

# Operator Norm-based Statistical Linearization to Bound the First Excursion Probability of Nonlinear Structures Subjected to Imprecise Stochastic Loading

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## ABSTRACT

This paper presents a highly efficient approach for bounding the responses and probability of failure of nonlinear models subjected to imprecisely defined stochastic Gaussian loads. Typically,

25 such computations involve solving a nested double loop problem, where the propagation of the  
26 aleatory uncertainty has to be performed for each realization of the epistemic parameters. Apart  
27 from near-trivial cases, such computation is generally intractable without resorting to surrogate  
28 modeling schemes, especially in the context of performing nonlinear dynamical simulations. The  
29 recently introduced operator norm framework allows for breaking this double loop by determining  
30 those values of the epistemic uncertain parameters that produce bounds on the probability of  
31 failure a priori. However, the method is in its current form only applicable to linear models due  
32 to the adopted assumptions in the derivation of the involved operator norms. In this paper, the  
33 operator norm framework is extended and generalized by resorting to the statistical linearization  
34 methodology, to account for nonlinear systems. Two case studies are included to demonstrate the  
35 validity and efficiency of the proposed approach.

36 **Keywords:** Uncertainty quantification; Imprecise probabilities; Operator norm theorem; Statisti-  
37 cal linearization

## 38 INTRODUCTION

39 Uncertainties about the true properties of, and loads acting on, structural systems are commonly  
40 encountered in the context of all fields of engineering, including structural dynamics. For instance,  
41 natural phenomena such as earthquakes or wind loads are especially hard to model exactly, since  
42 the corresponding dynamical loads acting on the system often cannot be described in a crisp way  
43 due to the sheer complexity of the underlying phenomena. Further, when designing structures with  
44 natural or highly engineered materials, such uncertainties may arise as well. To treat these issues  
45 effectively, stochastic processes (Shinozuka and Sato 1967, Vanmarcke and Grigoriu 1983) have  
46 been introduced as a rigorous framework to account for the aleatory uncertainties and corresponding  
47 correlations in space and time of uncertain loads and properties. This is obtained by resorting to  
48 the well-documented framework of probability theory, which is highly suited to treat aleatory  
49 uncertainties.

50 However, the definition of such stochastic processes may require prohibitive amounts of in-  
51 formative data to fully characterize the probabilistic descriptors, including the auto-correlation

52 function. In a practical engineering context, such information may not always be available due to  
53 scarcity, incompleteness or even conflicted nature of typically available data sources. As a potential  
54 remedy, one can model the additional (epistemic) uncertainty by means of subjective probability  
55 density functions, which might be a valid approach in case sufficient reasons are present to validate  
56 the considered assumptions. However, in general, this includes unwarranted subjectivity in the  
57 analysis, which might give a wrong sense of reliability to the model. Alternatively, set theoretical  
58 approaches, such as intervals (Faes and Moens 2019b) or fuzzy numbers (Beer 2004), can be  
59 used to include the epistemic uncertainty. By imposing such set-theoretical descriptors on top of  
60 probabilistic models for the uncertainty, a full set of probabilistic models that is consistent with  
61 the lack of knowledge is considered, which allows for an objective judgement on the bounds of the  
62 system reliability. In this context, utilizing the concept of imprecise probabilities (Beer et al. 2013)  
63 provides the analyst with a concrete theoretical framework to define and compute (with such hybrid  
64 forms) the uncertainties. In structural dynamics, for instance, given a set of stochastic processes that  
65 are consistent with the epistemic uncertainty, an imprecise probabilities-based solution treatment  
66 leads to bounds on the first excursion probability. The latter not only allows to assess the sensitivity  
67 of the model reliability to the existing epistemic uncertainty, but also yields an estimate of the lower  
68 bound of the reliability.

69 In engineering practice, however, the effective application of such methods is typically hindered  
70 by the corresponding computational cost. In essence, the propagation of the epistemic and aleatory  
71 uncertainty has to be performed such that their effects on the reliability are kept separated (Moens  
72 and Vandepitte 2004). This gives rise to double loop approaches, where the outer loop takes care of  
73 epistemic uncertainty while the inner loop deals with aleatory uncertainty. Many efficient methods  
74 have been introduced in recent years to alleviate this computational cost; see, indicatively, Faes  
75 et al. (2021a) for a recent review paper. Examples of such approaches are based on Extended  
76 Monte Carlo simulation (Wei et al. 2019), surrogate modeling schemes (Schöbi and Sudret 2017),  
77 Bayesian probabilistic propagation (Wei et al. 2021) or Line Sampling (de Angelis et al. 2015).  
78 A recent development in this context is based on operator norm theory to decouple the double

79 loop into a deterministic optimization, followed by a single reliability analysis per bound on the  
80 reliability (Faes et al. 2020; 2021b), which is capable of reducing the corresponding computational  
81 cost by several orders of magnitude. However, the methods based on operator norm theory are  
82 limited to linear systems subject to Gaussian loading, which renders their application to realistic  
83 engineering models impossible.

84 In this regard, directing attention to extending the operator norm framework to nonlinear  
85 dynamical systems subject to imprecise Gaussian loading, a new technique is developed herein for  
86 computing moderate to large failure probabilities. This is attained by resorting to the statistical  
87 linearization methodology (Roberts and Spanos 2003, Socha 2007), which is used for defining  
88 an equivalent linear system of equations to account for the nonlinear system under consideration.  
89 Then, an operator norm theory-based solution treatment (Faes et al. 2021b) is employed to obtain  
90 the bounds on the probability of failure. Two pertinent numerical examples demonstrate the validity  
91 and efficiency of the proposed methodology.

## 92 **BOUNDS ON THE RELIABILITY OF NONLINEAR DYNAMICAL SYSTEMS**

### 93 **Nonlinear stochastic dynamics**

94 A nonlinear dynamical system subjected to a stochastic load  $p(t, \xi)$  is represented using the  
95 Finite Element representation of the dynamical equation, by the following set of ordinary differential  
96 equations:

$$97 \quad \mathbf{M}\ddot{\mathbf{q}}(t) + \mathbf{C}\dot{\mathbf{q}}(t) + \mathbf{K}\mathbf{q}(t) + \mathbf{\Phi}(\ddot{\mathbf{q}}(t), \dot{\mathbf{q}}(t), \mathbf{q}(t)) = \boldsymbol{\rho}p(t, \xi), \quad (1)$$

98 where  $\mathbf{M} \in \mathbb{R}^{n_d \times n_d}$ ,  $\mathbf{C} \in \mathbb{R}^{n_d \times n_d}$  and  $\mathbf{K} \in \mathbb{R}^{n_d \times n_d}$  represent, respectively, the mass, damping and  
99 stiffness matrices of the system, and  $n_d$  denotes the degrees of freedom in the model. Further,  
100  $\xi$  represents a realization of a random variable vector, whereas the vector  $\boldsymbol{\rho} \in \mathbb{R}^{n_d \times 1}$  links the  
101 stochastic load  $p(t, \xi)$  to the appropriate degrees of freedom in the structure. The vectors  $\mathbf{q} \in \mathbb{R}^{n_d}$ ,  
102  $\dot{\mathbf{q}} \in \mathbb{R}^{n_d}$  and  $\ddot{\mathbf{q}} \in \mathbb{R}^{n_d}$  represent, respectively, the nodal displacements, velocities and accelera-  
103 tions, where a dot over a variable denotes differentiation with respect to time  $t \in \mathbb{R}$ . Finally,  
104  $\mathbf{\Phi}(\ddot{\mathbf{q}}(t), \dot{\mathbf{q}}(t), \mathbf{q}(t)) \in \mathbb{R}^{n_d}$  represents the nonlinear restoring force, which depends on the nodal

105 displacement, velocity and acceleration vectors.

106 In Eq. (1),  $p(t, \xi)$  represents the load to which the system is subjected, which in the context of  
 107 stochastic dynamical systems is usually modeled as a stochastic process. If  $p(t, \xi)$  is a stationary  
 108 zero-mean Gaussian process, it can be characterized using its power spectral density function  
 109  $S_{PP}(\omega)$ , where  $\omega \in \mathbb{R}$  denotes the circular frequency. The Wiener-Khinchine theorem allows for  
 110 the calculation of the autocorrelation function corresponding to  $S_{PP}(\omega)$ , and vice versa. This is  
 111 attained by utilizing the Fourier transforms:

$$112 \quad S_{PP}(\omega) = \frac{1}{2\pi} \int_{-\infty}^{+\infty} R_{PP}(\tau) e^{-i\omega\tau} d\tau, \quad R_{PP}(\tau) = \int_{-\infty}^{+\infty} S_{PP}(\omega) e^{i\omega\tau} d\omega, \quad (2)$$

where  $R_{PP}(\tau)$  denotes the autocorrelation function with time lag  $\tau \in \mathbb{R}$  and ‘i’ is the imaginary  
 unit. Sample paths of this stochastic process can be generated, for example, by applying the  
 Karhunen-Loève (KL) expansion (e.g., Schenk and Schuëller 2005, Stefanou 2009). In this regard,  
 assume that the loading is applied for time  $T$ , where  $t_k = (k - 1)\Delta t$ ,  $k = 1, 2, \dots, n_T$ , corresponds  
 to time discretization with step  $\Delta t$  and  $n_T$  denotes the number of discrete time steps. Then, the  
 associated discrete correlation matrix  $\mathbf{R}_{PP} \in \mathbb{R}^{n_T \times n_T}$  becomes:

$$\mathbf{R}_{PP} = \begin{bmatrix} R_{PP}(0) & R_{PP}(t_1 - t_2) & \dots & R_{PP}(t_1 - t_{n_T}) \\ R_{PP}(t_2 - t_1) & R_{PP}(0) & \dots & R_{PP}(t_2 - t_{n_T}) \\ \vdots & \vdots & \ddots & \vdots \\ R_{PP}(t_{n_T} - t_1) & R_{PP}(t_{n_T} - t_2) & \dots & R_{PP}(0) \end{bmatrix}. \quad (3)$$

