THE INDENTATION OF AN ANISOTROPIC HALF SPACE BY A RIGID PUNCH

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Eshelby et al. [1] and Stroh [2] have developed the theory of anisotropic elasticity for a three dimensional state of stress in which the stress is independent of one of the Cartesian coordinates. Various problems involving dislocations in an infinite anisotropic medium are solved in the first paper, while Stroh considers dislocation problems as well as determining the stresses round a crack subjected to an arbitrary non-uniform applied stress. In this note, which follows their treatment, we consider the problem of determining the stresses produced by the indentation of the plane surface of an anisotropic half space by a rigid punch. Problems of this type have been solved by Green and Zerna [3], Lekhnitskii [4], Brilla [5], Gallin [6] and Milne-Thomson [7] but if we take Cartesian coordinates x_1, x_2, x_3 and let the stress be independent of x_3 , then these authors all assume the $x_1 x_2$ plane to be one of elastic symmetry. The solution presented in this paper does not require this assumption so that it has a more general application than has been the case with previous solutions to problems of this type. The first part of the analysis given here is for general anisotropy, but in order to obtain a solution to the punch problem by the method of this paper, it is necessary to consider only a particular class of anisotropic materials. The indentation of such materials by a circular block is discussed in section 4 and the results are used in section 5 to examine the case when the circular block is on a transversely isotropic half space.

1. General equations

The stresses σ_{ij} are related to the elastic displacements u_k by the equations

(1)
$$\sigma_{ij} = c_{ijkl} \frac{\partial u_k}{\partial x_l}$$

where i, j, k, l = 1, 2, 3 and the convention of summing over a repeated Latin suffix is used. The elastic moduli c_{ijkl} have the symmetry properties

(2)
$$c_{ijkl} = c_{jikl} = c_{ijlk} = c_{klij}$$

On substituting (1) in the equilibrium equations

(3)
$$\frac{\partial \sigma_{ij}}{\partial x_i} = 0$$

we obtain

(4)
$$c_{ijkl}\frac{\partial^2 u_k}{\partial x_j \partial x_l} = 0$$

Now we suppose that u_k is independent of x_3 , and, following Eshelby *et al.* take

$$(5) u_k = A_k f(x_1 + px_2)$$

where f(z) is an analytic function of the complex variable z; (5) is a solution of the equations (4) provided the constant vector A_k satisfies the equations

(6)
$$(c_{i1k1} + pc_{i1k2} + pc_{i2k1} + p^2 c_{i2k2})A_k = 0$$

Values of A_k , not identically zero, can be found to satisfy these equations if p is a root of the sextic equation

(7)
$$|c_{i1k1} + pc_{i1k2} + pc_{i2k1} + p^2 c_{i2k2}| = 0$$

By applying the condition that the strain energy density

(8)
$$c_{ijkl} \frac{\partial u_i}{\partial x_j} \frac{\partial u_k}{\partial x_l}$$

should be positive for any state of strain Eshelby *et al.* have proved that equation (7) has no real root, so that the roots occur in complex conjugate pairs. The three roots with positive imaginary part will be denoted by p_{α} ($\alpha = 1, 2, 3$) with complex conjugates \bar{p}_{α} ; the corresponding values of A_k obtained from equation (6) are $A_{k\alpha}$ and $\bar{A}_{k\alpha}$. Summation over α will always be indicated explicitly. It will be assumed that the roots p_{α} are all distinct; equal roots may be regarded as the limiting case of distinct roots. A general expression for the displacement may then be written

(9)
$$u_k = \sum_{\alpha} A_{k\alpha} f_{\alpha}(z_{\alpha}) + \sum_{\alpha} \overline{A_{k\alpha}} \overline{f_{\alpha}(z_{\alpha})}$$

where $z_{\alpha} = x_1 + p_{\alpha} x_2$. From (1) we write the stresses as

(10)
$$\sigma_{ij} = \sum_{\alpha} L_{ij\alpha} f'_{\alpha}(z_{\alpha}) + \sum_{\alpha} \overline{L_{ij\alpha}} \overline{f'_{\alpha}(z_{\alpha})}$$

where

(11)
$$L_{ija} = (c_{ijk1} + p_a c_{ijk2})A_{ka}$$

and dashes denote differentiation with respect to z_{α} .

