the four wave numbers are defined by

$$k_{1,2} = \frac{v}{2\tau^2} (1 - 2\tau \pm \sqrt{1 - 4\tau}) \quad , \quad k_{3,4} = \frac{v}{2\tau^2} (1 + 2\tau \pm \sqrt{1 + 4\tau}) \quad (2.16)$$

For $\tau > 1/4$, k_1 and k_2 become complex in i .

Before proceeding further, let us discuss shortly the far-field motion due to the concentrated source. It is seen from (2.12)-(2.16) that for $\tau > 1/4$ the solution consists of four waves, viz one wave with wave number k_2 at $x = \infty$ and three waves with wave numbers k_1, k_3, k_4 at $x = -\infty$. For $\tau > 1/4$ the solution consists of no waves at $x = \infty$ and two waves with wave numbers k_3, k_4 at $x = -\infty$.

The various wave numbers are found as solutions of
 i) $\sigma = Uk \pm (gk)^{1/2}$ ($k=k_4, k_3$) and ii) $\sigma = (gk)^{1/2} - Uk$ ($k=k_1, k_2$) where σ is positive and known. The four solutions are indicated in figure 2. It is seen from the figure that, in order to get four waves, σ must be less than a certain maximal value, i.e. $\tau < 1/4$. Furthermore, in the relative frame of reference, both the k_1 wave and the k_2 wave have positive phase velocities which are larger than U . The k_1 wave has, however, a group velocity less than U , and is therefore located downstream. The k_2 wave has a group velocity larger than U , and is located upstream. The k_3 wave has positive phase velocity smaller than U , and the k_4 wave has negative phase velocity. These two waves are therefore located downstream.

Equation (2.17) is a (non-singular) Fredholm equation of second kind. The equation is solved by using a collocation method with cubic splines. A very good convergence is generally obtained by using 20-35 collocation points equally spaced. The more slender the body is, the more points are needed. The accuracy is about 1% or better in all results presented in this paper.

We shall be especially interested in the far-field. By contour integration we obtain from (2.11), applying (2.12)-(2.14), that

$$\lim_{x \rightarrow \infty} f_1(z) = A_2 \exp(-ik_2 z) \quad (2.21)$$

$$\lim_{x \rightarrow -\infty} f_1(z) = A_1 \exp(-ik_1 z) + A_3 \exp(-ik_3 z) - A_4 \exp(-ik_4 z) \quad (2.22)$$

where

$$A_{1,2} = i(1-ij) \frac{1}{\sqrt{1-4\tau}} \int_S \gamma(s) \exp(ik_{1,2} \zeta(s)) ds \quad (2.23)$$

$$A_{3,4} = i(1+ij) \frac{1}{\sqrt{1+4\tau}} \int_S \gamma(s) \exp(ik_{3,4} \zeta(s)) ds \quad (2.24)$$

For an elliptic contour it is appropriate to write the equation for the ellipse on parameter form as

$$\zeta(\theta) = R \cos(\theta) + ib \sin(\theta) - i(d+b) \quad (2.25)$$

and using θ as variable instead of s . (For definitions of R , b , d and θ , see fig. 1.)

3. THE OSCILLATING ELLIPTIC CYLINDER

Let us consider an elliptic cylinder oscillating in sway, heave and roll. The right hand side of the integral equation (2.17) is now given by (2.19) and (2.5) with f_0 chosen as zero. Thus $h(s')$ is given by

and $U = 0$. As expected, the maximum relative amplitude is obtained in heave and for the smallest value of b/R , i.e. $b/R = 0.05$. For later reference we note that the maximum amplitude in heave for $b/R = 0.3$ occurs for $vR \approx 1.2$.

To examine the effect of increasing the depth of the cylinder, d/R is in fig. 4 chosen as 2 (the only case where d/R is not one). It is noticed that the maximum amplitude now is about 40% of the maximum amplitude for $d/R = 1$.

In figs 5a, 5b, 5c are displayed the relative amplitudes for sway, heave and roll, respectively for the Froude number $Fr = U/(gR)^{\frac{1}{2}}$ equal to 0.2. The corresponding curves for $Fr = 0.4$ are shown in figs 6a, 6b, 6c. We notice that for $Fr = 0.2$ the k_3 wave has a vanishing amplitude. This is also true for the k_1 wave except very close to $\tau = 1/4$ (corresponding to $vR = 1.5625$). Hence, practically speaking, for vR less than 1.5625, the motion for $Fr = 0.2$ consists of one wave at $x = \infty$ (the k_2 wave) and one wave at $x = -\infty$ (the k_4 wave). For larger values of vR only the k_4 wave occurs.

For $Fr = 0.4$ three waves occur when $\tau < 1/4$, viz the k_1 wave, the k_2 wave and the k_4 wave. For $Fr = 0.7$ and $Fr = 1.0$ we find that all four waves have appreciable amplitudes. To get an idea of which of the waves that are important for a given Froude number, and for which values of vR these waves obtain their maximum amplitudes, we examine first the corresponding values for $Fr = 0$. As an example, let us consider the heave motion for $Fr = 0.2$ (fig. 5b). We have noticed above that for $Fr = 0$ the maximum amplitude for heave occurs when $vR = 1.2$ (fig. 3b). This corresponds to a far-field motion with $kR = 1.2$ where k is the wave number. It seems reasonable to expect that also for $Fr \neq 0$,

$$\begin{aligned}
 & \delta\gamma(s') + \\
 & \frac{1}{\pi} \int_S \gamma(s) \operatorname{Im}_i \left\{ \exp(i\beta(s')) (i+j) \left[k_1 \int_{-\infty}^{\zeta(s')} \frac{\exp(-ik_1(\zeta(s')-u)) du}{u-\zeta(s)} \right. \right. \\
 & \quad \left. \left. - k_2 \int_{-\infty}^{\zeta(s')} \frac{\exp(-ik_2(\zeta(s')-u)) du}{u-\zeta(s)} \right] \right\} \\
 & = O(\delta) \tag{3.4}
 \end{aligned}$$

To the same order of accuracy we may set $k_1 = k_2 = k$, say. The inner integral is then evaluated by contour integration. Thereby

$$\begin{aligned}
 & \delta\gamma(s') - \\
 & 2k \operatorname{Im}_i \left\{ (i+j) \exp(i\beta(s')) - ik\zeta(s') \right\} \int_S \gamma(s) \exp(ik\overline{\zeta(s)}) ds \\
 & = O(\delta) \tag{3.5}
 \end{aligned}$$

We notice that the integral in (2.23), with $k_1, k_2 = k$, is exactly the same as the integral in (3.5). Hence, if γ remains finite, we deduce that $A_{1,2}$ is finite in the limit, in spite of the factor δ^{-1} . This result is valid for an arbitrary smooth body.

3.2 The forces

In many practical problems where a body is oscillating, for example due to incoming waves, it is important to know the magnitude of the damping force. Usually this force is mainly due to viscous effects. However, if the body is located near the free surface, the damping due to the radiated waves, may be of practical importance. To examine the magnitude of this force, we first note that it can be written as

$$F = D \sin \sigma \tau \tag{3.6}$$

fig. 7 that it is important to take into account the effect of the current.