113 Note that the framework described above can be also extended to account for non-stationary  
 114 Gaussian processes, see e.g. Li and Chen (2009a). Utilizing the matrix-vector form of the KL  
 115 expansion, i.e.:

$$116 \quad \mathbf{p}(\xi) = \mathbf{\Psi} \mathbf{\Lambda}^{1/2} \xi, \quad (4)$$

117 sample paths compatible with the stochastic ground acceleration are generated. In Eq. (4),  $\mathbf{p}$   
 118 denotes an  $n_T$ -dimensional vector containing the sample of the loading;  $\xi$  is a realization of

119 the random variable vector  $\Xi$ , which follows an  $n_{KL}$ -dimensional standard Gaussian distribution,  
120 where  $n_{KL}$  corresponds to the number of terms retained in the KL expansion;  $\Psi \in \mathbb{R}^{n_T \times n_{KL}}$   
121 is a matrix whose columns contain the eigenvectors associated with the largest  $n_{KL}$  eigenvalues  
122 of the discrete covariance matrix  $\mathbf{R}_{pp}$ ; and  $\Lambda \in \mathbb{R}^{n_{KL} \times n_{KL}}$  denotes a diagonal matrix whose  
123 elements contain the largest  $n_{KL}$  eigenvalues of  $\mathbf{R}_{pp}$ . A criterion for selecting the number  
124 of terms to be retained in the KL expansion is to find the minimum value of  $n_{KL}$ , such that  
125  $\sum_{p=1}^{n_{KL}} \lambda_p \geq p_v \sum_{p=1}^{n_T} \lambda_p$ , where  $p_v$  denotes the fraction of the total variance of the underlying  
126 stochastic process that is retained by the approximate representation, and  $\lambda_p$  is the  $p$ -th eigenvalue  
127 of  $\mathbf{R}_{pp}$  (Lee and Verleysen 2007). For a recent overview of numerical methods to solve the associ-  
128 ated Fredholm integral eigenvalue problem in a continuous case, the reader is directed to Betz et al.  
129 (2014). Alternatively, the sample paths can also be generated using frequency domain methods,  
130 such as described in Chen and Li (2013).

131 In a structural engineering context, one is usually interested in finding the reliability of the  
132 structure, which is related to its performance by means of Eq. (1). Practically, the structural  
133 reliability can be quantified by its complement, i.e., the failure probability  $P_f$ . In this context,  
134 failure is encoded in the performance function  $g(\xi)$ , i.e.,  $g(\xi) \leq 0$  indicates that the realization of  
135 values  $\xi$  leads to a structural failure. The probability of failure is calculated by solving the integral  
136 equation:

$$137 \quad P_f = \int_{\xi \in \mathbb{R}^{n_{KL}}} I_F(\xi) f_{\Xi}(\xi) d\xi, \quad (5)$$

138 where  $f_{\Xi}(\cdot)$  is a standard  $n_{KL}$ -dimensional Gaussian probability density function and  $I_F(\cdot)$  is the  
139 indicator function, whose value is equal to one in case  $g(\xi) \leq 0$  and zero otherwise. Note, in  
140 passing, that the exact formulation of  $g(\xi)$  is highly case dependent. For instance, when considering  
141 the first-passage problem, which is a classical problem in stochastic dynamics (e.g., Spanos and  
142 Kougioumtzoglou 2014, Spanos et al. 2016),  $g(\xi)$  is given by:

$$143 \quad g(\xi) = 1 - \max_{i=1, \dots, n_\eta} \left( \max_{k=1, \dots, n_T} \left( \frac{|\eta_i(t_k, \xi)|}{b_i} \right) \right). \quad (6)$$

144 where  $\eta_i(t_k, \xi)$ ,  $i = 1, 2, \dots, n_\eta$ , indicates the  $i$ -th response of the system at time instant  $t_k$  (e.g.,  $q_i$   
 145 or one of its time derivatives),  $|\cdot|$  denotes the absolute value and  $b_i$  is a predefined threshold value  
 146 above which a structural failure occurs (e.g., a maximally allowed displacement).

147 The integral in Eq. (5) usually comprises a high number of dimensions, as  $n_{KL}$  may be in the  
 148 order of hundreds or thousands for realistic stochastic processes. Furthermore,  $g(\xi)$ , and hence,  
 149  $I_F(\xi)$  is only known point-wise for realizations  $\xi$  of  $\Xi$ . Therefore, such an integral cannot be  
 150 solved analytically. In general, simulation methods should be applied to evaluate  $P_f$  (Schuëller and  
 151 Pradlwarter 2007). However, using simulation methods to calculate the probability of failure of a  
 152 non-linear dynamical system can become quite challenging (Pradlwarter et al. 2007). For instance,  
 153 the definition of appropriate importance sampling density functions to be used within the context of  
 154 Importance Sampling might not always be trivial in this case (Au 2009). Moreover, it is highlighted  
 155 that the nonlinear restoring force  $\Phi(\ddot{\mathbf{q}}(t), \dot{\mathbf{q}}(t), \mathbf{q}(t))$  in Eq. (1) hinders the determination of  
 156  $\eta_i(t_k), i = 1, 2, \dots, n_\eta, k = 1, 2, \dots, n_T$ , since its presence necessitates the employment of pertinent  
 157 numerical algorithms (Chopra 1995). In particular, combining simulation algorithms with these  
 158 nonlinear solvers potentially leads to solution frameworks of prohibitively high computational cost.

### 159 **Imprecise stochastic dynamical analysis**

160 The characterization of the stochastic process  $p(t, \xi)$  in Eq. (1) in terms of its power spectral  
 161 density, or autocorrelation function, usually relies on a prescribed model. This, in turn, depends on  
 162 a number of parameters, which are grouped in a vector  $\theta \in \mathbb{R}^{n_\theta}$ . In this case, the parameters that  
 163 determine the covariance matrix  $\mathbf{R}_{pp}(\tau|\theta)$  reflect some specific characteristics of the process, such  
 164 as dominant frequencies, amplitude, etc. When selecting the appropriate value of these quantities,  
 165 the analyst may be faced with considerable uncertainty, such as lack of knowledge, vague or  
 166 ambiguous information, etc., which leads to epistemic uncertainty concerning the correct parameter  
 167 value. Therefore, instead of selecting a crisp value, it is often preferred to explicitly account for this  
 168 epistemic uncertainty by resorting to non-traditional models for uncertainty quantification (Beer  
 169 et al. 2013).

170 In this regard, it is herein assumed that the epistemic uncertainty in the definition of  $\theta$  can be

171 bounded by an interval, i.e.,  $\theta \in \theta^I = [\underline{\theta}, \bar{\theta}]$ , where  $\underline{\theta}$  and  $\bar{\theta}$  denote, respectively, the lower and  
 172 upper bound between which the *true* parameter value is believed to lie. Techniques to infer these  
 173 bounds based on limited data have been reported; see, indicatively, Imholz et al. (2020). Taking  
 174 these uncertainties explicitly into account, Eq. (1) becomes:

$$175 \quad \mathbf{M}\ddot{\mathbf{q}}(t) + \mathbf{C}\dot{\mathbf{q}}(t) + \mathbf{K}\mathbf{q}(t) + \Phi(\ddot{\mathbf{q}}(t), \dot{\mathbf{q}}(t), \mathbf{q}(t)) = \rho p(t, \xi, \theta^I). \quad (7)$$

176 Close inspection of Eq. (7) reveals that both interval and random variables are present. The fact that  
 177 the input parameters of the stochastic loading model are described by means of intervals has impor-  
 178 tant implications on the evaluation of the structural reliability of the model under consideration. In  
 179 particular, both loading and the structural system responses become interval stochastic processes  
 180 (Faes and Moens 2019a). This, in turn, leads to an interval valued performance function, which  
 181 causes the failure probability to become interval valued as well. Therefore, instead of calculating  
 182 a single probability of failure associated with the structure (using Eq. (5)), given the epistemic  
 183 uncertainty represented by  $\theta^I$ , one has to estimate the bounds on  $P_f$ . These bounds are calculated  
 184 by solving the optimization problems:

$$185 \quad \underline{P}_f = \min_{\theta \in \theta^I} (P_f(\theta)) = \min_{\theta \in \theta^I} \left( \int_{\xi \in \mathbb{R}^{n_{KL}}} I_F(\xi, \theta) f_{\Xi}(\xi) d\xi \right), \quad (8)$$

$$186 \quad \bar{P}_f = \max_{\theta \in \theta^I} (P_f(\theta)) = \max_{\theta \in \theta^I} \left( \int_{\xi \in \mathbb{R}^{n_{KL}}} I_F(\xi, \theta) f_{\Xi}(\xi) d\xi \right). \quad (9)$$

188 In general, the solution of the optimization problems defined in Eqs. (8) and (9) is extremely  
 189 demanding from a computational perspective. Specifically, as pointed out earlier, the solution of  
 190 the reliability problem for nonlinear dynamical systems is rather cumbersome. In addition, solving  
 191 the corresponding optimization problems is not straightforward, since this constitutes a double loop  
 192 problem, where the inner loop comprises probability calculation, while the outer loop explores the  
 193 possible values of the parameters  $\theta$ . Hence, besides considering near-trivial simulation models,  
 194 such computation is generally intractable without resorting to surrogate modelling strategies.



