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To apply accurate procedures of structural analysis that are now available, the behavior of nailed joints in light-frame wood buildings under long-term loads needs to be studied. Such a behavior can best be evaluated by testing specimens under constant loads, which requires relatively simple testing arrangements. To provide for a practical use of the constant-load test results, theoretical models were developed that predict the behavior of nailed joints under varying loads that are subjected to wood structures in service.

Existing models and principles were used to develop five new general models, all of which account for the nonlinear viscous-viscoelastic behavior of nailed joints. The models incorporated the modified superposition principle and strain hardening principle. Heaviside function and Fourier series were also incorporated to describe varying loads that can be either discrete or mathematically defined continuous function.

To develop experimental data needed for the formulation and verification of the models developed,

joints made of Douglas-fir lumber, plywood and 6d nails were tested under four constant and four varying loads. The experimental data for constant loads were used to formulate specific theoretical models which were further modified for varying loads. The comparison between the predicted and the corresponding test results shows a very good agreement for all the specific models. The models that include the modified superposition principle are the most accurate and the simplest to apply to nailed joints under discrete load functions. Fourier series representation of varying-load functions shows a great potential for practical applications, because it can represent the service loads more accurately than the discrete approximation.

The specific models presented are limited to the type of joints used in this investigation. Other types of joints need to be tested under constant loads to develop appropriate equations that describe their creep behavior under varying loads.

PREDICTION MODELS FOR CREEP BEHAVIOR OF NAILED JOINTS BETWEEN DOUGLAS-FIR LUMBER AND PLYWOOD

by

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PREDICTION MODELS FOR CREEP BEHAVIOR OF NAILED JOINTS BETWEEN DOUGLAS-FIR LUMBER AND PLYWOOD

I. INTRODUCTION

In light-framed wood buildings, nailed joints connect framing and sheathing elements into a composite structural Important characteristics of composite behavior system. load sharing between elements of various stiffnesses are and partial composite action between framing-lumber and sheathing materials. These two characteristics have recently been incorporated into theoretical analysis of walls (32,33,34) and floors (35,37). However, the methods have not been applied practically on a wide scale, partially because the behavior of nailed joints under long-term loading lacks definition. Thus, deformation of nailed joints under long-term loading or creep should be studied. Theoretical models and procedures are needed to predict creep under varying load from the data obtained from constant-load tests. This investigation is aimed at developing and verifying such models. The models investigated are Five-element Model (5-E), Modified Fiveelement Model (M5-E), Viscous-viscoelastic Model 1 (V-VE1), Viscous-viscoelastic Models 2 (V-VE2), and Viscous-viscoelastic Model 3 (V-VE3).

1.1. Justification

In the actual environment, structural wood systems are under loads that are caused by gravity, winds, earthquakes and humans. The response of structural wood systems to these loads often is nonlinear and very complicated to define. Therefore, current design procedures for wood structures use simplified assumptions. For example, wood-stud wall system is represented by a set of identical, independent beams/columns. Such a representation does not account for load sharing among studs and partial composite action between wall coverings and studs. However, procedures were recently developed that consider load sharing and composite action in wall and floor systems (31,32,33,34,36). In order to apply these procedures, load-duration and creep effects on physical properties of lumber, sheathing materials and joints needed to be defined.

Load-duration behavior of lumber has been extensively studied (7,16,38). Similar studies have been also conducted on various types of composition boards (6,25,28, 29,30,31). On the other hand, studies on load-duration behavior of joints are only few (15,17,23,36), even though nailed joints are the most common type of joints in wood constructions. Therefore, studying creep behavior of nailed joints is essential in improving the existing design of wood structures. The composite action between wood frames and wall coverings is a substantial factor in overall strength and stiffness of light-framed wood buildings (36). The degree of composite action depends on the stiffness of joints which decreases under long-term loads. Therefore, longterm behavior of nailed joints should be known before the composite action can be included into a design procedure that gets approved by building codes.

Most studies on creep behavior of nailed joints have been limited to the behavior under constant loads, because constant loads are easy to apply in testing. However, the actual service loads on wood buildings are not constant but vary with time. Long-term varying loads are very difficult and expensive to apply in the laboratory, because each load change usually requires time-comsuming manual manipulation by research personnel. Furthermore, complexities are introduced by variation in material properties. A testing program that would include all the major materials and loads is not practical due to a prohibitively large number of specimens.

A practical alternative is offered by testing under constant loads, because they require simple arrangements and procedures. Thus, if creep behavior under varying loads could be predicted from the data of tests under constant loads, much time and money would be saved in evaluating the behavior of wood system under varying loads. One such a model had been developed and

3

demonstrated the possibility of predicting the long-term performance of nailed joints under varying loads (36). The model was developed and verified for increasing load function only. Therefore, it needs to be developed and verified for decreasing load functions. Furthermore, additional models which are more general, easy to use and perhaps more accurate are needed, because the model accuracy may depend on the type of joints and load functions.

1.2. Objectives

The overall objective was to theoretically model creep behavior of nailed joints. The specific objectives were :

- To evaluate the feasibility of existing theoretical concepts to model creep behavior of nailed joints,
- 2. To modify the most promising existing concepts and develop theoretical models and computer programs that accurately predict creep of nailed joints under varying load functions, and
- 3. To assess the accuracy of the models and computer programs by physical testing of typical nailed joints.

II. LITERATURE REVIEW

Wood has long been recognized and studied as a viscoelastic material. Most sheathing materials in U.S. housing construction are wood-based and, thus, can be regarded as viscoelastic. A few viscoelastic models have been developed to study the creep behavior of wood and wood-based materials, but studies on creep of nailed joints are scarce and strictly experimental (17,23). Only one example of theoretical modeling was found in the literature (36); its significance was in demonstrating the feasibility of using viscoelastic model for creep of nailed joints.

Next, the most important existing viscoelastic models are first introduced and then studied for possible applications to nailed joints. The models are fourelement, three-integral representation and viscousviscoelastic models.

2.1. Four-element model

The four-element or Burger's model is one of the most widely used mechanical models for viscoelastic materials. The basic elements of this model are spring and dashpot which represent elasticity and viscosity, respectively. Combinations of springs and dashpots can represent the behavior of many materials quite accurately. The simplest combination that has one spring and one dashpot in series is the Maxwell model (Figure 2.1a) and the Kelvin model with the spring and dashpot in parallel (Figure 2.1b). The Maxwell model simulates the instantaneous elastic displacement and recovery. The Kelvin model shows the delayed elasticity. But neither of these two models can describe the behavior of viscoelastic materials.

The simplest model that does describe the viscoelastic behavior successfully is a four-element model (Figure 2.2) which is a combination of Maxwell and Kelvin model. Its constitutive equation can be derived from the strain response of the model under constant load :

$$\epsilon(t) = \sigma/E_{1} + \sigma/E_{2} \{1 - EXP(-E_{2}t/n_{2})\} + \sigma t/n_{1} \quad (2-1)$$

in which ε (t) = strain as a function of time, t; σ = stress; E_1 and E_2 = spring stiffnesses; and n_1 and n_2 = dashpot viscosities. In four-element model, the first, second and third term represent instantaneous elastic, delayed elastic and viscous strain, respectively.

There are several procedures for evaluating parameters of the four-element model. For instance, Moslemi (27) applied such a model to describe creep behavior of hardboard under static ramp loading. He divided the total creep into three parts : instantaneous elastic, delayed elastic and viscous strain. The

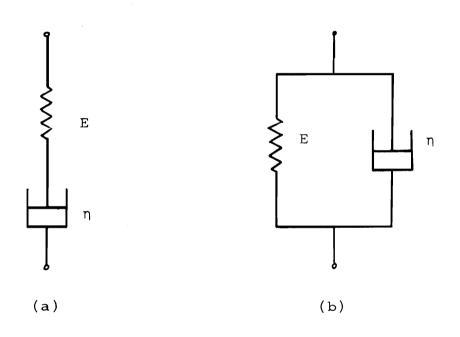


Figure 2.1. Maxwell (a) and Kelvin model (b). (E = spring stiffness, n = dashpot viscosity)

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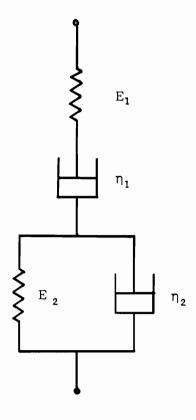


Figure 2.2. Four-element or Burger's model.

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constitutive equation developed by him under load function P(t) = At was :

$$\varepsilon$$
 (t) = At/E+[At/E+A n₁/E²{EXP(-Et/ n₁)-1}]+
At²/2 n₂ (2-2)

in which all symbols were defined earlier. In this model, the stiffness of both springs should be equal.

Szabo and Ifju (38) applied a four-element model to describe creep of wood beams under stresses due to moisture adsortion and desorption. They obtained the parameters of the constitutive equation by assuming that the slope of strain-time curve of specimens under constant load remained constant after t > 120 hours.

Pierce et al. (6,28,29,30) found the constitutive equation of the four-element model for chipboard by employing constant-load tests :

$$\varepsilon$$
 (t) = B₁+B₂ {1-EXP(-B₃t)}+B₁ t (2-3)

in which B_1 , B_2 , B_3 and B_4 are the parameters obtained by nonlinear least square curve fitting of experimental data. The model was applied to study the influence of moisture content, stress (6,29,30) and temperature (6) on creep of chipboard.

Pierce et al. (31) modified equation (2-3) to describe the nonlinear viscosity of chipboard. They replaced the linear dashpot in the Maxwell model by a nonlinear dashpot (Figure 2.3) in which the viscosity was expressed as a power function of time. The corresponding constitutive equation has five parameters, among which B and B are nonlinear :

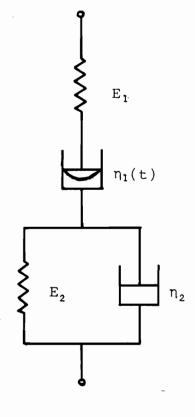
$$\varepsilon$$
 (t) = B₁+B₂{1-EXP(-B₃t)}+B₄t^{B₅} (2-4)

Again, nonlinearleast squares curve fitting was employed to get Bi, i = 1, ..., 5.

Thus, the existing investigations demonstrate that four-element model can successfully predict creep of wood and wood-based materials. Therefore, it was expected to be also successful to predict creep of nailed joints.

2.2. Three-integral representation

Multiple-integral representations have been widely used to predict the behavior of viscoelastic material, such as plastic, under long-term loads (8,11,12,26,27). One form is a three-integral representation which has been recognized as the one that provides adequate exactness and simplicity in representing nonlinear viscoelasticity. In deriving three-integral equations for continuous load functions, Findley et al. (11) applied first, a continuum mechanics approach and then, a modified superposition principle, both of which gave this expression :



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Figure 2.3. Nonlinear four-element model. (η_1 (t) = viscosity of nonlinear dashpot)

$$\varepsilon (t) = \int_{0}^{t} F_{1}(t-Z_{1}) \dot{\sigma}(Z_{1}) dZ_{1} + \int_{0}^{t} \int_{0}^{t} F_{2}(t-Z_{1}, t-Z_{2}) \dot{\sigma}(Z_{1}) \dot{\sigma}(Z_{2}) dZ_{1} dZ_{2} + \int_{0}^{t} \int_{0}^{t} \int_{0}^{t} F_{3}(t-Z_{1}, t-Z_{2}, t-Z_{3}) \dot{\sigma}(Z_{1}) \dot{\sigma}(Z_{2}) \\ \dot{\sigma}(Z_{3}) dZ_{1} dZ_{2} dZ_{3}$$
(2-5)

in which Z_1 , Z_2 and Z_3 = arbitrary time points between zero and t; $\dot{\sigma}$ (t) = derivative of stress with respect to t; and $F_1(t)$, F_2 (t) and F_3 (t) are known as kernel functions (8,11,12), which represent the time-dependent properties of material and can be derived by testing specimens under constant stress.

For constant stress, equation (2-5) becomes (11,24, 26,36) :

$$\varepsilon$$
 (σ ,t) = F, (t) σ +F₂(t) σ^2 +F₃(t) σ^3 (2-6)

in which $F_1(t)$, $F_2(t)$ and $F_3(t)$ are the same kernel functions as those in equation (2-5).

Several investigations (8,11,12,24,26,27) have shown that kernel functions can be expressed as power functions:

$$Fi(t) = Fi^{\circ} + Fi^{+}(t)$$
 (2-7)

where Fi° = constant; $Fi^{+}(t)$ = power function of time obtained from constant load tests; and i = 1,2,3.

In equation (2-5), F_1 , F_2 and F_3 are functions of one, two and three variables, respectively. Numerous tests are required to evaluate these functions and no one has determined all the functions that are required for any material under all possible stress conditions (11). Findley et al. (11,12) developed three simplified forms that can be evaluated from tests under constant loads. First is the product form :

$$F_{2}(t-Z_{1}, t-Z_{2}) = [F_{2}(t-Z_{1})F_{2}(t-Z_{2})]$$
(2-8)

$$F_{3}(t-Z_{1}, t-Z_{2}, t-Z_{3}) = [F_{3}(t-Z_{1})F_{3}(t-Z_{2})F_{3}(t-Z_{3})] (2-9)$$

Second is the first additive form :

$$F_{2}(t-Z_{1}, t-Z_{2}) = F_{2}(2t-Z_{1}-Z_{2})$$
 (2-10)

$$F_{3}(t-Z_{1}, t-Z_{2}, t-Z_{3}) = F_{3}(3t-Z_{1}-Z_{2}-Z_{3})$$
 (2-11)

and third is the second additive form :

$$F_{2}(t-Z_{1}, t-Z_{2}) = 1/2[F_{2}(t-Z_{1})+F_{2}(t-Z_{2})]$$
(2-12)
$$F_{3}(t-Z_{1}, t-Z_{2}, t-Z_{3}) = 1/3[F_{3}(t-Z_{1})+F_{3}(t-Z_{2})+ F_{3}(t-Z_{3})]$$
(2-13)

These forms of kernel functions were applied successfully to several man-made materials such as plastic (11,12) and nailed joint (36).

An alternative to these forms is the modified superposition principle which was developed on the basis of the Boltzmann superposition principle for linear materials (11). Because of its importance, it is discussed in detail next.

2.2.1. Modified superposition principle (MSP)

MSP was developed by Findley et al. (8,11) to describe the nonlinear behavior of viscoelastic materials, such as plastic (8,12,26), aluminum (9,10,19,20) and stainless steel (3,4,5,24).

The development was based on the Boltzmann linear superposition principle which is illustrated in Figure 2.4. The creep displacement in Figure 2.4b can be obtained by resolving the loading stress function (Figure 2.4a) into two constant stresses shown in Figures 2.4c and 2.4e and then, superimposing the corresponding creep displacements shown in Figures 2.4d and 2.4f. However, this principle can be used for linear materials only (11).

MSP is a nonlinear superposition of creep displacements as illustrated in Figure 2.5. Varying stresses in Figure 2.5a are resolved into two stresses shown in Figures 2.5c and 2.5e. Superposition of resulting creep displacements in Figures 2.5d and 2.5f produces the creep displacement in Figure 2.5b under stress function shown in Figure 2.5a.

For stepwise stresses, MSP can be expressed as the following sum(11) :

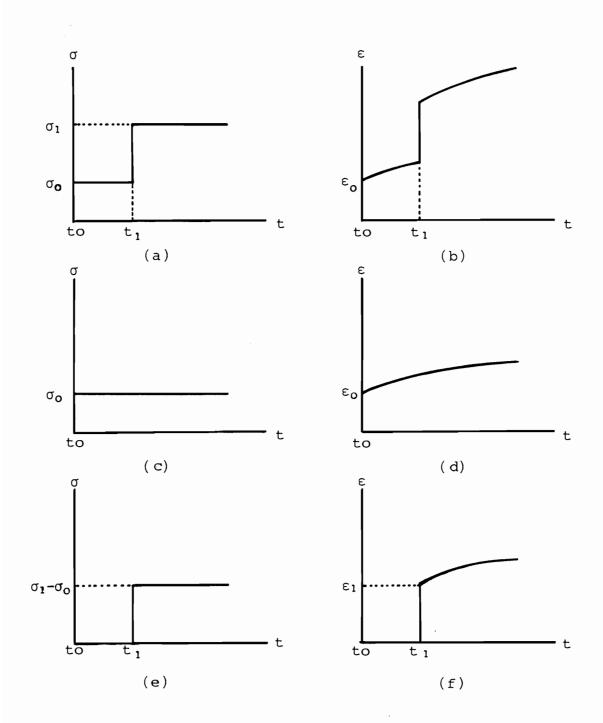


Figure 2.4. Boltzmannn superposition principle;(a) stress function, (b) strain-time curve, (c) first step in stress function, (d) strain under the first step, (e) second step in stress function, and (f) strain under the second step.

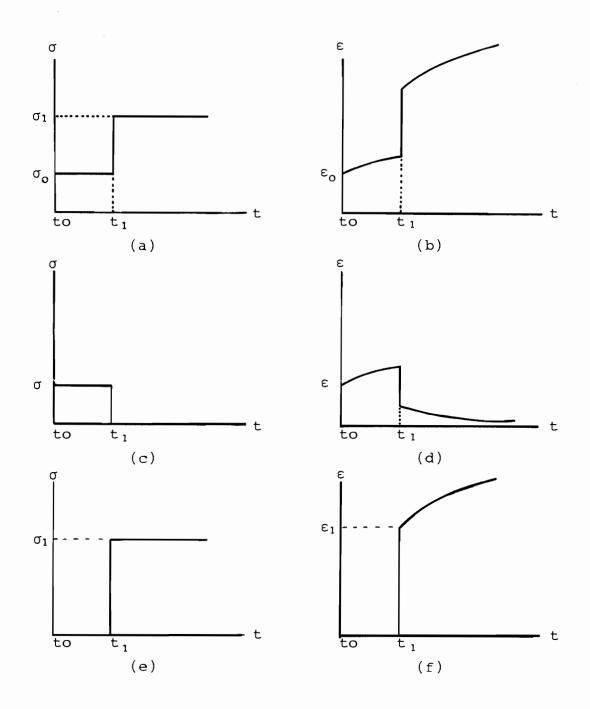


Figure 2.5. Modified superposition principle; (a) stress function, (b) strain-time curve, (c) first step in stress function, (d) strain under the first step, (e) second step in stress function, and (f) strain under the second step.

$$\varepsilon(t) = \sum_{i=0}^{N-1} [f(\sigma_i, t-t_i) - f(\sigma_{i-1}, t-t_i)], t > t_{N-1} (2-14)$$

where $\varepsilon = \text{strain}; N = \text{number of stressing steps}; \sigma_i = \text{stress level for i = 1,...,(N-1)}; and f(\sigma,t) = creep function. For continuous stress functions, the strain can be expressed as an integral form, that is, as an addition of an infinite number of infinitesimal steps of stress (11) :$

$$\varepsilon(t) = \int_{0}^{t} \frac{\partial f(\sigma(Z), t-Z)}{\partial \sigma(Z)} \frac{d \sigma(Z)}{dZ} dZ \qquad (2-15)$$

where Z is the integration variable identified in equation (2-5).

For instance, if a creep function is obtained by a three-integral form under constant load (equation(2-6)), then the creep displacement under varying stress function in Figure 2.5a can be obtained from equations (2-6) and (2-14) :

$$\varepsilon (t) = f(\sigma o, t-to) + f(\sigma_1, t-t_1) - f(\sigma o, t-t_1)$$

= $F_1(t-to) \sigma o + F_2(t-to) \sigma o^2 + F_3(t-to) \sigma o^3 +$
 $F_1(t-t_1) \sigma_1 + F_2(t-t_1) \sigma_1^2 + F_3(t-t_1) \sigma_1^3 -$
 $F_1(t-t_1) \sigma_0 - F_2(t-t_1) \sigma_0^2 - F_3(t-t_1) \sigma_0^3$ (2-16)

The equations obtained by MSP can be used to represent creep under varying-stress functions.

Originally, three-integral representation in equation (2-5) was proposed to represent creep under arbitrary stress functions. However, it is practically impossible to get the kernel functions in equation (2-5). Therefore, MSP provides a viable alternative in expressing the creep of nailed joints under varying stress.

2.3. Viscous-viscoelastic model

Most of the existing works have concentrated on creep of materials that have the same instantaneous displacement and recovery. However, there are many materials, including nailed joints, which do not fully recover the instantaneous displacement upon unloading. For such materials, the viscous-viscoelastic model developed by Findley et al. (3,4,5,9,10,19,20) is very useful.

The basis for this model is in subdividing the total creep strain into five components :

$$\varepsilon_{t} = \varepsilon_{e} + \varepsilon_{p} + \varepsilon_{pv} + \varepsilon_{NV} + \varepsilon_{de}$$
 (2-17)

where ε_t = total strain; ε_e = instantaneous elastic strain which is time-independent and recoverable; ε_p = instantaneous plastic strain which is time-independent and nonrecoverable; ε_{pv} and ε_{NV} = positive and negative viscous strains, respectively, which are time-dependent and nonrecoverable; and ε_{de} = delayed elastic strain which is time-dependent and recoverable. This concept was applied to develop theoretical models in this study.

In experiments, Findley et al. used viscoelastic materials, such as stainless steel (3,4,5) and aluminum (9,10,19,20), in which the instantaneous recovery was the same as the instantaneous displacement. Thus, they assumed that the instantaneous plastic strain was zero. For total creep strain, they employed a power function :

$$\varepsilon_{t} = \varepsilon^{\circ} + \varepsilon^{+} t^{n} = \varepsilon_{e} + \varepsilon_{VE} + \varepsilon_{v}$$
 (2-18)

in which ε° and ε^{+} are parameters obtained from tests; ε_{pv} and ε_{NV} are expressed as one term that is timedependent nonrecoverable strain, ε_{v} ; and ε_{de} is expressed as time-dependent recoverable strain, ε_{VE} . Three-integral representation was then applied to express the total creep :

$$\varepsilon_{t} = \mathbf{F}_{1}\sigma + \mathbf{F}_{2}\sigma^{2} + \mathbf{F}_{3}\sigma^{3}$$

= $(\mathbf{F}_{1}^{\circ} + \mathbf{F}_{2}^{+}(t))\sigma + (\mathbf{F}_{2}^{\circ} + \mathbf{F}_{2}^{+}(t))\sigma^{2} + (\mathbf{F}_{3}^{\circ} + \mathbf{F}_{3}^{+}(t))\sigma^{3}$ (2-19)

where $F_i = F_i^{\bullet} + F_i^{+}(t)$, i=1,2,3, are kernel functions; and F_i^{\bullet} and F_i^{+} are parameters obtained from experiments.

Findley et al. postulated that ε_{VE} and ε_v are functions of the same power of time, n. Therefore, equation (2-18) becomes :

$$\varepsilon_{+} = \varepsilon_{a} + Bt^{n} + Mt^{n} \tag{2-20}$$

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where B and M are experimental coefficients of recoverable and nonrecoverable strain terms, respectively. Equation (2-20) is often written as :

$$\varepsilon_{t} = \varepsilon_{e} + M(R+1)t^{n} \qquad (2-21)$$

where R = B/M. By comparing equations (2-18) and (2-21), expressions for $\varepsilon_{\rm VE}$ and $\varepsilon_{\rm v}$ can be written as :

$$\varepsilon_{VE} = \{R/(R+1)\} \varepsilon^+ t^n \qquad (2-22)$$

$$\varepsilon_v = \{1/(R+1)\} \varepsilon^+ t^n$$
 (2-23)

Expression for time-dependent strain, $F(\sigma)$, follows from equations (2-18) and (2-19) :

$$\epsilon^{+} t^{n} = F(\sigma) = F_{1}^{+}(t) \sigma + F_{2}^{+}(t) \sigma^{2} + F_{3}^{+}(t) \sigma^{3}$$
 (2-24)

Findley et al. found that no strain was developed under the stress less than certain limit. To incorporate this concept into the model, they further introduced the creep limit, σ *, into equation (2-24) :

$$F(\sigma) = F_{1}^{+}(t) (\sigma - \sigma *) + F_{2}^{+}(t) (\sigma - \sigma *)^{2} + F_{3}^{+}(t) (\sigma - \sigma *)^{3}$$
(2-25)

Next, Findley et al. applied MSP to recoverable strain of equation (2-22) and strain hardening principle to nonrecoverable strain of equation (2-23). The result was the expression for creep under varying loads. The constitutive equation for $\varepsilon_{\rm VE}$ under three-step increasing loads was found to be (19) :

$$\varepsilon_{VE} (t) = \{R/(R+1)\}[F(\sigma_1)\{t^n - (t-t_1)^n\} + F(\sigma_2)\{(t-t_1)^n - (t-t_2)^n\} + F(\sigma_3)(t-t_2)^n], t > t_2$$
(2-26)

where $F(\sigma)$ is the same as in equation (2-25).

2.3.1. Strain Hardening principle (SHP)

Many theories have been proposed to predict the creep rate in terms of other variables than stress (13). Among those, SHP has been the most successful (8,11,13). In SHP, strain rate is expressed as a function of strain and stress.

The basis of SHP is the postulation that the creep under constant stress can be represented by a function of stress and time (13), the simplest form of which is :

$$\varepsilon = \mathbf{A} \sigma^{\mathbf{n}} \mathbf{t}^{\mathbf{m}} \tag{2-27}$$

where A, n and m are constants. Differentiating equation

(2-27) gives the strain rate :

$$d\varepsilon/dt = \dot{\varepsilon} = mA \sigma^{n} t^{m-1}$$
(2-28)

Expression for time variable, t, can be obtained by imposing 1/m power on both sides of equation (2-27) and solving for t:

• · · ·

$$t = (\varepsilon)^{1/m} / (A \sigma^{n})^{1/m}$$
 (2-29)

which is then substituted into equation (2-28) to obtain :

$$d\varepsilon / dt = mA\sigma^{n} [\varepsilon^{1/m} / (A\sigma^{n})^{1/m}]^{m-1}$$
(2-30)

or

$$[\varepsilon^{(1/m-1)}/m]d\varepsilon = (A\sigma^n)^{1/m}dt \qquad (2-31)$$

Integrating both sides of equation (2-31) yields :

$$\varepsilon^{1/m} = \int (A\sigma^n)^{1/m} dt \qquad (2-32)$$

Therefore,

$$\varepsilon = \left[\int (A\sigma^{n})^{1/m} dt \right]^{m}$$
 (2-33)

Findley et al. (4,5,9,10) applied this principle to the nonrecoverable strain of equation (2-23) to get the expression for the nonrecoverable viscous strain under three-step increasing load :

$$\varepsilon_{v} = \{1/(R+1)\} [\{F(\sigma_{1})\}^{1/n}(t_{1}) + \{F(\sigma_{2})\}^{1/n}(t_{2}-t_{1}) + \{F(\sigma_{3})\}^{1/n}(t-t_{2})]^{n}, t > t_{2}$$
(2-34)

The expression for the total creep under three-step increasing load is the sum of equations (2-26), (2-34) and the instantaneous elastic strain at the time of interest. The prediction of this viscous-viscoelastic model agreed closely with the experimental data for aluminum (9,10,19,20) and stainless steel (3,4,5).