It may be of interest to compare the wave damping with the viscous drag for the restrained body. The latter may for an elliptic cylinder be written

$$F = \frac{1}{2} C_D \rho U^2 2b \quad (3.11)$$

where C_D is the drag coefficient. The damping force may according to fig. 7 be written

$$D = \lambda g \rho R \varepsilon \quad (3.12)$$

where λ is of order unity or smaller. Hence

$$\frac{D}{F} = \frac{\lambda}{C_D Fr^2} \frac{\varepsilon}{b} \quad (3.13)$$

C_D for various elliptic contours is discussed in Modern Developments In Fluid Dynamics (1938, p.415). A reasonable characteristic value for $b/R = 0.3$ is $C_D = 0.15$. Let us choose $Fr = 0.4$. The maximum value of λ is then 0.9, and the ratio between the maximum drag force and the viscous drag is

$$\frac{D}{F} \approx 40 \frac{\varepsilon}{b} \quad (3.14)$$

We see that even for very small ε , D/F may be larger than unity.

Obviously, when the depth d of the cylinder is increased, the damping force decreases. We have examined the magnitude of this effect for the case $U = 0$ by also computing the forces when $d/R = 2.0$. The damping force for heave in this case is displayed in fig. 4. Comparing this with the result for the damping force for $Fr = 0$ in fig. 7, we note that the maximum value in the deeper case is about 30% of the maximum value for $d/R = 1.0$.

(2.24). γ is obtained by solving (2.17). The right hand side of this equation is now (2.19) with $\partial\phi/\partial n = 0$ and $f_0(z)$ given by (4.3). In all examples below the distance between the mean free surface and the uppermost point of the ellipse is equal to R , as in almost all examples in section 3. Furthermore, b/R is 0.3, except in fig. 9.

In the relative frame of reference, the incoming wave will be a wave travelling in either the same or in the opposite direction as the body. In the latter case the wave is what we have termed a k_4 wave. In the first case the wave will be either a k_2 wave, k_1 wave or k_3 wave, depending on the magnitude of the wave number. From the discussion in section 2 it follows that a k_2 wave has a large wave length such that both the group velocity and phase velocity are larger than U , a k_1 wave has phase velocity larger and group velocity smaller than U and a k_3 wave has both group velocity and phase velocity smaller than U .

4.1. The far-field motion

Let us first consider the case $U = 0$. The amplitudes of the reflected waves are displayed in fig. 9 for various values of b/R . Since the amplitude of the reflected wave is zero for $b/R = 1$ (Dean 1948), we expect this amplitude to increase for decreasing values of b/R . This is seen to be true, and for $b/R = 0.05$ we find from the figure that the maximum reflected amplitude is about 0.3 times the incoming amplitude. This value may be compared with the result for the flat plate, recently discussed by Bjordal (1985). He obtains for $d/R = 1$ that the maximum reflected amplitude is 0.294 times the incoming amplitude.

Froude numbers a k_3 wave will be generated by an incoming k_4 wave. In this case we have no reflected wave, but the transmitted wave is splitted in two waves, viz a k_3 wave and a k_4 wave.

In fig. 11 are shown the wave amplitudes for $Fr = 0.4$. In fig. 11a the incoming wave is a k_1 wave and the generated waves is a k_2 wave and a k_4 wave. In fig. 11b the incoming wave is a k_2 wave, and a k_4 wave and a k_1 wave are generated. In both figs 11a and 11b the k_4 wave are, however, almost negligible. The most characteristic feature of these two figures is that for an incoming k_1 wave (k_2 wave) the amplitude tends towards zero approaching $\tau = 1/4$, whereas the generated k_2 wave (k_1 wave) obtains an amplitude for $\tau \rightarrow 1/4$ equal to the amplitude of the incoming wave. This is always found to be true. An interpretation of this result is that a k_1 wave or k_2 wave will be strongly reflected near $\tau = 1/4$ and as $\tau = 1/4$ is approached, we obtain with a very good approximation total reflection. Another remarkable feature in fig. 11b is that a_1/a_0 may be larger than one, i.e. we have overreflection. The overreflection is even stronger in fig. 12, where the diffraction properties for $Fr = 0.7$ for an incoming k_2 wave are displayed.

In fig. 11c the incoming wave is a k_4 wave. For $\tau < 1/4$ a k_1 wave and a k_2 wave are set up. Hence in this case the motion consists of an incoming wave travelling downstream (the k_4 wave), a transmitted wave (the k_4 wave), a reflected wave (the k_2 wave) and a generated k_1 wave travelling downstream. For $\tau > 1/4$ the k_4 wave passes by the submerged body without noticing it. If the submerged elliptic cylinder is replaced by a circular cylinder, the wave motion for $\tau < 1/4$ is very different. In this case the k_1 wave and the k_2 wave are exactly zero, as shown in I.

creasing Froude numbers for incoming waves travelling downstream and decreases for incoming waves travelling upstream.

For $Fr = 0.2$ and $Fr = 0.4$ we see that the maximum value of the first order horizontal force is about $0.4 \rho g R a_0$. The ratio between this force and the viscous drag (3.11) is

$$\frac{X_1}{F} = \frac{0.4}{C_D Fr^2} \frac{a_0}{b} \quad (4.6)$$

Introducing $C_D = 0.15$ and $Fr = 0.2$ we obtain that $X_1/F \approx 67a_0/b$. The maximum value of the first order vertical force is $1.2 \rho g R a_0$, obtained for $Fr = 0.4$ and incoming wave travelling downstream. The ratio between this force and the viscous drag is then $X_2/F \approx 200a_0/b$.

5. THE LEE-WAVE PROBLEM

To solve the radiation problem, it is necessary to find the lee-wave potential χ . The knowledge of the lee-wave solution is also important to examine the validity of our solution. A necessary condition for the linearized solution to be a good approximation, is that $\partial\chi/\partial x \ll 1$. In fig. 14 is displayed $\max|\frac{\partial\chi}{\partial x}|$ at $x = -\infty$ when $b/R = 0.3$. It is seen that $\max|\frac{\partial\chi}{\partial x}|$ is less than 0.1 for Froude numbers less than 0.6. For the sake of completeness we have also shown the amplitudes of the lee-waves for various parameters.