2. Indentation by a rigid punch

Consider an anisotropic elastic body which occupies the half space $x_2 < 0$. Let the area of contact between the punch and the elastic material be

$$(12) -b < x_1 < a, -\infty < x_3 < \infty$$

where a and b are positive constants. Then the boundary conditions on the plane $x_2 = 0$ are

(13)
$$\sigma_{12}(x_1, 0) = \sigma_{23}(x_1, 0) = 0 \quad -\infty < x_1 < \infty$$

(14)
$$\sigma_{22}(x_1, 0) = 0$$
 $x_1 < -b, x_1 > a$

and on $-b < x_1 < a$ only the normal component of displacement is given, so that

(15)
$$u_2(x_1, 0) = g(x_1) - b < x_1 < a$$

with

(16)
$$\int_{-b}^{a} \sigma_{22}(x_1, 0) dx_1 = -P$$

where P is the total applied force per unit length.

Let $f_{\alpha}(z_{\alpha}) = F_{\alpha}\phi(z_{\alpha})$ where F_{α} is a complex constant and $\phi(z)$ is analytic in $x_2 < 0$. Hence using (9) and (10) the components of stress and displacement are

(17)
$$u_{k} = \sum_{\alpha} A_{k\alpha} F_{\alpha} \phi(z_{\alpha}) + \sum_{\alpha} \overline{A_{k\alpha}} \overline{F_{\alpha}} \overline{\phi}(\overline{z_{\alpha}})$$

(18)
$$\sigma_{ij} = \sum_{\alpha} L_{ij\alpha} F_{\alpha} \phi'(z_{\alpha}) + \sum_{\alpha} \overline{L_{ij\alpha}} \overline{F_{\alpha}} \overline{\phi'(z_{\alpha})}$$

The boundary conditions (13) will be satisfied if

(19)
$$\sum_{\alpha} L_{12\alpha} F_{\alpha} = 0, \qquad \sum_{\alpha} L_{23\alpha} F_{\alpha} = 0$$

Suppose that there exist values of F_{α} such that the equations (19) are satisfied and

(20)
$$\sum_{\alpha} A_{2\alpha} F_{\alpha} = iM, \qquad \sum_{\alpha} L_{22\alpha} F_{\alpha} = N$$

where M and N are real constants. Then using (17) the condition (15) may be written as

(21) Re
$$[iM\phi(x_1)] = \frac{1}{2}g(x_1)$$
 $-b < x_1 < a$

Also using (18) and (20) it follows that on the boundary $x_2 = 0$

$$\sigma_{22}(x_1, 0) = 2N \operatorname{Re} [\phi'(x_1)]$$

so that conditions (14) and (16) may be used to give

(22) Re
$$[\phi(x_1)] = 0$$
 $x_2 = 0$, $x_1 < -b$
 $= -\frac{1}{2}P/N$ $x_2 = 0$, $x_1 > a$

The information (21) and (22) may be written in a single equation

Re
$$\left[\frac{\phi(x_1)}{(x_1-a)^{\frac{1}{2}}(x_1+b)^{\frac{1}{2}}}\right] = 0$$
 $x_1 < -b$
(23) $= \frac{\frac{1}{2}g(x_1)}{M(a-x_1)^{\frac{1}{2}}(x_1+b)^{\frac{1}{2}}}$ $-b < x_1 < a$
 $= \frac{-P}{2N(x_1-a)^{\frac{1}{2}}(x_1+b)^{\frac{1}{2}}}$ $x_1 > a$

Use of Cauchy's theorem in the form

$$f(z) = -\frac{1}{\pi i} \int_{-\infty}^{\infty} \frac{\operatorname{Re} f(x_1)}{x_1 - z} \, dx_1$$

for a function analytic in the lower half plane, gives

$$\frac{\phi(z_{\alpha})}{(z_{\alpha}-a)^{\frac{1}{2}}(z_{\alpha}+b)^{\frac{1}{2}}} = \frac{1}{\pi i} \int_{-b}^{a} \frac{-\frac{1}{2}g(x_{1})}{M(a-x_{1})^{\frac{1}{2}}(x_{1}+b)^{\frac{1}{2}}(x_{1}-z_{\alpha})} dx_{1} + \frac{1}{\pi i} \int_{a}^{\infty} \frac{P}{2N(x_{1}-a)^{\frac{1}{2}}(x_{1}+b)^{\frac{1}{2}}(x_{1}-z_{\alpha})} dx_{1}$$

where on the left, $0 \ge \arg (z_{\alpha} - a)^{\frac{1}{2}} \ge -\frac{1}{2}\pi$, $0 \ge \arg (z_{\alpha} + b)^{\frac{1}{2}} \ge -\frac{1}{2}\pi$ for $\operatorname{Im} z_{\alpha} < 0$. Hence