2.5. Creep and stiffness of nailed joints

There have been many studies (1,14,15,17,22,23,36) on the strength properties of nailed joints, but only a few (17,23,36) have dealt with creep of nailed joints.

Jenkins et al. (17) studied creep-related stiffness loss of nailed joints between Douglas-fir stud and plywood. They approximated the stiffness of nailed joints, K_i, by the secant modulus :

$$K_{j}(P,t) = (P_{j} - P_{j-1}) / [S_{t}(P,t_{j}) -S_{t}(P,t_{j-1})] (lb/in)$$
(2-35)

where $S_t = \text{total creep}$; P = load; t = time; and j-1 and j = successive intervals defining the secant modulus. The major problem with this study was the use of linear superposition for creep curves under constant loads to get creep under increasing loads by employing the assumption that creep behavior of nailed joints is linear. Another limitation was in restricting the study within the effect of increasing loads only.

Polensek (36) developed a theoretical model which can be used to predict creep of nailed joints under increasing loads from the data of constant load tests. He fitted the data to power functions :

$$Sc = at^n$$
 (2-36)

in which Sc = creep slip; a and n = constants obtained from experiments; and t = time. Mack (25) used the same power function. Polensek employed the three-integral representation, equation (2-6), to represent the creep under constant load :

$$S(t) = F_1(t)P+F_2(t)P^2+F_3(t)P^3$$
 (2-37)

where S(t) = creep slip; P = load; and F_i(t) = kernel functions for i = 1,2,3. The corresponding equation for stepwise load is :

$$S(t) = \prod_{i \neq 0}^{n} (\Delta P_{i}) F_{1}(t-t_{i}) + \prod_{i \neq 0}^{n} \prod_{j \neq 0}^{n} (\Delta P_{i}) (\Delta P_{j}) F_{2}(t-t_{i}, t-t_{j}) + \prod_{i \neq 0}^{n} \prod_{j \neq 0}^{n} (\Delta P_{i}) (\Delta P_{j}) (\Delta P_{k}) F_{3}(t-t_{i}, t-t_{j}, t-t_{k})$$
(2-38)

where $\Delta P_{i',j'k} = step size in stepwise load.$

To approximate the two- and three-variable kernel functions in equation (2-38), Polensek employed the product and two additive forms which were discussed in Section 2.2. In applying these approximations to nailed joints, he found that any of these three forms gives the accurate predictions of creep slip under increasing loads.

III. THEORETICAL PROCEDURE

It was not known at the beginning of this investigation whether the existing viscoelastic models for wood and wood-based materials can be directly applied to nailed joints. Therefore, several models for materials, such as plastic and aluminum, were studied for possible application to nailed joints. Three of the most promising models were the four-element, three-integral and viscousviscoelastic model, but they did not include all the mechanisms of nailed joint behavior. Therefore, the three models were modified in this investigation, which produced five new models : Five-element, Modified Five-element and Viscous-viscoelastic Model 1, 2 and 3. This chapter describes the development of each of these models.

3.1. Theoretical principles

The theoretical procedure employed in this study is summarized in Figure 3.1. Five models were developed from the existing models and concepts. To modify the models for varying loads, two approaches were chosen, one based on MSP and SHP, and the other on MSP only. The developments included three load functions : discrete function, Heaviside function and Fourier series. Among those three, discrete load function was used in both approaches, while the other two were used in Approach 2

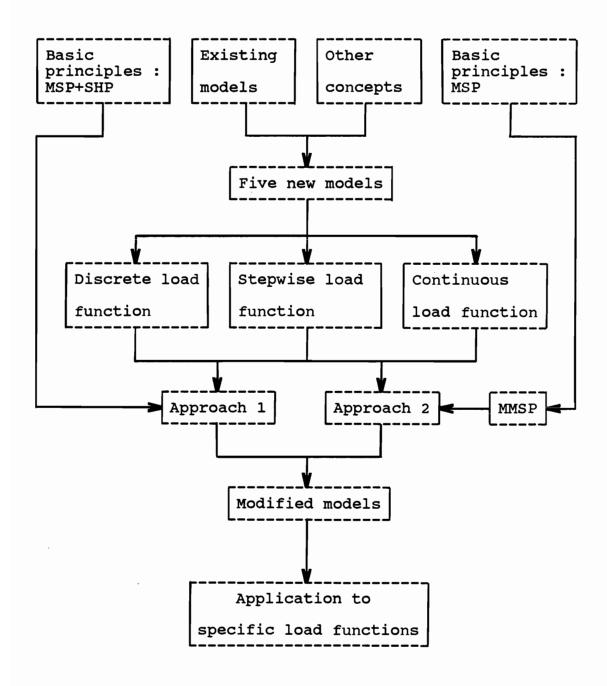


Figure 3.1. Theoretical procedure.

only. The developed models were then applied to specific load functions to get the corresponding expressions for the creep slip. The basic ideas incorporated into modeling are introduced first.

3.1.1. Basic concepts

It is often convenient to divide the time-dependent behavior of viscoelastic materials into instantaneous elastic, delayed elastic and viscous component. The first two components are recoverable but the third is not.

Past investigations have shown that wood and woodbased materials are viscoelastic materials (6,7,16,22,28, 29,30,38). Nails are made of common steel which is also viscoelastic material. Therefore, it may be postulated that nailed joints between lumber and sheathing material also behave viscoelastically. However, crushed wood around embedded nails introduces additional complexity. Preliminary testing in this investigation showed that the time-dependent behavior of nailed joints could not be described accurately by any of the existing viscoelastic models. For instance, Figure 3.2 illustrates a typical behavior of nailed joints under long-term load. The instantaneous recovery upon unloading, DG = BC, is much less than the instantaneous slip, AC, while the instantaneous displacement and recovery are the same for viscoelastic materials.

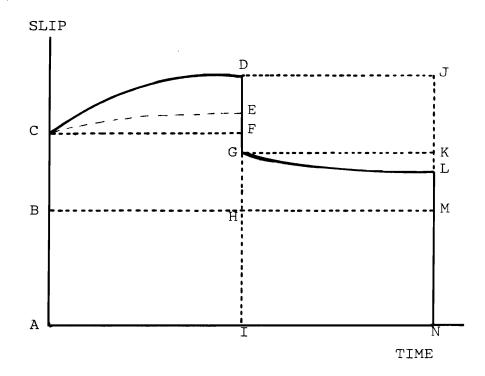


Figure 3.2. Typical creep curve of nailed joints.

In this investigation, three useful assumptions have been adopted. The first is that the instantaneous and delayed elastic displacement are totally recoverable. The second states that the plastic and viscous displacement are not recoverable. The third assumption reasons that because only elastic displacement is totally recoverable, the elastic displacement-versus-time curve should be the same as the recoverable displacement-versus-time curve. Thus, the total slip of nailed joints can be divided into four components, which are similar to the division introduced in the viscous-viscoelastic model discussed in Chapter 2.

In this study, the total slip is visualized as a combination of components that can be explained physically and represented mathematically (Figure 3.2). As mentioned before, the instantaneous elastic slip, BC, is the same as the instantaneous recovery, DG, and time-independent. The delayed elastic slip, CEF, is equal to the delayed recovery, GLK, and time-dependent. The rest in the instantaneous slip, AB, is plastic, time-independent and nonrecoverable. Under constant load, the instantaneous plastic slip is developed immediately upon loading and remains constant during loading and unloading period. The rest of total creep slip, CDE, is viscous, time-dependent and nonrecoverable. Therefore, the total creep slip, ACDI, can be divided into four components which are the instantaneous elastic, BCFH, instantaneous plastic, ABHI,

delayed elastic, CEF, and viscous slip, CDE.

The components in Figure 3.2 are similar to those employed by Findley et al. (3,4,5), whose model lacks the instantaneous plastic displacement. However, in nailed joints, the instantaneous plastic slip due to compressed damages of wood fibers around the nail is significant and must be included.

The models developed in this study consist of four components described above, which are connected in series and may be added to give the total creep slip :

$$S_{t} = Se + Sp + S_{de} + Sv \qquad (3-1)$$

where $S_t = \text{total creep slip}$; Se = instantaneous elasticslip; Sp = instantaneous plastic slip; $S_{de} = \text{delayed}$ elastic slip; and Sv = viscous slip.

3.1.2. Application of MSP to nailed joint (MMSP)

MSP discussed in Section 2.2 was proposed by Findley et al. (8) for materials having the same instantaneous displacement and recovery. In their model, the instantaneous plastic displacement was assumed to be zero, and all the creep displacement was assumed to be recoverable. Thus, the same equation was used to describe both creep displacement and recovery. However, in nailed joints, the instantaneous plastic slip is larger than the instantaneous elastic slip and the viscous slip is not recoverable (Figure 3.2). Therefore, MSP can be applied to elastic slip but not to total creep slip.

In viscous-viscoelastic model, Findley et al.(3,9,19) employed MSP and SHP for recoverable and nonrecoverable displacements, respectively. The same method was employed in this study and is referred to as Approach 1. However, the predictions by Approach 1 were smaller than the experimental data because the viscous slip predicted by SHP was too small. Thus, a further modification was carried out, which resulted in modified MSP or MMSP, and referred to as Approach 2.

Mathematically, MMSP was accomplished as follows. In Section 2.3.1, equation (2-33) defined the viscous displacement predicted by SHP as :

$$\varepsilon \mathbf{v} = \left[\int_{0}^{t} \left(\mathbf{A} \, \sigma^{n} \right)^{1/m} \mathrm{d}z \right]^{m} \tag{3-2}$$

Under stepwise load function, equation (3-2) becomes :

$$\varepsilon \mathbf{v} = [(A \sigma_0^n)^{1/m} t_1 + (A \sigma_1^n)^{1/m} (t_2 - t_1) + \dots + (A \sigma_{N-1}^n)^{1/m} (t - t_{N-1})]^m, t > t_{N-1}$$
(3-3)

where N = number of steps. The properties of m power are :

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$$(A+B+C+...)^m < \alpha$$
, = α and $> \alpha$ if m<1, m=1 and
m>1, respectively in which $\alpha = A^m + B^m + C^m + ...$ (3-4)

In equation (3-3), m is always less than one. Therefore, the maximum value of equation (3-3) equals :

$$\varepsilon \mathbf{v} = \left[\int_{0}^{t} (\mathbf{A} \sigma^{n})^{1/m} dz \right]^{m}$$

$$< \left[\mathbf{A} \sigma_{0}^{n} (\mathbf{t}_{1})^{m} + \mathbf{A} \sigma_{1}^{n} (\mathbf{t}_{2} - \mathbf{t}_{1})^{m} + \dots + \mathbf{A} \sigma_{N-1}^{n} (\mathbf{t} - \mathbf{t}_{N-1})^{m} \right], \mathbf{t} > \mathbf{t}_{N-1}$$
(3-5)

In Approach 2 for nailed joints, the maximum value of equation (3-5) was used to define the viscous slip, Sv, in which load, P, and slip, S, replace stress, σ , and strain, ε . Figure 3.3 graphically illustrates this approach. The load function in Figure 3.3a is composed of two steps. Under each step, the creep slip is divided into two parts, the recoverable and nonrecoverable slip. For the recoverable slip, Sr, MSP can be applied as shown in Figures 3.3c and 3.3d. For the nonrecoverable slip, Sn, the maximum value of equation (3-5) is used, which can be justified as follows. Upon unloading Po at time t1, the instantaneous plastic slip, Sp, is recovered and the viscous slip, Sv, remains constant (Figure 3.3e). Upon loading P_1 at time t_1 , the instantaneous plastic and viscous slip are developed (Figure 3.3f). Finally, combining all four graphs, Figures 3.3c, 3.3d, 3.3e and 3.3f, gives Figure 3.3b which is the total slip, S_{f} , under

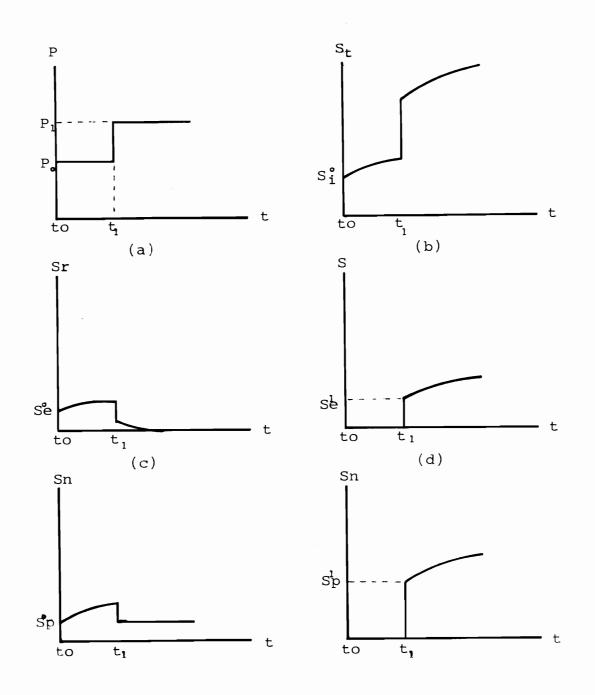


Figure 3.3. MMSP for nailed joints; (a) Load function, (b) slip under load function (a), (c) recoverable slip in the first step, (d) recoverable in the second step, (e) nonrecoverable slip in the first step, and (f) nonrecoverable slip in the second step.

load function in Figure 3.3a. Thus :

$$S_{t} = Sr(Po, t-to) + Sr(P_{1}, t-t_{1}) - Sr(Po, t-t_{1}) + Sp_{1} + Sv(Po, t_{1} - to) + Sv(P_{1}, t-t_{1}), t > t_{1}$$
(3-6)

Approach 1 and Approach 2 involve assuming that the stepwise decreasing load is the same as the stepwise increasing load acting in the opposite direction to the existing load and having magnitude of the maximum load just before unloading minus the actual load, that is, it is assumed as compressive force. It is further assumed that the compressive force induces elastic displacement or recovery only until the compressive force exceeds the existing load. By applying MSP to this compressive force, the recovery upon partial unloading can be calculated.

3.1.3. Stiffness loss due to creep

An additional concept must be included in the modeling of nailed joints. Previous studies (17,36) have shown that the stiffness of nailed joints decreases with creep magnitude. However, any specific definition of this effect has not been reported. The experimental data of this study showed that the creep-related stiffness loss affects the recovery under stepwise decreasing load. Therefore, the effect of stiffness loss should be included in the creep modeling. In this study, the stiffness of nailed joint will be defined by the secant slip modulus, that is the ratio of total load to the corresponding slip. For instance, if the load function in Figure 3.4a is applied to nailed joints, the creep slip curve becomes that shown in Figure 3.4b and the corresponding load-slip curve becomes Figure 3.4c. If load P_3 is applied instantaneously, the instantaneous slip will be Sa. However, under the load function in Figure 3.4a, the slip is developed as much as S_t (Figure 3.4b). The difference, Sc, between S_t and Sa is the delayed creep slip developed under the stepwise load shown in Figure 3.4a.

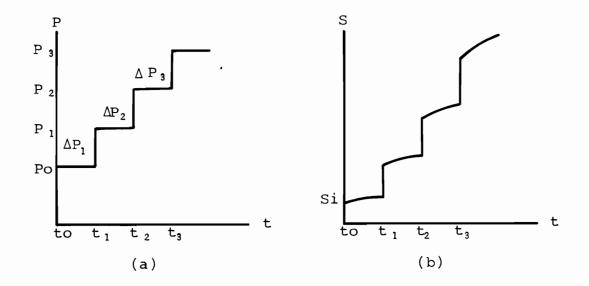
The stiffness of nailed joints in Figure 3.4c may be defined by the slope of line OA when P is applied instantaneously :

$$Ka = P_{3} / Sa$$
 (3-7)

where Ka = stiffness or slip modulus of nailed joints under instantaneous loading. When the stepwise increasing load function (Figure 3.4a) is applied, the slope of line OB in Figure 3.4c gives the slip modulus of nailed joints:

$$Ks = P_3 / S_+ = P_3 / (Sa + Sc)$$
 (3-8)

where Ks = slip modulus of nailed jonts under stepwise loading. Therefore, the stiffness reducing factor, Rs, is



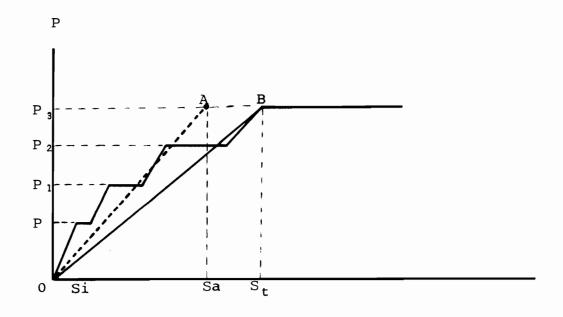


Figure 3.4. Stiffness loss due to creep in nailed joints; (a) load function, (b) slip under load function (a), and (c) load-slip curve of nailed joints under load function (a).

the ratio of Ks to Ka :

$$Rs = Ks / Ka = Sa / S_t = Sa / (Sa + Sc)$$
 (3-9)

Factor Rs depends on the shape of the load function and the duration of each step.

Under the decreasing stepwise loading, the recovery is affected by creep under each load step. The recovery rate is slowed because of creep acting opposite to recovery. The testing showed that the recovery under stepwise unloading was much less than the recovery under instantaneous unloading. Therefore, it was assumed in the modeling that, upon stepwise unloading, the recovery is reduced as much as the stiffness reducing rate under the reversed stepwise loading. For example, the following procedure would account for the stepwise decreasing load function in figure 3.5a. First, the stiffness loss under the reversed load function in Figure 3.5b and the corresponding stiffness reducing factor would be determined by equations (3-6) through (3-9) (Figure 3.5c). Then, the reduced recovery (dot line in Figure 3.5d) would be calculated by multiplying the stiffness reducing factor to the slip recovery obtained by MSP. In the experimental part of this study, it was shown that, in nailed joints, the recovery under partial unloading was almost negligible. Thus, the stiffness reducing factor was applied only to the complete unloading step at the end of

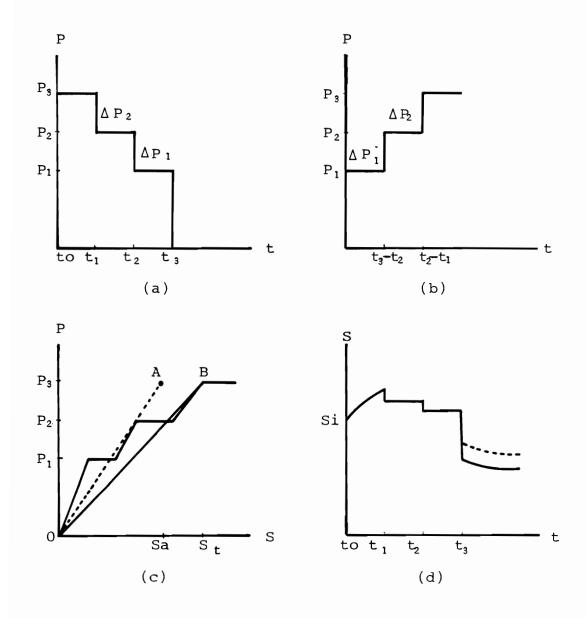


Figure 3.5. Recovery under stepwise decreasing load; (a) decreasing load function, (b) reversed increasing load function, (c) load-slip curve under load function (b), and (d) slip-time curve under load function (a).

the decreasing load function. The recovery predicted by this procedure agreed well with the experimental data.

3.2. Theoretical models

The development of the five new models are discussed next.

3.2.1. Five-element Model (5-E)

As mentioned before, the four-element model is too simple to represent the creep behavior of nailed joints. Therefore, a concept of nonlinear viscosity, originally introduced by Pierce et al. (31), was added to the fourelement model. Specifically, a nonlinear dashpot with variable flow rate replaced the single linear dashpot in four-element model (Figure 2.3). This was accomplished by expressing the viscosity of the nonlinear dashpot as a power function of time :

$$\eta_1$$
 (t) = atⁿ (3-10)

in which η_1 (t) = nonlinear viscosity of dashpot as a function of time, t, and a and n = constants obtained from experiments. Expressing the relation between load, P, and viscous slip, Sv, as :

$$P = n_1(t) (dSv / dt)$$
 (3-11)

and integrating gave :

$$Pt = n_1 (t) Sv$$
 (3-12)

$$Sv = Pt / n_1(t)$$
 (3-13)

Finally, the substitution of equation (3-10) into (3-13) resulted in :

$$Sv = Pt / at^n = APt^M$$
 (3-14)

where A = 1/a and M = 1-n.

However, such a model did not have the component describing the instantaneous plastic displacement, so that it still could not describe the behavior of nailed joints. Thus, a new element for the instantaneous plastic slip was added to the nonlinear four-element model. The result was the Five-element Model (5-E) which is shown in Figure 3.6. The new element, refered to as a nonlinear time-hardening element, describes the nonlinear instantaneous plastic slip upon loading and becomes a rigid element immediately after the instantaneous slip takes place and remains constant until the load increases.

The plasticity of the nonlinear time-hardening element, $\mu(P)$, was assumed to be a power function of load. Therefore, its load-slip relation was found to be :

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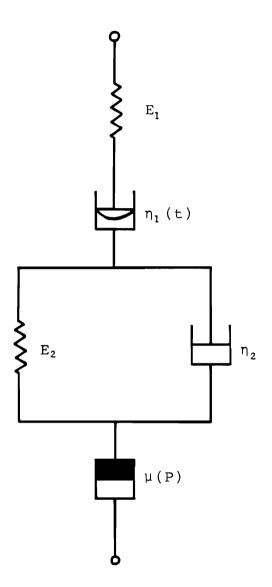


Figure 3.6. Five-element Model.

$$Sp = BP^{m}$$
 (3-15)

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where B and m are constants obtained from experiemnts. Now, the constitutive equation for Model 5-E could be written :

$$S_{t} = P/E_{1} + P/E_{2} [1 - EXP\{-(E_{2}/n_{2})t\}] + APt^{M} + BP^{m}$$
 (3-16)

where E , E , n_2 , A, B, M and m are parameters determined from experimental data.

Under constant load, equation (3-16) can be further simplified to give :

$$S = A_1 + A_2 \{1 - EXP(-A_3t)\} + A_4 t^M + A_5$$
 (3-17)

where A_1 , A_2 , A_3 , A_4 , A_5 and M are parameters obtained from the constant load tests. To apply this method, a set of different parameters is needed for each load level, which make Model 5-E difficult to apply to practical problems.

3.2.2. Modified Five-element Model (M5-E)

In applying Model 5-E, it is not practical to evaluate and use several sets of coefficients for various load levels when predicting the creep slip under varying load. Thus, Model 5-E was modified to allow its application with only one set of coefficients for all the load levels. The new model is refered to as the Modified Five-element Model (M5-E).

Tests have shown that the instantaneous elastic and viscous slip of nailed joints are not linear with respect to load. To include the nonlinearity of the instantaneous elastic slip, the linear single spring in Model 5-E was replaced by a nonlinear single spring (Figure 3.7). The stiffness of the nonlinear spring was assumed to be a power function of the load. Thus, the load-slip relation in this spring could be expressed as follows :

$$Se = B_r P^{N_1}$$
 (3-18)

where B_1 and N_1 are constants obtained from experiments.

The viscosity of nonlinear dashpot in Model M5-E, η_1 , was assumed to be the power function of both load and time to include the nonlinearity :

$$\eta_1$$
 (t) = bP²t^m (3-19)

where b, l and m again are constants obtained from experiments. Then, the load-slip relation was derived similarly to the procedure described in Section 3.2.1 :

$$Sv = B_{\mu}P^{N_2}t^{N_3}$$
 (3-20)

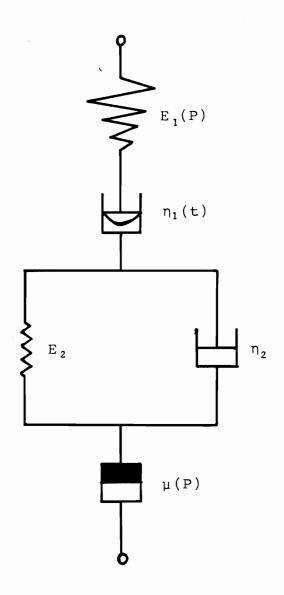


Figure 3.7. Modified Five-element Model.

where B_4 , N_2 and N_3 are constants obtained from experiments.

The constitutive equation of Model M5-E was obtained by incorporating these modifications into equations (3-16) and (3-17), which resulted in :

$$S_{t} = B_{1}P^{N_{1}} + B_{2}P\{1-EXP(-B_{3}t)\} + B_{4}P^{N_{2}}t^{N_{3}} + B_{5}P^{N_{4}}$$
(3-21)

in which B_1 , B_2 , B_3 , B_4 , B_5 , N_1 , N_2 , N_3 and N_4 are parameters obtained from the results of constant load tests. In Model M5-E, nonlinearity of the instantaneous elastic and viscous slip is included. Therefore, equation (3-21) can be employed for all load levels with only one set of parameters.

3.2.3. Viscous-viscoelastic Model 1 (V-VE1)

The three integral representation describing nonlinear viscoelastic behavior of materials cannot be directly applied to represent the time-dependent behavior of the material showing the instantaneous plastic displacement. Therefore, another approach was selected for nailed joints. It was based on equation (3-1) that defines the slip as a sum of four components :

$$S_t = Se + Sp + S_{de} + Sv$$
 (3-22)

which can also be expressed as a sum of two components, recoverable, Sr, and nonrecoverable slip, Sn, where :

$$Sr = Se + S_{de}$$
 (3-23)
 $Sn = Sp + Sv$ (3-24)

In nailed joints, the shapes of the curves of equations (3-23) and (3-24) are similar to the shape of the curve representing viscoelastic creep behavior. Therefore, it was possible to analogize that the threeintegral representation could be used for both Sr and Sn as follows :

$$Sr = F_1 (t) P + F_2(t) P^2 + F_3 (t) P^3$$
(3-25)

$$Sn = F_{4} (t) P + F_{5}(t) P^{2} + F_{6} (t) P^{3}$$
(3-26)

where $F_i(t)$, i=1-6, are kernel functions obtained from the results of constant load tests. Thus, the total slip equaled :

$$S_{t} = Sr + Sn = F_{1} (t) P+F_{2}(t) P^{2}+F_{3} (t) P^{3} + F_{4} (t) P+F_{5}(t) P^{2}+F_{6} (t) P^{3}$$
(3-27)

Equation (3-27) will be called the Viscous-viscoelastic Model 1 (V-VE1).

3.2.4. Viscous-viscoelastic Model 2 (V-VE2)

Model V-VE2 consists of three-integral representation for recoverable slip and power functions for nonrecoverable slip. Thus, equation (3-25) was first applied to define Sr and then, equations (3-15) and (3-20) were used to define Sp and Sv, respectively, to give :

$$S_{t} = F_{1}(t)P+F_{2}(t)P^{2} + F_{3}(t)P^{3}+B_{4}P^{N_{2}}t^{N_{3}}+B_{5}P^{N_{4}}$$
 (3-28)

Equation (3-28) is a combination of Models M5-E and V-VE1 and referred to as the Viscous-viscoelastic Model 2 (V-VE2).