6. SUMMARY AND CONCLUSION

The two-dimensional radiation problem and diffraction problem are discussed for submerged elliptic cylinders when a current is present. The amplitudes of the generated waves, the damping force, the first order horizontal force and the mean second order horizontal force are computed. It is found that the singularity at $\tau = 1/4$ has a great influence on the magnitude of amplitudes and forces, but both the wave amplitudes and forces remain finite when

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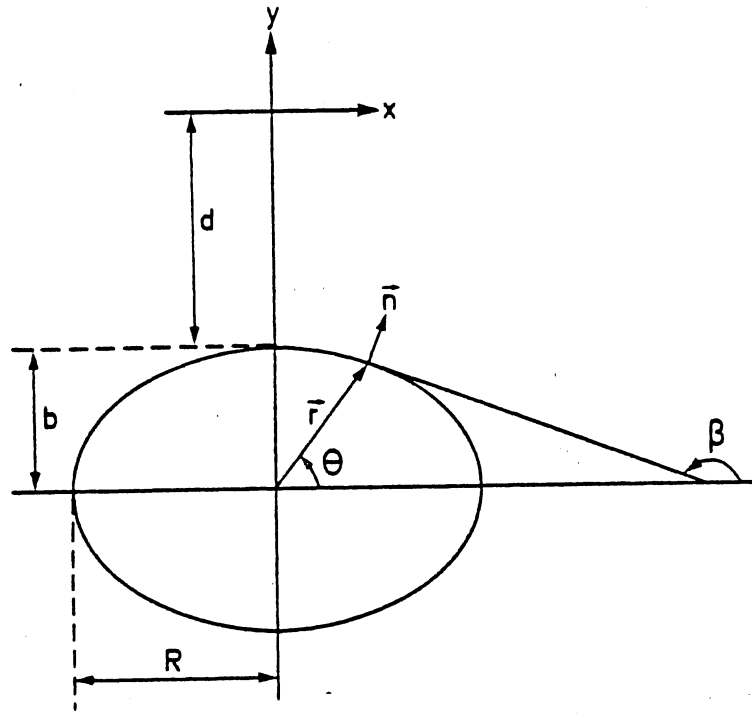


Fig. 1 Geometry of the problem.

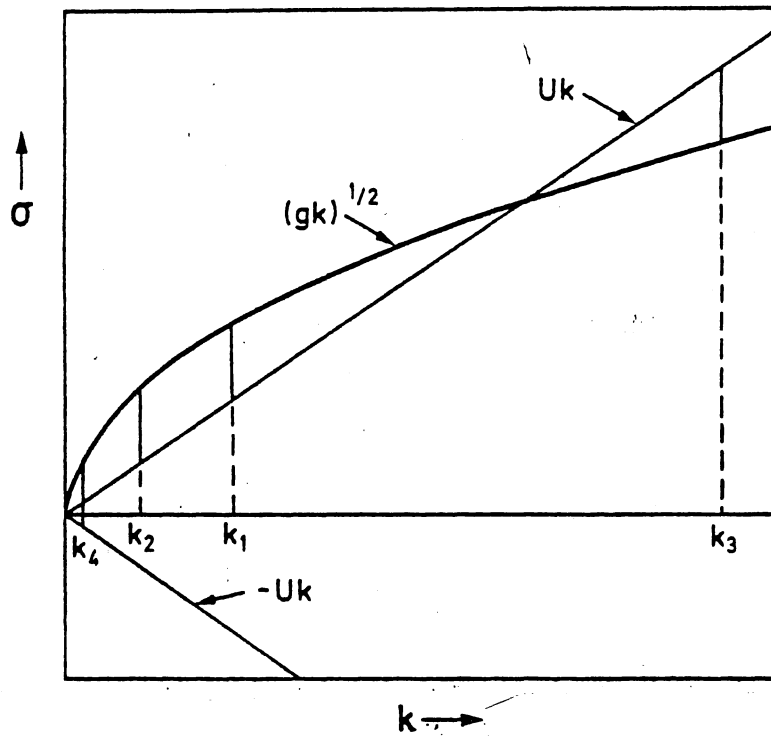


Fig. 2 The four wave numbers for given σ and U .

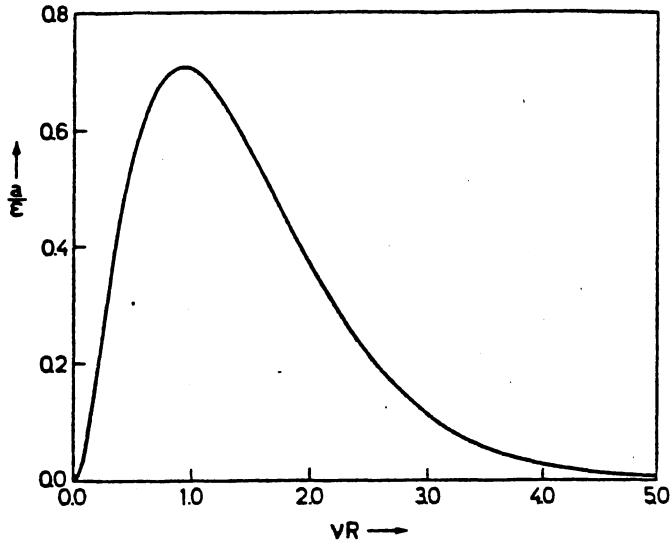


Fig. 3a. Amplitudes for radiated waves in sway or heave for a circle ($b/R=1.0$, $d/r=1.0$), $Fr=0$.

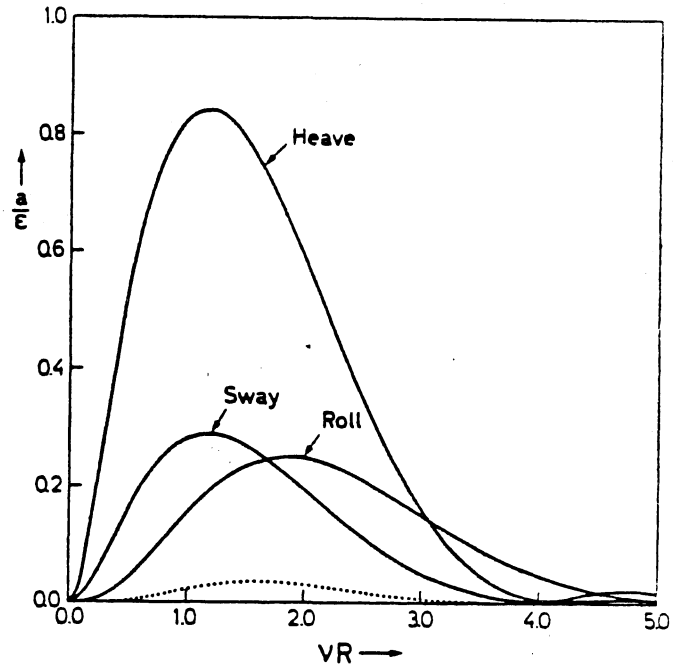


Fig. 3b. Amplitudes for radiated waves in heave, sway and roll ($b/R=0.3$, $d/R=1.0$), $Fr=0$. The dotted curve is the damping force $D/\rho g R E$ in roll.

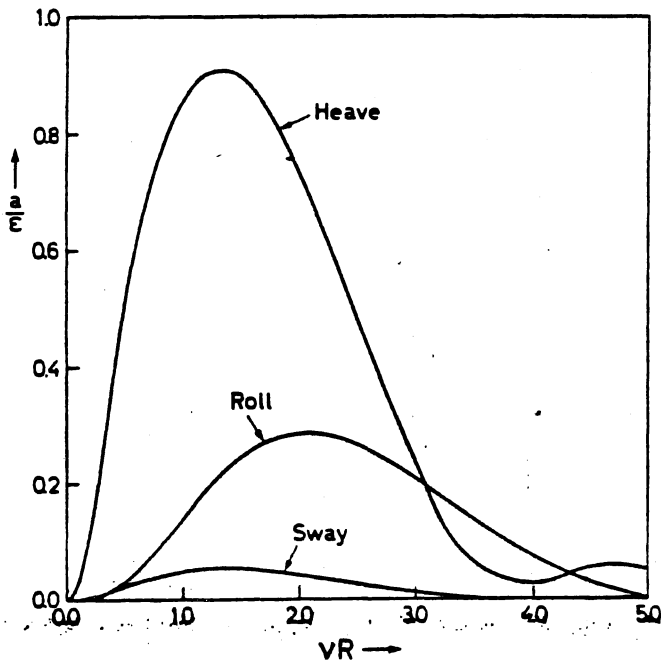


Fig. 3c. Amplitudes for radiated waves in heave, sway and roll ($b/R=0.05$, $d/R=1.0$), $Fr=0$.