(24)

$$\phi(z_{\alpha}) = \frac{(z_{\alpha} - a)^{\frac{1}{2}}(z_{\alpha} + b)^{\frac{1}{2}}}{\pi i M} \int_{-b}^{a} \frac{-\frac{1}{2}g(x_{1})}{(a - x_{1})^{\frac{1}{2}}(x_{1} + b)^{\frac{1}{2}}(x_{1} - z_{\alpha})} dx_{1}$$

$$-\frac{P}{\pi i N} \log \{i(a + b)^{-\frac{1}{2}}[(z_{\alpha} - a)^{\frac{1}{2}} + (z_{\alpha} + b)^{\frac{1}{2}}]\}.$$

3. Restriction to a particular class of anisotropic materials

The solution presented in section 2 is only applicable if suitable values of the F_{α} can be found to satisfy equations (19) and (20). We must therefore restrict our attention to the class of anisotropic materials for which such values of the F_{α} can be found. In this section we show that it is possible to find suitable F_{α} for the class of materials for which the constants c_{1112} , c_{1222} , c_{1123} , c_{2223} and c_{1323} are zero. This is not necessarily the most general class of materials for which the solution of section 2 is valid, but with these six constants zero the sextic in p (7) becomes a cubic in p^2 and with this simplification it is reasonably easy to establish the existence of values of the F_{α} to satisfy (19) and (20). Also it is a class of materials of some interest since it includes orthotropy and transverse isotropy as special cases.

Putting the six constants equal to zero the matrix equation (6) reduces to

(25)
$$\begin{bmatrix} c_{1111} + p^2 c_{1212} & (c_{1122} + c_{1221})p & c_{1131} + p^2 c_{1232} \\ (c_{1122} + c_{1221})p & c_{1212} + p^2 c_{2222} & (c_{2231} + c_{2132})p \\ c_{1131} + p^2 c_{3212} & (c_{3122} + c_{3221})p & c_{3131} + p^2 c_{3232} \end{bmatrix} \begin{bmatrix} A_1 \\ A_2 \\ A_3 \end{bmatrix} = 0$$

where the values of p are obtained by solving equation (7) which is a cubic in p^2 . We consider two possible cases.

CASE 1. Suppose the roots of the cubic are real. Then since p cannot be real the cubic in p^2 must have negative roots. Hence the six values of p are complex numbers with zero real part. Then if the $A_{3\alpha}$ are put equal to 1 equation (25) has a solution for which the $A_{1\alpha}$ have zero imaginary part and the $A_{2\alpha}$ have zero real part. Also using (11) it may easily be shown that with this choice of the $A_{i\alpha}$ the $L_{12\alpha}$ and $L_{23\alpha}$ have zero real part and the $L_{22\alpha}$ have zero imaginary part. Hence we may choose $F_3 = 1$ and then real and unique values of F_1 and F_2 may be obtained from (19) provided $L_{121}L_{232}-L_{122}L_{231} \neq 0$. Values of M and N may then be obtained from (20).

CASE 2. Suppose only one of the roots $(p_1 \text{ say})$ of the cubic is real and negative. Then the other roots form a conjugate pair and $p_2 = -\bar{p}_3$. Putting the $A_{3\alpha}$ equal to 1 equation (25) has a solution for which A_{11} has zero imaginary part and A_{21} has zero real part. Also $A_{22} = -\bar{A}_{23}$ and $A_{12} = \bar{A}_{13}$. Then using (11) it may easily be shown that L_{121} and L_{231} have zero real part and that $L_{122} = -\bar{L}_{123}$ and $L_{232} = -\bar{L}_{233}$. Also L_{221} has zero imaginary part and $L_{222} = \bar{L}_{223}$. Hence we may choose $F_1 = 1$ and then unique values of F_2 and F_3 such that $F_2 = \bar{F}_3$ may be obtained from (19) provided $L_{122}L_{233} - L_{123}L_{232} \neq 0$. Values of M and N may then be obtained from (20).