3.2.5. Viscous-viscoelastic Model 3 (V-VE3)

This is another combination of Models M5-E and V-VE1. In Model V-VE3, power functions are used to express the recoverable slip and the nonrecoverable slip is defined by the three-integral representation. Thus, using the first two terms in equation (3-21) to define Sr and equation (3-26) to define Sn gave :

$$S_{t} = B_{1}P^{N_{1}} + B_{2}P\{1 - EXP(-B_{3}t)\} + F_{4}(t)P + F_{5}(t)P^{2} + F_{6}(t)P^{3}$$
(3-29)

This equation is refered to as the Viscous-viscoelastic

Model 3 (V-VE3).

3.2.6. Model applications to varying load

To predict the creep of nailed joints under varying load, further modifications of the models developed were needed. In this investigation, two modification concepts, identified as Approach 1 and Approach 2, which were previously introduced in Section 3.1, were used to further modify the models.

3.2.6.1. Application of Approach 1

N.

Table 3.1 summarizes the models modified according to Approach 1. Each model is divided into two parts, recoverable and nonrecoverable slip, to apply MSP and SHP, respectively, as given in Table 3.1 in which equations (3-30) through (3-33) are defined as follows :

$$Sr = B_1 P^{N_1} + B_2 P\{1 - EXP(-B_3 t)\}$$
(3-30)

$$Sr = F_1 (t) P + F_2 (t) P^2 + F_3 (t) P^3$$
(3-31)

$$Sn = B_{\mu} P^{N_2} t^{N_3} + B_5 P^{N_4}$$
(3-32)

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$$Sn = F_{4} (t) P + F_{5} (t) P^{2} + F_{6} (t) P^{3}$$
(3-33)

Approach 1 could not be applied to Model 5-E which is discrete, because Approach 1 requires the integration of Heaviside function that is not possible for Model 5-E.

	Equation No. for								
Model	Basic part		Modifi	cation part	Loading part				
	Sr	Sn	Ss	ਤ ਤੰ	S 1 ^(a)	S ₂ ^(b)			
M5-E	3-30	3-32	3-34	3-37 or 3-38	3-49	3-51			
V-VE1	3-31	3-33	3-34	3-45 or 3-46	3-50	3-51			
V-VE2	3-31	3-32	3-34	3-37 or 3-38	3-49	3-51			
V-VE3	3-30	3-33	3-34	3-45 or 3-46	3-50	3-51			

Table 3.1. Numbers identifying equations whose sum defines the models developed by Approach 1.

- (a) Slip under the four-step increasing load function in figure 3.4a.
- (b) Slip under the three-step decreasing load function in figure 3.5a.

By applying equation (2-18) of MSP to the recoverable slip in equations (3-30) and (3-31), the recoverable slip under stepwise load, S¹, was obtained :

$$S_{s}^{1} = \sum_{i=0}^{N-1} [Sr(P_{i}, t-t_{i}) - Sr(P_{i-1}, t-t_{i})], t > t_{N-1}$$
(3-34)

where Sr is given by equation (3-30) for Models M5-E and V-VE3 and by equation (3-31) for Models V-VE1 and V-VE2, and N = number of steps.

The nonrecoverable slip under varying load, S^{\pm}, for Models M5-E and V-VE2 was obtained by applying SHP to equation (3-32) as follows. Equation (3-32) was differentiated first :

$$dSn/dt = B_{\mu}N_{3}P^{N_{2}}t^{(N_{3}-1)}$$
(3-35)

and solved for time :

$$t = [(Sn-B_5P^{N_4})/(B_4P^{N_2})]^{1/N_3}$$
(3-36)

Substituting equation (3-36) into equation (3-35) and integrating gave :

$$Ss^{2} = B_{5}P^{N_{4}} + \left[\int_{0}^{t} (B_{4}P^{N_{2}})^{1/N_{3}} dt \right]^{N_{3}}$$
(3-37)

which, under stepwise load function, became :

$$SS^{2} = B SP^{N_{4}} + [(B_{4} PO_{1}^{N_{2}})^{\frac{1}{N_{3}}} t_{1} + (B_{4} P^{N_{2}})^{\frac{1}{N_{3}}} (t_{2} - t_{1}) + \dots + (B_{4} P_{N-1}^{N_{2}})^{\frac{1}{N_{3}}} (t - t_{N-1})]^{N_{3}}, t > t_{N-1}$$
(3-38)

SHP was applied to equation (3-33) to obtain the nonrecoverable slip under varying load, S⁻³³, for Models V-VE1 and V-VE3. This was made possible by having the same power of time in all the kernel functions in equation (3-33) as follows :

$$F_{4}(t) = F_{4}^{a} + F_{4}^{+}t^{m}$$
 (3-39)

$$F_5(t) = F_5 + F_5 t^m$$
 (3-40)

$$F_6(t) = F_8 + F_6^{+}t^{m}$$
 (3-41)

where F_{4}^{o} , F_{5}^{o} , F_{6}^{o} , F_{4}^{+} , F_{5}^{+} , F_{6}^{+} and m are parameters obtained from constant load tests. Finally, equation (3-33) became :

$$Sn = (F_{4}^{\circ} P + F_{5}^{\circ} P^{2} + F_{6}^{\circ} P^{3}) + (F_{4}^{\dagger} P + F_{5}^{\dagger} P^{2} + F_{6}^{\dagger} P^{3}) t^{m}$$

= G₁(P) + G₂(P)t^m (3-42)

which was again solved for time :

$$t = [\{Sn-G_1(P)\}/G_2(P)]^{1/m}$$
(3-43)

and differentiated to give :

$$dSn/dt = mG_2(P)t^{(m-1)}$$
 (3-44)

Substituting equation (3-43) into equation (3-44) and integrating the result yielded :

$$s\hat{s} = G_1(P) + [\int_0^t \{G_2(P)\}^{t/m} dt]^m$$
 (3-45)

which under stepwise load function equaled :

$$S_{2}^{2} = G_{1} (P_{N-1}) + [\{G_{2} (P_{0})\}^{1/m} t_{1} + \{G_{2} (P_{1})\}^{1/m} (t_{2} - t_{1}) + \dots + \{G_{2} (P_{N-1})\}^{1/m} (t - t_{N-1})]^{m}, t > t_{N-1}$$
(3-46)

The total creep slip under varying load, Ss, is the sum of Ss^1 and Ss^2 :

$$Ss = Ss^{1} + Ss^{2}$$
 (3-47)

As discussed in Section 3.1.2, the decreasing load was considered as the compressive load acting in the opposite direction to the existing load, so that MSP could be applied in the same way as for increasing load. Thus, the total slip under decreasing load function was found to be:

$$SS = Smax - SS^{1}$$
(3-48)

where Smax = maximum slip before unloading and S_{s}^{1} is given by equation (3-34). The stiffness reducing factor, Rs, was applied to the last step of the load function as discussed in Section 3.1.3. Next, the total creep slip was evaluated for the four-step increasing load function defined in Figure 3.4a and the three-step decreasing load function defined in Figure 3.5a. The results are shown in Table 3.1 in which equations (3-49), (3-50) and (3-51) are given as follows :

$$Ss = Sr(Po, t-to) + Sr(P_1, t-t_1) - Sr(Po, t-t_1) + Sr(P_2, t-t_2) - Sr(P_1, t-t_2) + Sr(P_3, t-t_3) - Sr(P_2, t-t_3) + G_1(P_3) + [{G_2(Po)}^{1/m}(t_1 - to) + {G_2(P_1)}^{1/m}(t_2 - t_1) + {G_2(P_2)}^{1/m}(t_3 - t_2) + {G_2(P_3)}^{1/m}(t-t_3)]^m, t>t_3$$
(3-50)

$$Ss = Smax-Sr(\Delta P_2, t-t_1) - Sr(\Delta P_2 + \Delta P_1, t-t_2) +$$

$$Sr(\Delta P_2, t-t_2) - Rs[Sr(P_3, t-t_3) -$$

$$Sr(\Delta P_2 + \Delta P_1, t-t_3)], t>t_3$$
(3-51)

3.2.6.2. Application of Approach 2

In Approach 2, MMSP is applied to the total slip. Thus, MSP was applied to the recoverable slip and for the nonrecoverable slip, the procedure described in Section 3.1.2 was used. The general expression for Approach 2 under stepwise load function was derived from equation (3-6) :

$$Ss = \frac{N_{-1}}{\sum_{i=0}^{N-1}} [Sr(P_{i}, t-t_{i}) - Sr(P_{i-1}, t-t_{i}) + Sp(P_{N-1})] + \frac{N_{-1}}{\sum_{i=0}^{N-1}} Sv(P_{i}, t_{i+1} - t_{i}) + Sv(P_{N-1}, t-t_{N-1}), t > t_{N-1} (3-52)$$

To use this equation in the models, the model constitutive equations should be divided into three parts, the recoverable, Sr, instantaneous plastic, Sp, and viscous slip, Sv. The definition for each part of the models is given in table 3.2 in which equations (3-53) through (3-61) are defined as follows :

- $Sr = A_1 + A_2 \{1 EXP(-A_3 t)\}$ (3-53)
- $Sr = B_1 P^{N_1} + B_2 P\{1 EXP(-B_3 t)\}$ (3-54)
- $Sr = F_{1}(t)P+F_{2}(t)P^{2}+F_{3}(t)P^{3}$ (3-55)

$$Sp = A_5$$
 (3-56)

$$Sp = B_5 p^{N_4}$$
 (3-57)

$$Sp = F_{4}^{\circ}P + F_{5}^{\circ}P^{2} + F_{6}^{\circ}P^{3}$$
 (3-58)

$$Sv = A_{\mu}t^{M}$$
(3-59)

$$Sv = B_{\mu}P^{N_2}t^{N_3}$$
 (3-60)

$$Sv = (F_{\mu}^{+}P + F_{5}^{+}P^{2} + F_{5}^{+}P^{3})t^{m}$$
(3-61)

Equation (3-52) of Approach 2 was applied to all five models where Sr, Sp and Sv are defined by equation numbers given in Table 3.2. For decreasing load, MSP was applied

	Equation No. for									
No do 1	Basic part			Modification part	Loading part					
Model	Sr	Sp	Sv	$S\dot{s}^{1} + S\dot{s}^{2}$	$S_1^{(a)}$	S ₂ (b)				
5 - E	3-53	3-56	3-59	3-52	3-62	3-51				
M5-E	3-54	3-57	3-60	3-52	3-62	3-51				
V-VE1	3-55	3-58	3-61	3-52	3-62	3-51				
V-VE2	3-55	3-57	3-60	3-52	3-62	3-51				
V-VE3	3-54	3-58	3-61	3-52	3-62	3-51				

Table 3.2. Numbers identifying equations whose sum defines the models developed by Approach 2.

- (a) Slip under the four-step increasing load function in figure 3.4a.
- (b) Slip under the three-step decreasing load function in figure 3.5a.

to the recoverable slip only. Consequently, equation (3-48) also can be used for deceasing load in which Sr is given in Table 3.2.

For instance, the total creep slip under load functions in Figures 3.4a and 3.5a were evaluated as described above and is presented in Table 3.2 in which equation (3-62) is defined by :

$$Ss = Sr(Po, t-to) + Sr(P_1, t-t_1) - Sr(Po, t-t_1) +$$

$$Sr(P_2, t-t_2) - Sr(P_1, t-t_2) + Sr(P_3, t-t_3) -$$

$$Sr(P_2, t-t_3) + Sv(Po, t_1-to) + Sv(P_1, t_2 - t_1) +$$

$$Sv(P_2, t_3 - t_2) + Sv(P_3, t-t_3) + Sp(P_3), t > t_3$$
(3-62)

3.3. Representation of load functions

In Section 3.2.6, the models were modified under discrete loads. These discrete loads can be represented by a equation using Heaviside step function or Fourier approximation. In this Section, two modification ideas for the models are discussed. Two ideas are the applications of Approach 2 with the load functions represented by Heaviside function and Fourier approximation, respectively.

3.3.1. Stepwise load function represented by Heaviside function

The procedure described in Section 3.1 is a discrete

analysis in which the load function is not defined as a continuous equation. Stepwise varying load is not continuous, but it can be expressed as a continuous equation in terms of Heaviside unit step function (11,18). The application of Heaviside and Dirac delta function to the models developed is presented next.

3.3.1.1. Difinition

Heaviside unit step function (Figure 3.8a) is defined by (11) :

$$H(t-a) = \begin{cases} 1 & \text{if } t > a \\ 1/2 & \text{if } t = a \\ 0 & \text{if } t < a \end{cases}$$
(3-63)

where t = time and a = constant.

Dirac delta function (Figure 3.8b) is defined as follows (11,18) :

$$\delta(t-a) = \begin{cases} \infty & \text{if } t = a \\ 0 & \text{if } t \neq a \end{cases}$$
(3-64)

and

$$\int_{-\infty}^{+\infty} (t-a)dt = \int_{a-}^{a+} (t-a)dt = 1$$
 (3-65)

Dirac delta function actually is the derivative of Heaviside unitstep function :

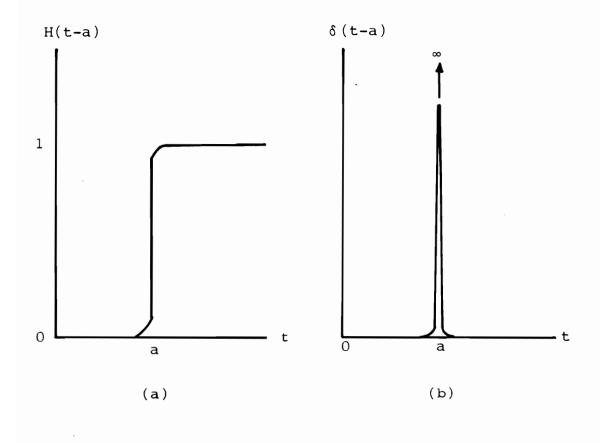


Figure 3.8. Heaviside unit step function (a) and Dirac delta function (b).

$$d{H(t-a)} / dt = \dot{H}(t-a) = \delta(t-a)$$
 (3-66)

The integral of Heaviside and Dirac delta function has following properties :

$$\int_{0}^{\infty} H(t-a) \ \delta(t-b) dt = \begin{cases} 0 & \text{if } a > b \\ 1/2 & \text{if } a = b, \ a \ge 0 \\ 0 & \text{if } a = b, \ a < 0 \\ 1 & \text{if } a < b, \ b \ge 0 \\ 0 & \text{if } a < b, \ b < 0 \end{cases}$$

$$\int_{0}^{\infty} \{H(t-a)\}^{N} \delta(t-a) dt = \begin{cases} 1/(N+1) & \text{if } a > 0 \\ 0 & \text{if } a < 0 \end{cases}$$
(3-68)

where a, b and N are constants.

3.3.1.2. Application

Because it was expected that the results of this procedure were similar to the results of discrete analysis described in Section 3.1, the application was performed for load function 7 only. The resulting models were then modified by Approach 2.

For period A, the load function of Figure 3.9 is :

$$P(t) = PoH(t-to) = \begin{cases} 0 & \text{if } t < to \\ Po & \text{if } t > to \end{cases}$$
(3-69)

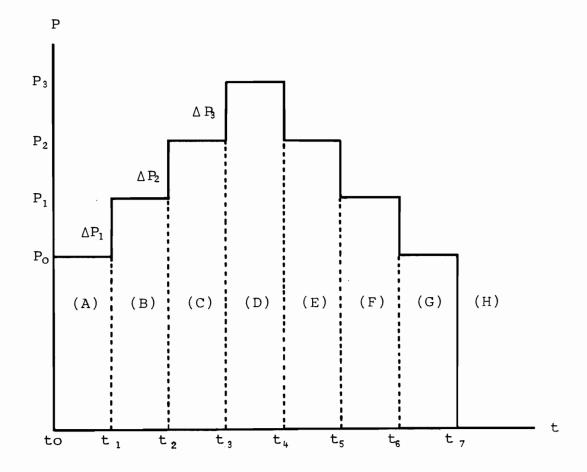


Figure 3.9. Load function 7.

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where Po = load in the first step and to = initial time of the first step. The load function in period A and B of Figure 3.9 equals :

$$P(t) = PoH(t-to) + \Delta P_1 H(t-t_1)$$
(3-70)

where $P_1 = Po + \Delta P_1 = load level during the second$ step. By repeating this procedure for all steps, theoverall load function equals :

$$P(t) = PoH(t-t_{0}) + \Delta P_{1} H(t-t_{1}) + \Delta P_{2} H(t-t_{2}) + \Delta P_{3} H(t-t_{3}) - \Delta P_{3} H(t-t_{4}) - \Delta P_{2} H(t-t_{5}) - \Delta P_{1} H(t-t_{6}) - PoH(t-t_{7}), t > t_{7}$$
(3-71)

where $P_2 = Po + \Delta P_1 + \Delta P_2 = load level during the third step;$ $P_3 = Po + \Delta P_1 + \Delta P_2 + \Delta P_3 = load level during the fourth step;$ and t_i = initial time of each step for i = 0,1,...,7.

Equation (3-52) of MMSP could be rewritten for stepwise increasing-load function to give :

$$Ss = \frac{N_{\pm 1}}{i=0} [Sr(P_{i}, t-t_{i}) - Sr(P_{i-1}, t-t_{i}) + Sp(P_{i}) - Sp(P_{i-1})] + \frac{N_{\pm 2}}{i=0} Sv(P_{i}, t_{i+1} - t_{i}) + Sv(P_{N-1}, t-t_{N-1}), t > t_{N-1}$$
(3-72)

The first four terms in this equation are the expression for MSP and can be represented by an integral formfor a continuous load function. Thus, equation (3-72) became :

$$Ss = \int_{0}^{t} [\partial \{Sr(P(z), t-z)\} / \partial P(z) + d\{Sp(P(z))\} / dP(z)\}] \dot{P}(z) dz + \frac{N-2}{i \sum_{0}^{2} Sv(P_{i}, t_{i+1} - t_{i}) + Sv(P_{N-1}, t-t_{N-1}), t > t_{N-1} (3-73)$$

The integrals in equation (3-73) were solved by direct integration. The remaining terms in equation (3-73) were solved in the same way as discussed in Section 3.2.6. Model 5-E was not employed because it could not be integrated. Therefore, only the remaining models were modified by this procedure.

3.3.1.2.1. Modified five-element Model (M5-E)

Model M5-E was divided into three components represented by equations (3-53), (3-56) and (3-59). By employing equations (3-53) and (3-56), the integrals in equation (3-73) were expressed as :

$$S^{\frac{1}{2}} = \int_{0}^{t} [B_{1}N_{1} \{P(z)\}^{N_{1}-1} + B_{2}\{1-EXP(-B_{3}(t-z))\} + B_{5}N_{4} \{P(z)\}^{N_{4}-1}] \dot{P}(z)dz$$

$$= B_{1}N_{1} \int_{0}^{t} \{P(z)\}^{N_{1}-1} \dot{P}(z)dz + B_{2} \int_{0}^{t} \dot{P}(z)dz - B_{2} \int_{0}^{t} EXP(-B_{3}(t-z))\dot{P}(z)dz + B_{5}N_{4} \int_{0}^{t} \{P(z)\}^{N_{4}-1} \dot{P}(z)dz \qquad (3-74)$$

in which load function, P(t), was given by equation (3-69) and its derivative, $\dot{P}(t)$, equalled :

$$\dot{\mathbf{P}}(\mathbf{z}) = \mathbf{Po} \,\delta(\mathbf{t}) + \Delta \mathbf{P}_1 \,\delta(\mathbf{t} - \mathbf{t}_1) + \Delta \mathbf{P}_2 \,\delta(\mathbf{t} - \mathbf{t}_2) + \\ \Delta \mathbf{P}_3 \,\delta(\mathbf{t} - \mathbf{t}_3) - \Delta \mathbf{P}_3 \,\delta(\mathbf{t} - \mathbf{t}_4) - \Delta \mathbf{P}_2 \,\delta(\mathbf{t} - \mathbf{t}_5) - \\ \Delta \mathbf{P}_1 \,\delta(\mathbf{t} - \mathbf{t}_6) - \mathbf{Po} \,\delta(\mathbf{t} - \mathbf{t}_7)$$
(3-75)

Next, equation (3-74) was solved for each step of load function 7 by substituting equations (3-71) and (3-75)into equation (3-74), and by applying equations (3-65), (3-66) and (3-67).

The second and third terms in equation (3-73) were expressed by applying equation (3-59):

$$Ss^{2} = \sum_{i=0}^{N-2} B_{4} P_{i}^{N_{2}} (t_{i+1} - t_{i})^{N_{3}} + B_{4} P_{N-1}^{N_{2}} (t - t_{N-1})^{N_{3}}$$
(3-76)

Creep slip during the loading part of load function 7 is a combination of equations (3-74) and (3-76). For example, the creep slip in period D of Figure 3.9 was found to be :

$$Ss = B_{1} [Po^{N_{1}} + N_{1} \{ (P_{1})^{N_{1}-1} \Delta P_{1} + (P_{2})^{N_{1}-1} \Delta P_{2} + (P_{3})^{N_{1}-1} \Delta P_{3} \}] + B_{2} [P_{3} - \{PoEXP(-B_{3} (t-t_{0})) + \Delta P_{1}EXP(-B_{3} (t-t_{1})) + \Delta P_{2}EXP(-B_{3} (t-t_{2})) + \Delta P_{3}EXP(-B_{3} (t-t_{3}))] + B_{5} [Po^{N_{4}} + N_{4} \{ (P_{1})^{N_{4}-1} \Delta P_{1} + (P_{2})^{N_{4}-1} \Delta P_{2} + (P_{3})^{N_{4}-1} \Delta P_{3} \}] + 4 [Po^{N_{2}} (t_{1}-to)^{N_{3}} + P_{1}^{N_{2}} (t_{2}-t_{1})^{N_{3}} + P_{2}^{N_{2}} (t_{3}-t_{2})^{N_{3}} + P_{3}^{N_{2}} (t-t_{3})^{N_{3}}], t_{3} < t < t_{4}$$
(3-77)

The solution for the unloading portion was based on the basic assumption that the decreasing load could be considered as a compressive force applied in the opposite direction to the existing load. The compressive-force function, Pc(t), and its derivative, $\dot{P}c(t)$, were derived as :

$$Pc(t) = \Delta P_3 H(t-t_4) + \Delta P_2 H(t-t_5) + \Delta P_1 H(t-t_6) +$$

$$PoH(t-t_7) \qquad (3-78)$$

$$\dot{\mathbf{P}}_{\mathbf{C}}(\mathbf{t}) = \Delta \mathbf{P}_3 \quad \delta(\mathbf{t}-\mathbf{t}_4) + \Delta \mathbf{P}_2 \quad \delta(\mathbf{t}-\mathbf{t}_5) + \Delta \mathbf{P}_1 \quad \delta(\mathbf{t}-\mathbf{t}_6) + \mathbf{P}_0 \quad \delta(\mathbf{t}-\mathbf{t}_7)$$

$$(3-79)$$

For unloading, MSP can be applied, because only recoverable slip is related to unloading steps. Thus :

$$Ss = Smax-Rs[\int_{t_{4}}^{t} \frac{\partial \{Sr(P_{c}(z), t-z)\}}{\partial P_{c}(z)dz]}, t > t_{4} (3-80)$$

in which Smax = maximum slip at time t4 before unloading; Rs = stiffness reducing factor; and Sr = recoverable slip defined by equation (3-53). As suggested in Section 3.1.3, stiffness reducing factor, Rs, was applied only at the end of the last step. The integration in equation (3-80) can be solved either directly or numerically. For example, the creep slip for period H of Figure 3.9 was found directly to be :

Ss = Smax - Rs[B₁ [
$$\triangle P_3^{N_1} + N_1$$
 { ($\triangle P_3 + \triangle P_2$)^{N₁-1} $\triangle P_2 +$
($\triangle P_3 + \triangle P_2 + \triangle P_1$)^{N₁-1} $\triangle P_1 + (P_3)^{N_1-1}Po$]+B₂ [P₃-

{
$$\triangle P_3 EXP(-B_3 (t-t_4)) + \triangle P_2 EXP(-B_3 (t-t_5)) +$$

 $\triangle P_1 EXP(-B_3 (t-t_6)) + POEXP(-B_3 (t-t_7)) \}] (3-81)$

3.3.1.2.2. Viscous-viscoelastic Model 1 (V-VE1)

Model V-VE1 is composed of three parts which are represented by equations (3-54), (3-57) and (3-60). MMSP and the corresponding equation (3-73) were applied to Model V-VE1. The integrals in equation (3-73) were written as :

$$SS^{1} = \int_{0}^{t} F_{1} (t-z) \dot{P}(z) dz + 2 \int_{0}^{t} F_{2} (t-z) P(z) \dot{P}(z) dz + 3 \int_{0}^{t} F_{3} (t-z) \{P(z)\}^{2} \dot{P}(z) dz + \int_{0}^{t} \{F_{4}^{\circ} + 2F_{5}^{\circ} P(z) + 3F_{6}^{\circ} (P(z))^{2} \} \dot{P}(z) dz \qquad (3-82)$$

The load function, P(t), and its derivative, $\dot{P}(t)$, were defined by equation (3-71) and (3-75), respectively. Equation (3-82) can be integrated either directly or numerically. The rest terms in equation (3-73) were determined as :

$$SS^{2} = \sum_{i=0}^{N-2} (F_{4}^{+} P_{i}^{+} + F_{5}^{+} P_{i}^{2}^{+} + F_{6}^{+} P_{i}^{3}^{-}) (t_{i+1}^{-} - t_{i}^{-})^{m} + (F_{4}^{+} P_{N-1}^{-} + F_{5}^{+} P_{N-1}^{2} + F_{6}^{+} P_{N-1}^{3}) (t_{i+1}^{-} - t_{i}^{-})^{m}, t > t_{N-1} (3-83)$$

Adding equation (3-82) to equation (3-83) gave creep slip under load function 7 for Model V-VE1.