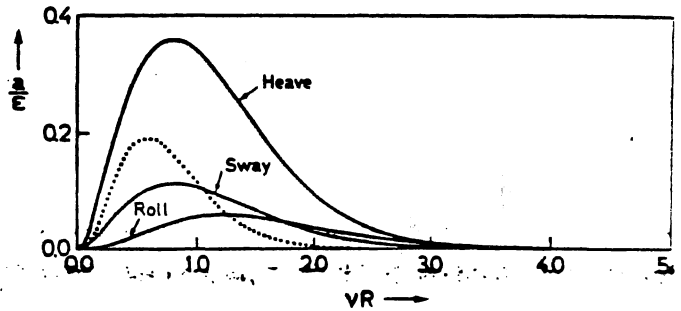


Fig. 4. Amplitudes for radiated waves in heave, sway and roll ($b/R=0.3$, $d/R=2.0$), $Fr=0$. The dotted curve is the damping force $D/\rho g R E$ in heave.

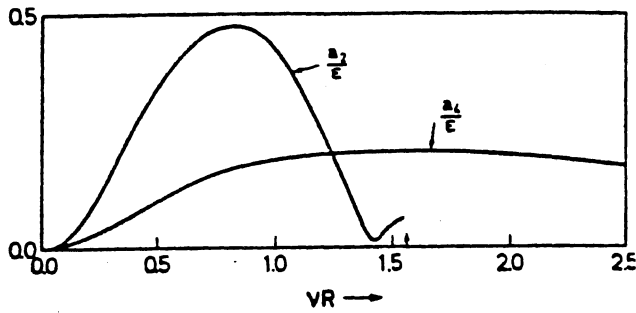


Fig. 5a.

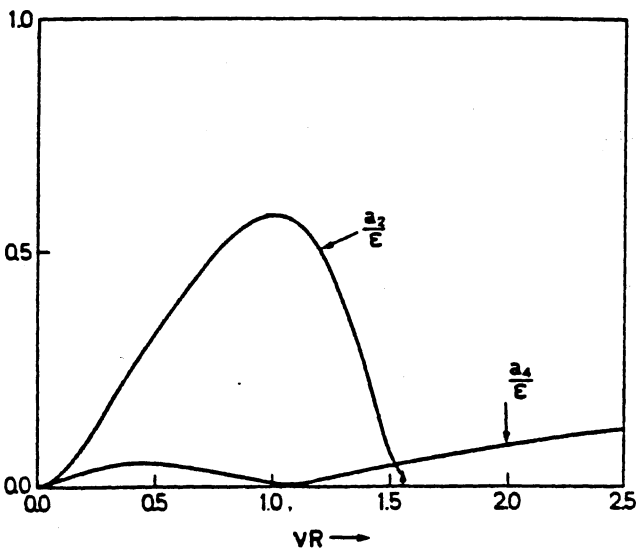


Fig. 5c.

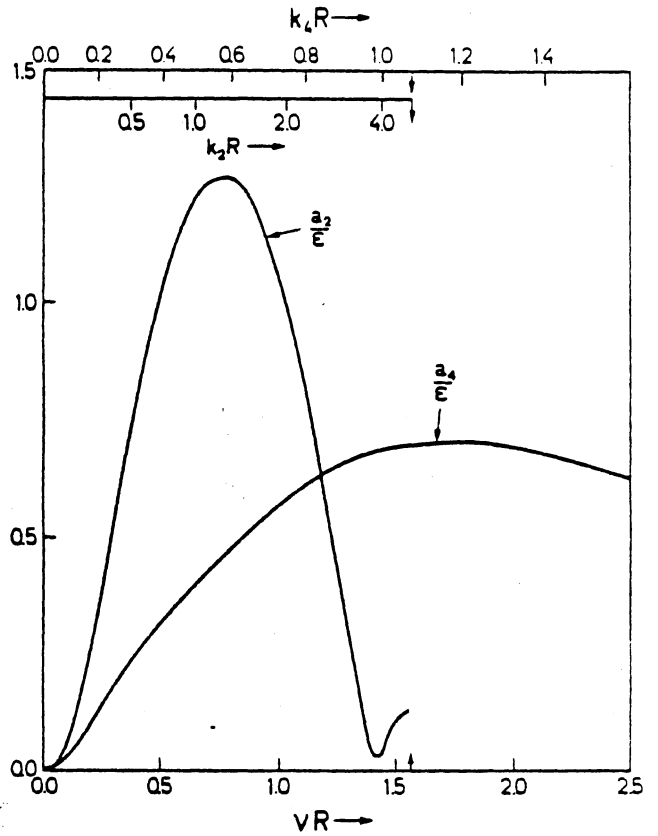


Fig. 5b.

Figs 5 Amplitudes for radiated waves in sway (5a), heave (5b) and roll (5c) for $Fr = 0.2$ ($b/R = 0.3$, $d/R = 1.0$). The arrows at the vR -axis indicate $\tau = 1/4$. On fig. 5b are indicated the values of the wave numbers k_2 and k_4 .

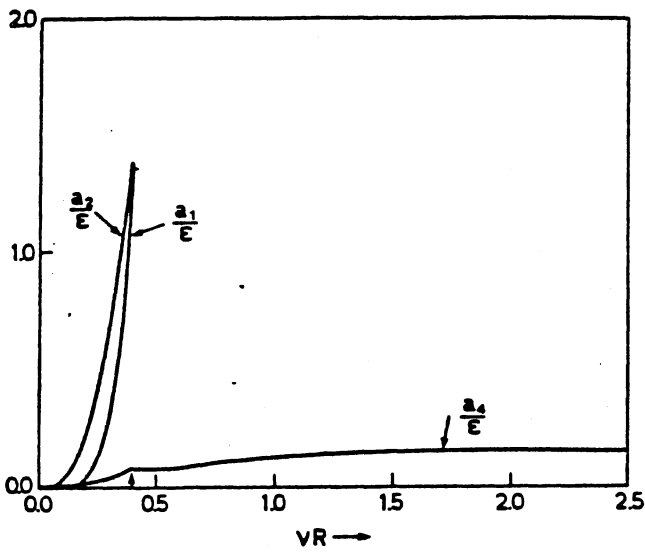


Fig. 6a.