## OPERATOR NORM THEORY AS A TOOL TO DECOUPLE THE DOUBLE LOOP

A highly efficient operator norm theory-based approach to decouple the double loop associated with the solution of Eqs. (8) and (9) has already been developed by some of the authors of the present paper (Faes et al. 2021b; 2020). In this section, a concise presentation of the results in Faes et al. (2021b, 2020) is provided for completeness. Then, directing attention to computing the bounds on the probability of failure of the nonlinear system given by Eq. (7), a novel methodology is proposed, which is based on the combination of the statistical linearization method (Roberts and Spanos 2003) with the theoretical framework described above.

### Linear problems

The operator norm method introduced in Faes et al. (2021b, 2020), specifically focuses on models whose relation between the response  $\eta$  and the uncertain inputs  $\theta$  and  $\xi$  is given by:

$$\eta(\theta, \xi) = \mathbf{A}\mathbf{B}(\theta)\xi. \quad (10)$$

In Eq. (10),  $\mathbf{A} : \mathbb{R}^{n_r} \mapsto \mathbb{R}^{n_\eta}$  denotes a continuous linear map that represents the translation of the model input to the responses of interest, whereas  $\mathbf{B} : \mathbb{R}^{n_{KL}} \mapsto \mathbb{R}^{n_r}$  is a linear map that transforms the random vector  $\xi$  to the sample paths of the stochastic process which serves as model input. For instance, using the KL series expansion,  $\mathbf{B}$  is given in its discrete form as:

$$\mathbf{B} = \mathbf{\Psi}\mathbf{\Lambda}^{1/2}, \quad (11)$$

where  $\mathbf{\Psi}$  and  $\mathbf{\Lambda}$  are the matrices which contain, respectively, the eigenvectors and eigenvalues of the matrix  $\mathbf{R}_{pp}$  (see also section “Bounds on the reliability of nonlinear dynamical systems”). Note that eq. (10) allows modeling the dynamic response of linear structural systems comprising classical or non-proportional damping subject to dynamic loading. Details about the numerical formulation of eq. (10) can be found in, e.g., Chopra (1995); Jensen and Valdebenito (2007).

Considering the linear map defined in Eq. (10) and also defining  $\mathbf{D}(\theta) = \mathbf{A}\mathbf{B}(\theta)$  for simplicity,

218 it can be shown that the inequality:

$$219 \quad \|\mathbf{D}(\boldsymbol{\theta})\boldsymbol{\xi}\|_{p_1} \leq |c| \|\boldsymbol{\xi}\|_{p_2}, \quad (12)$$

220 with  $\|\cdot\|_p$  denoting a certain  $L_p$  norm, always holds. In essence, this equation states that the length  
 221 of the uncertain model input  $\boldsymbol{\xi}$ , quantified via a prescribed  $L_{p_i}$  norm, can be amplified at most by  
 222 a factor  $c$  towards the model responses  $\boldsymbol{\eta}$  when applying the linear mapping defined by  $\mathbf{D}(\boldsymbol{\theta})$ . A  
 223 measure for *how much* a certain deterministic linear map  $\mathbf{D}(\boldsymbol{\theta})$  increases the length of the uncertain  
 224 model input  $\mathbf{v}$  in the maximum case, is given by the operator norm  $\|\mathbf{D}(\boldsymbol{\theta})\|_{p_1, p_2}$ , which is defined  
 225 in a deterministic sense (i.e., for one realization of the uncertain parameters) as:

$$226 \quad \|\mathbf{D}(\boldsymbol{\theta})\|_{p_1, p_2} = \inf \{c \geq 0 : \|\mathbf{D}(\boldsymbol{\theta})\mathbf{v}\|_{p_1} \leq |c| \cdot \|\mathbf{v}\|_{p_2}, \forall \mathbf{v} \in \mathbb{R}^{n_v}\}, \quad (13)$$

227 or, equivalently:

$$228 \quad \|\mathbf{D}(\boldsymbol{\theta})\|_{p_1, p_2} = \sup \left\{ \frac{\|\mathbf{D}(\boldsymbol{\theta})\mathbf{v}\|_{p_1}}{\|\mathbf{v}\|_{p_2}} : \mathbf{v} \in \mathbb{R}^{n_v} \text{ with } \mathbf{v} \neq 0 \right\}. \quad (14)$$

229 Clearly, the calculation of a specific value  $\|\mathbf{D}(\boldsymbol{\theta})\|_{p_1, p_2}$  depends on the choice of  $p_1$  and  $p_2$ . The  
 230 interested reader is directed to Faes et al. (2021b, 2020) for an analytical presentation of the method  
 231 and for guidance on the optimal selection of  $p_1$  and  $p_2$ ; and to Faes and Valdebenito (2020, 2021)  
 232 for a practical application of the framework in the context of reliability-based design optimization.

233 In case of calculating first excursion probabilities, taking into account Eq. (6), experience  
 234 shows that selecting  $p_1 \rightarrow \infty$  and  $p_2 = 2$  provides a good correlation between the operator  
 235 norm  $\|\mathbf{D}(\boldsymbol{\theta})\|_{\infty, 2}$  and the failure probability  $P_f$ . This happens since the operator norm  $\|\mathbf{D}(\boldsymbol{\theta})\|_{\infty, 2}$   
 236 describes the amount of ‘energy’ amplification in the random signal towards the ‘extremes’ of  
 237 the responses  $\boldsymbol{\eta}_i$ , and hence, its corresponding effect on  $P_f$ . Thus, it is readily seen that finding  
 238 those values of the epistemic uncertain parameters  $\boldsymbol{\theta}$  that minimize and maximize, respectively,  
 239  $\|\mathbf{D}(\boldsymbol{\theta})\|_{\infty, 2}$  will provide a good approximation of the realizations that minimize and maximize  $P_f$ .  
 240 Hence, the double loop that is presented in Eqs. (8) and (9) can be efficiently decoupled, first, by

241 determining  $\theta^U$  via:

$$242 \quad \theta^U = \operatorname{argmax}_{\theta \in \theta^I} \|\mathbf{D}(\theta)\|_{\infty,2} \quad (15)$$

243 to find the parameters that yield  $\bar{P}_f$ , and then, by determining  $\theta^L$  via:

$$244 \quad \theta^L = \operatorname{argmin}_{\theta \in \theta^I} \|\mathbf{D}(\theta)\|_{\infty,2} \quad (16)$$

245 to find the parameters that yield  $\underline{P}_f$ . Next, the bounds on  $P_f$ , i.e.,  $\underline{P}_f$  and  $\bar{P}_f$ , are obtained by  
246 solving Eq. (5) twice, corresponding to  $\theta^U$  and  $\theta^L$ . It is noted that any pertinent optimization solver  
247 can be employed to solve Eqs. (15) and (16). Further, it is readily seen that recasting the problem  
248 in the form given by Eq. (10) is critical for the application of the method. In essence, this means  
249 that the underlying model must be linear, and that the aleatory uncertainty can only be present in  
250 the load description (Faes et al. 2021b). This feature of the method hinders its direct application to  
251 nonlinear systems defined by Eq. (7). Nevertheless, this limitation is addressed in the following by  
252 resorting to the statistical linearization method, i.e., by defining an equivalent linear system for the  
253 nonlinear system of Eq. (7).

## 254 **Statistical linearization methodology**

255 In this section, a concise presentation of the statistical linearization methodology is provided for  
256 completeness. The main objective of the method is to replace the originally considered nonlinear  
257 system with an equivalent linear one and minimize (in some sense) the difference between the  
258 two systems. Clearly, the readily available solution frameworks for treating the equivalent linear  
259 system are used to estimate the stochastic response of its nonlinear counterpart. In general, several  
260 variations of the method have been used to solve approximately and efficiently nonlinear stochastic  
261 differential equations associated with engineering applications; see, indicatively, Fragkoulis et al.  
262 (2016b); Kougioumtzoglou et al. (2017); Fragkoulis et al. (2019); Spanos and Malara (2020);  
263 Pasparakis et al. (2021); Ni et al. (2021) and references therein. Its extensive utilization in  
264 stochastic dynamics is associated with its capacity to treat a wide range of nonlinear behaviors in a  
265 straightforward manner.

266 The statistical linearization method is invoked herein to obtain an equivalent linear system that  
 267 is compatible with the operator norm framework. For the application of the method, the nonlinear  
 268 system in Eq. (1) is replaced by an equivalent linear system of the form:

$$269 \quad (\mathbf{M} + \mathbf{M}_e) \ddot{\mathbf{q}}(t) + (\mathbf{C} + \mathbf{C}_e) \dot{\mathbf{q}}(t) + (\mathbf{K} + \mathbf{K}_e) \mathbf{q}(t) = \boldsymbol{\rho} p(t, \boldsymbol{\xi}). \quad (17)$$

270 In Eq. (17),  $\mathbf{M}_e$ ,  $\mathbf{C}_e$  and  $\mathbf{K}_e$  denote, respectively, the mass, damping and stiffness  $n_d \times n_d$  matrices  
 271 of the equivalent linear system that account for neglecting the nonlinearity from Eq. (1). Next, the  
 272 error  $\boldsymbol{\varepsilon} \in \mathbb{R}^{n_d}$  is defined as the difference between Eqs. (1) and (17), i.e.:

$$273 \quad \boldsymbol{\varepsilon} = \boldsymbol{\Phi}(\ddot{\mathbf{q}}(t), \dot{\mathbf{q}}(t), \mathbf{q}(t)) - \mathbf{M}_e \ddot{\mathbf{q}}(t) - \mathbf{C}_e \dot{\mathbf{q}}(t) - \mathbf{K}_e \mathbf{q}(t), \quad (18)$$

274 and its mean square is minimized. Note that although several criteria are available for minimizing  
 275  $\boldsymbol{\varepsilon}$  (e.g., Socha 2007, Elishakoff and Andriamasy 2012), adopting a mean square error minimization  
 276 in conjunction with the Gaussian assumption for the system response probability density functions  
 277 (Roberts and Spanos 2003) facilitates the determination of the equivalent linear system in Eq. (17).  
 278 Specifically, the elements of matrices  $\mathbf{M}_e$ ,  $\mathbf{C}_e$  and  $\mathbf{K}_e$  are given in closed form by:

$$279 \quad m_{ij}^e = \mathbb{E} \left[ \frac{\partial \Phi_i}{\partial \ddot{q}_j} \right], \quad c_{ij}^e = \mathbb{E} \left[ \frac{\partial \Phi_i}{\partial \dot{q}_j} \right], \quad k_{ij}^e = \mathbb{E} \left[ \frac{\partial \Phi_i}{\partial q_j} \right], \quad (19)$$

280 where  $\mathbb{E}[\cdot]$  is the expectation operator and the indices  $i, j = 1, 2, \dots, n_d$  denote the corresponding  
 281 element of the  $n_d \times n_d$  matrices and  $n_d$ -dimensional vectors.