Next, Model V-VE1 and equations (3-82) and (3-83)

were used to evaluate the slip for period D of Figure 3.9. The resulting equation was found to be :

$$Ss = F_{1}(t)Po+F_{1}(t-t_{1}) \Delta P_{1}+F_{1}(t-t_{2}) \Delta P_{2}+F_{1}(t-t_{3}) \Delta P_{3} + F_{2}(t)Po^{2}+2F_{2}(t-t_{1})P_{1} \Delta P_{1}+2F_{2}(t-t_{2})P_{2} \Delta P_{2} + 2F_{2}(t-t_{3})P_{3} \Delta P_{3}+F_{3}(t)Po^{3}+3F_{3}(t-t_{2})P_{1}^{2} \Delta P_{1} + 3F_{1}(t-t_{2})P_{2}^{2} \Delta P_{2}+3F_{3}(t-t_{3})P_{3}^{2} \Delta P_{3}+F_{4}^{\circ}P_{3} + 2F_{5}^{\circ}(Po^{2}+P_{1} \Delta P_{1}+P_{2} \Delta P_{2}+P_{3} \Delta P_{3})+3F_{6}^{\circ}(Po^{3}+P_{1}^{2} \Delta P_{1}+P_{2}^{2} \Delta P_{2}+P_{3}^{2} \Delta P_{3})+(F_{4}^{+}Po+F_{5}^{+}Po^{2}+F_{6}^{+}Po^{3})t^{m}+(F_{4}^{+}P_{1}+F_{5}^{+}P_{1}^{2}+F_{6}^{+}P_{1}^{3})(t_{2}-t_{1})^{m}+(F_{4}^{+}P_{2}+F_{5}^{+}P_{2}^{2}+F_{6}^{+}P_{3}^{3})(t-t_{3})^{m}, t>t_{3}$$

$$(3-84)$$

For unloading, decreasing loads were again regarded as a compressive forces. The bases for determining the slip under unloading function were the force function and its derivative given in equations (3-78) and (3-79). For MSP, the resulting equation was identical to equation (3-80) in which Sr was given by equation (3-54).

The same development was also applied to period H in Figure 3.9. The total slip was found to be :

$$Ss = Smax - Rs[F_{1}(t-t_{4}) \Delta P_{3} + F_{1}(t-t_{5}) \Delta P_{2} + F_{1}(t-t_{6}) \Delta P_{1} + F_{1}(t-t_{7})Po+F_{2}(t-t_{4}) \Delta P_{3}^{2} + 2F_{2}(t-t_{5}) (\Delta P_{3} + \Delta P_{2}) \Delta P_{2} + 2F_{2}(t-t_{6}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1}) \Delta P_{1} + 2F_{2}(t-t_{7})P_{3} Po+ F_{3}(t-t_{4}) \Delta P_{3}^{3} + 3F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2})^{2} \Delta P_{2} + 3F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{1})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{2})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{2})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{2})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{2})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2} + \Delta P_{2})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2})^{2} \Delta P_{1} + 2F_{3}(t-t_{5}) (\Delta P_{3} + \Delta P_{2})^{2} \Delta P_{2} + 2F_{3}(t-t_{5})^{2} \Delta P_{2} + 2F_{3}(t-t_{5})^{2} \Delta P_{2} + 2F_{3}(t-t_{5})^{2} \Delta P_{3} + 2F_{3}(t-t_{5})^{2} \Delta P_{3} + 2F_{3}(t-t_{5})^{2} \Delta P_{3} + 2F_{3}(t-t_{$$

$$3F_3(t-t_7)P_3^2$$
 Po], t>t₇ (3-85)

Models V-VE2 and V-VE3 are combinations of Models M5-E and V-VE1. The corresponding total slip could be easily obtained from equations (3-74), (3-76), (3-82) and (3-83) following the procedure of Section 3.2.6.2, but the development was not carried out in this dissertation.

3.3.2. Continuous load function represented by Fourier series

Next, a procedure was developed for loading represented by continuous functions. Such a procedure offers advantages when the load is difficult to be represented by a simple equation that is easy to integrate. The development was based on a Fourier series approximation, because such an approximation can easily be used for varying-load functions.

3.3.2.1. Definition of Fourier approximation

Fourier series approximation of load function P(t) is a trigonometric series defined by (18) :

$$Fp(t) = (1/Tp) \int_{0}^{Tp} P(t) dt +$$

$$\sum_{i=1}^{\infty} [(2/Tp) \{ \int_{0}^{Tp} P(t) \cos(i\omega t) dt \} \cos(i\omega t) +$$

$$(2/Tp) \{ \int_{0}^{Tp} P(t) \sin(i\omega t) dt \} \sin(i\omega t)]$$

$$= Ao + \sum_{i=1}^{\infty} [Ai \cos(i\omega t) + Bi \sin(i\omega t)] \qquad (3-86)$$

where P(t) = load function; Tp = period; and $\omega = 2\pi / Tp$.

To illustrate such a procedure, Fourier series approximation was applied to load function 7 of figure 3.9. First, load function 7 was divided into two parts (Figure 3.10), part a consisting of a constant load of 60 lb and part b consisting of stepwise varying load which was load function 7 minus 60 lb.

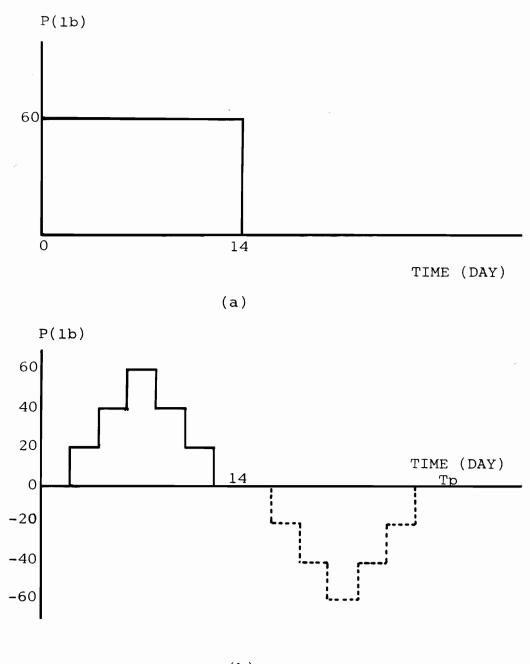
Part b which shows odd periodic extension by dot line in Figure 3.10 was approximated by a Fourier sine series (18). In this case, equation(3-86) was valid, in which :

Ao = 60
Ai = 0
Bi =
$$(2/Tp) \int_{0}^{Tp} P(t) \sin(i\omega t) dt$$

= $(2/Tp) \int_{0}^{Tp} P(t) \sin(2i\pi/Tp) t dt$ (3-87)

For load function 7, t7 is 14 days or 20,160 minutes and Tp is 28 days or 40,320 minutes, for which equation (3-87) was found to be :

$$Bi = -(1/i \pi) [P_1 \{\cos(2i \pi/7) + \cos(6i \pi/7) + \cos(8i \pi/7) + \cos(8i \pi/7) + \cos(12i \pi/7) - \cos(12i \pi/7) - \cos(5i \pi/7) + \cos(5i \pi/7) - \cos(13i \pi/7) \} + P_2 \{\cos(3i \pi/7) + \cos(5i \pi/7) + \cos(9i \pi/7) + \cos(11i \pi/7) - \cos(2i \pi/7) - \cos(4i \pi/7) - \cos(10i \pi/7) - \cos(12i \pi/7) \} + P_3 \{\cos(4i \pi/7) + \cos(10i \pi/7) - \cos(3i \pi/7) - \cos(11i \pi/7) \}] (3-88)$$



(b)

Figure 3.10. Division and odd periodic extension of load function 7 for Fourier approximation.

After having defined all the coefficients, load function 7 was fully defined :

$$Fp(t) = Ao + \sum_{i=1}^{\infty} Bi \sin(i\omega t)$$
 (3-89)

In programming, "i" was selected to be 20, because preliminary calculations showed that 20 assures convergence at sufficient accuracy.

3.3.2.2. Application of Fourier approximation

Next, Fourier sine series was applied to the models developed and the models were modified by Approach 2. For varying load, MMSP was represented by equation (3-73) in which the derivative of the load function equalled :

$$\dot{\mathbf{F}}\mathbf{p}(\mathbf{t}) = \sum_{i=1}^{\infty} i\omega \mathbf{B}_{i} \cos(i\omega \mathbf{t})$$
(3-90)

Model 5-E could not be used in this procedure because it is discrete.

Model M5-E can be divided into three parts that are associated with equations (3-53), (3-56) and (3-59). The general expression for MMSP under continuous load function was found to be equation (3-73), in which the integrals equalled :

$$Ss^{1} = B_{1} N_{1} \int_{0}^{t} \{Fp(z)\}^{N_{1}-1} \hat{F}p(z) dz + B_{2}[\int_{0}^{t} \{1-EXP(-B_{3}(t-z))\} \hat{F}p(z) dz] + B_{5} N_{4} \int_{0}^{t} \{Fp(z)\}^{N_{4}-1} \hat{F}p(z) dz$$
(3-91)

The remaining terms in equation (3-73) were expressed as :

$$Ss^{2} = \sum_{i=1}^{N_{2}^{2}} B_{4}Fp^{N_{2}} (t_{i+1}-t_{i})^{N_{3}} + B_{4}Fp_{N-1}^{N_{2}} (t-t_{N-1})^{N_{3}}, t > t_{N-1}$$
(3-92)

For continuous load function, the viscous slip was obtained by equation (3-92). The total slip under load function 7 was expressed by a Fourier series as a combination of equations (3-91) and (3-92). When a numerical method is used to solve equation (3-91), equations (3-91) and (3-92) should both have the same step size.

Next, the unloading part of the function was expressed by :

$$Fc(t) = Fmax - Fp(t), t > tmax$$
(3-93)

where Fc(t) = unloading part of the equation; Fmax = maximum load at time tmax; and Fp(t) is defined by equation (3-89). The derivative of equation (3-93) equalled :

$$\dot{F}_{c}(t) = -\dot{F}_{p}(t), t > tmax \qquad (3-94)$$

where Fp(t) was defined by equation (3-90).

Next, MSP was applied to the unloading part of the function in which the recoverable slip was defined as :

$$Ss = Smax - Rs[\int_{tmax}^{t} \{ \partial Sr(Fc(z), t-z) / \partial Fc(z) \} \dot{F}c(z) dz]$$

= Smax - Rs[B₁N₁ $\int_{tmax}^{t} \{Fc(z)\}^{N_1-1} \dot{F}c(z) dz +$
B [$\int_{tmax}^{t} \{1-EXP(-B_3(t-z))\} \dot{F}c(z) dz]$], t>tmax (3-95)

where Smax = maximum slip at time tmax and the other symbols were previously defined. The integration of equation (3-95) can be carried out either by direct or numerical method. As before, the stiffness reducing factor, Rs, was applied to the last step only.

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3.3.2.2.2. Viscous-viscoelastic Model 1 (V-VE1)
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Model V-VE1 is divided into three parts that were defined by equations (3-54), (3-57) and (3-60). Again, the slip under varying load was defined by equation (3-73) of MMSP in which the integrals equalled :

$$SS^{1} = \int_{0}^{t} F_{1} (t-z) \dot{F}p(z) dz + 2 \int_{0}^{t} F_{2} (t-z) Fp(z) \dot{F}p(z) dz +$$

$$3 \int_{0}^{t} F_{3}(t-z) \{Fp(z)\}^{2} \dot{F}p(z) dz + \int_{0}^{t} \{F_{4}^{\circ} + 2F_{5}^{\circ} Fp(z) +$$

$$3F_{5}^{\circ} \{Fp(z)\}^{2} \} \dot{F}p(z) dz \qquad (3-96)$$

where Fp(t) and $\dot{F}p(t)$ were given by equations(3-89) and (3-90), respectively. The remaining terms in equation (3-73) were found to be :

$$Ss^{2} = \sum_{i=0}^{N-2} (F_{4} + F_{p_{i}} + F_{5} + F_{p_{i}}^{2} + F_{6} + F_{p_{i}}^{3}) (t_{i+1} - t_{i})^{m} + (F_{4} + F_{p_{N-1}} + F_{5} + F_{p_{N-1}}^{2} + F_{6} + F_{p_{N-1}}^{3}) (t_{-} - t_{N-1})^{m}, \\ t > t_{N-1}$$
(3-97)

Equation (3-97) is a discrete approximation of a continuous load function. The sum of equations (3-96) and (3-97) gives the total slip under load function 7 approximated by Fourier series. When equation (3-95) is evaluated by a numerical integration, it is convenient to use the same step size for both, equations (3-96) and (3-97).

For unloading, the force function and its derivative were expressed by equations (3-93) and (3-94), respectively. When MSP was applied, the total slip became :

$$Ss = Smax - Rs[\int_{tmax}^{t} \{ \partial Sr(Fc(z), t-z) / \partial Fc(z) \} \dot{F}c(z) dz$$

= Smax - Rs[$\int_{tmax}^{t} F_1(t-z) \dot{F}c(z) dz$ +
 $2 \int_{tmax}^{t} F_2(t-z) Fc(z) \dot{F}c(z) dz$ +
 $3 \int_{tmax}^{t} F_3(t-z) \{Fc(z)\}^2 \dot{F}c(z) dz]$ (3-98)

Again, equation (3-98) can be evaluated by direct or numerical integration. The stiffness reducing factor, Rs,

was applied to the last step only.

Models V-VE2 and V-VE3 can be easily obtained from equations (3-91), (3-92) and (3-95) for Model M5-E and from equations (3-96), (3-97) and (3-98) for Model V-VE1. However, these derivations were not presentated in this dissertation.

VI. EXPERIMENTAL PROCEDURE

To formulate and verify the models developed, creep experimental values are required for nailed joints under constant and varying loads. Therefore, a testing study was carried out, which included the most common joint type in building construction. The joint type selected was nailed joint between framing member and plywood sheathing.

4.1. Joint specimens

Material selection and joint specimen construction are introduced in this section.

4.1.1. Material selection

Douglas-fir lumber of nominal size 2-by 4-inches was selected as representative framing member. Initially, 350 12-inch long pieces of lumber were cut from studs left over from another project at the Forest Products Laboratory, Oregon State University. The pieces containing too much pith, checks, splits or knots were excluded because these defects could produce joints that were not typical of the overall statistical population.

The sheathing material consisted of 3/8-inch thick Douglas-fir plywood of sheathing grade, because it represented commonly used sheathing materials in construction stock. It was selected from 4-by 8-foot panels bought in a local lumber yard. In the laboratory, sections of 4-by 15-inches were cut from the panels. The sections having weak adhesion, checks, splits and knots were excluded, because they would not produce typical joints.

Next, the materials were stored in a conditioning room at 12-percent equillibrium moisture content (EMC) with forced air circulation until they reached EMC. Their moisture content were checked periodically by an electric moisture meter to monitor their equilibration.

Six penny galvanized box nails were selected to make joints, because this type is commonly used in wall construction. The length and diameter of the nail are 2.0 in. and 0.98 in., respectively. The nails having a crushed head, shank or extremely rough surface were excluded.

4.1.2. Evaluation of specific gravity

Specific gravity is often visualized as an easily measured property of wood, which is closely related to other mechanical properties. Therefore, it was evaluated for all 350 pieces of lumber selected.

In this investigation, specific gravity was based on the volume at 12 % moisture content and evaluated by :

$$SG = WW / (1.12 V)$$
 (4-1)

where Ww = weight of wood and V = volume of wood at 12percent moisture content. In equation (4-2), moisture contents of all pieces of lumber were assumed to be 12percent and the volumes were obtained by measuring the thickness, width and length of each piece.

4.1.3. Evaluation of elastic bearing constant

Elastic bearing constant is related to the stiffness of nailed joints. It is defined as the elastic spring modulus for the wood under the nail and expressed in psi per inch of wood deformation. In nailed joint, the wood can be visualized as a foundation and the nail as a beam. Under small deflection, the nail and the wood behave elastically. Therefore, the elastic spring constant of wood is important in indicating the stiffness of nailed joints.

The elastic bearing constant depends on the testing method and the equation of calculating it. In this investigation, elastic bearing constants were evaluated by the modulus of subgrade reaction of soil foundation (2), because it was suitable for nailed joints and relatively easy to test. In this method, the elastic bearing constant is defined by :

$$Ko = q / y (lb/in.^3)$$
 (4-2)

where Ko = elastic bearing constant; q = stress in wood (lb/in²); and y = deflection under nail (in.). For nailed joints, q equals :

$$q = P / A \tag{4-3}$$

where :

$$\mathbf{A} = \mathbf{D} \mathbf{L} \tag{4-4}$$

in which P = load (lb); A = contact area between nail and wood (in.); D = diameter of nail (in.); and L = contact length between nail and wood (in.). The contact length, L, was obtained by :

$$L = Ln - Lp \tag{4-5}$$

where Ln = length of nail (in.) and Lp = thickness of plywood (in.). Substituting equations (4-3), (4-4) and (4-5) into equation (4-2) resulted in :

$$Ko = P / [yD(Ln-Lp)] (lb/in3)$$
 (4-6)

in which P, y, D, Ln and Lp are variables measured in experiments.

The testing arrangement for the elastic bearing constant is shown in Figure 4.1. The nails were placed on the cross section of the lumber pieces and compression load parallel to the grain was applied. A steel loading

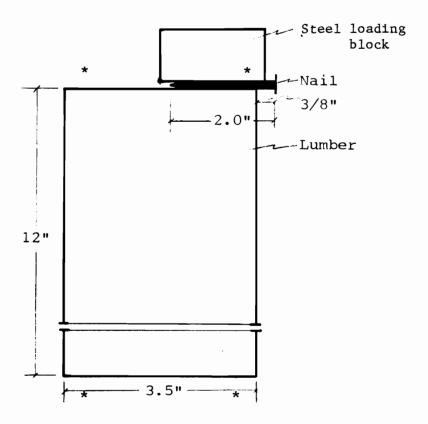


Figure 4.1. Testing arrangement for evaluating elastic bearing constant.

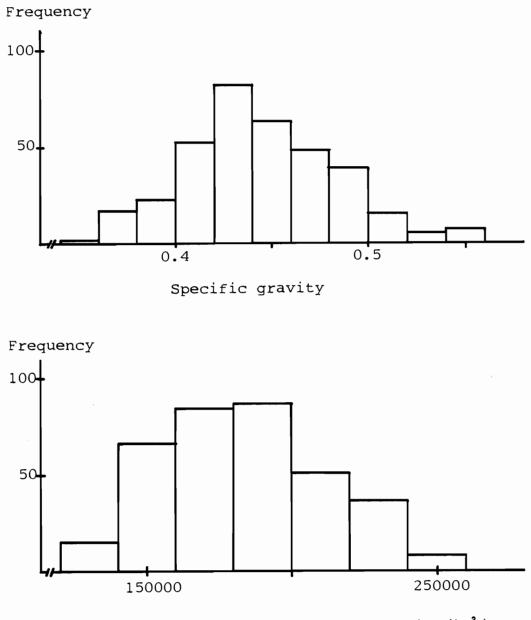
block was used to apply load to the nail. The contact length of the nail equalled the actual penetrating length, that is the length of nail minus the thickness of plywood. In Figure 4.1, the tests were performed on four corners (* marks) of each section of lumber and the average of four was taken as the elastic bearing constant of that section.

4.1.4. Specimen construction

The specific gravity and elastic bearing constant were the basis for selecting 25 lumber sections used for nailed joints specimens. The specific gravities and elastic bearing constants of 350 lumber sections are shown in Figure 4.2. These histograms were assumed to represent the Douglas-fir lumber. The average was 0.45 for specific gravity and 183,929 lb/in.³ for elastic bearing constant. The 25-specimen lumber sample was selected from 350 sections, so that the histograms of this sample matched those in Figure 4.2. The average for the 25-specimen sample was 0.45 for specific gravity and 185,693 lb/in³. for elastic bearing constant. Furthermore, the selected lumber specimens were checked visually to assure that they were free of defects.

Nailed joints were constructed by hammer-driving until the stud and plywood made firm contact and the nail head was flat with the surface of the plywood.

Each of 25 lumber and plywood specimens was used



Elastic bearing constant (lb/in³)

Figure 4.2. Distribution of specific gravities and elastic bearing constants of Douglas-fir lumber specimens.

eight times, once for each of eight load functions, to reduce the variation in material properties. After testing each nailed joint, the nail was carefully pulled out, and a new joint was assembled by hammer-driving a new nail into the plywood and lumber at the point one inch apart from the previous nailing point. The framing members were used four times on each narrow edge (Figure 4.3) and the plywood sections were used eight times (Figure 4.3). The reuse of lumber and plywood to make new specimen was expected to minimize the variation in creep readings for the eight load functions due to material variability.

4.2. Testing arrangement

The nailed-joint specimens (Figure 4.4) were the same as those developed in an earlier investigation (17,22). The type selected introduces almost pure shear and has considerably less moment than the joint in the ASTM testing arrangement (17,22). Steel straps were used to connect specimen to the frame supporting at the top and to the weight at the bottom in Figure 4.4.

In constant-load tests, dial gauges were used to monitor the creep slip instead of linear variable differential transformers (LVDTs). To compensate the difference in measuring instruments between constant-and varying-load tests, the measurements of the dial gauges



Figure 4.3. Douglas-fir lumber and plywood specimens used in testing.



Figure 4.4. Testing arrangement for varying-load tests; A = LVDT, B = string to LVDT core, C = stud and D = plywood. and LVDTs were calibrated by one vernier caliper. The readings were taken visually at intervals of 1, 2.5, 5, 10, 20, 30, 60, 120, 240, 480, 1440 minutes and daily afterwards.

In varying-load tests, LVDTs were used. They were connected to 14-channel analogue input box (A in Figure 4.5) which had eight AC-LVDT and six DC-LVDT channels. The signals were then sent to a 21X-micrologger (B in Figure 4.5) which was used to collect data at a preprogrammed scanning rate of one second for the first two hours and 30 minutes afterwards. Finally, the data were stored on cassette tapes by a tape recorder (C in Figure 4.5) which was connected to the 21X-micrologger. The data on the cassette tapes were recovered by an IBM microcomputer for the analysis.

The loads were applied and removed by a small lifting device that was made specifically for this testing (Figure 4.6). The device minimized the jerk and vibration which could develop during the loading and unloading processes.

4.3. Testing procedure

All the tests were performed in the conditioning room at 12-percent EMC, because the behavior of nailed joints under load is affected by the change in EMC. During the tests, checking the dry and wet bulb temperatures showed that EMC varied between 11.5 % and 12.5 %, which is within

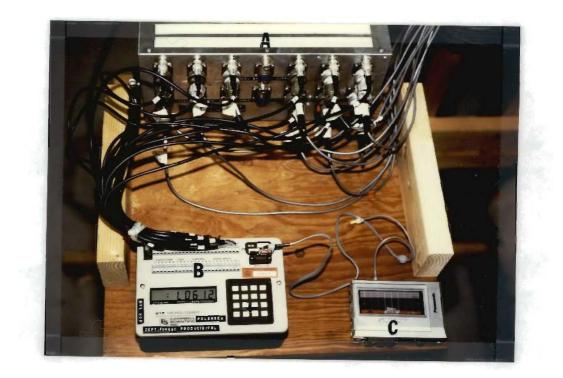


Figure 4.5. The arrangement for data acquisition system; A = 16 channel analogue input box, B = 21 Xmicrologger, C = cassette tape recorder.



Figure 4.6. Loading and unloading device.

the acceptable limits.

4.3.1. Load functions

The eight load functions employed in the tests are defined in Figures 4.7 (constant-load functions) and 4.8 (varying-load functions). The load range between 60 and 120 lb was selected, because this range represents the load which can be applied to the wall in the actual environment. The varying-load functions were selected to include loading, unloading, increasing, decreasing and cyclic loading, because these are basic patterns of the actual loads developed by several natural occurances.

4.3.2. Description of constant-load tests

In constant-load tests, sets of two or three specimens were loaded in series to reduce the time and space required for the testing. Therefore, the specimens at the top had six and those at the middle had three more pounds of weight than those at the bottom. The test results were corrected for this over-loading by using piecewise-linear interpolation to bring the slip to the target load. Piecewise-linear interpolation was sufficient because of the small corrections involved.

The loading schedule in this tests is given in Table 4.1. The specimens were tested in 10 series sets, among

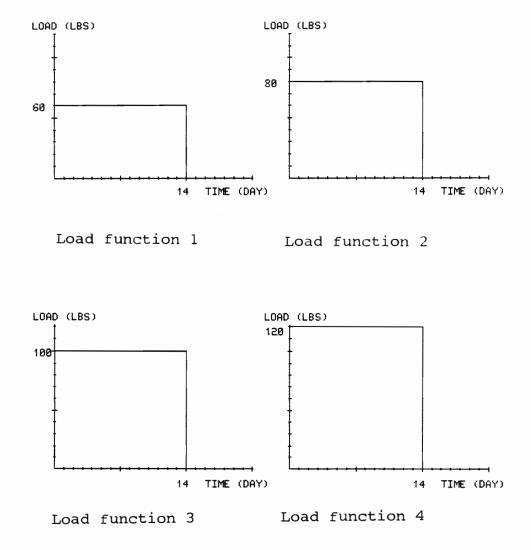


Figure 4.7. Constant-load functions used in testing to develop data needed to form the models.

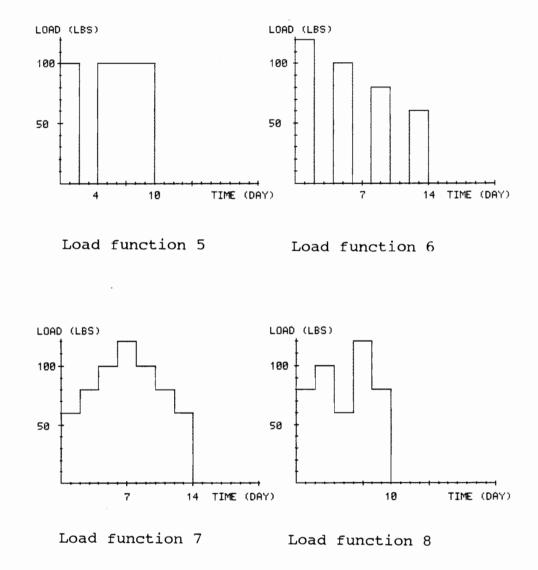


Figure 4.8. Varying-load functions used in testing to develop data for model verification.

which the first five sets had three specimens and the remaining five had two specimens. The specimens in two or three sets were simultaneously subjected to the same load function. The tests were run with all four load functions simultaneously to reduce the potential variation in the room conditioning.

After the loads were removed, the recovery was measured for one week. Among 25 specimens, five were subjected to more dynamic load than others due to the unexpected mistakes during the loading processes and showed much more instantaneous slip than the specimens which had only static loads. Therefore, the results of those five were excluded from the analysis, and varyingload tests were also conducted with the other 20 specimens.

4.3.3. Description of varying-load tests

Ten specimens were tested with only one specimen in each set. In Table 4.1, test No. 5, 6, 7 and 8 were scheduled for the first ten specimens and were the same as test No. 9, 10, 11 and 12 which were scheduled for the remaining ten specimens. Load functions 5 and 8 took 10 days and load functions 6 and 7 took 14 days to complete and the recovery was measured for one week afterwards.

The loading or unloading was accomplished with the help of a lifting device to assist with the gradual

Table 4.1. The assignment of load-function numbers (defined in Figures 4.7 and 4.8) in the testing schedule.

Test	Test										sed 10
type	No.	1	2	3	4	5	6	7	8	9	10
Constant	1	1	1	1	2	2	3	3	3	4	4
load	2	- 3	3	3	4	4	1	1	1	2	2
test	3	2	2	2	1	1	4	4	4	3	3
	4	4	4	4	3	3	2	2	2	1	1
Varying load test	5	5	5	5	6	6	7	7	7	8	8
	6	7	7	7	8	8	5	5	5	6	6
	7	6	6	6	5	5	8	8	8	7	7
	8	8	8	8	7	7	6	6	6	5	5
	9	5	5	5	6	6	7	7	7	8	8
	10	7	7	7	8	8	5	5	5	6	6
	11	6	6	6	5	` 5	8	8	8	7	7
	12	8	8	8	7	7	6	6	6	5	5
	L	L									

application and minimize the jerk. The partial loading and partial unloading were performed manually. Preliminary tests showed that the device reduced the loading and unloading jerk.