4. Indentation by a circular block

Here we take

(26)
$$g(x_1) = \mathscr{B}x_1^2 - \gamma$$

which corresponds to the case of a rigid circular punch of large radius R with $\mathscr{B} = 1/2R$. Substituting for $g(x_1)$ in (24) and integrating we get

(27)
$$\phi(z_{\alpha}) = + \frac{i}{2M} \left[\frac{1}{2} \mathscr{B}(a-b+2z_{\alpha})(z_{\alpha}-a)^{\frac{1}{2}}(z_{\alpha}+b)^{\frac{1}{2}} - \mathscr{B}z_{\alpha}^{2} + \gamma \right] - \frac{P}{\pi i N} \log \left\{ i(a+b)^{-\frac{1}{2}} \left[(z_{\alpha}-a)^{\frac{1}{2}} + (z_{\alpha}+b)^{\frac{1}{2}} \right] \right\}$$

and it is easily checked that this function has the required property (23) and is $O(\log |z_a|)$ as $|z_a| \to \infty$. For this particular case we require that the stresses are

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finite at the ends a, b of the arc of contact of the punch with the elastic half space. This will be the case if $\phi'(z_{\alpha})$ is finite at $z_{\alpha} = a$, -b. Differentiate (27) with respect to z_{α} . Then the coefficient of $(z_{\alpha}-a)^{-\frac{1}{2}}(z_{\alpha}+b)^{-\frac{1}{2}}$ is

$$+\frac{1}{4}i\frac{\mathscr{B}}{2M}(4z_{\alpha}^{2}-(a-b)^{2})-\frac{P}{2\pi iN}$$

For finite stresses, this expression must vanish at $z_{\alpha} = a$, $z_{\alpha} = -b$ and this condition will be satisfied if

(28)
$$a = b = \left\{\frac{-PM}{\pi N\mathscr{B}}\right\}^{\frac{1}{2}}$$

4. A circular block on a transversely isotropic solid

It follows from equation (28) that for a circular punch the arc of contact with the half space is proportional to $(-M/N)^{\frac{1}{2}}$. In this section we calculate $(-M/N)^{\frac{1}{2}}$ for a circular punch on a transversely isotropic half space with the boundary of the half space normal to the transverse plane. The elastic behaviour of transversely isotropic materials is characterized by five elastic constants which will be denoted by A, N, F, C and L. If the x_2 -axis lies in the transverse plane and the x_1 -axis makes an angle α with the transverse plane (Figure 1) then the only non zero c_{ijkl} which are of interest are given by

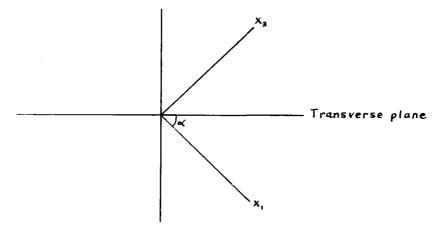


Figure 1

$$c_{1111} = A \cos^{4} \alpha + 2(F+2L) \cos^{2} \alpha \sin^{2} \alpha + C \sin^{4} \alpha$$

$$c_{1122} = N \cos^{2} \alpha + F \sin^{2} \alpha$$

$$c_{1131} = (A-F-2L) \cos^{3} \alpha \sin \alpha + (F-C+2L) \cos \alpha \sin^{3} \alpha$$

$$c_{2222} = A, \qquad c_{2231} = (N-F) \cos \alpha \sin \alpha$$

$$c_{3311} = (A + C - 4L) \cos^{2} \alpha \sin^{2} \alpha + F(\cos^{4} \alpha + \sin^{4} \alpha)$$

$$c_{3322} = F \cos^{2} \alpha + N \sin^{2} \alpha$$

$$c_{3331} = (A - F - 2L) \cos \alpha \sin^{3} \alpha + (F - C + 2L) \cos^{3} \alpha \sin \alpha$$

$$c_{1331} = L(\cos^{2} \alpha - \sin^{2} \alpha)^{2} - (2F - C - A) \cos^{2} \alpha \sin^{2} \alpha$$