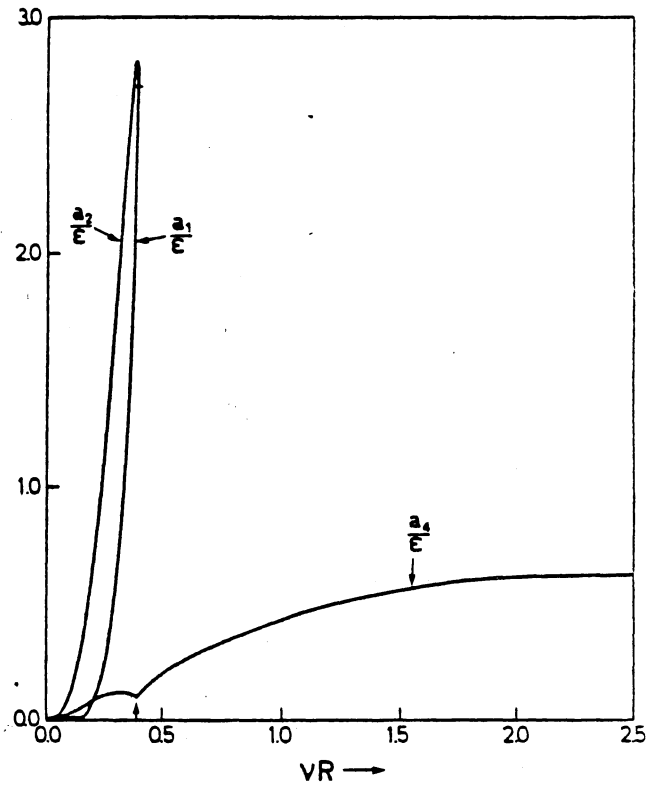


Fig. 6b.

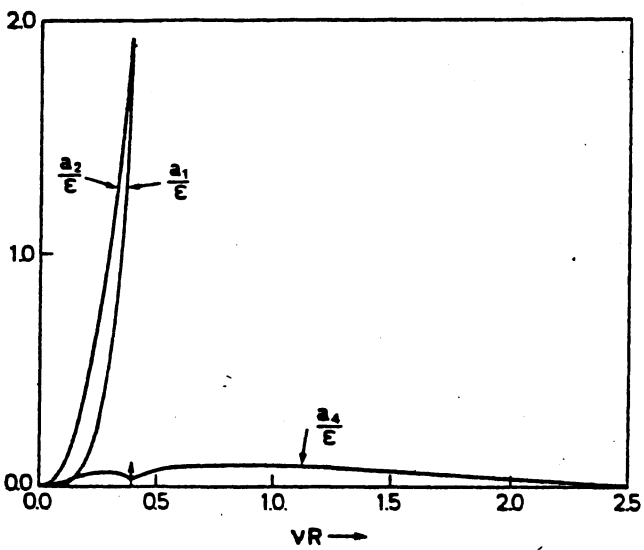


Fig. 6c.

Figs 6 Amplitudes for radiated waves in sway (6a), heave (6b) and roll (6c) for $Fr = 0.4$ ($d/R = 1.0$). The arrows at the vR -axis indicate $\tau = 1/4$.

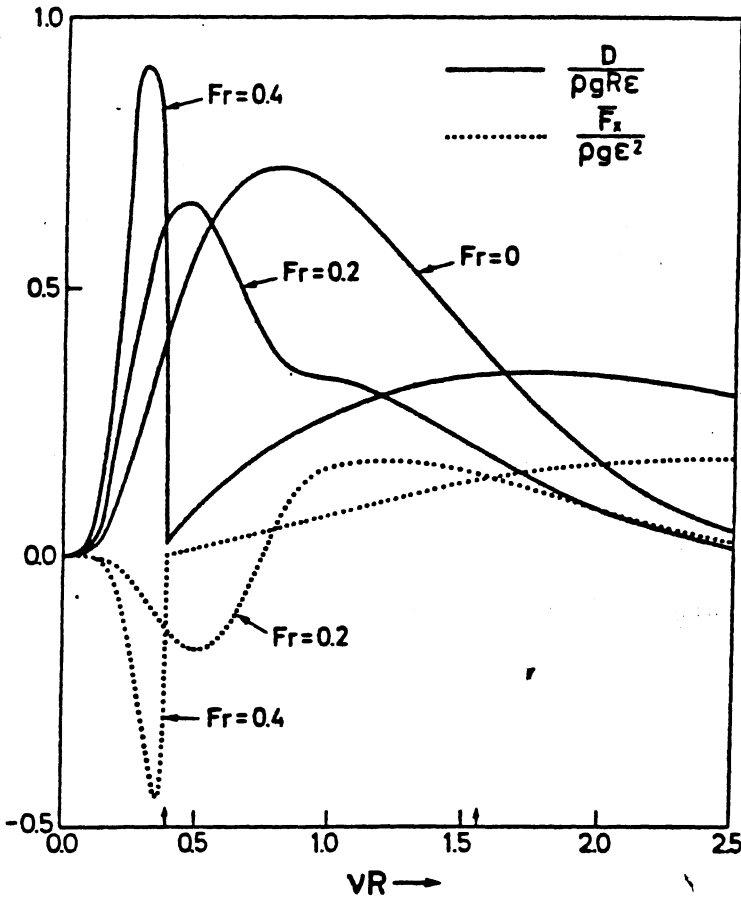


Fig. 7. Damping force $D/\rho g R \epsilon$ and mean second order horizontal force $\bar{F}_x/\rho g \epsilon^2$ in heave for $Fr=0, 0.2$ and 0.4 . The first arrow at the VR -axis indicates $\tau=1/4$ for $Fr=0.4$ and the second arrow $\tau=1/4$ for $Fr=0.2$.

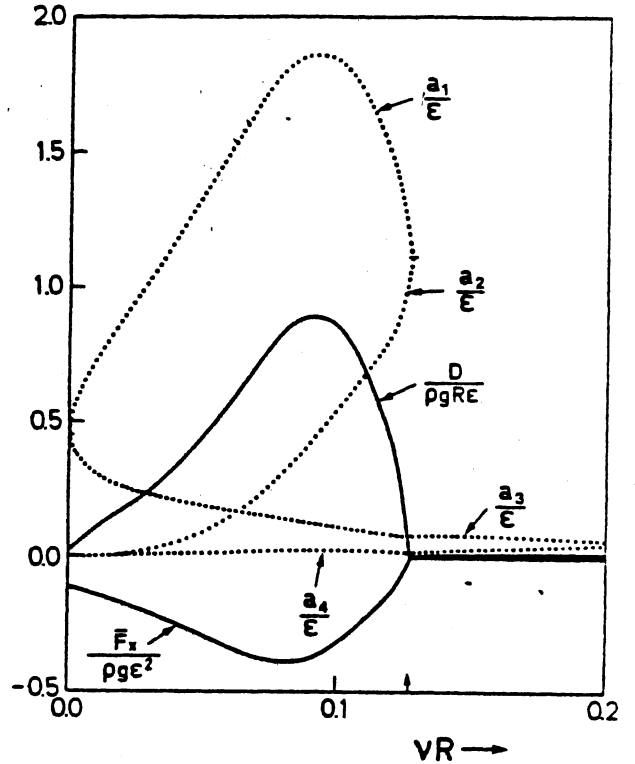


Fig. 8. Damping force $D/\rho g R \epsilon$ and mean second order horizontal force $\bar{F}_x/\rho g \epsilon^2$ in roll for $Fr=0.7$. Radiated amplitudes are dotted. The arrow at the VR -axis indicates $\tau=1/4$.