4.3.4. Procedure of comparing experimental and theoretical data

To compare the theoretical predictions with the experimental data, the square of the correlation coefficient, R^2 , and the sum of the squares of errors, SSE, were calculated, which are defined as follows :

$$R^{2} = \frac{\left[\sum_{i=1}^{N} (S-\overline{S}) (S^{*}-\overline{S}^{*}) \right]^{2}}{\left[\sum_{i=1}^{N} (S-\overline{S})^{2} \sum_{i=1}^{N} (S^{*}-\overline{S}^{*})^{2} \right]}$$
(4-7)

SSE =
$$\sum_{i=1}^{N} (S-S^{*})^{2}$$
 (4-8)

where S = experimental data; \overline{S} = mean of experimental data; S * = predicted data; \overline{S} * = mean of predicted data; and N = number of observations.

V. RESULTS AND DISCUSSION

To develop and verify the workable models of the theoretical concepts introduced in Chapter 3, experimental observations for creep of nailed joints under constant and varying loads were needed. This chapter describes the formulation and verification of the workable models for nailed joints tested in Chapter 4. The models can predict the behavior of nailed plywood-to-wood joints under any type of loading function.

5.1. Experimental results

This section contains the results of testing described in Chapter 4.

5.1.1. Creep under constant load

The raw data obtained from the tests under constant loads of Chapter 4, which are the mean values of 20 specimens, are given in Appendix B.1. The graphs that are based on the data in Appendix B and presented in this chapter were drawn by an IBM PC/XT microcomputer and TEKTRONICS graphic plotter using the software developed in another FRL study in the Forest Products Department.

The mean time-slip traces from these data are shown in Figure 5.1. Under 60-lb load, the delayed slip and

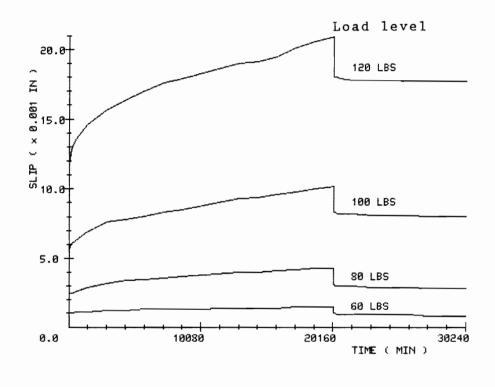


Figure 5.1. Creep of nailed joints under constant-load functions.

	Creep slip (0.001 in.) under constant load (lb)					
Time(min.)	60	80	100	120		
						
0.0	0.5	1.1	1.7	2.7		
1.0	0.5	1.2	1.7	2.7		
2.5	0.5	1.2	1.8	2.8		
5.0	0.5	1.2	1.8	2.8		
10.0	0.5	1.2	1.9	2.9		
20.0	0.5	1.2	1.9	2.9		
30.0	0.5	1.2	1.9	2.9		
60.0	0.5	1.2	1.9	2.9		
120.0	0.5	1.3	1.9	2.9		
240.0	0.6	1.3	2.0	2.9		
480.0	0.6	1.3	2.0	3.0		
1440.0	0.6	1.3	2.0	3.1		
2880.0	0.6	1.4	2.1	3.1		
4320.0	0.6	1.4	2.1	3.1		
5760.0	0.6	1.4	2.1	3.2		
7200.0	0.7	1.5	2.2	3.2		
10080.0	0.7	1.5	2.2	3.2		
12960.0	0.7	1.5	2.2	3.2		
15840.0	0.7	1.6	2.2	3.3		
20160.0	0.7	1.6	2.2	3.3		

Table 5.1.	Recoverable	slip	of	nailed	joints	under
	constant-load	functi	ons	•		

	Creep slip (0.001 in.) under constant load (lb)					
Time(Min.)	60	80	100	120		
0.0	0.4	1.0	2.9	6.4		
1.0	0.5	1.0	3.2	7.4		
2.5	0.5	1.1	3.3	7.6		
5.0	0.5	1.1	3.4	8.0		
10.0	0.5	1.1	3.4	8.2		
20.0	0.5	1.2	3.6	8.6		
30.0	0.5	1.2	3.7	8.8		
60.0	0.5	1.3	3.9	9.2		
120.0	0.5	1.3	4.0	9.6		
240.0	0.5	1.3	4.1	10.1		
480.0	0.5	1.3	4.3	10.5		
1440.0	0.5	1.6	4.9	11.5		
2880.0	0.6	1.8	5.5	12.6		
4320.0	0.6	2.0	5.7	13.3		
5760.0	0.6	2.1	5.9	13.8		
7200.0	0.6	2.1	6.1	14.4		
10080.0	0.6	2.3	6.6	15.1		
12960.0	0.7	2.5	7.1	15.8		
15840.0	0.7	2.6	7.4	16.2		
20160.0	0.8	2.7	8.0	17.6		

Table 5.2. Nonrecoverable slip of nailed joints under constant-load functions.

even the total slip were negligible. However, it increased as the load level increased. Under 120-1b load, the delayed slip was almost twice the magnitude of the instantaneous slip.

The slip recovery was relatively small compared to the slip under loading. The slip recovered immediately after unloading was about 1/2 to 1/3 of the instantaneous slip. At 60 and 80 lb, the immediate slip recovery was close to 1/2 of the instantaneous slip. The delayed recovery was negligible for all four load levels.

The total slip under each load was divided into four components : Se, S_{de} , Sp and Sv which are defined in Chapter 3. The four components are given in Table 5.1 for recoverable slip and in Table 5.2 for nonrecoverable slip. Three components : Se, Sp and Sv display nonlinear relationship with the load. The experimental slip in Tables 5.1 and 5.2 were used to evaluate the parameters for the workable creep models.

5.1.2. Creep under varying load

The means of the raw data for 20 nailed joints specimens tested under varying-load functions are presented in Appendix B.2. The mean time-slip traces for load functions 5, 6, 7 and 8 are presented in Figures 5.2, 5.3, 5.4 and 5.5, respectively.

When removing the load at t=2880 minutes (2 days),

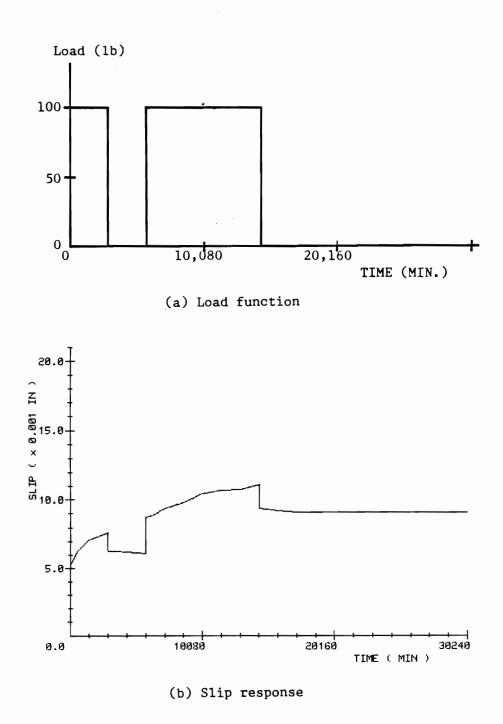
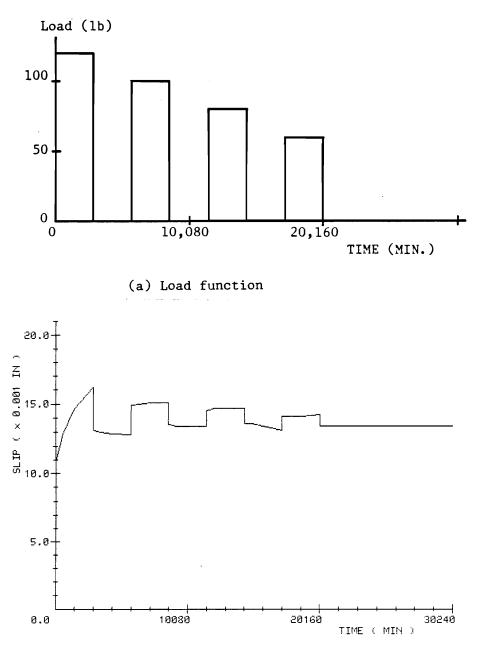
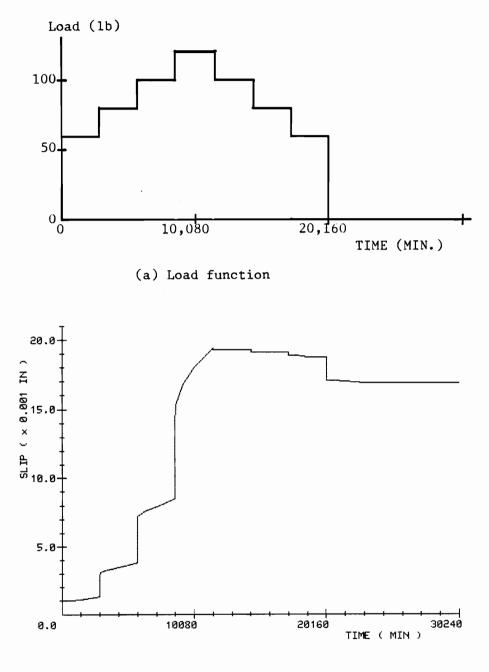


Figure 5.2. Creep of nailed joints under load function 5.



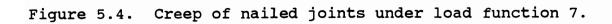
(b) Slip response

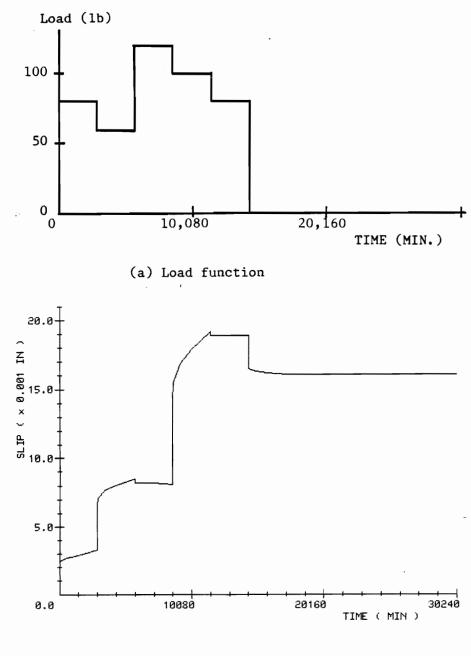
Figure 5.3. Creep of nailed joints under load function 6.



,

(b) Slip response





(b) Slip response

Figure 5.5. Creep of nailed joints under load function 8.

the trace in Figure 5.2 displays a recovery of about 1/3 of the instantaneous slip, which remained almost constant until reloading. Upon reloading at t=5760 minutes (4 days), the slip quickly reached the magnitude observed just before unloading. After unloading at the end of the load function, about 1/3 of the instantaneous slip observed in the previous step was recovered again and the subsequent delayed recovery was negligible. From the observations discussed, it was postulated that the recovery was not affected by the loading history.

Figure 5.3 depicts the slip under uniform loads of 120, 100, 80 and 60 lb. After the initial loading of 120 lb, the slip increased stiffly. Upon unloading, about 1/3 of the instantaneous slip was recovered, which was followed by a small amount of delayed recovery. In the 100-lb step, both the slip and recovery were smaller than that in the first step. The same trend was observed in the subsequent loading and unloading steps of 80 and 60 The instantaneous slip and recovery were almost the 1b. same in each step except the first step, and their magnitudes were similar to the instantaneous elastic slip under constant-load functions. These observations underlined the general understanding that the prior loading history affects the creep. In this study, the following specific observation was made : if the present load is smaller than the previous maximum load, only elastic slip is developed upon loading and is recovered

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upon unloading.

Figure 5.4 depicts the creep slip under load function 7. During stepwise load increase, the slip increased nonlinearly. Even though the load increment was the same, the slip increase got progressively larger with higher loads. Therefore, it was deduced that the present load level and the previous loading history affected the slip in next step. The recovery during the stepwise load decrease was considerably smaller than that in the complete unloading step at the end of decreasing load.

Figure 5.5 shows the slip response to load function 8. Partial unloading of 40 lb at t=5,760 minutes (4 days) gave a negligible but almost the same recovery as that of a 40 pound partial unloading at t=11,520 minutes (8 days). This indicates that the recovery under decreasing load is not affected by the present load level and the previous loading history. By observing the creep slip under load functions 7 and 8, it can be postulated that the decreasing load can be treated as the compressive force, which acts in the opposite direction to the existing load and causes the elastic displacement only.

The observations and postulations presented in Figures 5.2 through 5.5 were included in theoretical modeling discussed in Chapter 3.

5.2. Formulation of theoretical models

The parameters of five models were obtained from the results of constant load tests. The data were fitted by a nonlinear least squares procedure (21,28,31) for Models 5-E and M5-E and by the procedure reported earlier (36) for Model V-VE1. The curve fitting procedures are discussed next.

5.2.1. Five-element Model (5-E)

The constitutive equation for Model 5-E is equation (3-17) which is composed of two parts, Sr and Sn. Each part includes two linear and one nonlinear parameters. The nonlinear parameters are A for Sr and m for Sn. The experimental data for each component are given in Tables 5.1 and 5.2.

The curve fitting was performed as follows. The recoverable part in equation (3-17) was rewritten as :

$$Sr = A_1 + A_2 \{1 - EXP(-A_3 t)\} + e$$
 (5-1)

where e = error in curve fitting. Inversion of equation (5-1) gives :

$$e = Sr-[A_1 + A_2 \{1 - EXP(-A_3t)\}]$$
(5-2)

The sum of the squares of the errors equalled :

$$SSE = \sum_{i=1}^{N} e_{i}^{2} = \sum_{i=1}^{N} [S_{ri} - A_{1} - A_{2} \{1 - EXP(-A_{3} t)\}]^{2}$$
(5-3)

where N is the number of observations. To minimize equation (5-3), the partial derivatives should be zeros :

$$\frac{3}{3} SSE / \frac{3}{4} = -2 \sum_{i=1}^{N} [S_{ri} - A_{i} - A_{2} \{1 - EXP(-A_{3} t_{i})\}] = 0 \quad (5-4)$$

$$\frac{3}{3} SSE / \frac{3}{4} = -2 \sum_{i=1}^{N} \{1 - EXP(-A_{3} t_{i})\} [S_{ri} - A_{1} - A_{2} \{1 - EXP(-A_{3} t_{i})\}] = 0 \quad (5-5)$$

$$\frac{3}{3} SSE / \frac{3}{4} = -2A_{2} \sum_{i=1}^{N} t_{i} EXP(-A_{3} t_{i}) [S_{ri} - A_{1} - A_{2} \{1 - EXP(-A_{3} t_{i})\}] = 0 \quad (5-6)$$

Substituting in equations (5-4) and (5-5) as follows :

$$E_{i} = EXP(-A_{3} t_{i})$$
 (5-7)

results in :

$$NA_{1}+A_{2}(N-\underset{i=1}{\overset{N}{\Sigma}}E_{i}) = \underset{i=1}{\overset{N}{\Sigma}}S_{ri}$$
(5-8)

$$A_{1}(N-\sum_{i=1}^{N}E_{i})+A_{2}(N-2\sum_{i=1}^{N}E_{i}+\sum_{i=1}^{N}E_{i}^{2}) = \sum_{i=1}^{N}S_{i}-\sum_{i=1}^{N}S_{i}E_{i}$$
(5-9)

Equations (5-8) and (5-9) are linear with respect to parameters A1 and A2, and they were rewritten in a matrix form :

$$\begin{bmatrix} N & N - \sum_{i=1}^{N} E_{i} \\ N - \sum_{i=1}^{N} E_{i} & N - 2 \sum_{i=1}^{N} E_{i} + \sum_{i=1}^{N} E_{i}^{2} \end{bmatrix} \begin{bmatrix} A_{1} \\ A_{2} \end{bmatrix} = \begin{cases} \sum_{i=1}^{N} S_{ri} \\ N \\ i = 1 \end{cases}$$
(5-10)
$$\begin{bmatrix} N \\ i = 1 \end{bmatrix}$$
(5-10)

which was solved by Gauss elimination procedure to give :

$$\begin{bmatrix} 1 & 1 - \frac{N}{i \stackrel{\leq}{=} 1} E_{i} / N \\ 0 & \sum_{i \stackrel{\leq}{=} 1}^{N} E_{i}^{2} - \left(\sum_{i \stackrel{\leq}{=} 1}^{N} E_{i} \right)^{2} / N \end{bmatrix} \begin{bmatrix} A_{1} \\ A_{2} \end{bmatrix} = \begin{bmatrix} \sum_{i \stackrel{\leq}{=} 1}^{N} S_{i} / N \\ \sum_{i \stackrel{\leq}{=} 1}^{N} S_{i} \sum_{i \stackrel{\leq}{=} 1}^{N} E_{i} / N - \sum_{i \stackrel{\leq}{=} 1}^{N} S_{i} E_{i} \end{bmatrix} (5-11)$$

from which :

$$A_{2} = [1/\{\sum_{i=1}^{N} E_{i}^{2} - (\sum_{i=1}^{N} E_{i}^{2})^{2}N\}](\sum_{i=1}^{N} S_{ri}\sum_{i=1}^{N} E_{i}/N - \sum_{i=1}^{N} S_{ri} E_{i}) (5-12)$$

$$A_{1} = \sum_{i=1}^{N} S_{ri}/N - A_{2}(1 - \sum_{i=1}^{N} E_{i}/N) (5-13)$$

Equations (5-12) and (5-13) were solved by first assuming a reasonable estimate for A_3 in equation (5-7). Based on assumed A_3 , A_1 and A_2 were calculated and then the estimated A_3 should be checked by applying equation (5-6). If equation (5-6) was not equal to zero, the next estimate of A_3 could be obtained by Newton's formula :

$$\mathbf{A}_{2}^{"} = \mathbf{A}_{3}^{"} - [(\partial SSE/\partial \mathbf{A}_{3})/(\partial^{2}SSE/\partial \mathbf{A}_{3}^{2})]$$
(5-14)

where A'_3 = current estimate, and A''_3 = estimate in the next step. The second derivative of SSE with respect to A_3 in equation (5-14) was obtained by differentiating equation (5-6) :

$$\partial^{2} SSE / \partial A^{2} = 2A_{2} \begin{bmatrix} N \\ i = 1 \\ 2 \end{bmatrix} S_{ri} t_{1}^{2} E_{1} - (A_{1} + A_{2}) \sum_{i = 1}^{N} t_{1}^{2} E_{1} + 2A_{2} \sum_{i = 1}^{N} t_{1}^{2} E_{1}^{2} \end{bmatrix}$$
(5-15)

LOAD	A1	 A2	 A3	A4	A5	 m
60	0.5118	0.19014	0.0002981	0.0004536	0.2941	0.57
80	1.2130	0.40501	0.0001523	0.0288380	1.0509	0.41
100	1.8438	0.34536	0.0005148	0.1451320	3.0947	0.35
120	2.8434	0.42717	0.0003385	0.5136500	7.1030	0.30

Table 5.3. Parameters in equation (3-17) defining Model 5-E.

Substituting the current estimate and equation (5-15) into equation (5-14) calculated the estimate of A₃ in the next step, which became again the current estimate in the next step. This process was repeated until equation (5-6) approached zero.

The nonrecoverable part of equation (3-17) was obtained by the same method as the recoverable part. A BASIC program for an IBM microcomputer was developed for above nonlinear least square procedure (Appendix C.1). The parameters of the models for nailed joints between Douglas-fir lumber and plywood under constant load were evaluated. They are given in Table 5.3.

5.2.2. Modified Five-element Model (M5-E)

The constitutive equation for Model 5-E is equation (3-21) in which B_1 , B_2 , B_4 and B_5 are linear and B_3 , N_1 , N_2 , N_3 and N_4 are nonlinear. Nonlinear least square fitting of the experimental data could not be used, because the number of nonlinear parameters is too high. Therefore, the parameters of Model M5-E were obtained from the parameters of Model 5-E because Models 5-E and M5-E are obtained from the same data and have the same components except the nonlinear terms for the instantaneous elastic and viscous slip of Model M5-E.

Equation (3-17) of Model 5-E and equation (3-21) of Model M5-E consist of four components, each of which has a

Table 5.4. Parameters in equation (3-21) defining Model M5-E.

Parameter	Value
B ₁	5.7464 X 10
B ₂	0.003812
B 3	3.2590 X 10
B ₄	3.1916 X 10
B ₅	4.8121 X 10
N 1	2.2838
N ₂	4.3212
N ₃	0.3500
N 4	4.9026

similar form for both models. Therefore, by comparing each part of equation (3-21) with the corresponding part of equation (3-17), the parameters of Model M5-E were obtained in terms of the parameters of Model 5-E. They were found to be :

$B_1 P^{N_1}$	$= A_1$	(5-16)

$B_2 P$	$= \mathbf{A}_2$		(5-17)

B₂

$$\mathbf{D}_{\mathbf{\mu}}\mathbf{P} = \mathbf{A}_{\mathbf{\mu}} \tag{5-19}$$

$$N_3 = m$$
 (5-20)

$$B_{5}P^{N_{4}} = A_{5}$$
 (5-21)

Equations (5-17), (5-18) and (5-20) show linear relation between the parameters of Model M5-E, B₂, B₃ and N₃, and the parameters of Model 5-E, A₂, A₃ and m. Therefore, B₂, B₃ and N₃ were obtained as the averages of the values for A₂, A₃ and m given in Table 5.3, because the table shows values for each of the four load levels. In equations (5-16), (5-19) and (5-21), parameters B₁, B₄ B₅, N₁, N₂ and N₄ have nonlinear relation with A₁, A₄ and A₅, which are the parameters of Model 5-E. Thus, they were evaluated by nonlinear least square fitting of the values for A₁, A₄ and A₅ given in Table 5.3. The analysis resulted in Table 5.4.

5.2.3. Viscous-viscoelastic Model 1 (V-VE1)

(5-18)

Model V-VE1 can be represented by equation (3-27), in which the first three terms represent recoverable slip and the remaining represent nonrecoverable slip. For the nailed joints investigated, the experimental data needed to formulate equation (3-27) are given in Tables 5.1 and 5.2.

To obtain the kernel functions for the three integral representation, the procedure proposed by Polensek (36) was employed. For instance, the recoverable slip was represented by :

$$Sr = F_{1}(t) P + F_{2}(t) P^{2} + F_{3}(t) P^{3}$$
(5-22)

and rewritten in a matrix form :

$$\{Sr\} = [P] \{F(t)\}$$
 (5-23)

where :

$$\{Sr\} = \begin{cases} S_{r1} \\ S_{r2} \\ S_{r3} \end{cases}$$
(5-24)
$$[P] = \begin{bmatrix} P_1 & P_1^2 & P_1^3 \\ P_2 & P_2^2 & P_2^3 \\ P_3 & P_3^2 & P_3^3 \end{bmatrix}$$
(5-25)
$$\{F(t)\} = \begin{cases} F_1(t) \\ F_2(t) \\ F_3(t) \end{cases}$$
(5-26)

and kernel functions were obtained by inversion of equation (5-23) :

$$\{F(t)\} = [P]^{-1} \{Sr\}$$
 (5-27)

To get kernel functions, the data under load levels P1 = 60 lbs, P2 = 100 lbs and P3 = 120 lbs were used. Therefore, the inverse of load matrix, $[P]^{-1}$, gave :

$$[P]^{-1} = \begin{bmatrix} 1/12 & -9/100 & 1/24 \\ -11/7200 & 9/4000 & -1/900 \\ 1/14400 & -1/80000 & 1/144000 \end{bmatrix}$$
(5-28)

in which the unit of matrix elements is (1/1b). In equation (5-23), Sr was expressed by power functions :

$$\{Sr\} = \{Sr^{\circ} + Sr^{\dagger}(t)\}$$
 (5-29)

and kernel functions, $\{F(t)\}$, were represented by equations (3-39), (3-40) and (3-41). Thus, equation (5-27) equaled :

$$\{F^{\circ} + F^{+}(t)\} = [P]^{-1} \{Sr^{\circ} + Sr^{+}(t)\}$$
 (5-30)

in which $\{F^{+}F^{+}(t)\} = \{F^{+}\}+\{F^{+}(t)\}$, and $\{F^{+}\}$ and $\{F^{+}(t)\}$ were obtained as follows :

$$\{F^{\circ}\} = [P]^{-1} \{Sr^{\circ}\}$$
 (5-31)

$$\{F^{+}(t)\} = [P]^{-1} \{Sr^{+}(t)\}$$
 (5-32)

Next, these procedures were applied to the nailed joints investigated in this study. The experimental data given in Table 5.1 were fitted to the power functions by applying nonlinear least squares analysis to give :

$$S_{r1} = 0.477 + 0.013222 t^{0.30}$$
, for P=60 lb (5-33)

$$S_{r^2} = 1.701 + 0.104614 t^{0.17}$$
, for P=100 lb (5-34)

$$S_{r_3} = 2.544 + 0.217091 t^{0.12}$$
, for P=120 lb (5-35)

and kernel functions were obtained in the same way as before. They are :

$$F_{1}(t) = [-7.340+1.1020 t^{0.30}-9.4153 t^{0.17} + 9.0455 t^{0.12}] 0.001 (5-36)$$

$$F_{2}(t) = [0.272-0.0202 t^{0.30}+0.2354 t^{0.17} - 0.2412 t^{0.12}] 0.001 (5-37)$$

$$F_{3}(t) = [-0.00028+0.00009 t^{0.30}-0.0013 t^{0.17} + 0.0015 t^{0.12}] 0.001 (5-38)$$

For nonrecoverable slip, the method was the same. The experimental data for nonrecoverable slip (Table 5.2) were fitted to the power functions that had the same power of time:

$$S_{n_1} = 0.2825 + 0.003878 t^{0.35}$$
, for P=60 lb (5-39)

$$S_{n^2} = 3.0947 + 0.145132 t^{0.35}$$
, for P=100 lb (5-40)

$$S_{n_3} = 7.5054 + 0.306340 t^{0.35}$$
, for P=120 lb (5-41)

from which the following kernel functions were obtained :

$$F_{+}(t) = [57.74367+0.025453 t^{0.35}] 0.001$$
 (5-42)

$$F_5(t) = [-1.80786 - 0.019755 t^{0.35}] 0.001$$
 (5-43)

$$F_6(t) = [0.01540 + 0.00034 t^{0.35}] 0.001 (5-44)$$

Next, kernel functions evaluated in equations (5-36), (5-37) and (5-38) for Sr and (5-42), (5-43) and (5-44) for Sn were substituted into equation (3-27) to give the creep slip of Model V-VE1 for nailed joints under constant load.