$$c_{2332} = L \cos^{2} \alpha + \frac{1}{2}(A - N) \sin^{2} \alpha$$

$$c_{2312} = (\frac{1}{2}(A - N) - L) \cos \alpha \sin \alpha$$

$$c_{1212} = \frac{1}{2}(A - N) \cos^{2} \alpha + L \sin^{2} \alpha$$

Consider firstly the special cases when $\alpha = 0$ and $\frac{1}{2}\pi$. For transversely isotropic materials equation (7) is a cubic in p^2 and when $\alpha = 0$ the cubic has three equal roots so the solution to the punch problem may be obtained by comparison with the solution for the isotropic case. When $\alpha = \frac{1}{2}\pi$ (25) becomes

$$\begin{bmatrix} C+Lp^2 & (F+L)p & 0\\ (F+L)p & L+Ap^2 & 0\\ 0 & 0 & L+\frac{1}{2}(A-N)p^2 \end{bmatrix} \begin{bmatrix} A_1\\ A_2\\ A_3 \end{bmatrix} = 0$$

Also the determinant (7) may be expanded to give

$$(\frac{1}{2}(A-N)p^{2}+L)(ALp^{4}-(F^{2}+2FL-AC)p^{2}+CL)=0$$

Let $p_1^2 = -2L/(A-N)$. Then $A_{11} = A_{21} = 0$ and A_{31} is arbitrary so let $A_{31} = 0$. If the roots of the quadratic term are denoted by p_2^2 and p_3^2 then choosing $A_{22} = A_{23} = i$ it follows that

$$A_{12} = \frac{-i(F+L)p_2}{C+Lp_2^2}, \qquad A_{13} = \frac{-i(F+L)p_3}{C+Lp_3^2}$$
$$A_{32} = A_{33} = 0.$$

Also using equation (11) $L_{23\alpha} = 0$ and

$$L_{121} = 0, \quad L_{122} = iL \left[\frac{C - Fp_2^2}{C + Lp_2^2} \right], \quad L_{123} = iL \left[\frac{C - Fp_3^2}{C + Lp_3^2} \right]$$
$$L_{221} = 0, \quad L_{222} = ip_2 \left[A - \frac{F(F + L)}{C + Lp_2^2} \right], \quad L_{223} = ip_3 \left[A - \frac{F(F + L)}{C + Lp_3^2} \right]$$

If p_2^2 and p_3^2 are real then real values of F_2 and F_3 which satisfy (19) are given by

$$F_2 = \frac{C + Lp_2^2}{C - Fp_2^2}, \qquad F_3 = \frac{-(C + Lp_3^2)}{C - Fp_3^2}$$

Hence

$$M = \frac{C + Lp_2^2}{C - Fp_2^2} - \frac{C + Lp_3^2}{C - Fp_3^2}$$
$$N = ip_2 \left[\frac{A(C + Lp_2^2) - F(F + L)}{C - Fp_2^2} \right] - ip_3 \left[\frac{A(C + Lp_3^2) - F(F + L)}{C - Fp_3^2} \right]$$

If p_2^2 and p_3^2 are complex conjugates and $p_2 = -\bar{p}_3$ then suitable values of F_2 and F_3 which satisfy (19) are given by

$$F_2 = i \left[\frac{C + Lp_2^2}{C - Fp_2^2} \right], \qquad F_3 = -i \left[\frac{C + Lp_3^2}{C - Fp_3^2} \right]$$

Hence

$$iM = -\frac{C+Lp_2^2}{C-Fp_2^2} + \frac{C+Lp_3^2}{C-Fp_3^2}$$
$$N = -p_2 \left[\frac{A(C+Lp_2^2) - F(F+L)}{C-Fp_2^2}\right] + p_3 \left[\frac{A(C+Lp_3^2) - F(F+L)}{C-Fp_3^2}\right]$$

When $0 < \alpha < \frac{1}{2}\pi$ the calculation of M and N becomes more complicated. However the task may be carried out reasonably easily with the aid of a computer and this has been done for zinc which has the elastic constants A = 16.5, N = 3.1, F = 5, C = 6.2 and L = 3.96. If each of these numerical values is multiplied by 10^{11} then the units for constants are dynes/cm². Table 1 shows the variation in $(-M/N)^{\frac{1}{2}}$ as α assumes the values $n\pi/18$ $(n = 1, 2, \dots, 9)$.

TABLE 1 Values of $(-M/N)^{\frac{1}{2}}$ for $a = n\pi/18$

n	1	2	3	4	5	6	7	8	9
$(-M/N)^{\frac{1}{2}}.$.356	.360	.367	.377	.389	.401	.411	.417	.419

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