282 Next, note that the equivalent linear system response variance is also required to compute the  
 283 elements of the equivalent matrices given by Eq. (19). This is attained by employing either a time-  
 284 or a frequency-domain solution framework (Roberts and Spanos 2003, Fragkoulis et al. 2016b,  
 285 Kougiumtzoglou et al. 2017). For instance, following the latter, the system response variance is

286 determined by resorting to the input-output relationship of random vibration theory:

$$287 \quad \mathbf{S}_{\mathbf{q}\mathbf{q}}(\omega) = \boldsymbol{\alpha}(\omega)\mathbf{S}_{\mathbf{p}\mathbf{p}}(\omega)\boldsymbol{\alpha}^{\text{T}*}(\omega), \quad (20)$$

288 where  $\mathbf{S}_{\mathbf{q}\mathbf{q}}(\omega)$  and  $\mathbf{S}_{\mathbf{p}\mathbf{p}}(\omega)$  denote, respectively, the response and excitation power spectrum, and  
 289 ‘T\*’ corresponds to the conjugate transpose matrix operator. Further,  $\boldsymbol{\alpha}(\omega)$  is the frequency  
 290 response function matrix of the equivalent system in Eq. (17), i.e.:

$$291 \quad \boldsymbol{\alpha}(\omega) = \left[ -\omega^2(\mathbf{M} + \mathbf{M}_e) + i\omega(\mathbf{C} + \mathbf{C}_e) + (\mathbf{K} + \mathbf{K}_e) \right]^{-1}, \quad (21)$$

292 Thus, taking into account Eqs. (20) and (21), the system response variance is determined by:

$$293 \quad \mathbb{E} [q_i^2(t)] = \int_{-\infty}^{\infty} S_{q_i q_i}(\omega) d\omega, \quad \mathbb{E} [\dot{q}_i^2(t)] = \int_{-\infty}^{\infty} \omega^2 S_{q_i q_i}(\omega) d\omega, \quad \mathbb{E} [\ddot{q}_i^2(t)] = \int_{-\infty}^{\infty} \omega^4 S_{q_i q_i}(\omega) d\omega, \quad (22)$$

294 where  $S_{q_i q_i}(\omega)$ ,  $i = 1, 2, \dots, n_d$ , are the diagonal elements of the system response spectrum  
 295  $\mathbf{S}_{\mathbf{q}\mathbf{q}}(\omega)$ . Clearly, Eq. (19) and Eq. (22) define a coupled set of nonlinear equations to be solved  
 296 for determining  $\mathbf{M}_e$ ,  $\mathbf{C}_e$  and  $\mathbf{K}_e$ . For its solution, the following iterative scheme is used. First, the  
 297 equivalent parameter matrices in Eq. (17) are set equal to null matrices. Then, initial values for the  
 298 response variance are computed by Eq. (22). Next, the latter are used in conjunction with Eq. (19)  
 299 to update the values for  $\mathbf{M}_e$ ,  $\mathbf{C}_e$  and  $\mathbf{K}_e$ . The last two steps are repeated until convergence.

300 Finally, it is noted that since the linearization is performed in a mean-square error minimization  
 301 sense, the approximation of the true system is generally less accurate in the tails of the distribution.  
 302 Hence, the accuracy of the method tends to decrease when considering smaller failure probabilities.  
 303 That is, using the equivalent linear system does not generally provide sufficiently accurate estimates  
 304 for smaller failure probabilities. In this regard, in the proposed approach the equivalent linear system  
 305 is only used for identifying the epistemic parameter values that yield the extrema of  $P_f$ . After these  
 306 values have been identified, they are used to obtain the corresponding lower and upper bounds of  $P_f$   
 307 for the original nonlinear system by means of direct Monte Carlo simulation. Nonetheless, as it is

308 shown in the numerical examples section, the proposed framework provides practical advantages in  
 309 the sense that the failure probability bounds can be computed with significantly greater numerical  
 310 efficiency.

### 311 **Solution of the equivalent linear system**

312 Clearly, Eq. (17) represents a linear structural system subject to stochastic Gaussian loading.  
 313 However, it is noted that, depending on the form of nonlinearity  $\Phi(\ddot{\mathbf{q}}(t), \dot{\mathbf{q}}(t), \mathbf{q}(t))$  in Eq. (7),  
 314 the parameter matrices of the equivalent system in Eq. (17) are no longer necessarily symmetric.  
 315 Nevertheless, this poses no difficulty in applying the proposed methodology. In general, new ap-  
 316 proaches have been recently developed for treating linear and nonlinear multi-degree-of-freedom  
 317 systems which lack mathematically appealing properties, such as symmetry and positive definite-  
 318 ness; see, indicatively, Fragkoulis et al. (2016a,b). Further, note that matrix  $\mathbf{C} + \mathbf{C}_e$  represents a  
 319 ‘full’ damping matrix. Therefore, commonly applied solution schemes based on convolution, as  
 320 described in Chopra (1995) cannot be applied directly.

321 In this regard, Eq. (17) is recast into a state-space form (Chopra 1995; Jensen and Valdeben-  
 322 ito 2007):

$$323 \quad \mathbf{M}^* \dot{\mathbf{q}}^*(t) + \mathbf{K}^* \mathbf{q}^*(t) = \mathbf{P}^*(t, \xi), \quad (23)$$

324 where  $\mathbf{M}^* \in \mathbb{R}^{2n_d \times 2n_d}$ ,  $\mathbf{K}^* \in \mathbb{R}^{2n_d \times 2n_d}$  and  $\mathbf{P}^* \in \mathbb{R}^{2n_d \times 1}$  are block matrices given by:

$$325 \quad \mathbf{M}^* = \begin{bmatrix} \mathbf{0} & \mathbf{M} + \mathbf{M}_e \\ \mathbf{M} + \mathbf{M}_e & \mathbf{C} + \mathbf{C}_e \end{bmatrix}, \quad \mathbf{K}^* = \begin{bmatrix} -(\mathbf{M} + \mathbf{M}_e) & \mathbf{0} \\ \mathbf{0} & \mathbf{K} + \mathbf{K}_e \end{bmatrix}, \quad \mathbf{P}^* = \begin{bmatrix} \mathbf{0} \\ \rho p(t, \xi) \end{bmatrix}, \quad (24)$$

326 and  $\mathbf{q}^*(t)$  denotes the  $2n_d$ -dimensional vector

$$327 \quad \mathbf{q}^*(t) = \begin{bmatrix} \dot{\mathbf{q}}(t) \\ \mathbf{q}(t) \end{bmatrix}. \quad (25)$$

328 The impulse response function  $h_i(t)$  corresponding to the system in Eq. (23) is defined as:

$$329 \quad h_i(t) = \sum_{r=1}^{2n_d} \frac{\boldsymbol{\beta}_i^T \boldsymbol{\Phi}_r \boldsymbol{\Upsilon}_{pr}^T \boldsymbol{\rho}}{(2\lambda_r T_r + S_r)} e^{\lambda_r t}, \quad (26)$$

330 where  $i = 1, 2, \dots, n_r$  denotes the number of responses, and  $\boldsymbol{\beta}_i$  is a constant vector such that a  
 331 response of interest  $\eta_i$  is generated as  $\eta_i = \boldsymbol{\beta}_i^T \mathbf{q}$ . Variables  $T_r$  and  $S_r$  are the modal energies given  
 332 by:

$$333 \quad T_r = \boldsymbol{\Upsilon}_{pr}^T (\mathbf{M} + \mathbf{M}_e) \boldsymbol{\Phi}_{pr}, \quad S_r = \boldsymbol{\Upsilon}_{pr}^T (\mathbf{C} + \mathbf{C}_e) \boldsymbol{\Phi}_{pr}, \quad (27)$$

334 where  $\boldsymbol{\Upsilon}_{pr}$  and  $\boldsymbol{\Phi}_{pr}$  are, respectively, the position parts (i.e., the last  $n_d$  components) of the right  
 335 and left eigenvectors, associated with the right and left eigenproblems of Eq. (23);  $\lambda_r$  contains the  
 336 corresponding eigenvalues.