5.2.4. Viscous-viscoelastic Model 2 (V-VE2) and Model3 (V-VE3)

Models V-VE2 and V-VE3 are combinations of Models M5-E and V-VE1. The constitutive equations of Models V-VE2 and V-VE3 are given in equations (3-28) and (3-29), respectively, whose parameters can be obtained from Models M5-E and V-VE1. The procedure is easy to carry out and is not presented in this dissertation.

5.2.5. Fourier series approximation of load function 7

Load function 7 was applied as an example of continuous-load function. The solution involved the approximation of load function 7 by equation (3-89) of Fourier sine series in Section 3.3. The coefficients were obtained by equation (3-88) and are given in Table 5.5,

Coefficient	Value
Ао	60.0
Bı	44.48648
В з	-11.05122
B ₅	0.27989
B 7	-3.63783
B 9	0.15550
B ₁₁	-3.01394
B ₁₃	3.42207
B ₁₅	2.96574
B ₁₇	-1.95024
B.i 9	0.07365

Table 5.5. Fourier sine-series coefficients in equation (3-88) defining load function 7

 $B_i \simeq 0.0$ when i = 2, 4, 6, 8, 10, 12, 14, 16, 18, 20

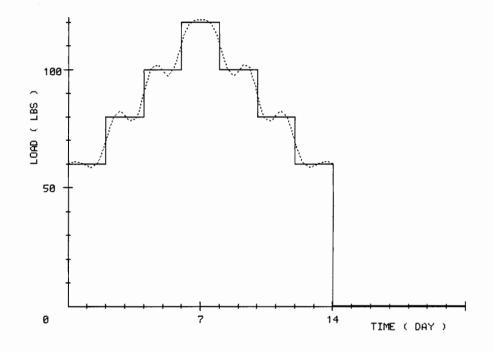


Figure 5.6. Load function 7 and its Fourier sine series approximation.

in which the number of terms in the series was 20. The approximated curve is shown in Figure 5.6.

5.3. Verification of theoretical models

The theoretical models for varying loads were obtained according to the procedure described in Chapter 3, and the specific parameters for nailed joints investigated were developed in Section 5.2. In this Section, the predictions of the theoretical models developed were compared to each other and to the experimental data.

5,3.1. Response to constant-load functions

The predictions of the models under constant-load functions are presented in Figure 5.7. Although the data under 80-lb loading were not used to construct Models V-VE1, V-VE2 and V-VE3, their predictions were also valid for 80-lb loading as shown in Figure 5.7. Tables 5.6 and 5.7 show the statistical indicators, R² and SSE, for each of the five models under constant-load functions, respectively.

5.3.2. Accurary of Solutions developed

Legend :

	Experimental		Model V-VEl
••••	Model 5-E		Model V-VE2
·	Model M5-E	···	Model V-VE3

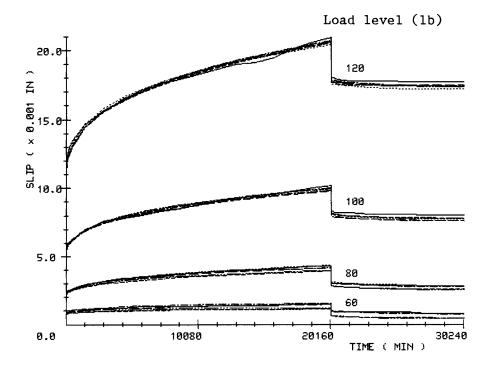


Figure 5.7. Experimental and theoretical data obtained under constant-load functions.

 MODEL	60 lbs	80 lbs	100 lbs	120 lbs
5-E	0.7421	0.9065	0.9545	0.9734
M5-E	0.8219	0.9139	0.9500	0.9724
V-VE1	0.7601	0.9102	0.9598	0.9757
V-VE2	0.7507	0.9100	0.9560	0.9759
V-VE3	0.7275	0.9069	0.9546	0.9722

Table 5.6. The values of R^2 for the models under constant-load functions.

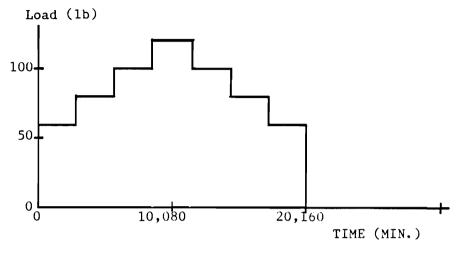
Table 5.7. The values of SSE for the models under constant-load functions.

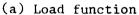
MODEL	60 lbs	80 lbs	100 lbs	120 lbs
5-E	3.7566	1.3866	4.0542	11.3995
M5-E	0.5280	1.1841	5.4727	10.4951
V-VE1	3.5773	3.0573	3.5035	10.0897
V-VE2	0.6271	1.2769	5.2385	9.5246
V-VE3	2.8768	2.7416	3.8514	10.8314

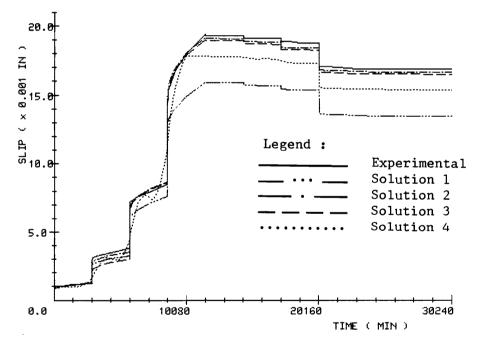
For modifications of the models, four Solutions were proposed in chapter 3. They are :

- Solution 1) Application of Approach 1 with discrete load function;
- Solution 2) Application of Approach 2 with discrete load function;
- Solution 3) Application of Approach 2 with load function represented by Heaviside function; and
- Solution 4) Application of Approach 2 with load function represented by Fourier series.

These Solutions were applied to the models except Model 5-E for load function 7 and then compared to one another and to the experimental data. The creep values predicted by each Solution and each model were computed by BASIC programs for an IBM microcomputer which are recorded in Appendices C.6 (Solution 1), C.4 (Solution 2), C.7 (Solution 3) and C.8 (Solution 4). The predicted results are given in Appendices B.7 (Solution 1), B.5 (Solution 2), B.8 (Solution 3) and B.9 (Solution 4). The results are also presented graphically in Figures 5.8, 5.9, 5.10 and 5.11 for Models M5-E, V-VE1, V-VE2 and V-VE3, respectively. Model 5-E was expressed by Solution 2 because all Solutions except Solution 2 require integration, while Model 5-E is discrete and cannot be

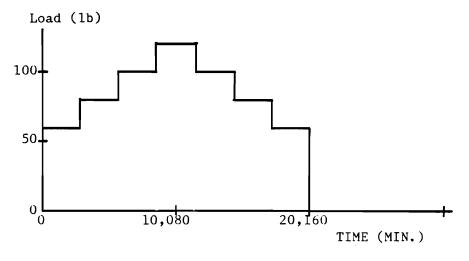




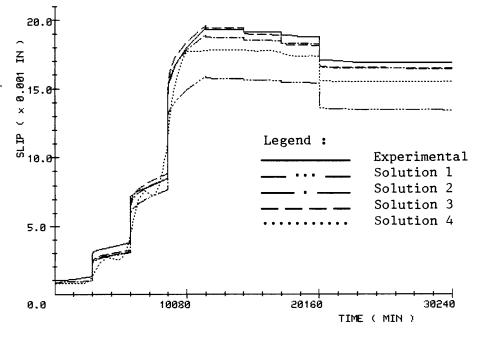


(b) Slip response

Figure 5.8. Effect of Solutions used on the creep of Model M5-E under load function 7.

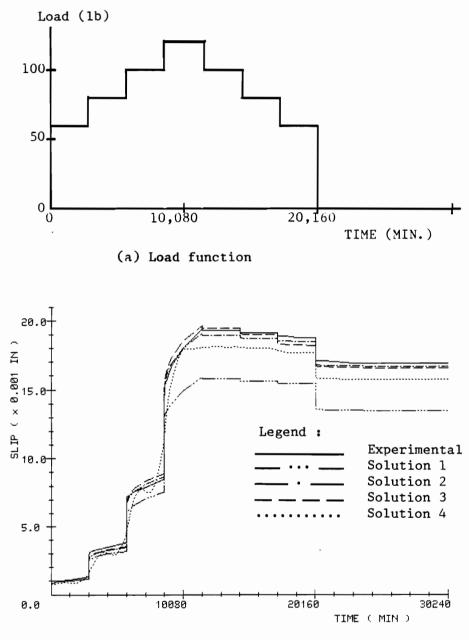


(a) Load function



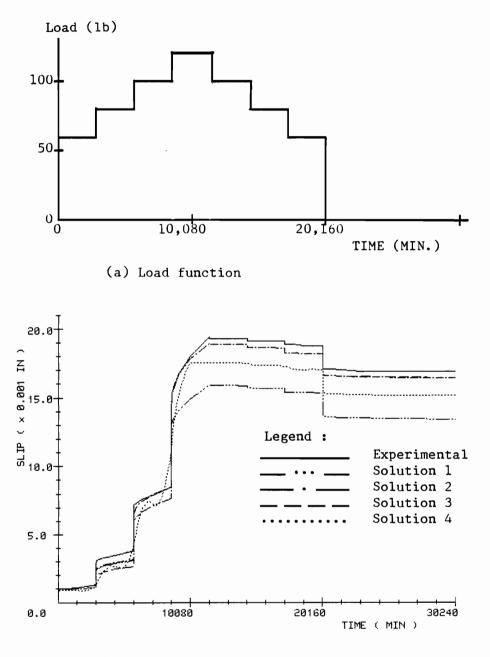
(b) Slip response

Figure 5.9. Effect of Solutions used on the creep of Model V-VE1 under load function 7.



(b) Slip response

Figure 5.10. Effect of Solutions used on the creep of Model V-VE2 under load function 7.



(b) Slip response

Figure 5.11. Effect of Solutions used on the creep of Model V-VE3 under load function 7.

integrated.

For all four models, Solution 1 predicted the lowest slip values, which were the worst among the four Solutions as illustrated in Figures 5.8 through 5.11, especially at high load levels. Solution 4 predicted intermediate values, but the predicted curves were wavy. The predictions by Solutions 2 and 3 were superior to the other two and very close to the experimental data. For Model M5-E (Figure 5.8), Solution 2 yielded slightly better agreement than that of Solution 3. For Models V-VE1 (Figure 5.9) and V-VE2 (Figure 5.10), Solution 3 gave a slightly closer agreement than Solution 2, but both Solutions gave results that were very close to each other for Model V-VE3 (Figure 5.11). Thus, results by Solutions 2 and 3 appear to give about the same accuracy. However, Solution 2 is much easier to apply than Solution 3, because the latter involves complicated integration of Heaviside function. For these reasons, Solution 2 is the preferred one among the four Solutions.

Solution 4 gave a relatively accurate prediction although it was wavy and underpredicted the slip during unloading steps. The underprediction was due to the shape of the approximated load function by Fourier sine series (Figure 5.6), which had its maximum at t=10,080 minutes (7 days) and decreased from that point on. However, the actual load function had its maximum between t=8,640 minutes (6 days) and t=11,520 minutes (8 days). Therefore, the approximated load function had shorter loading period under maximum load than the actual load function. For this reason, the predictions of the models by Solution 4 were below the experimental data at the maximum load and during unloading. Therefore, if the load function could be approximated more exactly by Fourier series, the results would have agreed better. Random load functions, such as those caused by winds or earthquakes, cannot be applied to the models directly, because they cannot be integrated. However, for such functions, Solution 4 is relatively easy to apply and expected to produce acceptable aaccuracy.

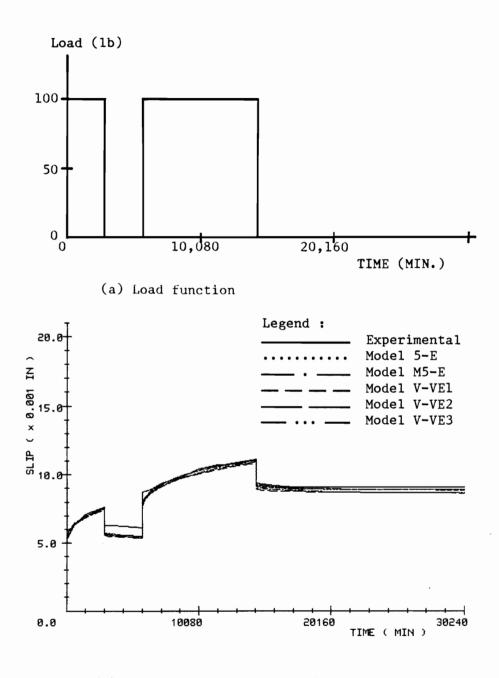
The comparisons in Figures 5.8 through 5.11 show that Solution 2 is not only the most accurate but also the easiest to apply for stepwise loads. Thus, only Solution 2 was extended to the remaining experimental load functions.

5.3.3. Response to varying-load functions

In this section, the predictions of the developed models that were modified by Solution 2 are compared with the experimental observations for the creep under varyingload functions.

5.3.3.1. Creep under load function 5

The five models developed in this study were solved



(b) Slip response

Figure 5.12. Comparison between the experimental and theoretical data for load function 5.

Model	R ²	SSE (0.000001 in.)
5-E	0.9612	5.80
M5-E	0.9562	8.39
V-VE1	0.9610	6.15
V-VE2	0.9583	8.84
V-VE3	0.9583	5.92

Table 5.8. The values of R^2 and SSE for the models developed for load function 5

Table 5.9. The values of R^2 and SSE for the models developed for load function 6.

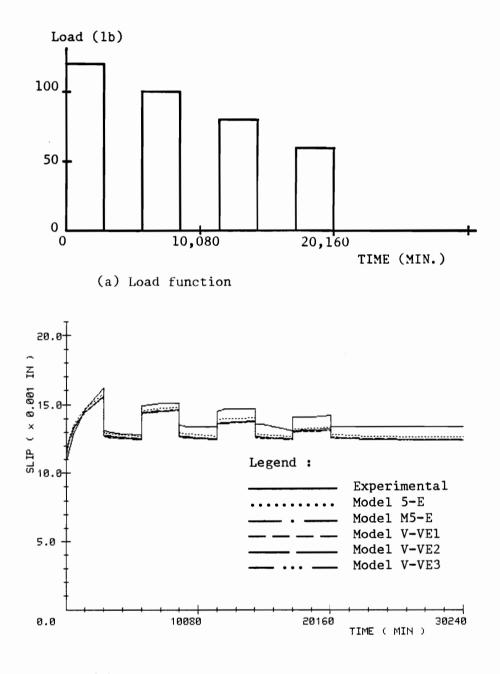
Model	R ²	SSE (0.000001 in.)
5-E	0.8462	25.64
M5-E	0.7982	39.17
V-VE1	0.8204	40.39
V-VE2	0.8222	38.32
V-VE3	0.7916	41.18

for load function 5 (Figure 4.8). Their predictions were computed by an IBM microcomputer and the program is recorded in Appendix C.2. The predicted data are given numerically in Appendix B.3 and graphically in Figure 5.12. The statistical indicators, R^2 and SSE, measuring the closeness of theoretical and experimental data are presented in Table 5.8.

As shown in Figure 5.12, the predictions are very close to each other and to the experimental data for all the models. Although the predictions are somewhat lower than the experimental data during the period from t=2,880 minutes (2 days) to t=5,760 minutes (4 days), the overall predictions agree quite well with the experimental data. Their R²-values are all greater than 0.95 (Table 5.8). The values of R² indicates that Model 5-E is slightly better than others, and the predictions by Model V-VE1 are similar to Model V-VE3 and slightly better than Models M5-E and V-VE2. However, these differences are almost negligible and, for all practical purposes, the correlation is excellent and about the same for all the models.

5.3.3.2. Creep under load function 6

Again, an IBM microcomputer was used to compute the creep for the models developed under the load function 6 (Figure 4.8). The numerical results are given in Appendix



(b) Slip response

Figure 5.13. Comparison between the experimental and theoretical data for load function 6.

B.4 and graphical ones are shown in Figure 5.13. The computer program is presented in Appendix C.3. The corresponding R^2 and SSE are summarized in Table 5.9.

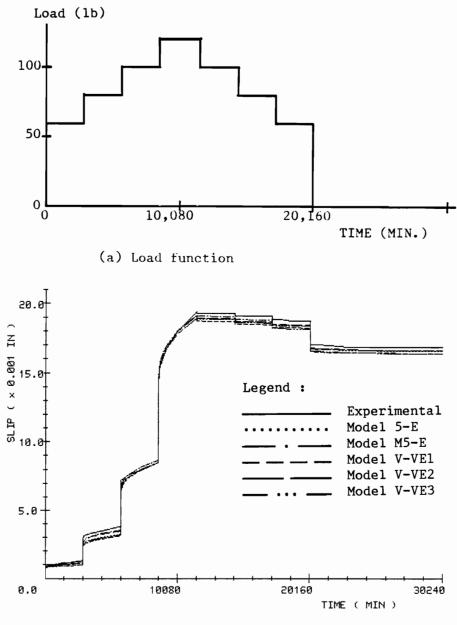
The correlation for the load function 6 is not as good as those for other load functions, although all the R^2 -values of the models are about 0.8 which still indicates a good fit. Again, the prediction by Model 5-E is superior, while Models M5-E, V-VE1 and V-VE2 are similar to each other but slightly better than Model V-VE3.

The inferior correlation for load function 6 in comparison to other functions was caused by the tedious loading and unloading procedure used in testing. Although the loading process was performed carefully with a special device, the application of large load in each cycle of the load function 6 induced dynamic jerk, which in turn caused additional instantaneous slip. Thus, the slip upon loading was greater than the recovery upon unloading when only the action of the static load was removed. This difference was accumulated with the number of steps. In the theoretical models, however, only static-loading effect was considered. This observation is evidenced by the difference between the experimental and theoretical data, which is getting larger with the number of steps.

5.3.3.3. Creep under load function 7

Creep under the load function 7 was already discussed

133



(b) Slip response

Figure 5.14. Comparison between the experimental and theoretical data for load function 7.

Table 5.10.	The	valu	es d	of R ²	and	SSE	for	the	models
	deve	loped	for	load	funct	ion 7	7.		

Model	R ²	SSE (0.000001 in.)	
5 - E	0.9986	7.63	
M5-E	0.9979	6.45	
V-VE1	0.9977	13.24	
V-VE2	0.9981	6.60	
V-VE3	0.9977	11.40	
Figure 5.11.	The values of R^2 developed for load	and SSE for the r function 8.	nodels
Model	R ²	SSE (0.000001 in.)	
5-E	0.9967	22.36	
M5-E	0.9966	18.70	
V-VE1	0.9972	14.83	
		,	
V-VE2	0.9969	15.07	
V-VE2 V-VE3	0.9969 0.9968		

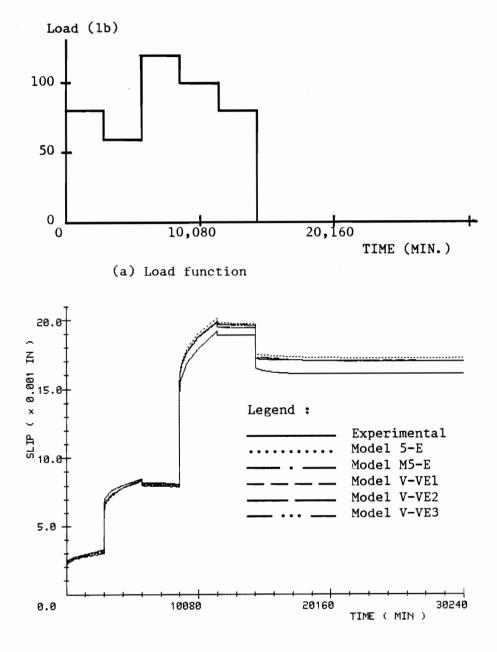
in Section 5.3.2. In this section, only the predictions of the models by Solution 2 are presented. The results are given numerically in Appendix B.5 and graphically in Figure 5.14. The statistical indicators, R^2 and SSE, are shown in Table 5.10.

The predictions of the models developed agree closely with the experimental data, although they are slightly below the experimental data in the second loading step and in the unloading steps. All the R²-values are greater than 0.99. It appears that Models 5-E, M5-E and V-VE2 are similar to each other and slightly more accurate than Models V-VE1 and V-VE3.

5.3.3.4. Creep under load function 8

As for the other functions, a program (Appendix C.5) and an IBM microcomputer were used to predict the creep under load function 8 (Figure 4.8). The results are presented in Appendix B.6 and Figure 5.15. The corresponding R^2 and SSE are given in Table 5.11.

The predictions of the models are somewhat larger than but quite close to the experimental data under the 120-pound load and in the unloading steps. All the R^2 values are greater than 0.99. Models V-VE1 and V-VE2 are slightly better than the other models.



(b) Slip response

Figure 5.15. Comparison between the experimental and theoretical data for load function 8.

IV. CONCLUSION AND RECOMMENDATION

6.1. Conclusions

The conclusions reached in this investigation pertain to the experimental and theoretical findings. The conclusions derived on the basis of the experimental investigation are :

- The delayed slip under low levels of constant loads is negligible.
- The recovery that occurs immediately after unloading is about 1/2 to 1/3 of the instantaneous slip developed upon loading.
- 3. In constant-load tests, the instantaneous and delayed recovery can be regarded as the instantaneous and delayed elastic slip due to loading. The rest in the instantaneous and delayed slip due to loading can be regarded as the instantaneous plastic and viscous slip, respectively. The assumption that only elastic slip is recovered produces accurate theoretical predictions, but potential errors due to possible recovery of plastic or viscous slip cannot be detected because of the small overall recovery.
- 4. If the same weight is applied again after unloading, the slip quickly reaches the magnitude

obtained just before unloading.

- 5. If the subsequent load in the repetitive uniform loading is less than the maximum load in the previous loading steps, only elastic slip develops and recovers in both loading and unloading.
- Under stepwise increasing load, the instantaneous elastic, plastic and viscous slip increase nonlinearly with respect to load.
- 7. Loading history has an effect on the slip under increasing load, but it has no effect on the recovery under decreasing load.

The following conclusions were derived on the basis of the theoretical investigation :

- Generally, the creep predictions of the five models developed in this study agree closely with the experimental data.
- In the theoretical modeling, stepwise decreasing load can be regarded as the compressive force acting opposite to the existing load.
- 3. Stiffness of nailed joints is reduced due to the creep under long-term load. The recoveryduring decreasing loads is reduced as much as stiffness reducing factor obtained under the reversed increasing-load function.

- 4. Solution 2, which is the modification of the models by modified superposition principle applied to nailed joints (MMSP), gives the most accurate predictions of the creep slip under varying-load functions investigated and is the easiest to use for stepwise-load functions.
- 5. Solution 4 in which the models are modified by MMSP and the load function is represented by Fourier series gives the results which are quite accurate but somewhat wavy.

6.2. Recommendations

To make the theoretical models developed practically useful, the following recommendations are made :

- The behavior of nailed joints under actual service load functions, such as those caused by winds and earthquakes, should be studied and the models should be verified for these functions.
- 2. The representation of the service load functions by simple equations that can be integrated should be studied, so that they can be incorporated into the models.
- The procedure of applying the models to the theoretical analysis of wood building should be developed.
- The effect of creep in nailed joints on the overall behavior of wood components or structures under long-termload should be studied.
- 5. The models developed are accurate for the nailed joints between Douglas-fir lumber and plywood. It is anticipated that they will work equally well for other joint types and materials, but additional investigations are needed to evaluate material constants and demonstrate the accuracy.

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APPENDICES

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Appendix A. List of symbols used in this study.

	=	Equillibrium moisture content
EXP	=	Exponential function
5-E	=	Five-element Model
FRL	=	Forest Research Laboratory
M5-E	=	Modified Five-element Model
MSP	=	Modified superposition principle
MMSP	=	Modified MSP for nailed joints
osu	=	Oregon State University
SHP	=	Strain hardening principle
V-VE1	=	Viscous-viscoelastic Model 1
V-VE2	=	Viscous-viscoelastic Model 2
V-VE3	=	Viscous-viscoelastic Model 3
A, Ao, A_1, A_2, \dots	=	Constants
A '3, A3"	=	First and second estimate for A_3
a	=	Constant
a B, B_1 , B_2 ,, b		
$B, B_1, B_2,, b$	=	Constants
B, B ₁ , B ₂ ,, b C	=	Constants Constant
B, B ₁ , B ₂ ,, b C D	-	Constants Constant Diameter of nail
B, B ₁ , B ₂ ,, b C D E, E ₁ , E ₂	=	Constants Constant Diameter of nail Stiffnesses of springs
B, B ₁ , B ₂ ,, b C D E, E ₁ , E ₂ Ei		Constants Constant Diameter of nail Stiffnesses of springs EXP(-A ₃ ti)
<pre>B, B₁, B₂,, b C D E, E₁, E₂ Ei e, ei</pre>		Constants Constant Diameter of nail Stiffnesses of springs EXP(-A ₃ ti) Errors in curve fitting

Fp(t),	=	Load function and its derivative approximated by Fourier series
Fc(t), Fc(t)	=	Reversed compressive force function and its derivative for decreasing load functions approximated by Fourier series
$G_1(P)$, $G_2(P)$	=	Function of load
H(t-a), Ḣ(t-a)	=	Heaviside unit step function and its derivative
i, j, k	=	Constants
Ka	=	Stiffness modulus of nailed joints for instantaneous loading
Кј	=	Stiffness modulus of nailed joints
Ко	=	Elastic bearing constant
Ks	=	Stiffness modulus of nailed joints for stepwise loading
L	-	Length of nail penetration into framing members
Ln		Length of nail
Lp	=	Thickness of plywood
M, m	=	Constants
N	=	Number of steps or observations
N ₁ , N ₂ ,	-	Constants
Р	=	Load
Po, P ₁ ,	=	Load level in stepwise load function
P(t), P(t)	=	Load function and its derivative
Pc(t), Pc(t)	H	Reversed compressive load function and its derivative for decreasing load function.
đ	=	Stress applied to wood from nail
R	=	Constant
R ²	=	Square of the correlation coefficient

Rs	=	Stiffness reducing rate					
s, s	=	Experimental data and their mean					
s*, s *	=	Theoretical data and their mean					
SG	=	Specific gravity					
SSE	=	Sum of the squares of errors					
Sa	=	Instantaneous slip					
Sc	=	Delayed or creep slip					
s _{de}	=	Delayed elastic slip					
Se	=	Instantaneous elastic slip					
Smax	=	Maximum slip just before unloading					
Sn	-	Nonrecoverable slip under constant load					
Sp	=	Instantaneous plastic slip					
Sr	=	Recoverable slip under constant load					
SS	=	Recoverable slip under stepwise load					
Ss ²	=	Nonrecoverable slip under stepwise load					
st	=	Total slip					
Sv	=	Viscous slip					
S_1 , S_2	=	Slip under example load functions					
t	=	Time					
to, t ₁ ,	=	Initial time for each step in stepwise-load function					
tmax	=	Time when the maximum slip occurs					
Тр	=	Period used in Fourier approximation of load function 7					
v		Volume of wood at 12-percent moisture content					
Vs	=	Volume of wood at specified moisture content					

Wo	=	Ovendry weight of wood
ŴW	=	Weight of wood at 12-percent moisture content
ω	=	Angular velocity = $2 \pi / Tp$
У	=	Deflection under the nail
z, z ₁ , z ₂ ,	=	Integration variables
α	=	Constant = $A^m + B^m + C^m + \dots$
ε,ε	=	Strain and strain rate
ε [•] , ε ⁺	=	constants
ε _{de} , ε _{VE}	=	Delayed elastic strain
ε _e	=	Instantaneous elastic strain
$\epsilon_{\rm NV}$	=	Negative viscous strain
ε _p	=	Instantaneous plastic strain
ε _{pv}	=	Positive viscous strain
ε _t	=	Total strain
ε	=	Viscous strain
ε ₀ , ε ₁ ,	=	Strain under stepwise stressing
η ₁ , η ₂	-	Viscosity of linear dashpot
η ₁ (t)	=	Viscosity of nonlinear dashpot
μ(Ρ)	=	Plasticity of nonlinear time-hardening element
σ, σ	=	Stress and stress rate
σ ₀ , σ ₁ ,	=	Stress levels in stepwise stress function
σ*	=	Creep limit stress
^{∆P} i,j,k	=	Step sizes in stepwise load functions
δ(t-a)	=	Dirac delta function

TIME	60 LBS	80 LBS	100 LBS	120 195
$\begin{array}{c} 0 & 0 \\ 1 & 250 \\ 0 & 0 \\ 1 & 250 \\ 0 & 0 \\ 0 &$	900000000000000000000000000000000000000	10000000000000000000000000000000000000	44600000000000000000000000000000000000	114846715056740690151697211000000988877777 900011110233456777799000888888888887777777777777777777

AppendixB.1. Experimental data for creep of nailed joints under constant-load functions.