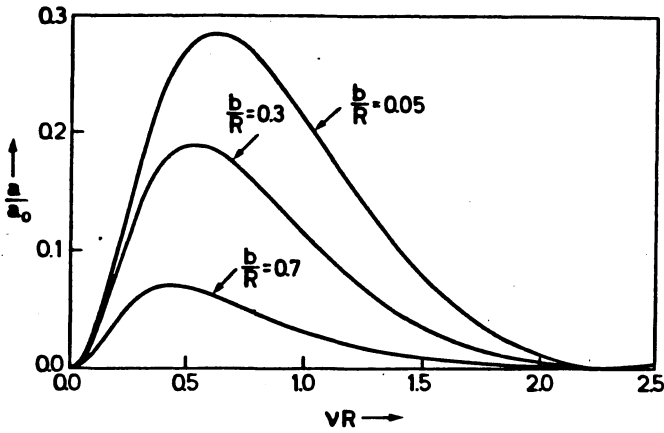


Fig. 9. Amplitudes of reflected waves in the diffraction problem for various values of b/R ($d/R=1.0$), $Fr=0$.

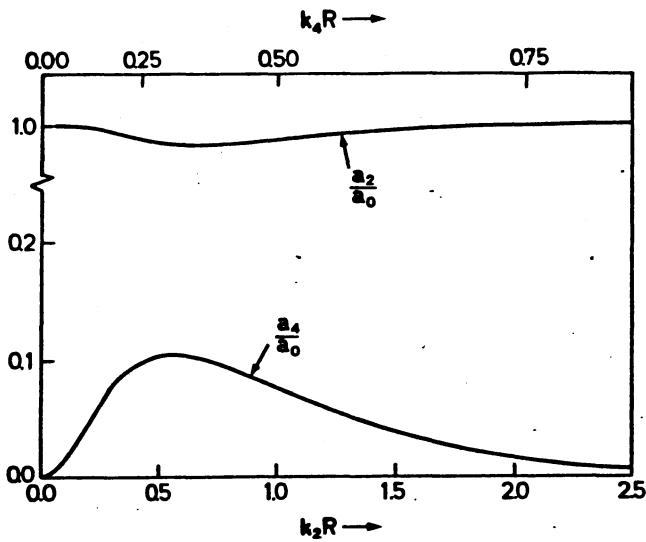


Fig. 10a. Amplitudes of the reflected k_4 wave and transmitted k_2 wave when a k_2 wave is incident upon the cylinder ($b/R=0.3$, $d/R=1.0$), $Fr=0.2$.

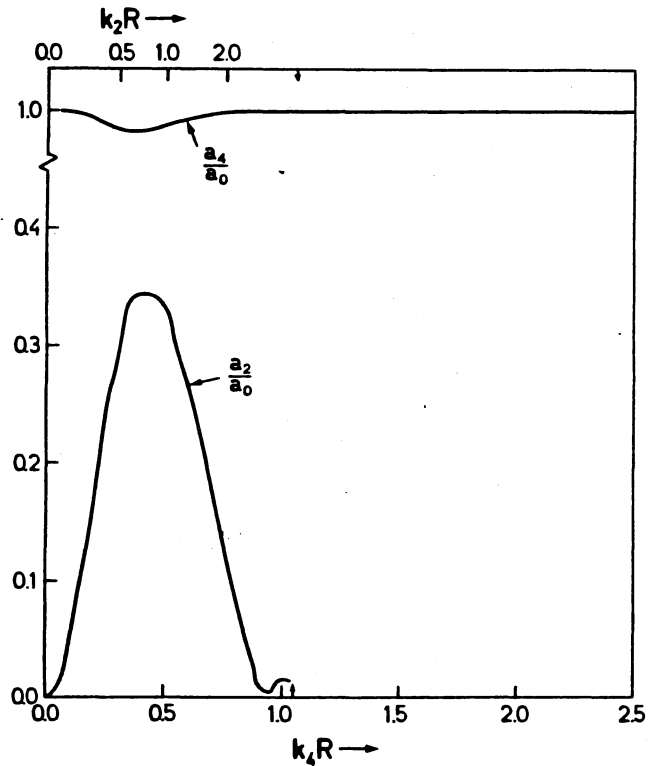


Fig. 10b. Amplitudes of the reflected k_2 wave and transmitted k_4 wave when a k_4 wave is incident upon the cylinder ($b/R=0.3$, $d/R=1.0$), $Fr=0.2$. The arrows at the kR -axis indicate $\tau=1/4$.

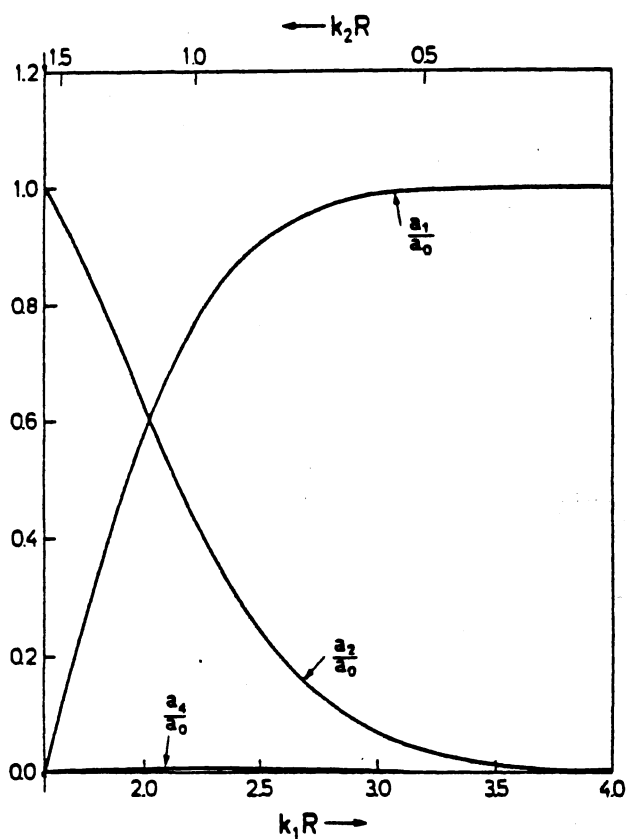


Fig. 11a. Amplitudes of the reflected k_2 wave, generated k_4 wave and transmitted k_1 wave when a k_1 wave is incident upon the cylinder ($b/R=0.3$, $d/R=1.0$), $Fr=0.4$. The arrows at the kR -axis indicate $\tau=1/4$.