The dynamic responses  $\boldsymbol{\eta}_i, i = 1, 2, \dots, n_\eta$ , that solve Eq. (17) are calculated by apply-  
 ing the convolution integral between the corresponding unit impulse response functions  $h_i(t)$ ,  
 $i = 1, 2, \dots, n_\eta$ , and the stochastic loading  $p(t, \boldsymbol{\xi})$ , i.e.:

$$\eta_i(t, \boldsymbol{\xi}) = \int_0^t h_i(t - \tau) p(t, \boldsymbol{\xi}) d\tau, \quad i = 1, 2, \dots, n_\eta. \quad (28)$$

337 In view of the excitation model introduced in Eq. (4), evaluating Eq. (28) at time  $t_k$  yields:

$$338 \quad \eta_i(t_k, \boldsymbol{\xi}) = \sum_{l_1=1}^k \Delta t \epsilon_{l_1} h_i(t_k - t_{l_1}) \left( \sum_{l_2=1}^{n_{KL}} \psi_{l_1, l_2} \sqrt{\lambda_{l_2}} \xi_{l_2} \right) = \boldsymbol{\gamma}_{i, k} \boldsymbol{\xi}, \quad (29)$$

for  $i = 1, 2, \dots, n_\eta$ ,  $k = 1, 2, \dots, n_T$ , where  $\psi_{l_1, l_2}$  is the  $(l_1, l_2)$ -th element of matrix  $\boldsymbol{\Psi}$ ;  $\boldsymbol{\gamma}_{i, k}$  is a  
 $n_{KL}$ -dimensional vector such that:

$$\boldsymbol{\gamma}_{i, k} = \left[ \sum_{l_1=1}^k \Delta t \epsilon_{l_1} h_i(t_k - t_{l_1}) \psi_{l_1, 1} \sqrt{\lambda_1} \quad \dots \quad \sum_{l_1=1}^k \Delta t \epsilon_{l_1} h_i(t_k - t_{l_1}) \psi_{l_1, n_{KL}} \sqrt{\lambda_{n_{KL}}} \right] \quad (30)$$

339 and  $\epsilon_{l_1}$  is a coefficient depending on the numerical integration scheme used in the evaluation of the  
 340 convolution integral. When the trapezoidal integration rule is chosen (Gautschi 2012),  $\epsilon_{l_1} = 1/2$ ,

341 if  $l_1 = 1$  or  $l_1 = k$ ; otherwise,  $\epsilon_{l_1} = 1$ . As such,  $\boldsymbol{\eta}_i$  is calculated as a linear transformation that maps  
 342 the standard normal random vector  $\boldsymbol{\xi}$  to the responses  $\boldsymbol{\eta}_i$  for each time instant:

$$343 \quad \boldsymbol{\eta}_i(\boldsymbol{\xi}) = \boldsymbol{\Gamma}_i(\boldsymbol{\theta})\boldsymbol{\xi}, \quad (31)$$

344 where:

$$345 \quad \boldsymbol{\eta}_i(\boldsymbol{\xi}) = \begin{bmatrix} \eta_i(t_1, \boldsymbol{\xi}) \\ \eta_i(t_2, \boldsymbol{\xi}) \\ \vdots \\ \eta_i(t_{n_T}, \boldsymbol{\xi}) \end{bmatrix}, \quad \boldsymbol{\Gamma}_i(\boldsymbol{\theta}) = \begin{bmatrix} \gamma_{i,1}(\boldsymbol{\theta}) \\ \gamma_{i,2}(\boldsymbol{\theta}) \\ \vdots \\ \gamma_{i,n_T}(\boldsymbol{\theta}) \end{bmatrix}, \quad (32)$$

346 in which  $\boldsymbol{\Gamma}_i(\boldsymbol{\theta})$  is a  $n_T \times n_{KL}$  matrix that represents a linear map from the standard normal random  
 347 vector  $\boldsymbol{\xi}$  to the  $i$ -th response of interest. Note that  $\boldsymbol{\Gamma}_i(\boldsymbol{\theta})$  depends directly on the epistemic uncertain  
 348 parameters  $\boldsymbol{\theta}$  through the eigenvalues and eigenvectors of the KL series expansion.

### 349 **Bounds on the first excursion probability**

350 As explained in section ‘‘Linear problems’’, the operator norm theorem can be used to bound the  
 351 probability of failure of linear models under epistemic uncertainty in the definition of the load. To  
 352 extend the method towards treating nonlinear dynamical simulation models, a framework based on  
 353 the combination of the operator norm-based treatment and the statistical linearization methodology  
 354 is proposed. Hereto, the linearized system of Eq. (31) is considered. Specifically, the epistemic  
 355 uncertain parameters of the imprecisely defined stochastic load that bound  $P_f$  are defined as:

$$356 \quad \boldsymbol{\theta}^U = \operatorname{argmax}_{\boldsymbol{\theta} \in \boldsymbol{\theta}^I} \max_{i=1,2,\dots,n_\eta} \|\boldsymbol{\Gamma}_i(\boldsymbol{\theta})\|_{\infty,2} \quad (33)$$

357 and:

$$358 \quad \boldsymbol{\theta}^L = \operatorname{argmin}_{\boldsymbol{\theta} \in \boldsymbol{\theta}^I} \max_{i=1,2,\dots,n_\eta} \|\boldsymbol{\Gamma}_i(\boldsymbol{\theta})\|_{\infty,2}, \quad (34)$$

359 with  $\boldsymbol{\Gamma}_i$  as defined in Eq. (32). These parameter realizations are used for finding the parameters that  
 360 yield  $\bar{P}_f$  and  $\underline{P}_f$ , respectively. Note that the explicit dependence of  $\boldsymbol{\Gamma}_i$  on  $\boldsymbol{\theta}$  is highlighted in these



361 equations. The parameters  $\theta$  influence  $\Gamma_i$  through the eigenfunctions and corresponding eigenvalues  
 362 of the KL expansion shown in Eq. (4) and the interaction with the structural nonlinearities. Based  
 363 on the derivations in Tropp (2004), Eqs. (33) and (34) are recast into:

$$364 \quad \theta^U = \operatorname{argmax}_{\theta \in \theta^I} \max_{i=1,2,\dots,n_\eta} \max_{j=1,2,\dots,n_T} \|\Gamma_i^{j:}(\theta)\|_2 \quad (35)$$

365 and:

$$366 \quad \theta^L = \operatorname{argmin}_{\theta \in \theta^I} \max_{i=1,2,\dots,n_\eta} \max_{j=1,2,\dots,n_T} \|\Gamma_i^{j:}(\theta)\|_2, \quad (36)$$

367 respectively, where the superscript ‘ $j$  :’ denotes the  $j$ -th row of matrix  $\Gamma_i$  and  $\|\cdot\|_2$  denotes the  
 368 regular  $L_2$  vector norm.

369 To summarize, the proposed procedure can be described as follows:

- 370 1. Represent the nonlinear model including the epistemic uncertainty by using Eq. (7).
- 371 2. Solve the optimization problems in Eqs. (35) and (36) to identify  $\theta^U$  and  $\theta^L$ , by using  
 372 any appropriate algorithm. Then, compute matrix  $\Gamma(\theta)$  for a given realization  $\theta$ . This  
 373 is done in two steps. First, applying the statistical linearization method, solve iteratively  
 374 Eqs. (19)-(22). Secondly, taking into account Eqs. (24)-(32), perform modal analysis over  
 375 the equivalent linear system to derive matrix  $\Gamma(\theta)$ .
- 376 3. Once  $\theta^U$  and  $\theta^L$  are identified, perform reliability analysis using the full nonlinear model  
 377 in order to determine the upper and lower bounds of the failure probability.

## 378 NUMERICAL EXAMPLES

### 379 Case study 1: two-degrees-of-freedom nonlinear system

380 In this case study, the two-degrees-of-freedom (DOF) system in Fig. 1 is considered. The  
 381 system consists of masses  $m_1$  and  $m_2$ , which are connected to each other by a linear damper of  
 382 damping coefficient  $c_2$  and a linear spring of stiffness coefficient  $k_2$ . Further, mass  $m_1$  connects  
 383 to the foundation by a linear damper of damping coefficient  $c_1$  and a nonlinear spring of stiffness  
 384 coefficient  $k_1$ .

385 Next, considering the coordinates vector  $\mathbf{q}^T = [q_1 \quad q_2]$  and following the standard Newtonian  
 386 approach to derive the system governing equations of motion (Roberts and Spanos 2003), Eq. (1)  
 387 is formulated. The system parameter matrices are given by:

$$388 \quad \mathbf{M} = \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}, \quad \mathbf{C} = \begin{bmatrix} c_1 + c_2 & -c_2 \\ -c_2 & c_2 \end{bmatrix}, \quad \mathbf{K} = \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 \end{bmatrix}, \quad (37)$$

389 whereas:

$$390 \quad \rho p(t, \xi) = \begin{bmatrix} 1 \\ 0 \end{bmatrix} p(t, \xi) \quad (38)$$

391 denotes the stochastic excitation. Further, the nonlinear restoring force of the system is given by:

$$392 \quad \Phi(\ddot{\mathbf{q}}, \dot{\mathbf{q}}, \mathbf{q}) = \begin{bmatrix} k_1 \nu q_1^3 \\ 0 \end{bmatrix}, \quad (39)$$

393 where  $\nu$  corresponds to the intensity of the nonlinearity. Finally, the load  $p(t, \xi)$  acting on the  
 394 system is modeled as a zero-mean Gaussian stochastic process, described by the Clough-Penzien  
 395 spectrum (Li and Chen 2009b):

$$396 \quad S_{PP}(\omega) = \frac{\omega^4 \left( \omega_g^4 + (2\zeta_g \omega_g \omega)^2 \right) S_0}{\left( (\omega_g^2 - \omega^2)^2 + (2\zeta_g \omega_g \omega)^2 \right) \left( (\omega_f^2 - \omega^2)^2 + (2\zeta_f \omega_f \omega)^2 \right)}. \quad (40)$$

397 The following parameter values are considered for the system in Fig. 1,  $m_1 = m_2 = 1$  [kg],  
 398  $c_1 = c_2 = 0.2$  [N·s/m],  $k_1 = k_2 = 1$  [N/m], whereas the intensity of the nonlinearity is  $\nu = 1$  and  
 399 the nominal parameters of the excitation spectrum are  $[\omega_g, \omega_f, \zeta_g, \zeta_f, S_0] = [4\pi, 0.4\pi, 0.7, 0.7, 3 \times$   
 400  $10^{-4}]$ . Failure of the system is considered as the first passage of any of the displacements of the  
 401 masses over a threshold value of  $b = 0.040$  [m]. Further, it is considered that the analyst is unsure  
 402 about the exact values of the stochastic load acting on the system. Specifically, the definition of the  
 403 parameters of the Clough-Penzien spectrum is subject to epistemic uncertainty. The intervals that

404 are applied for bounding this epistemic uncertainty are shown in Table 1.