TIME	5	6	7	8
$\begin{array}{c} 0 & 0 \\ 30 & 0 \\ 30 & 0 \\ 0 & 0 \\ 30 & 0 \\ 0 & 0 \\ 30 & 0 \\ 0 & 0 \\ 128889430000000000000000000000000000000000$	283531644355211447948 4455775365668865899 11 11 11 11 11 11 11 9999 9999	51096242100807901160054444567777760410111116554444444444	011111122288080473980473333751111111999888887111099999 90001358808888888899999999999999999999888888777777	

Appendix B.2. Experimental data for creep of nailed joints under varying-load functions.

Appendix B.3. Theoretical predictions by Solution 2 for the creep of nailed joints under load function 5.

TIME	5-E	M5-E	V-VE1	V-VE2	V-VE3
$\begin{array}{c} 0.00\\ 1.000\\ 1.000\\ 0.0$	45000000000000000000000000000000000000	4 bibliobibl	45565555666557555555555555777777868889999999999	40000000000000000000000000000000000000	

Appendix B.4.

Theoretical predictions by Solution 2 for the creep of nailed joints under load function 6.

TIEM	5-E	М5-Е	V-VE1	V-VE2	V-VE3
$\begin{array}{c} 0.00\\ 1.000\\ 1.000\\ 0.0$	11111111111111111111111111111111111111	56057529935499999986312299999012563255554432981244444455557 0001111213345222222222222244444444444444557777777666656666666666	11111111111111111111111111111111111111	11111111111111111111111111111111111111	11111111111111111111111111111111111111

.

Appendix B.5.

Theoretical predictions by Solution 2 for the creep of naild joints under load function 7. $\begin{array}{c} 12120 & 000\\ 112960 & 000\\ 114400 & 000\\ 1144405 & 000\\ 1144405 & 000\\ 1144405 & 000\\ 1144405 & 000\\ 11444360 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 11444460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 & 000\\ 114460 &$

TIME	 5-Е	 М5-Е	V-VE1	V-VE2	V-VE3
$\begin{array}{c} 0.00\\ 1.000\\ 10$	492482856016489626366841777777777765530457660125990666665554 22248285601648962636684177777777777765530457660125990666665554	61583854615104526967456066666544177816872880888888888876	32829532489055852635445876554432119734072296572198877665444444	22222222222222222222222222222222222222	22222222222222222222222222222222222222

				10 mm - 10 mm - 100 mm
TIME	M5-E	V-VE1	V-VE2	V-VE3
$\begin{array}{c} 0 \\ 1 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\$	000000001111122222222233666666666773353353554455222 3538889999001255555670411557677816377600112385858206645222 3538889999000125555567041155767781637760011228858206645222	6890124689375935163-2655179162270967346841362831888	368724705642368665077111524594367606780286920733000 7777788889996014444455555 	00000000001122222222223483270079778940352843000 **********************************

Appendix B.7. Theoretical predictions by Solution 1 under load function 7.

 $\begin{array}{c} 115550 \\ 000 \\ 0111550 \\ 000$

13.44

16Ò

Appendix B.8. Theoretical predictions by Solution 3 for load function 7.

TIME	М5-Е	V-VE1	V-VE2	V-VE3
$\begin{array}{c} 0.00\\ 1.000\\ 1000\\ 0.00$	35679036022200714041915693784928967793472428424017777766665	00000000000000000000000000000000000000	00000000000111010000000000000000000000	7777889013407646837/4126678230696866820048184979550000000000000000000000000000000000

TIME	M5-E	V-VE1	V-VE2	V-VE3
$\begin{array}{c} 1.000\\ 1000000\\ 1000000\\ 1240000000\\ 1240000000000000\\ 12480889140000000000000000\\ 128888914000000000000000000000000000000000$	4457893689866795013325055931432558524118345302111111 90000000000111111112222384444455567771111122223500000 1111112222386898667950133250559314325585241188345302111111	00000000000011111111122224444444455677111111111111111111111111111	334578267977891723687151250810943739198511207764444	7777889012197789749887558289275598088403855589940776666 88888999992838393445713772224444445568771111111111111111111111111111111111

i.

Appendix B.9. Theoretical predictions by Solution 4 under load function 7.

 $\begin{array}{c} 11550.000\\ 115640.000\\ 112640.000\\ 112640.000\\ 112700.000\\$

DEFINT I-N OPEN "A; INDATA.001" FOR INPUT AS #1 INPUT "Nc. of points and initial value ?";N,B DIM X(N),Y(N),Z(N),V(N) FOK (=1 TO N INPUT #1,X(I),Y(I) NEXT I LPRINT X(N),Y(N) =0 $100 \\ 110 \\ 120 \\ 130 \\ 130$ L=0 REM "Make summations." SUM1=0 | SUM2=0 | SUM3=0 SUM4=0 SUM4=0 SUM4=0 SUM5=0 SUM5=0 SUM5=0 FOR I=1 TO N Z(I)=X(I)^8 SUM1=SUM1+Z(I) SUM2=SUM2+(Z(I)^2) SUM3=SUM3+Y(I) SUM4=SUM4+Y(I)*Z(I) NEXT I REM "Get coefficient A and C." C=(SUM3+(SUM1+Y(I))/SUM2)/(N-(SUM1^2)/SUM2) A=SUM4/SUM2-(SUM1/SUM2)*C REM "Check if dY/d8=0.0." FOR I=1 TO N V(I)=LOG(X(I)) SUM5=SUM5+(Y(I)*Z(I)-A*(Z(I)^2)-C*Z(I))*V(I) NEXT I NEXT I SUM5=SUM5+(Y(I)*Z(I)-A*(Z(I)^2)-C*Z(I))*V(I) NEXT I REM "If dY/dB=0.0, print answers." DY1=-2!*(A)*SUM5 IF ABS(DY1)<1E-10 THEN GOTO 550 REM "if dY/dB is not equal to 0.0, repeat _" REM "Newton's method." FUR I=1 TO N SUM6=SUM6+(Y(I)*Z(I)-2!*A*(Z(I)^2)-C*Z(I))*(V(I)^2) NEXT I DY2=-2!*(A)*SUM6 M=L/10 IF L=M*10 THEN LFRINT A,B,C,DY1,L L=L+1 44445555555555 E=L+1 B=B-(DY1/DY2) GDTO 190 LPRINT A,8,C END

Appendix C.1. Program for nonlinear least squares curve fitting.

REM "Creep slip by Solution 2 under loading function 5." OPEN "A:CRVARY.505" FOR OUTPUT AS #1 DIM T(12) N=12 FOR I=1 TO N INPUT "Time interval";T(I) NEXT I REM "Input some constants." DI=2880! HT=1440! DF=20! PT=0! 100 REM 110 $\frac{120}{130}$ 140150 $\frac{160}{170}$ 11122223456789 PT=0! PL00=100! A=.35 REM "Define some functions for 5-E." DEF FNFP(PL)=4.8121E-10*(PL^4.9026) DEF FNFF100(TM)=1.844+.34536*(1!-EXP((-.0005148)*TM)) SEE EVETTOO(TM)= 14513*(TM^.35) NREm 'Det Ine some functions for 3.c.' DEF FMFP(FL)=4.8121E-10*(FL^4, 9026) DEF FMFP(FL)=4.8121E-10*(FL^4, 9026) DEF FMFT100(TM)=.14513*(TM^.35) REM 'Define some functions for M5-E." EM=-.0003259 AM=5.07464E-05 BM=3.1916*-10 DEF FMD0(PL,TM)=BM*(PL^2.28375)+(PL*.003012)*(1!-EXP(EN*TM)) DEF FMD0(PL,TM)=BM*(PL^2.28375)+(PL*.003012)*(1!-EXP(EN*TM)) DEF FMD0(PL,TM)=BM*(PL^2.28375)+(PL*.003012)*(1!-EXP(EN*TM)) DEF FMD0(PL,TM)=BM*(PL^2.28375)+(PL*.003012)*(1!-EXP(EN*TM)) DEF FMF2(TM)=(.272-.0202*(TM^.3)-.0153(TM^.17)+0.0455*(TM^.12))*.001 DEF FMF2(TM)=(.272-.0202*(TM^.3)+.2354*(TM^.17)+0.045*(TM^.12))*.001 DEF FMF2(TM)=(.272-.0202*(TM^.3)+.2354*(TM^.17)+.0015*(TM^.12))*.001 DEF FMF2(TM)=(.0003*(0009*(TM^.3)-.0013*(TM^.17)+.0015*(TM^.12))*.001 DEF FMF5(TM)=(.103780+.00375*(TM^A))*.001 DEF FMF5(TM)=(.101777430*PL-1.80786*(PL^2)+EMF3(TM)*(PL^3))*.001 DEF FM00(PL]=(.57.7430*PL-1.80786*(PL^2)+.015395*(PL^3))*.001 DEF FM00(PL]=(.57.7430*PL-1.80786*(PL^2)+.015395*(PL^3))*.001 DEF FM00(PL]=(.57.7430*PL-1.80786*(PL^2)+.015395*(PL^3))*.001 DEF FM00(PL]=(.57.7430*PL-1.80786*(PL^2)+.015395*(PL^3))*.001 DEF FM00(PL]=(.57.7430*PL-1.80786*(PL^2)+.015395*(PL^3))*.001 DEF FM00(PL]=(.57.7430*PL-1.80786*(PL^2))*.001 DEF FM00(PL]=(.57.7430*PL+1.80786*(PL^2))*.001 DEF FM00(PL]=(.57.7430*PL+1.80786*(PL^2))*.001 DEF FM00(PL]=(.57.7430*PL+1.80786*(PL^2))*.001 DEF FM00(PL]=(.57.7430*PL+1.80786*(PL^2))*.001 DEF FM00(PL]=(.57.7430*PL+1.80786*(PL^2))*.001 DEF FM00(PL]=(.57.7430*PL+1.80786*(PL^2))*.001 DEF FM00(PL]=(.57.7430*PL+1.80786*(PL^2))*.00 300 360 370 370 370 370 400 4100 4100 4100 4100 4100 460 580 590 600 640 350

Appendix C.2. Program for the creep predicted by Solution 2 under load function 5.

SEM "Compute the creep slip under the second step of loads." FOR "=1 TO N FOR "=1 TO N FOR "=1 TO N TIME = FINEFLOOTINT) VE = NDE(FLOD.TIMI) VE = NDE(FLOD.TIMI) VE = NDE(FLOD.TIMI) VE = NDE(FLOD.TIMI) SALESS - VE SV2=654-EV VV=654-EV VV=654-

REM "Creep slip by Solution 2 under loading function 6." OPEN "A:CRVARY.606" FOR OUTPUT AS #1 DEFINT I-N DIM T(12) 100 DIM T(12) N=12 FOR (=1 TO N INPUT "Time interval";T(I) NEXT I REM "Input some constants." DT=2880! HT=1440! DP=20! PT=0! PT=0! FL00=120! A=.35 101=1 1423=1 1023=1 0EFF FNFF(PL)=4.8121E-10*(PL^4.9026) DEFF FNFF(0CITM)=.5120001+.17014*(11-EXP((-.0002781)*TM)) DEFF FNFF80(TM)=1.2134+.340536*(11-EXP((-.0003523)*TM)) DEFF FNFF80(TM)=2.343+.340536*(11-EXP((-.0003385)*TM)) DEFF FNFF120(TM)=2.343+.34717*(11-EXP((-.0003385)*TM)) DEFF FNFF120(TM)=2.343+.34717*(11-EXP((-.0003385)*TM)) DEFF FNFF120(TM)=2.3435*(TM^4.31-EXP((-.0003812)*(11-EXP(EN*TM))) DEFF FNFF120(TM)=5.3155*(TM^4.31-EXP((-.0003812)*(11-EXP(EN*TM))) DEFF FNFF120(TM)=5.3155*(TM^4.31-EXP((-.0003812)*(11-EXP(EN*TM))) DEFF FNFF120(TM)=5.3155*(TM^4.31-EXP((-.0003812)*(11-EXP(EN*TM))) DEFF FNDE(PL,TM)=BN*(PL^2D*(TM^4.31-2003812)*(11-EXP(EN*TM))) DEFF FNDE(PL,TM)=BN*(PL^2D*(TM^4.31-2004)*(TM^4.17))+.0455*(TM^4.12))*.001 DEFF FNDE(PL,TM)=EN*(PL^2D*(TM^4.31+23541*(TM^4.17))-.2412*(TM^4.12))*.001 DEFF FNF2(TM)=(-.0003+.00009*(TM^4.3)+.23541*(TM^4.17))+.0015*(TM^4.12))*.001 DEFF FNF3(TM)=(-.0003+.00009*(TM^4.3)+.20113*(TM^4.17))+.0015*(TM^4.12))*.001 DEFF FNF3(TM)=(-.0003+.00009*(TM^4.3)+.20113*(TM^4.17))+.0015*(TM^4.12))*.001 DEFF FNF3(TM)=(-.0003+.00009*(TM^4.3)+.2001 DEFF FNF3(TM)=(-.0003+.00009*(TM^4.3)+.2001 DEFF FNF3(TM)=(-.0003+.00009*(TM^4.3))*.001 DEFF FNF3(TM)=(-.000000*(TM^4.3))*.001 DEFF FNF3(TM)=(-.00000*(TM^4.3))*.001 DEFF FN73(TM)=(-.00000*(TM^4.3))*.001 DEFF FN73(TM)=(-.00000*(TM^4.3))*.001 DEFF FN73(TM)=(-.00000*(TM^4.3))*.001 DEFF FN73(TM)=(-.00000*(TM^4.3))*.001 DEFF FN73(TM)=(-.00000*(TM^4.3))*.001 DEFF FN73(TM)=(-.00000*(TM^4.3)) 102=1 103=1 310 320 330 340 340 350 370 370 370 370 370 40004420044420 7450000 74500000 745000000 SME=VE+VM \$V1=EV+VV \$V2=EV+VM \$V3=VE+VV PRIMT_H1, USING "#######.##":TIME:\$S5F;SME:SV1;SV2:SV3 400 610 620 630 156123 1756123 17560223 640 550 5600 5600 5600 5700 590

REM "Compute the creep slip under the second step of losos." F1=TIME F3 T=1 TO N TIME=F1+UL SGN =S1=FNPF H2C(TIM1) UP=FNDF(PLOD:TIM1) UP=FNDF(PLOD:TIM1) UP=FNDF(PLOD:TIM1) SGN =S3-EV

REM "Creep slip by Solution 2 under loading function 7." 100 DEFIN(I-N DIM T(12) N=12 FOR I=1 TO N INPUT "Time interval";T(I) $\begin{array}{c}110\\120\\130\end{array}$ $\frac{140}{150}$ 133 MEVE _ 1 funct from the functions for field 170 REM "Input some constants." 190 HT=1440! 200 DF=20! 210 FT=0! 230 A= .35 240 CF=.465 240 CF=.765 240 DEF FNFF(PL)=4.8121E=10*(FL44.9026) 270 DEF FNFF(PL)=4.8121E=10*(FL4.9026) 270 DEF FNFF(PL)=1/1+.0338*(1)-EXP((-.0002781)*TM)) 280 DEF FNFF(TM)=.3/1+.12676*(1)-EXP((-.0002781)*TM)) 290 DEF FNFF(0(TM)=.3/24.120001+.110*(1)-EXP((-.0001523)*TM)) 210 DEF FNFF60(TM)=1.213*.40501*(1)-EXP((-.0001523)*TM)) 210 DEF FNFF60(TM)=1.213*.40501*(1)-EXP((-.0001523)*TM)) 210 DEF FNFF60(TM)=1.213*.44533*(1)-EXP((-.0001523)*TM)) 210 DEF FNFF10(TM)=2.3843*.42717*(1)-EXP((-.0001523)*TM)) 320 DEF FNFF10(TM)=1.844*.34533*(1)-EXP((-.0003385)*TM)) 320 DEF FNFF120(TM)=2.343*.42717*(1)-EXP((-.0003385)*TM)) 320 DEF FNF120(TM)=2.343*.42717*(1)-EXP((-.0003385)*TM)) 320 DEF FNF120(TM)=2.343*.42717*(1)-EXP((-.0003385)*TM)) 320 DEF FNF120(TM)=2.343*.42717*(1)-EXP((-.0003385)*TM)) 320 DEF FNF120(TM)=2.343*.42717*(1)-EXP((-.0003385)*TM)) 320 DEF FNF120(TM)=2.343*.42717*(1)-EXP((-.0003812)*(1)-EXP(EN*TM)) 320 DEF FNF120(TM)=2.343*.42717*(1)-EXP((-.0003812)*(1)-EXP(EN*TM)) 320 DEF FNDE(DL,TM)=AN*(PL^2.238375)+(PL*.003812)*(1)-EXP(EN*TM)) 320 DEF FNDE(DL,TM)=AN*(PL^2.238375)+(PL*.003812)*(1)-EXP(EN*TM)) 320 DEF FNDE(DL,TM)=AN*(PL^2.238375)+(PL*.003812)*(1)-EXP(EN*TM)) 320 DEF FNDE(DL,TM)=AN*(PL^2.238375)+(PL*.003812)*(1)-EXP(EN*TM)) 320 DEF FNDE(DL,TM)=(-.00028+.0000072*(TM^3.3)-9.41530(TM^3.17)+9.0455*((M^3.12))*.001 320 DEF FNF3(TM)=(-.00028+.0000072*(TM^3.3)-.001308*(TM^3.17)+.001508*(TM^3.12))*.001 320 DEF FNF3(TM)=(-.00028+.0000072*(TM^3.3)-.001308*(TM^3.17)+.001508*(TM^3.12))*.001 320 DEF FNF3(TM)=(-.00028+.0000072*(TM^3.3)-.001308*(TM^3.17)+.001508*(TM^3.12))*.001 320 DEF ENE2(TM)=(-.00028+.0000072*(TM $\frac{160}{170}$ NEXT I REM "Input some constants." DEF FNF4(TM) = (57.74367+.025453*(TM^A))*.001 DEF FNF5(TM)=(-1.80786-.019755*(TM^A))*.001 DEF FNF6(TM)=(.015399+.00034*(TM^A))*.001 DEF FNVE(PL.TM)=FNF1(TM)*PL+FNF2(TM)*(PL^2)+FNF3(TM)*(PL^3) DEF FNVU(PL.TM)=FNF4(TM)*PL+FNF5(TM)*(PL^2)+FNF6(TM)*(PL^3) DEF FNVG(PL)=(57.74367*FL-1.80786*(PL^2)+.015399*(PL^3))*.001 REM "Compute the creep slip under the first step of loads." LPRINT "TIME 5-E MS-E V-VE1 V-VE2 V-VE3" FOR I=1 TO N TIME=FT+T(I) SSF=FNFF60(TIME)+FNFT60(TIME)+FP1 VM=FNDV(PLOD,TIME)+FNFT60(TIME)+FP1 VE=FNDE(PLOD,TIME)+FNFT60(TIME)+FP1 VE=FNVE(PLOD,TIME) EV=FNVU(PLOD,TIME) SME=VE+VM 500 610 620 630 640 650 660 V-FRVV(FC00,TIME) SME=VE+VM SV1=EV+VM SV2=EV+VM SV3=VE+VV LPRINT USING "+######.##";TIME,S5F,SME,SV1,SV2,3V3 NEXT I 670 670 690 700 710 720

Appendix C.4. Program for the creep predicted by Solution 2 under load function 7.

REM "Compute the creep slip under the second step of loads." PI=TIME P(1=PLOD+OP) = P(1=PLOD+OP)730 740 750 268 778 780 780 780 έóŏ 810 820 830 840 850 85000 86700 89900 89900 NEXT I REM_"Compute the creep slip under the third step of lasds." 900 REM "Compute the creep slip under the third step of losds." 910 PT=TIME 920 PL2=PL1+DP 930 FOR I=1 TO N 940 TIME=PT+T(I) 950 TIM1=TIME-OT 960 TIM2=TIME-2*OT 970 S5F=ENFF30(TIME)+FNFF80(TIM1)-FNFF60(TIM1)+FNFF100(TIM2)-FNFF80(TIM2)+FNFT30 (DT)+FNFT80(DT)+FNFT100(TIM2)+FP3 980 VM=FNFF(PL2)+FNDV(PL0D,DT)+FNDV(PL1,DT)+FNDV(PL2,TIM2) 980 VM=FNFP(PL2)+FNDV(PL0D,DT)+FNDV(PL1,DT)+FNDV(PL2,TIM2) 970 VE=FNDE(PL0D,TIME)+FNDE(PL1,TIM1)-FNDE(PL0D,TIM1)+FNDE(FL2,TIM2)-FNVE(FL1,TIM2) 1000 EV=FNVE(PL0D,TIME)+FNVE(PL1,TIM1)-FNVE(PL0D,TIM1)+FNVE(PL2,TIM2)-FNVE(FL1,TIM2) IUDU EV=FNVE(PLUD,IIME)+FNVE(PL1,TIM1)-FNVE(PL0D,TIM1)+FNVE(PL2,TIM2)-FNVE(FL1,T IM2) IM2) U005 SVE=VE+VM I005 SVI=CV+VM I005 SVI=SVI+VM I005 SVI=SVI+VM I005 SVI=SVI+VM I005 SVI=SVI I005 SVI I005

 $1350 \\ 1360 \\ 1370 \\ 1390 \\ 1300 \\ 1300 \\ 1300 \\ 1300 \\ 1300 \\ 1300 \\ 1300 \\ 1300 \\ 1300 \\ 1000 \\$ 1460 1460 1470 1470 1300 1210 1620 1630 $\frac{1640}{1650}$ 1660 1670 1680 1680 1700 1700 SEPESI-ENPERTINA)+FNFC(TIMS)-FNFE(TIMS)+FNFFS(TIMS)-FNFC(TIM6)-FNFC(TIM6))
1710 VE=(FNVE(PLOD,TIM4)+FNVE(PLOD,TIM5)+FNVE(PLOD,TIM5)+FNVE(PLOD,TIM6)-FNVE(PLOD,TIM6)*1)+
1720 VED=(FNVE(PLOD,TIM4)+FNVE(PLOD,TIM5)+2!-FNVE(PLOD,TIM5)+-VF(PLOD,TIM6)*1)+
1730 SME=52-VE
1730 SME=52-VE
1730 SV2=84-VED
1740 SV1=83-VED
1740 SV1=83-VE ĜOT O END 1990 2000

REM "Creep slip by Solution 2 u OPEN "A:CRVARY.808" FOR OUTPUT DEFINT I-N DIM T(12) under loading function 8." 100 110120130140AS #1 N=12 FOR I=1 TO N INPUT_"Time interval";T(I) ĩ50 160 170 NEAT I nout some constants." REM "Input some constants." HT=1440! DP=20! DDP=40! F)=0! d = 47.0° prove 440 450 460 470 480 490 600 610 620 630 640 650 3300 3670 3670 3670 300 3670 300 7720 720

Appendix C.5. Program for the creep predicted by Solution 2 under load function 8.

750 7760 7780 7780 7780 REM "Compute the creep slip under the second step of loads." FI=TIME PL1=PLOD+OP FOR 1=1 TO N FUR I=1 TO R TIME=PT+T(I) TIM1=TIME-DT S5F=FNFF80(TIME)+FNFF100(TIM1)-FNFF80(TIM1)+FNFT80(DT)+FNFT100(TIM1)+FP3 VM=FNFF(PL1)+FNDV(PLDD,DT)+FNDV(FL1,TIM1) VE=FNDE(PL0D,TIME)+FNDE(PL1,TIM1)-FNDE(FL0D,TIM1) EV=FNVE(PL0D,TIME)+FNVE(FL1,TIM1)-FNVE(PL0D,TIM1) UV=FNVV(PL0C,DT)+FNVV(PL1,TIM1)-FNVG(PL0D) CMC=UCTUM 820 830 910 920 930 1110 1120 1130

REM "Compute the creep slip under the fifth step of loads."
PT=TIME
S1=SSF
S2=S04
F07
F1=F1#
F1=T1ME_4+01
S1=S1=FNFC(IIM4)
UE=FNDE(ODP,TIM4)
U=FNDE(ODP,TIM4)
U=FNDE(ODP,TIM4)
U=FNDE(ODP,TIM4)
S1=S1=U
S1=S3=U
S1=S3=U
S1=S3=U
S1=S3=U
S1=S1=U
S $\begin{array}{r} 1640\\ 1650\\ 1660\\ 1680\\ 1680\\ 1690\\ 1770\\ 17710\\ 1720 \end{array}$ END

REM "Creep slip by Solution 2 under loading function 7." DEFINE I-N DIM T(12) N=12 FOR I=1 TO N INPUT "Time interval";T(I) 100 $\begin{array}{c}
 110 \\
 120 \\
 130
 \end{array}$ $\frac{140}{150}$ 130 170 NEXT I REM "Input some constants." DT=2080 HT=1440! DP=20! PT=0! 180 190 $\frac{200}{210}$ $\frac{200}{220}$ DP=20* PI=0: PI=0: PLOD=60! A=:35 REM "Define some functions for 5-E;" DEF FNFFP(FL)=4.8121E-10*(FL^4.9026) DEF FNFFP(TM)=:171+.06338*(1!-EXP((-.0002981)*TM)) DEF FNFFP(TM)=:5120001+.19014*(1'-EXP((-.0002981)*TM)) DEF FNFF80(TM)=FNFF80(TM)+.0004536*(TM^.57)+.294 DEF FNFF80(TM)=FNFF80(TM)+.0004536*(TM^.41)+.051 DEF FNFF100(TM)=FNFF80(TM)+.028838*(TM^.41)+.051 DEF FNFF100(TM)=FNFF80(TM)+.14513*(TM^.35)+3.095 DEF FNFF100(TM)=2.843+.42717*(1!-EXP((-.0003385)*TM)) DEF FNFF120(TM)=2.843+.42717*(1!-EXP((-.0003385)*TM)) DEF FNFF120(TM)=2.843+.42717*(1!-EXP((-.0003385)*TM)) DEF FNFF120(TM)=FNFF120(TM)+.51365*(TM^.3)+7.103 REM "Define some functions for MS-E." EN=-.0003254 AN=5.07464E-03 SN=0.1916E-10 CN=4.3212 DEF FNDS(PL,TM)=AN*(PL^2.28375)+(PL*.003812)*(1!-EXP(EN*TM)) DEF FNDS(PL,TM)=EN*(PL^CN)*(TM^A)+FNFP(PL) DEF FNDS(PL,TM)=TM*((BNFL4CN)*(TM^A)+FNFP(PL)) DEF FNF1(TM)=(-2,37+,2020*(TM^A,3)+.23538*(TM^A,17)+.2412*(TM^A,12))*.001 DEF FNF4(TM)=(57,74367+.025453*(TM^A))*.201 u_5)+(PL*.003812)*(1!-EXP(EN*TM)) .ude(EN*FL*CN)^(1/A); .ude(

Appendix C.6. Program for the creep predicted by Solution 1 under load function 7.