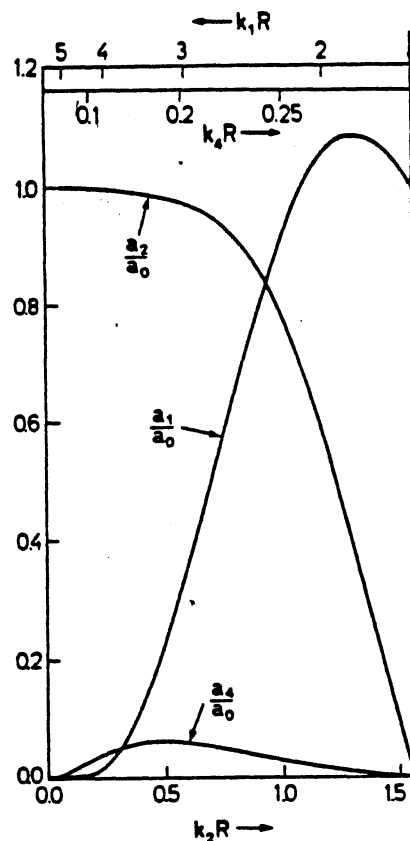


Fig. 11b. Amplitudes of the reflected k_4 wave and k_1 wave and transmitted k_2 wave when a k_2 wave is incident upon the cylinder ($b/R=0.3$, $d/R=1.0$), $Fr=0.4$. The arrows at the kR -axis indicate $\tau=1/4$.

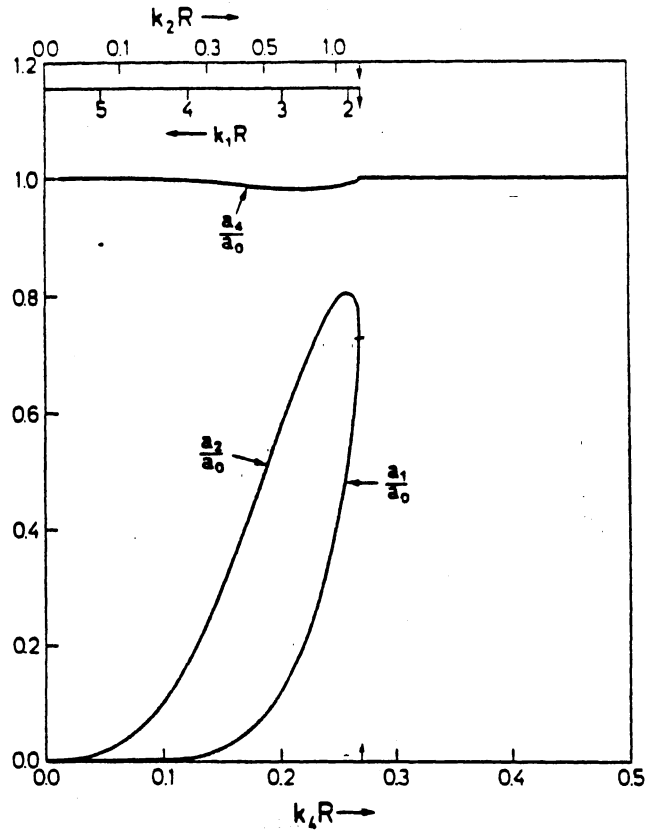


Fig. 11c. Amplitudes of the generated k_1 wave, reflected k_2 wave and transmitted k_4 wave when a k_4 wave is incident upon the cylinder ($b/R=0.3$, $d/R=1.0$), $Fr=0.4$. The arrows at the kR -axis indicate $\tau=1/4$.

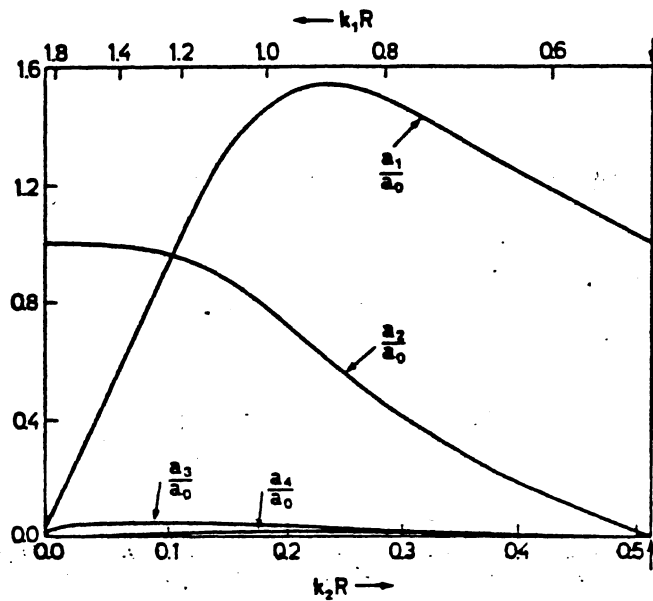


Fig. 12. Amplitudes of the reflected k_4 wave, k_1 wave and k_3 wave and transmitted k_2 wave when a k_2 wave is incident upon the cylinder ($b/R=0.3$, $d/R=1.0$), $Fr=0.7$. The arrows at the kP -axis indicate $\tau=1/4$.

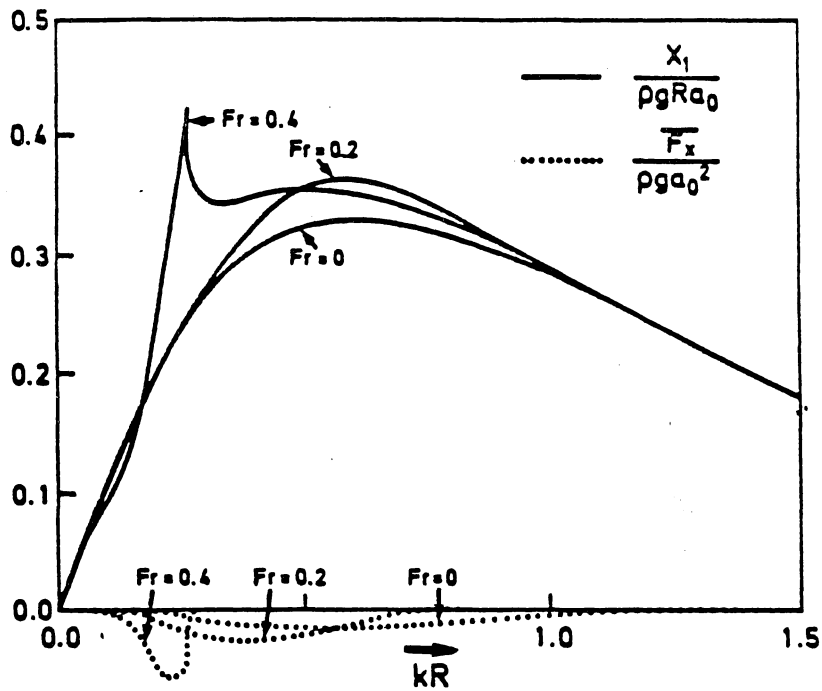


Fig. 13a. First order horizontal force $X_1/\rho g R a_0$ and mean second order horizontal force $\bar{F}_x/\rho g a_0^2$ for $Fr=0, 0.2$ and 0.4 with incoming wave travelling downstream ($b/R=0.3, d/R=1.0$). For $Fr=0.2$ and 0.4 the incoming wave is a k_4 wave.