405 Next, the herein proposed operator norm theory-based statistical linearization framework is  
 406 employed for computing the bounds on the probability of failure. In this regard, first, the governing  
 407 equation of motion with parameter matrices and nonlinear vector given by Eqs. (37) and Eq. (39),  
 408 respectively, is replaced by an equivalent linear system of the form of Eq. (17). Then, considering  
 409 the error function in Eq. (18) and adopting a mean square minimization of the error, Eq. (19) leads  
 410 to the equivalent parameter matrices:

$$411 \quad \mathbf{M}_e = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}, \quad \mathbf{C}_e = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}, \quad \mathbf{K}_e = \begin{bmatrix} 3k_1\nu\sigma_{q_1}^2 & 0 \\ 0 & 0 \end{bmatrix}. \quad (41)$$

412 Regarding the numerical implementation, considering as stopping criterion  $\left| \frac{\mathbf{K}_e^{i+1} - \mathbf{K}_e^i}{\mathbf{K}_e^i} \right| < 10^{-5}$ ,  
 413 where the index ‘ $i$ ’ denotes the  $i$ -th iteration and the initial value  $\mathbf{K}_e^0$  is set equal to zero, the  
 414 iterative scheme described in the section “Statistical linearization methodology” converges after  
 415 three iterations. Thus, the nonlinear system shown in Fig. 1 is approximated by the equivalent  
 416 linear system whose governing equations of motion are given by Eq. (17).

417 Next, the augmented state-space system in Eq. (23) is formulated and taking into account  
 418 Eqs. (26)-(31), the linear map  $\mathbf{\Gamma}_i(\boldsymbol{\theta})$  is calculated. Then, following the presentation in the section  
 419 “Bounds on the first excursion probability”, and considering the derived equivalent linear matrices,  
 420 the operator norm that corresponds to any given realization of the epistemically uncertain Gaussian  
 421 process load is computed. In addition, the optimization over the operator norm can be performed  
 422 using the Matlab built-in patternsearch optimization tool. Finally, two optimization problems have  
 423 to be solved; the first one for determining  $\boldsymbol{\theta}^U$  (see Eq. (35)) and the second one for determining  $\boldsymbol{\theta}^L$   
 424 (see Eq. (36)), which require approximately 100 iterations to converge.

425 So far, the operator norm-based statistical linearization framework is used for determining the  
 426 bounds on  $P_f$ . Next, the validity of the obtained results is verified by using a brute-force implemen-  
 427 tation of the double-loop problem. Hereto, the Newmark solver is considered in conjunction with  
 428 Monte Carlo simulation (MCS) as the ‘inner loop’ in Eqs. (8) and (9) for computing  $P_f$  for each

429 realization of the epistemic uncertainty. It is noted that a total of 1000 samples are considered for  
 430 estimating the failure probability at each realization of the epistemic parameters. A patternsearch  
 431 optimization algorithm (Kolda et al. 2003) is used to solve the optimization problem in the ‘outer  
 432 loop’. This result serves as the benchmark for the bounds on  $P_f$  against which the result of the  
 433 proposed operator norm-based statistical linearization framework is compared.

### 434 *Results and discussion*

435 The functional relationship between the operator norm  $\|\Gamma\|_{2,\infty}$ , as computed over the linearized  
 436 system, and  $P_f$ , as computed using MCS combined with the Newmark solver, is shown in Fig. 2.  
 437 The black dots in this figure are obtained by drawing 1000 uniformly distributed samples in between  
 438 the bounds of  $\theta^l$ . First, it is noted that the relation between the operator norm  $\|\Gamma_i(\theta)\|_{\infty,2}$  and  $P_f$  is  
 439 not bijective. In addition, there is a clear trend between these two quantities, where higher operator  
 440 norm values correspond to higher probability of failure values and vice-versa. This illustrates  
 441 the validity of the proposed approach in the sense that minimizing (or maximizing) the operator  
 442 norm also yields a minimum (or maximum) of the failure probability. Further, Table 2 shows the  
 443 parameters that yield an extremum in  $P_f$  by optimizing directly over  $P_f$  (indicated DL), as well as  
 444 over the operator norm (indicated ON). These parameters are grouped in the rows indicated with  
 445  $\theta$ . Furthermore, the corresponding optima are reported, as well as the number of required function  
 446 calls ( $n^0$ ). It is important to stress that to obtain a value for the operator norm, only the linear map  $\Gamma$   
 447 (see Eq. (31)) needs to be assembled and the corresponding operator norm needs to be calculated.  
 448 On the other hand, the calculation of one value of  $P_f$  requires the full solution of Eq. (5).

449 Finally, in order to evaluate the performance of the proposed approach for different threshold  
 450 levels, Fig. 3 presents the failure probability bounds obtained by the proposed method (denoted ON)  
 451 and the reference bounds obtained by a direct double loop implementation (denoted DL) for different  
 452 values of  $b$ . First, note that the failure probability values tend to decrease for higher threshold levels,  
 453 as expected. In addition, it is seen that the lower bounds for the failure probability obtained by  
 454 the proposed approach agree very well with the reference values for smaller threshold levels, i.e.,  
 455  $b \leq 0.040$  m. On the other hand, the deviations between the operator norm-based estimates for the

456 lower bounds and the corresponding reference values tend to increase for larger values of  $b$ , which  
 457 are associated with smaller failure probabilities. For instance, the proposed scheme overestimates  
 458 the lower failure probability bound in 30% for the case  $b = 0.050$  m. This illustrates that the  
 459 proposed statistical linearization-based method is more suitable for problems involving moderate  
 460 to large failure probabilities, as already pointed out. In this regard, the integration of the ON-based  
 461 framework with alternative linearization techniques can (potentially) improve the performance of  
 462 the proposed scheme for smaller failure probabilities.

### 463 **Case study 2: six degrees-of-freedom structure**

464 In this example, a 6-DOF system of rigid masses  $m_i$  ( $i = 1, 2, \dots, 6$ ) connected to each other  
 465 by nonlinear dampers as shown in Fig. 4 is considered. In this regard, considering the coordinates  
 466 vector  $\mathbf{q}^T = [q_1 \ q_2 \ q_3 \ q_4 \ q_5 \ q_6]$ , the matrix form of the system governing equations of  
 467 motion is formulated (see Eq. (1)), whose parameter matrices are given by:

$$468 \quad \mathbf{M} = \begin{bmatrix} m_1 & 0 & 0 & 0 & 0 & 0 \\ m_2 & m_2 & 0 & 0 & 0 & 0 \\ m_3 & m_3 & m_3 & 0 & 0 & 0 \\ m_4 & m_4 & m_4 & m_4 & 0 & 0 \\ m_5 & m_5 & m_5 & m_5 & m_5 & 0 \\ m_6 & m_6 & m_6 & m_6 & m_6 & m_6 \end{bmatrix}, \mathbf{C} = \begin{bmatrix} c_1 & -c_2 & 0 & 0 & 0 & 0 \\ 0 & c_2 & -c_3 & 0 & 0 & 0 \\ 0 & 0 & c_3 & -c_4 & 0 & 0 \\ 0 & 0 & 0 & c_4 & -c_5 & 0 \\ 0 & 0 & 0 & 0 & c_5 & -c_6 \\ 0 & 0 & 0 & 0 & 0 & c_6 \end{bmatrix} \quad (42)$$

469 and:

$$470 \quad \mathbf{K} = \begin{bmatrix} k_1 & -k_2 & 0 & 0 & 0 & 0 \\ 0 & k_2 & -k_3 & 0 & 0 & 0 \\ 0 & 0 & k_3 & -k_4 & 0 & 0 \\ 0 & 0 & 0 & k_4 & -k_5 & 0 \\ 0 & 0 & 0 & 0 & k_5 & -k_6 \\ 0 & 0 & 0 & 0 & 0 & k_6 \end{bmatrix}. \quad (43)$$

471 Further, it is assumed that the system is subjected to ground acceleration, which is modeled as a  
 472 stochastic process, whose corresponding power spectrum is given by:

$$473 \quad \mathbf{S}(\omega) = \begin{bmatrix} S_1(\omega) & 0 & 0 & 0 & 0 & 0 \\ 0 & S_2(\omega) & 0 & 0 & 0 & 0 \\ 0 & 0 & S_3(\omega) & 0 & 0 & 0 \\ 0 & 0 & 0 & S_4(\omega) & 0 & 0 \\ 0 & 0 & 0 & 0 & S_5(\omega) & 0 \\ 0 & 0 & 0 & 0 & 0 & S_6(\omega) \end{bmatrix}, \quad (44)$$