REM "Compute the creep slip under the second step of loads." PI=TIME PL1=PLOD+OP FOR I=1 TO N TIME=PT+T(I) TIM1=TIME-OT SSF=FNFT60(TIME)+FNFT80(TIM1)-FNFT60(TIM1) VM=FNFP(PL1)+((FNDS(PLOD,DT)+FNDS(PL1,TIM1))^A) VE=FNDE(PL0D,TIME)+FNDE(PL1,TIM1)-FNDE(PL0D,TIM1) EV=FNVE(PL0D,TIME)+FNVE(PL1,TIM1)-FNUE(PL0D,TIM1) EV=FNVE(PL0D,TIME)+FNVE(PL1,TIM1)-FNUE(PL0D,TIM1) VU=FNVG(PL1)+((FNVH(PL0D,OT)+FNVH(PL1,TIM1))^A) SME=VE+VM 780 790 SME=VE+VM SV1=EV+VM SV2=EV+VM SV3=VE+VV LPRINT USING "+########";TIME,S5F,SME,SV1,SV2,SV3 NEXT I REM "Compute the creep slip under the third step of loads." 800 810 820 830 3403400 8500 8700 8700 8700 8700 8700 REM "Compute the creep slip under the third step of loads." PI=TIME PL2=PL1+DP FOR I=1]0 N TIME=PT+T(I) TIM1=(IME-DT TIM2=TIME-2*DT S5F=FNFT60(TIME)+FNFT80(TIM1)-FNFT60(TIM1)+FNFT100(TIM2)-FNFT80(TIM2) VM=FNFP(PL2)+((FND5(PL0),DT)+FND5(PL1,DT)+FND5(PL2,TIM2))^A) VE=FNDE(PL00,TIME)+FNDE(PL1,TIM1)-FNDE(PL00,TIM1)+FNDE(PL2,TIM2)-FNDE(PL1,TI 910 920 230 940 950 M2) 960 EV=FNVE(PLOD,TIME)+FNVE(PL1,TIM1)-FNVE(PLOD,TIM1)+FNVE(PL2,TIM2)-FNVE(PL1,TI M2) 970 VU=FNVG(PL2)+((FNVH(PLOD,DT)+FNVH(PL1,DT)+FNVH(PL2,TIM2))^A) 980 SME=VE+VM 990 SV1=EV+VV 400 SV1=EV+VV 980 SML=vL+vM 970 SV1=EV+VM 1010 SV3=VE+VV 1020 LPRINT USING "+#####.##";TIME,S5F,SME,SV1,SV2,SV3 1030 PEXT I 1040 REM "Compute the creep slip under the forth step of loads." 1050 PT=TIME 1060 FI=TIME 1060 FI=TIME-DT 1070 FON I=1 10 N 1080 TIME=PT+T(I) 1090 TIME=TIME-DT 1100 TIM2=TIME-2*DT 1100 TIM3=TIME-3*DT 1100 TIM3=TIME-3*DT 1120 A5F=FNFT60(TIM2)+FNFT80(TIM1)+FNFT60(TIM2)-FNFT80(TIM2) 1130 S5F=FNFT60(TIM3)-FNFT100(TIM3) 1140 VM=FNFP(PL3)+t(FND5(PLD,OT)+FND5(PL1,DT)+FND5(PL2,DT)+FND5(PL3,TIM3))^A) 1150 VE=FNDE(PL0D,TIME)+FNDE(PL1,TIM1)-FNVE(PL0D,TIM1)+FNVE(PL2,TIM2)-FNDE(PL1,TIM2)+FNDE(PL3,TIM3)-FNDE(PL2,TIM3) 1150 AEV=FNVE(PL0D,TIME)+FNVE(PL1,TIM1)-FNVE(PL0D,TIM1)+FNVE(PL2,TIM2)-FNVE(PL1,TIM2)+FNDE(PL3,TIM3)-FNDE(PL2,TIM3) 1160 AEV=FNVE(PL3)+t((FNVE(PL3,TIM3)-FNVE(PL2,TIM3))^A) 1170 EV=AEV+FNVE(PL3)+t((FNVH(PL0D,DT)+FNVH(PL1,DT)+FNVH(PL2,DT)++NVH(PL3,TIM3))^A) 1180 VU=FNVG(PL3)+t((FNVH(PL0D,DT)+FNVH(PL1,DT)+FNVH(PL2,DT)++NVH(PL3,TIM3))^A) 1190 SME=VE+VM VV=FNVG(PL3)+((FNVH(FL0D;DT)+FNVH(FL1,DT)+FNVH(PL2,DT)+FNVH SME=VE+VM SV1=EV+VV SV2=EV+VM SV3=VE+VV LPRINT USING "+##########";TIME,S5F,SME,SV1,SV2,SV3 NEXT I REM "Compute the creep slip under the first step of loads." PT=TIME S1=S5F S2=SME S3=SV1 S3=SV1 S4=SV2 S5=SV3

FOR I=1 TO N TIME=PT+T(I) (IM4=TIME-4*DI S5F=S1-FNEE(IIM4)*,765 VED=FNVE(PLOD,TIM4)*,765 VED=FNVE(PLOD,TIM4)*,765/3: SME=S2-VE SV1=S3-VED SV2=S4-VED SV3=S5-VE LPRINT USING "+#####.##";TIME.S5F,SME.SV1,SV2,SV3 DEVT LYRINT USING "+######.##";TIME.SSF.SME.SV1.SV2.SV3 NEXT I REM "Compute the creep slip under the sixth step of loads." PT=TIME DDP=2!*DP FOR [=1 TO N TIME=PT+T(I) TIM5=TIME-5*DT SSF=S1-(FNF8(TIM4)+FNF8(TIM5))*.765 VE=(FNDE(DP.TIM4)+FNDE(DDP.TIM5)-FNDE(DP.TIM5))*.765 VED=(FNVE(PLOD.TIM4)+FNVE(PLOD.TIM5))*.765/3! SME=S2-VE SV1=S3-VED SV2=S4-VED SV3=S5-VE LPRINT USING "+#######";TIME.SSF.SME.SV1.SV2.SV3 NEXT I REM "Compute the creep slip under the seventh step of loads." PI=TIME REM "Compute the creep slip under the seventh step of loads," PT=TIME FOR I=1 TO N TIME=PT+T(I) TIM4=TIME-4*DT IIM5=TIME-5*DT TIM6=TIME-6*DT S5F=S1-(FNFB(TIM4)+FNFB(TIM5)+FNFB(TIM6))*.765 VE=(FNDE(DP,TIM4)+FNDE(DDP,TIM5)-FNDE(DP,TIM5)+FNDE(PLOD,TIM6)-FNDE(DDP,TIM 745 161001610016200163001640016500 $1330 \\ 1670 \\ 1680$ 1370 SEF=S1=(FNPB(TIM4)+FNPB(TIM5)+FNPB(TIM6))*.765 1680 VE=(FNVE(PLOD,TIM4)+FNVE(DDP,TIM5)+FNVE(OP,TIM5)+FNDE(PLOD,TIM6))*.765/3! 1700 SVI=S3-VED 1710 SVI=S3-VED 1720 SVJ=S4-VED 1720 SVJ=S4-VED 1730 SVJ=S5-VE 1730 SVJ=S5-VE 1730 SVJ=S5-VE 1730 SVJ=S5-VE 1730 FOR I=1 TO N 1740 LPRINT USING "+#####.##"fIME.S5F.SME.SV1.SV2.SV3 1750 NEXT I 1750 FOR I=1 TO N 1790 TIM4=TIME-5*D1 1810 TIM5=TIME-6*D1 1830 TIM5=TIME-6*D1 1830 TIM5=TIME-6*D1 1850 VIE (FNPB(TIM4)+FNPB(TIM5)+FNFB(TIM6)+FNFF120(TIM7)-FNFF60(TIM7))*.765 1850 VE=(FNDE(DP,TIM4)+FNDE(DDP,TIM5)-FNDE(DP,TIM5)+FNDE(FLOD,TIM6)-FNDE(DDP,TIM 6)+FNDE(PL3.TIM7)-FNDE(PLOD,TIM5)+FNVE(PLOD,TIM6))/3!+FNVE(PL3.TIM7)-FN 1860 VE=S2-VE 1860 SV=S4-VED 1870 SME=S2-VE 1860 SV=S4-VED 1870 SME=S2-VE 1880 SV=S4-VED 1890 SV3=S5-VE 1880 SV=S4-VED 1900 SV3=S5-VE 1880 THE=2100 THE SF.SME.SV1.SV2.SV3 1890 FOR I=100 THE ST.SME 1890 FOR I=100 THE SF.SME 1890 SV=S4-VED 1990 SV3=S5-VE 1990 SV3=S5-VE 1990 THE=1100 THE ST.SME 1990 SV3=S5-VE 1990 THE=1100 THE ST.SME 1990 SV3=S5-VE 1990 SV3=S5-VE 1990 SV3=S5-VE 1990 THE=1100 THEN GOTO 1990 1990 NEXT I 1995 TF TIME>30240 THEN GOTO 1970 NEXT I 23040 THEN GOTO 1940 TIME=TIME+HT IF TIME>30240 THEN GOTO 1970 GOTO 1800 END 940 950 960 970 Series .

Appendix C.7. Program for the creep predicted by Solution 3 under load function 7.

```
"Creep slip by Solution 2 under loading function 7."
"Load function is expressed as a continuous function which is sum of
Heaviside unit step functions."
N_"A:CRVARY.D07" FOR OUTPUT AS #1
    HI=1440:
DP=20!
DDP=40!
PloD=60!
A=.35
CE=.765
KEM "Define some functions for M5-E."
EN=-.0003254
E2=.003812
AN=5.07464E-05
8N=3.1916E-10
CN=4.8121E-10
CN=4.8121E-10
CN=4.8121E-10
CN=4.8121E-10
DEF FNDI(PL)=AN*(PL^AM)
DEF FNDI(PL)=AN*(PL^AM)
DEF FNDE(PL)=AN*AM*((PL+.5*DP)^(AM-1!))*0P
DEF FNDE(PL)=AN*AM*((PL+.5*DP)^(AM-1!))*0P
DEF FNDV(PL,TM)=PL*EXP(EN*TM)
DEF FNDV(PL,TM)=BN*(PL^BM)*(TM^A)
DEF FNDV(PL,TM)=BN*(PL^BM)*(TM^A)
DEF FNDP(PL)=CN*CM*((PL+.5*DP)^(CM-1!))*0P
PI=CN*(PL0D^CM)
REM "Define some functions for U-VE1."
    310
320
330
    2340
3340
3350
3370
     380
390
     400
    410
420
                                           "Define same functions for U-VE1."
FNF1(TM)=(-7.34+1.102*(TM^.3)-9.415301*(TM^.17)+9.0455*(TM^.12))*.001
FNF2(TM)=(.272-.0202*(TM^.3)+.23538*(TM^.17)-.2412*(TM^.12))*.001
FNF3(TM)=(.00028+.000092*(TM^.3)-.001308*(TM^.17)+.001508*(TM^.12))*.00
                      -ROAD
     430
    44ð
450
     43Ō
                    DEF FNF4(TM)=(57.74367+.025453*(TM^A))*.001
DEF FNF5(TM)=(-1.80786-.019755*(TM^A))*.001
DEF FNF5(TM)=(.015399+.00034*(TM^A))*.001
DEF FNV6(FL)=(.015399+.00034*(TM^A))*.001
DEF FNV0(PL,TM)=2!*FNF1(TM)*FL*FNF2(TM)*(PL^2)+FNF3(TM)*(PL^A)
DEF FNV0(PL,TM)=2!*FNF2(TM)*FL*DP+3!*FNF3(TM)*(PL^A)*(PL^A)*.001
DEF FNV0(PL,TM)=(.025453*PL-.019755*(PL^2)+.00034*(PL^A))*.001
DEF FNV0(PL)=(.77.74367*PL-1.80786*(PL^2)+.00034*(PL^A))*.001
DEF FNV0(FL)=(.2!*1.80786*PL*DP+3!*.015399*PL*(PL+DP)*DF)*.001
DEF FNV0(FL)=(.2!*1.80786*PL*DP+3!*.015399*PL*(PL+DP)*DF)*.001
REM "Compute the creep slip under the first step of loads."
FOR I=1 TO N
TIME=FT+T(I)
Vm=FNDV(PL0D.TIME)+FI
VE=FNDI(PL0D)*PL0D*E2-FNDD(PL0D.TIME)*E2
EV=FNV0(PL0D.TIME)+FNVG(PL0D)
SME=VE+VM
    470
   44555555555
   560
570
   580
590
   600
610
620
630
                    VV=*NVV(PLOG,TIME)+FNVG(PLOD)
SME=VE+VM
SVI=EV+VV
SV2=EV+VM
SV3=VE+VV
PRINT #1. UGING "#######.##":TIME:SME;SV1:SV2:SV3
NEXT I
  640
650
660
             Û
   67
```

SEM "Compute the creep slip under the second step of loads." Pl=TIME PL1=PL0D+DP FOR I=1 TO N TIME=PT+T(I) IIM1=(IME=DT VM=PI+FNDP(PLOD)+FNDV(PLOD,DT)+FNDV(PL1.TIM1) VE=FNDI(FLOD)+FNDE(PLOD,DT)+FNDD(PLOD,TIME)+FNDD(DP,TIM1))*E2 EV=FNVE(PLOD)+FNDE(PLOD)+PL1*E2-.FNDD(PLOD,TIME)+FNVD(DP,TIM1))*E2 EV=FNVE(PLOD,TIME)+FNVE(DP,TIM1)+FNVD(PLOD,TIM1) Wv=FNVV(PLOD,DT)+FNVV(PL1,TIM1)+FNVG(FLOD)+FNVG(DP)+FNVP(PLUD) SMF=UF+VM 680 690 200 710 720 730 749 25¥ 7807780 SME=VE+VM SV1=EV+VV SV2=EV+VM SV3=VE+VV 30Ô 810 820 830 PRINT #1, USING "#########;TIME;SME;SV1;SV2;SV3 NEXT I REM_"Compute the creep slip under the third step NEXT I REM "Compute the creep slip under the third step of loads." PI=TIME PL2=FL1+DP FOR I=1 IO N TIME=PT+T(I) TIME=TIME-OT TIME=TIME-OT TIM2=TIME-OT TIM2=TIME-2*DT VM=PI+FNDP(FL0D)+FNDP(PL1)+FNDV(PL0D,OT)+FNDV(PL1,OT)+FNDV(PL2,TIM2) VM=PI+FNDP(FL0D)+FNDP(PL1)+FNDV(PL0D,OT)+FNDV(PL1,OT)+FNDV(PL2,TIM2) VM=PI+FNDP(FL0D)+FNDP(PL1)+FNDV(PL0D,OT)+FNDV(PL1,OT)+FNDV(PL2,TIM2) VM=PI+FNDP(FL0D)+FNDP(PL1)+FNDV(PL0D,OT)+FNDV(PL1,OT)+FNDV(PL2,TIM2) 900 TIM2=TIME-2*DT 910 VM=PI+FNDP(PL0D)+FNDP(PL1)+FNDV(PL0D,DT)+FNDV(PL1,DT)+FNDV(PL2,T1m2) 920 VE=FNDI(PL0D)+FNDE(PL0C)+FNDE(PL1)+PL2*E2-(FNDD(PL0D,TIME)+FNDD(DP,TIM1)+FND D(DP,TIM2))*E2 930 EV=FNVE(PL0D,TIME)+FNVE(DP,TIM1)+FNVE(DP,TIM2)+FNVO(PL0D,TIM1)+FNVD(PL1,TIM2) 930 EV=FNVE(PL0D,TIME)+FNVE(DP,TIM1)+FNVE(DP,TIM2)+FNVO(PL0D)+FNVG(DP)*2+FNVP(PL0) /40 VV=FNVV(PL0D,DT)+FNVV(PL1,DT)+FNVV(PL2,TIM2)+FNVG(PL0D)+FNVG(DP)*2!+FNVP<PL0 D)+FNVP(PL1) 950 SME=VE+VM 950 SV1=EV+VV 970 SV2=EV+VV 970 SV2=EV+VM 980 SV3=VE+VV 990 PRINT #1, USING "##########;TIME;SME;SV1;SV2;SV3 940 FKIN; #1, USING "######.##";[IME;SM2;SV1;SV2;SV3 1000 NEXT I 1010 REM "Compute the creep slip under the forth step of loads." 1020 PT=TIME 1030 PL3=PL2*DP 1040 FOR I=I 10 N 1050 TIME=PI+T(I) 1040 (IM1=TIME=DT 1070 TIM2=TIME=2*DT 1080 (IM1=TIME=3*DT 1080 (IM1=TIME=3*DT 1080 (IM1=TIME=3*DT 10/0 IIM2=TIME-3*DT 1080 IIM3=TIME-3*DT 1090 UM=PI+FNDP(PL0D)+FNDP(PL1)+FNDP(PL2)+FNDV(PL0D,DT)+FNDV(PL1,DT)+FNDV(PL2,DT 1+FNDDVPL3,TIM3) 1100 UE=FNDI(PL0D)+FNDE(PL0D)+FNDE(PL1)+FNDE(PL2)+FN3*E2-(FNDD(PL0D,TIME)+FNDD(D) 7 TIM1/+FNDD(DP,TIM2)+FNDE(PL7,TIM3))*E2 1110 EV=FNVE(PL0D,TIME)+FNVE(OP,TIM1)+FNVE(DP,TIM2)+FNVE(OP,TIM3)+FNVO(PL0D,TIM1) +FNUD(PL1,TIM2)+FNUO(PL2,TIM3) 1120 WU=FNVU(PL0D)+FNUP(PL2,TIM3) 1120 WU=FNUV(PL0D)+FNUP(PL1)+FNUV(PL2,DT)+FNVU(PL3,TIM3)+FNVG(PL0D)+FNUG(PL0D)+FNUG(PL0D)+FNUG(PL2) 1130 SME=VE+VM 1140 SV1=EV+VM 1140 SV1=EV+VM 1150 SV2=EV+VM 1160 SV3=VE+VU 1170 PRINT #1, USING "##########;TIME;SME;SV1;SV2;SV3 1180 NEXT I 1190 REXT I 1190 SS3=SV3 1200 FT=TIME 1220 SS3=SV1 1230 SS3=SV3 1250 FOR I=T TO N 1260 TIME=FT#H(I) 1270 TIM4=TIME-4*DT

REM "Creep slip by Solution 2 under load function 7." REM "Load function 7 is approximated by a Fourier series." DEFINT I-N OPEN "CRVARY.F07" FOR DUTPUT AS #1 INPUT "No. of terms in the Fourier series";N DIM B(100),1(20) FOR I=1 TO 12 INPUT "Time interval";T(I) NEXT I DT=2880! TP=40320! PHI=3.14159 P0=0! P1=20! P2=40! P3=60! Sta-0! 100110120130 $\frac{140}{150}$ $\frac{160}{170}$ 180 190 r3112481248 = 0 3112481248 = 0 3112481248 = 0 ĭĎ=0 UME=0! VUE=6: Ad=60: PL=66: REM "Get coefficients of the Fourier series." GOSUE 1360 FOR I=1 TO N PREMIT " A" TAB(6) I TAB(10) "=" TAB(12) B(I) NEAT I REM "Get the creep slip under the Fourier series load function." REM "Give some parameters for theoretical models." ITME=0: B1=1:28375 B1=1:28375 B1=1:28375 B1=1:28375 B1=1:28375 B1=1:28375 B4=4.9026 AN=2.07464E=05 BA=4.9026 AN=2.07464E=10 EX=.000312 EN=.0003239 A1=.3 A2=.17 A3=.12 A4=1:7.35 F=.001 CF1=.765 REM "Define some functions. DEF FNF1(X)=(-7.34+1.102*(XA1)+9.415301*(XA2)+9.0455*(XA3))*F DEF FNF2(X)=(-222-.0202*(XA1)+.2354*(XA2)-.2412*(XA3))*F DEF FNF2(X)=(-222-.0202*(XA1)+.2354*(XA2)+.0013*(XA2)+0013*(XA3))*F DEF FNF3(X)=(-0003*.00009*(XA1)-.0013*(XA2)+.0013*(XA3))*F DEF FNF3(X)=(-2245*X-.01976*(XA2)+.0154*(XA3))*((YA3))*.062)** SI=AN*(A0AB1) V = -7.34*A0+.272*(A0A2)-.0003*(ADA3))*F ĂŎ=6Ŏ! 2444444444444 640 650 630 670 670 690 710 710

Appendix C.8. Program for the creep predicted by Solution 4 under load function 7.

0 GON_L=1.T0 12 0 IF = TTE_= T(L) = THEN (DTO 950) 0 IF = TTE_= THEN (DTO 800 ELSE DH=10) 0 IF = TTE_= THEN (DTO 800 ELSE GOTO 620) 0 IF = TTE_= THEN (DTO 800 ELSE GOTO 620) 0 IF = TTE_= THEN (DTO 800 ELSE GOTO 620) 0 IF = TTE_= THEN (DTO 800 ELSE SCOTO 620) 0 IF = TTE_= THEN (DTO 800 ELSE SCOTO 620) 0 IF = TTE_= THEN (DTO 800 ELSE SCOTO 620) 0 IF = TTE_= THEN (DTO 800 ELSE SCOTO 620) 0 IF = TTE_= Z38 750 750 750 750 790 810 820 830 850 870 890 890 910 920 930 9400 9700 9700 9700 9900 1010 1020 1030 1040 1050 1060 1070 1080 1100
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REM "This subroutine computes coefficients of the Fourier series." FOR I=1 TO N C1=COS(I*PHI/7!) C3=COS(2!*I*PHI/7!) C4=COS(3!*I*PHI/7!) C4=COS(4!*I*PHI/7!) C5=COS(5!*I*PHI/7!) C5=COS(6!*I*PHI/7!) C6=COS(6!*I*PHI/7!) B(I)=(PI*(C2-C5+C6-C1)+P2*(C3-C4+C5-C2)+P3*(C4-C3))*(-2!)/(I*PHI) NFYT T $1360 \\ 1370 \\ 1380 \\ 1390 \\ 1300 \\ 1000 \\$ NEXT I RETURN REM "This subroutine computes load at given time from the Fourier series." $\frac{1460}{1470}$ PL=0! PD=0! PD=0! FOR I=1 TO N CPT=2!*I*PHI/TP SC=SIN(CPT*TN) CC=COS(CPT*TN) PL=FL+B(I)*SC PD=PD+B(I)*CC*CPT NEXT I PL=PL+A0 RETURN RETURN REM "This subroutine evaluates integral by the trapezoidal rule." TN=TIM2 TN=TIM2 ST1=0 ST2=0 ST3=0 STS=0: STS=0! ST6=0! ST7=0! TT=TIME-TN REM "Compute load at given time from the Fourier series." $\frac{1640}{1650}$ PK=PL PK=PL GOSUB 1470 PA=(PK+PL)/2! IF ID=0 THEN GOTO 1760 PL=PM-PL IF PL<0! THEN PL=0! PD=-PD IF TN=TIM2 THEN GOTO 1770 ELSE GOTO 1790 FTD=PL FTT=PD FTI=PO IF TN=>TIN THEN GOID 1860 IF TN=>TIN THEN GO(O 1860 ST3=ST3+EXP(EN*I)*PO ST5=ST5+FNF1(TT)*PD ST5=ST5+FNF1(TT)*PD ST7=ST7+FNF3(TT)*(PL^2)*PD TN=TN+DH GOTO 1670 IF TN=TIN THEN GOTO 1870 ELSE GOTO 1890 FPO=PL FPI=PD ST1=ST1+(PL^B11)*FD ST2=ST2+PD ST3=ST3+EXP(EN*)T)*PD ST5=SI5+FNF1(TT)*PD ST6=SI5+FNF1(TT)*PD ST6=SI5+FNF2(TT)*PL*PD ST7=ST7+FNF3(TT)*(PL^2)*FD IF TN=TIN THEN GOTO 1990 ELSE GOTO 1960 1900 1910192019301940 1950

) IF ID=1 THEN GOTO 1990 ELSE GOTO 1970) ST4=ST4+BN*(PA^B2)*((DH^B3)*.062)) ST8=ST8+FNFH(PA,DH)) IF TN>TIME THEN GOTO 2010 ELSE GOTO 1670) TS=TIME-TIN) ST1=DH*(ST1-((FPO^B11)*FPI+(PL^B11)*PD)/2!)+SS1 (ST2=DH*(ST2-(FPI+PD)/2')+SS2 (ST3=DH*(ST3-(EXP(EN*TS)*FTI+EXP(EN*TT)*PD)/2!) (ST5=DH*(ST5-(FNF1(TS)*FTI+FNF1(TT)*PD)/2!) (ST5=DH*(ST5-(FNF1(TS)*FTI+FNF1(TT)*PD)/2!) (ST5=DH*(ST5-(FNF1(TS)*FTI+FNF1(TT)*PD)/2!) (ST5=DH*(ST5-(FNF3(TS)*(FTO^2)*FTI+FNF3(TT))*(FL^2)*F0)/2!) (SS2=ST2 (SS3=ST3) (SS4=ST4 (SS5=ST5) (SS4=ST4) (SS5=ST5) (SS6=ST6) (SS5=ST6) (SS5=ST6) (SS6=ST6) (SS6=ST6 1970 1980 1980