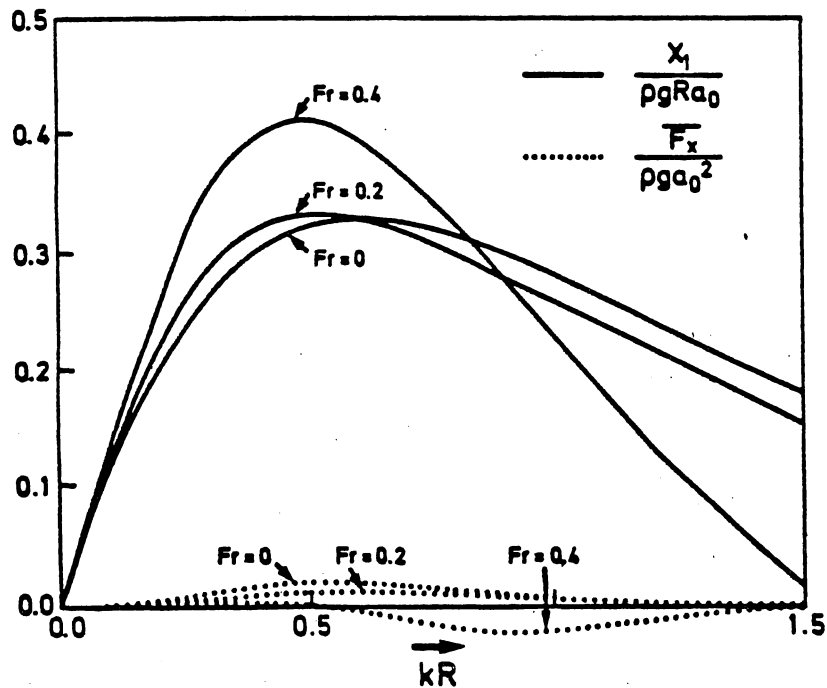


Fig. 13b. First order horizontal force $X_1/\rho g R a_0$ and mean second order horizontal force $\bar{F}_x/\rho g a_0^2$ for $Fr=0, 0.2$ and 0.4 with incoming wave travelling upstream ($b/R=0.3, d/R=1.0$). For $Fr=0.2$ and 0.4 the incoming wave is a k_2 wave.

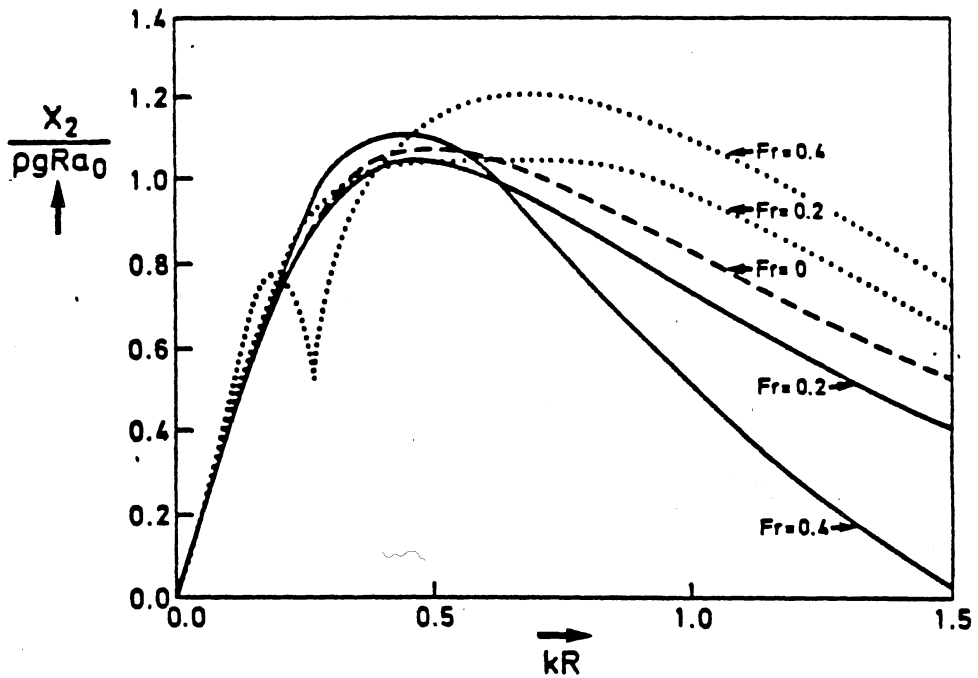


Fig. 13c. First order vertical force $X_2/\rho g R a_0$ for $Fr=0$ (---), for $Fr=0.2$ and 0.4 with incoming wave travelling downstream (\cdots) and for $Fr=0.2$ and 0.4 with incoming wave travelling upstream (—), ($b/R=0.3$, $d/R=1.0$).

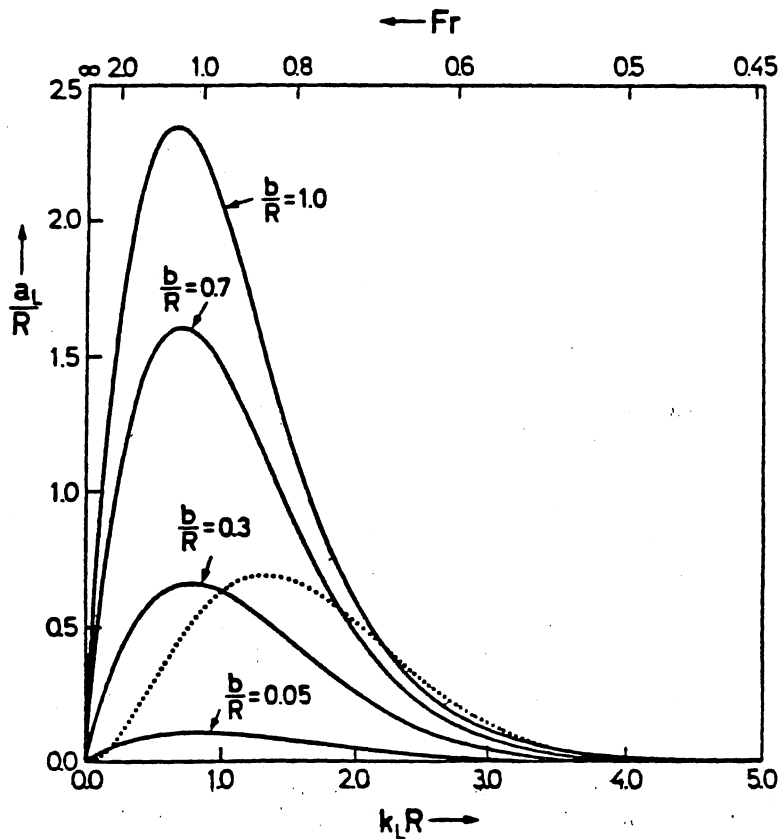


Fig. 14. Amplitude of the lee-wave at $\chi=-\infty$ for various values of b/R ($d/R=1.0$). The lee-wave number k_L is given by $k_L=g/U^2=1/Fr^2R$. The dotted curve is $\max|\partial\chi/\partial x|$ at $x=-\infty$ for $b/R=0.3$ ($d/R=1.0$).