474 where  $S_i(\omega)$ ,  $i = 1, 2, \dots, 6$ , is modeled as a Clough-Penzien spectrum (see Eq. (40)) with the  
 475 epistemic uncertainty on the parameters  $\omega_g$ ,  $\omega_f$ ,  $\zeta_g$  and  $\zeta_f$  characterized by the intervals given  
 476 in Table 1, whereas the epistemic uncertainty on parameter  $S_0$  is characterized by the interval  
 477  $[0.8, 1.2] \times 0.05$ . In addition, the nonlinear function  $\Phi(\ddot{\mathbf{q}}, \dot{\mathbf{q}}, \mathbf{q})$  takes the form:

$$478 \quad \Phi^T(\ddot{\mathbf{q}}, \dot{\mathbf{q}}, \mathbf{q}) =$$

$$479 \quad \begin{bmatrix} c_1 \nu \dot{q}_1^3 - c_2 \nu \dot{q}_2^3 & c_2 \nu \dot{q}_2^3 - c_3 \nu \dot{q}_3^3 & c_3 \nu \dot{q}_3^3 - c_4 \nu \dot{q}_4^3 & c_4 \nu \dot{q}_4^3 - c_5 \nu \dot{q}_5^3 & c_5 \nu \dot{q}_5^3 - c_6 \nu \dot{q}_6^3 & c_6 \nu \dot{q}_6^3 \end{bmatrix},$$

$$480 \quad (45)$$

481 with  $\nu$  describing the intensity of the nonlinearity in Eq. (45). The system parameter values are  
 482  $m_1 = m_2 \cdots = m_6 = 1$ ,  $c_1 = c_2 \cdots = c_6 = 0.2$ ,  $k_1 = k_2 \cdots = k_6 = 1$  and  $\nu = 3$ . In addition, failure  
 483 is defined in this case as the first passage of any interstory drift beyond the maximum allowable  
 484 threshold  $b = 0.6$  m.

485 Then, the herein proposed operator norm theory-based statistical linearization framework is  
 486 applied. In this regard, the equivalent linear mass and stiffness  $6 \times 6$  matrices take the form:

$$487 \quad \mathbf{M}_e = \mathbf{K}_e = \mathbf{0}, \quad (46)$$

488

whereas the equivalent linear damping  $6 \times 6$  matrix becomes:

489

$$\mathbf{C}_e = \begin{bmatrix} 3c_1\nu\sigma_{\dot{q}_1}^2 & -3c_2\nu\sigma_{\dot{q}_2}^2 & 0 & 0 & 0 & 0 \\ 0 & 3c_2\nu\sigma_{\dot{q}_2}^2 & -3c_3\nu\sigma_{\dot{q}_3}^2 & 0 & 0 & 0 \\ 0 & 0 & 3c_3\nu\sigma_{\dot{q}_3}^2 & -3c_4\nu\sigma_{\dot{q}_4}^2 & 0 & 0 \\ 0 & 0 & 0 & 3c_4\nu\sigma_{\dot{q}_4}^2 & -3c_5\nu\sigma_{\dot{q}_5}^2 & 0 \\ 0 & 0 & 0 & 0 & 3c_5\nu\sigma_{\dot{q}_5}^2 & -3c_6\nu\sigma_{\dot{q}_6}^2 \\ 0 & 0 & 0 & 0 & 0 & 3c_6\nu\sigma_{\dot{q}_6}^2 \end{bmatrix}. \quad (47)$$

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The elements of the equivalent matrix in Eq. (47) are determined by utilizing the iterative scheme described in the section ‘Statistical linearization methodology’. Specifically, using  $\left| \frac{\mathbf{C}_e^{i+1} - \mathbf{C}_e^i}{\mathbf{C}_e^i} \right| < 10^{-5}$  as stopping criterion, where ‘ $i$ ’ denotes the  $i$ -th iteration of the scheme, and also considering the initial value  $\mathbf{C}_e^0 = \mathbf{0}$ , the scheme converges after five iterations. Thus, the nonlinear system shown in Fig. 4 is approximated by the equivalent linear system whose governing equations of motion are given by Eq. (17).

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Next, the augmented state-space system in Eq. (23) is formulated and taking into account Eqs. (26)-(31), the linear map  $\mathbf{\Gamma}_i(\boldsymbol{\theta})$  is calculated. Subsequently, following the presentation in the section ‘Bounds on the first excursion probability’, and considering the derived equivalent linear matrices, the operator norm that corresponds to a certain realization of the epistemically uncertain Gaussian process load is computed. In addition, the optimization over the operator norm is performed using the Matlab built-in patternsearch optimization tool. Finally, two optimization problems have to be solved; the first one for determining  $\boldsymbol{\theta}^U$  (see Eq. (35)) and the second one for determining  $\boldsymbol{\theta}^L$  (see Eq. (36)), which require approximately 200 iterations to converge.

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### Results and discussion

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The results of the herein proposed framework are shown in Table 3, which shows the parameters that yield an extremum in  $P_f$  by either optimizing directly over  $P_f$  (indicated DL) or over the operator norm (indicated ON). These parameters are grouped in the rows indicated with  $\boldsymbol{\theta}$ . Clearly, the proposed method is capable of adequately approximating the true bounds on  $P_f$ . The results

509 are compared to a brute-force double loop implementation using Newmark method to solve the  
510 nonlinear ODE, MCS to calculate  $P_f$ , and patternsearch in Matlab to optimize over the epistemic  
511 parameter space. It is highlighted that the results obtained by following the proposed approach  
512 are in reasonable agreement with the corresponding results obtained by following a classic double  
513 loop approach. The small discrepancy between the results is expected and is due to adopting  
514 an approximate linearization scheme to enable the application of the operator norm framework.  
515 Nonetheless, it can be argued that these bounds are highly reasonable given the immense reduction  
516 in computational cost that is required to calculate them. For instance, considering the upper bound  
517 on  $P_f$ , the required number of deterministic model solutions can be reduced from 292000 to just  
518 626, with 1000 additional samples for computing the associated failure probability.

## 519 **CONCLUSIONS**

520 In this paper, a novel technique has been developed for bounding the responses and probability of  
521 failure of nonlinear structural models subjected to imprecisely defined stochastic Gaussian loads.  
522 The proposed technique can be construed as a generalization of a recently developed operator  
523 norm-based method to account for nonlinear dynamical systems. This is attained by resorting to  
524 the statistical linearization methodology for defining a linear system equivalent to the nonlinear  
525 system under consideration. In this regard, the double loop that is typically associated with  
526 estimating the bounds on the probability of failure of nonlinear dynamical systems is effectively  
527 decoupled and the associated computational cost is reduced by several orders of magnitude. Thus,  
528 it can be argued that integrating statistical linearization into the operator norm framework allows  
529 for bounding the probability of failure of nonlinear systems with acceptable accuracy and at greatly  
530 reduced numerical cost. The validity and numerical efficiency of the proposed technique has  
531 been demonstrated by considering two nonlinear structural systems. It is noted, however, that  
532 since the linearization scheme has been performed in a mean-square error minimization sense, the  
533 representation of the nonlinear system is less accurate in the tails of the distribution. This aspect  
534 renders the proposed approach mostly suitable for estimating the bounds of moderate to large failure  
535 probabilities. Nevertheless, future work is directed towards developing an enhanced operator norm-



536 based linearization scheme capable of estimating bounds on smaller failure probabilities. This can  
537 be achieved, in principle, by combining the application of the statistical linearization methodology  
538 with a stochastic averaging treatment. Further, the proposed framework can be integrated with more  
539 advanced simulation methods, such as importance sampling or subset simulation. Another path  
540 for future work consists of extending the range of application of the proposed framework to more  
541 general models for stochastic loading (other than Gaussian). Finally, the evaluation of the proposed  
542 approach for more complex and numerically demanding structural models involving multiple types  
543 of nonlinearities constitutes an additional subject for future research.

#### 544 **DATA AVAILABILITY STATEMENT**

545 All data, models, or code that support the findings of this study are available from the corre-  
546 sponding author upon reasonable request.

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#### 558 **REFERENCES**

559 Au, S. (2009). “Stochastic control approach to reliability of elasto-plastic structures.” Journal Of  
560 Structural Engineering and Mechanics, 32(1), 21–36.

561 Beer, M. (2004). “Uncertain structural design based on nonlinear fuzzy analysis.” Journal of  
562 Applied Mathematics and Mechanics (ZAMM), 84(10-11), 740–753.

563 Beer, M., Ferson, S., and Kreinovich, V. (2013). “Imprecise probabilities in engineering analyses.”  
564 Mechanical Systems and Signal Processing, 37(1-2), 4–29.

565 Betz, W., Papaioannou, I., and Straub, D. (2014). “Numerical methods for the discretization of  
566 random fields by means of the Karhunen-Loève expansion.” Computer Methods in Applied  
567 Mechanics and Engineering, 271, 109–129.

568 Chen, J. and Li, J. (2013). “Optimal determination of frequencies in the spectral representation of  
569 stochastic processes.” Computational Mechanics, 51(5), 791–806.

570 Chopra, A. (1995). Dynamics of structures: theory and applications to earthquake engineering.  
571 Prentice Hall.

572 de Angelis, M., Patelli, E., and Beer, M. (2015). “Advanced line sampling for efficient robust  
573 reliability analysis.” Structural Safety, 52, Part B, 170–182.

574 Elishakoff, I. and Andriamasy, L. (2012). “The tale of stochastic linearization technique: Over half  
575 a century of progress.” Nondeterministic mechanics, 115–189.

576 Faes, M. and Moens, D. (2019a). “Imprecise random field analysis with parametrized kernel  
577 functions.” Mechanical Systems and Signal Processing, 134, 106334.

578 Faes, M. and Moens, D. (2019b). “Recent Trends in the Modeling and Quantification of Non-  
579 probabilistic Uncertainty.” Archives of Computational Methods in Engineering.

580 Faes, M. and Valdebenito, M. (2021). “Fully decoupled reliability-based optimization of lin-  
581 ear structures subject to gaussian dynamic loading considering discrete design variables.”  
582 Mechanical Systems and Signal Processing, 156, 107616.

583 Faes, M. G. and Valdebenito, M. A. (2020). “Fully decoupled reliability-based design optimization  
584 of structural systems subject to uncertain loads.” Computer Methods in Applied Mechanics and  
585 Engineering, 371, 113313.

586 Faes, M. G., Valdebenito, M. A., Moens, D., and Beer, M. (2020). “Bounding the first excursion  
587 probability of linear structures subjected to imprecise stochastic loading.” Computers &  
588 Structures, 239, 106320.

589 Faes, M. G. R., Daub, M., Marelli, S., Patelli, E., and Beer, M. (2021a). “Engineering analysis with  
590 probability boxes : a review on computational methods.” Structural Safety.

591 Faes, M. G. R., Valdebenito, M. A., Moens, D., and Beer, M. (2021b). “Operator norm theory as an  
592 efficient tool to propagate hybrid uncertainties and calculate imprecise probabilities.” Mechanical  
593 Systems and Signal Processing.

594 Fragkoulis, V. C., Kougioumtzoglou, I. A., and Pantelous, A. A. (2016a). “Linear random vibra-  
595 tion of structural systems with singular matrices.” Journal of Engineering Mechanics, 142(2),  
596 04015081.

597 Fragkoulis, V. C., Kougioumtzoglou, I. A., and Pantelous, A. A. (2016b). “Statistical linearization  
598 of nonlinear structural systems with singular matrices.” Journal of Engineering Mechanics,  
599 142(9), 04016063.

600 Fragkoulis, V. C., Kougioumtzoglou, I. A., Pantelous, A. A., and Beer, M. (2019). “Non-stationary  
601 response statistics of nonlinear oscillators with fractional derivative elements under evolutionary  
602 stochastic excitation.” Nonlinear Dynamics, 97(4), 2291–2303.

603 Gautschi, W. (2012). Numerical Analysis. Birkhäuser Boston, 2nd edition.

604 Imholz, M., Faes, M., Vandepitte, D., and Moens, D. (2020). “Robust uncertainty quantification  
605 in structural dynamics under scarce experimental modal data: A bayesian-interval approach.”  
606 Journal of Sound and Vibration, 467, 114983.

607 Jensen, H. and Valdebenito, M. (2007). “Reliability analysis of linear dynamical systems using  
608 approximate representations of performance functions.” Structural Safety, 29(3), 222–237.

609 Kolda, T., Lewis, R., and torczon, V. (2003). “Optimization by direct search: New perspectives on  
610 some classical and modern methods.” SIAM Review, 45(3), 385–482.

611 Kougioumtzoglou, I., Fragkoulis, V., Pantelous, A., and Pirrotta, A. (2017). “Random vibration of  
612 linear and nonlinear structural systems with singular matrices: A frequency domain approach.”  
613 Journal of Sound and Vibration, 404, 84–101.

614 Lee, J. and Verleysen, M. (2007). Nonlinear Dimensionality Reduction. Springer.

615 Li, J. and Chen, J. (2009a). Stochastic Dynamics of Structures. John Wiley & Sons, Singapore.

616 Li, J. and Chen, J. (2009b). Stochastic dynamics of structures. John Wiley & Sons.

617 Moens, D. and Vandepitte, D. (2004). “An interval finite element approach for the calculation  
618 of envelope frequency response functions.” International Journal for Numerical Methods in  
619 Engineering, 61(14), 2480–2507.

620 Ni, P., Fragkoulis, V. C., Kong, F., Mitseas, I. P., and Beer, M. (2021). “Response determination  
621 of nonlinear systems with singular matrices subject to combined stochastic and deterministic  
622 excitations.” ASCE-ASME Journal of Risk and Uncertainty in Engineering Systems, Part A:  
623 Civil Engineering, 7(4), 04021049.

624 Pasparakis, G., Fragkoulis, V., and Beer, M. (2021). “Harmonic wavelets based response evolu-  
625 tionary power spectrum determination of linear and nonlinear structural systems with singular  
626 matrices.” Mechanical Systems and Signal Processing, 149, 107203.

627 Pradlwarter, H., Schuëller, G., Koutsourelakis, P., and Charmpis, D. (2007). “Application of  
628 line sampling simulation method to reliability benchmark problems.” Structural Safety, 29(3),  
629 208–221.

630 Roberts, J. B. and Spanos, P. D. (2003). Random vibration and statistical linearization. Courier  
631 Corporation.

632 Schenk, C. and Schuëller, G. (2005). Uncertainty Assessment of Large Finite Element Systems.  
633 Springer-Verlag, Berlin/Heidelberg/New York.

634 Schöbi, R. and Sudret, B. (2017). “Structural reliability analysis for p-boxes using multi-level  
635 meta-models.” Probabilistic Engineering Mechanics, 48, 27–38.

636 Schuëller, G. and Pradlwarter, H. (2007). “Benchmark study on reliability estimation in higher  
637 dimensions of structural systems – An overview.” Structural Safety, 29, 167–182.

638 Shinozuka, M. and Sato, Y. (1967). “Simulation of nonstationary random process.” Journal of the  
639 Engineering Mechanics Division, 93(1), 11–40.

640 Socha, L. (2007). Linearization methods for stochastic dynamic systems, Vol. 730. Springer  
641 Science & Business Media.

642 Spanos, P. D., Di Matteo, A., Cheng, Y., Pirrotta, A., and Li, J. (2016). “Galerkin scheme-based  
643 determination of survival probability of oscillators with fractional derivative elements.” Journal  
644 of Applied Mechanics, 83(12).

645 Spanos, P. D. and Kougioumtzoglou, I. A. (2014). “Survival probability determination of nonlinear  
646 oscillators subject to evolutionary stochastic excitation.” Journal of Applied Mechanics, 81(5).

647 Spanos, P. D. and Malara, G. (2020). “Nonlinear vibrations of beams and plates with frac-  
648 tional derivative elements subject to combined harmonic and random excitations.” Probabilistic  
649 Engineering Mechanics, 59, 103043.

650 Stefanou, G. (2009). “The stochastic finite element method: Past, present and future.” Computer  
651 Methods in Applied Mechanics and Engineering, 198(9-12), 1031–1051.

652 Tropp, J. A. (2004). “Topics in Sparse Approximation.” Ph.D. thesis, The University of Texas at  
653 Austin, , <<http://www.lib.utexas.edu/etd/d/2004/troppd73287/troppd73287.pdf>>.

- 654 Vanmarcke, E. and Grigoriu, M. (1983). “Stochastic finite element analysis of simple beams.”  
655 Journal of Engineering Mechanics, 109(5).
- 656 Wei, P., Liu, F., Valdebenito, M., and Beer, M. (2021). “Bayesian probabilistic propagation  
657 of imprecise probabilities with large epistemic uncertainty.” Mechanical Systems and Signal  
658 Processing, 149, 107219.
- 659 Wei, P., Song, J., Bi, S., Broggi, M., Beer, M., Lu, Z., and Yue, Z. (2019). “Non-intrusive  
660 stochastic analysis with parameterized imprecise probability models: I. performance estimation.”  
661 Mechanical Systems and Signal Processing, 124, 349 – 368.

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**TABLE 1.** Tested values for  $\theta^I$ .

$\omega_g^I$	$\omega_f^I$	$\zeta_g^I$	$\zeta_f^I$	$S_0^I$
$[0.8, 1.2] \times 4\pi$	$[0.8, 1.2] \times 0.4\pi$	$[0.8, 1.2] \times 0.7$	$[0.8, 1.2] \times 0.7$	$[0.8, 1.2] \times 3 \times 10^{-4}$

**TABLE 2.** Results of the optimization problems. Case study 1.

parameter		$\underline{P}_f$ (DL)	$\underline{P}_f$ (ON)	$\overline{P}_f$ (DL)	$\overline{P}_f$ (ON)
$\theta$	$S_0^*$	$2.409 \cdot 10^{-04}$	$2.409 \cdot 10^{-04}$	$3.534 \cdot 10^{-04}$	$3.591 \cdot 10^{-04}$
	$\omega_g^*$	11.782	15.080	11.195	10.056
	$\omega_f^*$	1.507	1.508	1.007	1.005
	$\zeta_g^*$	0.700	0.840	0.575	0.840
	$\zeta_f^*$	0.825	0.840	0.575	0.560
	Output	$P_f$	0.084	0.088	0.977
	$ON$	0.0072	0.0069	0.0354	0.0375
	$n^0$	354000	520 + 1000	28900	595 + 1000

**TABLE 3.** Results of the optimization problems. Case study 2.

parameter		$\underline{P}_f$ (DL)	$\underline{P}_f$ (ON)	$\overline{P}_f$ (DL)	$\overline{P}_f$ (ON)
$\theta$	$S_0^*$	0.040	0.040	0.060	0.060
	$\omega_g^*$	12.557	12.684	14.570	10.053
	$\omega_f^*$	1.507	1.508	1.007	1.005
	$\zeta_g^*$	0.809	0.840	0.700	0.560
	$\zeta_f^*$	0.827	0.840	0.567	0.560
	Output	$P_f$	0.097	0.123	0.859
	$ON$	0.081	0.079	0.307	0.319
	$n^0$	281000	1804 + 1000	292000	626 + 1000



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**Fig. 1.** A two-degrees-of-freedom nonlinear system under stochastic excitation.

**Fig. 2.** Comparison of the operator norm, computed on the linearized system with the probability of failure as computed by Monte Carlo simulation in combination with Newmark method.

**Fig. 3.** Failure probability bounds for different threshold levels obtained by the proposed method (ON) and a double loop implementation (DL).

**Fig. 4.** A six-degrees-of-freedom nonlinear system under stochastic excitation.